



This is a repository copy of *Predicting ash deposition behaviour for co-combustion of palm kernel with coal based on CFD modelling of particle impaction and sticking*.

White Rose Research Online URL for this paper:
<http://eprints.whiterose.ac.uk/91182/>

Version: Accepted Version

Article:

Ma, L., Yang, X., Ingham, D. et al. (2 more authors) (Accepted: 2015) Predicting ash deposition behaviour for co-combustion of palm kernel with coal based on CFD modelling of particle impaction and sticking. *Fuel*. ISSN 1873-7153

<https://doi.org/10.1016/j.fuel.2015.10.056>

Reuse

Unless indicated otherwise, fulltext items are protected by copyright with all rights reserved. The copyright exception in section 29 of the Copyright, Designs and Patents Act 1988 allows the making of a single copy solely for the purpose of non-commercial research or private study within the limits of fair dealing. The publisher or other rights-holder may allow further reproduction and re-use of this version - refer to the White Rose Research Online record for this item. Where records identify the publisher as the copyright holder, users can verify any specific terms of use on the publisher's website.

Takedown

If you consider content in White Rose Research Online to be in breach of UK law, please notify us by emailing eprints@whiterose.ac.uk including the URL of the record and the reason for the withdrawal request.



eprints@whiterose.ac.uk
<https://eprints.whiterose.ac.uk/>

1 Predicting ash deposition behaviour for co-combustion of
2 palm kernel with coal based on CFD modelling of particle
3 impaction and sticking

4 Xin Yang ^a, Derek Ingham ^a, Lin Ma ^{a*}, Alan Williams ^b, Mohamed Pourkashanian ^a

5 ^a Energy Engineering Group, Energy-2050, Department of Mechanical Engineering,
6 University of Sheffield, Sheffield S10 2TN, UK

7 ^b Energy Research Institute, School of Chemical and Process Engineering, University of
8 Leeds, Leeds LS2 9JT, UK

9 *Corresponding author:

10 Email: lin.ma@sheffield.ac.uk Phone: +44 (0) 114 21 57212.

11 **Abstract:**

12 A CFD model that simulates particle impaction and sticking has been developed for predicting the ash
13 deposition characteristics for the co-combustion of South African Coal (SAC) and palm kernel expeller
14 (PKE) in an entrained flow reactor. The numerical related errors, caused by interception and the improper
15 resolving of the flow-field within the boundary layer near the deposition surface, are investigated. In order
16 to minimize the numerical related errors without excessive meshing, a new revised particle impaction
17 model has been developed and accomplished using an impaction correction factor. The particle sticking is
18 predicted based on the molten fraction results that have been obtained from the chemical equilibrium
19 calculations using the chemical fractionation data in order to consider the short residence time of fly ash
20 particles. The simulation results show that a reasonable coarse mesh, coupled with the revised particle
21 impaction model, is suitable to accurately resolve the particle impaction without using a prohibitive
22 meshing size. The ash deposition behaviour is determined by the particle impaction and sticking properties.
23 Good agreements are obtained between the predicted results and the experimental data for the ash
24 deposition behaviour.

25 **Keywords:** Ash deposition, CFD, revised particle impaction model, particle sticking, co-firing.

26 **1 Introduction**

27 Co-combustion of biomass with coal has been used as a near term measure to reduce CO₂ emission
28 from coal fired power plants [1, 2]. Currently, co-firing 10-20% (thermal) biomass with coal has been
29 widely used in power stations in the UK and Europe and a higher co-firing rate is also used. Further, some
30 power stations, such as the Drax in the UK, are being converted to firing 100% biomass. With the recent
31 announcement of the new EU targets of reducing gas emissions, fuel flexibility is likely to be one of the
32 key factors influencing the operation of the power stations in the future, and the uses of various biomass
33 and waste for power in the EU are expected to substantially increase. Currently most large scale power
34 stations are using relatively clean biomass such as wood pellets, and to some extent straw, olive stones and
35 palm kernel expeller (PKE). An addition of up to about 10-20% biomass has only moderate effects on the
36 ash deposition in the furnace. However, with an increased co-firing rate and the use of a wide range of
37 biomass sources, ash related problems are ranking high on the list of significant operational constraints in
38 co-firing power plants [3, 4]. Ash deposition reduces the efficiency of the heat transfer through the water
39 walls and heat exchangers and causes corrosion of boiler tubes, which may lead to reduced generating
40 capacity and unscheduled outages [5]. Therefore, an improved understanding of the ash deposition in firing
41 various types of biomass is imperative for an efficient boiler operation and optimization in the future [6].

42 The optimum biomass co-firing rate in coal-fired boilers has still been mainly determined by
43 experiments up to now [4]. Computational Fluid Dynamics (CFD) has been widely used for solid fuel
44 combustion simulations and various sub-models have been developed for predicting ash depositions in lab-
45 scale test facilities as well as for full scale boilers [3-13]. Considerable progress has been made in the last
46 decades in developing ash deposition models for CFD simulations [3-5, 14-17], and more detailed and
47 accurate sub-models for combustion, fuel/ash particle transport, and sticking and deposition rate
48 predictions have been developed [3, 16]. Typically, Lagrangian methods are employed to compute the
49 trajectories of ash particles, coupled with an Eulerian method for flow and gaseous phase reaction, where
50 the inertial impaction of particles are often considered as the only or main mechanism for ash deposition
51 formation. Therefore, accurate prediction of the particle impaction is a critical factor that affects the
52 modelling of the ash deposition. The impaction efficiency of the particles is usually assumed to be unity
53 which represents the worst scenario in terms of ash deposition rate [3, 16]. In practice the impaction
54 efficiency can be much lower than this depending on the size, shape and density of the particles and the

55 nature of the depositing surface. Weber et al. [17] investigated the requirements for accurate predictions of
56 the impaction efficiency of fly ashes in a 2D geometry using the RANS- based CFD methods. It was
57 concluded that only when the flow-field in the neighbourhood of the deposition surfaces is accurately
58 resolved can accurate predictions of the particle impaction be obtainable by using the RANS- based CFD
59 methods, especially for small particles since their trajectories are strongly affected by the boundary layer
60 development. Haugen et al. [18, 19] applied direct numerical simulation (DNS) to investigate the particle
61 impaction behaviour on cylinders and superheater tube bundles in a crossflow in order to accurately
62 resolve the boundary layers around the cylinders. It should be noted that in these models [17-19],
63 extremely fine grid for RANS or DNS is needed. However, this requirement is difficult to be satisfied in
64 the simulation of an industrial boiler in order to predict ash deposition behaviours [17]. Often predicting
65 major operational parameters such as the boiler temperatures and/or combustion properties may be
66 achieved by using a reasonably coarse computer mesh. However, this reasonably coarse mesh can still lead
67 to a significant error in ash particle impaction efficiency calculation [17].

68 In addition to the particle impaction, the stickiness of the ash particles plays a critical role in the
69 formation of ash deposit and related slagging and fouling. The stickiness of an ash particle can be
70 determined based on such as the viscosity, kinetic energy and degree of molten of fly ash particles. In
71 terms of viscosity based sticking models, a reference viscosity is used to determine the stickiness. The
72 value of the reference viscosity ranges within $8-10^8$ Pa.s which makes the sticking model strongly sensitive
73 to the reference viscosity and may contribute to an inaccurate stickiness prediction [4, 20]. In addition, the
74 kinetic energy thresholding sticking model, based on the Johnson-Kendall-Roberts (JKR) theory [14, 21,
75 22], has been proposed which takes into account the kinetic energy of the particles and the surface energy
76 of both the particle and the impacted surface. However, a fitting process was necessary to develop the
77 effective Young's modulus versus the particle temperature and the particle diameter by matching the
78 experimental data with the simulation results [14]. Further, the molten fraction-based sticking model has
79 been developed using slag calculations based on the chemical equilibrium of the ash composition and it
80 was found that deposition models based on the molten fraction of ash particles calculated from chemical
81 equilibrium are promising [20].

82 Therefore, this paper aims to develop an improved ash deposition CFD model through (i) a new revised
83 particle impaction model to minimize the numerical related errors with an affordable number of

84 computational mesh, and (ii) an appropriate particle sticking model based on the ash chemistry and the
85 particle momentum for the PKE where there is relatively a scarce amount of data available. The model
86 developed has been tested using the experimental data from Imperial College's entrained flow reactor [23,
87 24], where PKE with the high level of phosphorus has been considered.

88 **2 Source of experimental data**

89 Figure 1 shows a schematic geometry of the entrained flow reactor (EFR) located at Imperial College
90 London. It consists of four electrically heated furnaces with a diameter of 0.1 m and a length of about 5 m.
91 The burner consists of a primary inlet through which the pulverized coal and the primary air are fed, and a
92 secondary inlet for the heated air. The uncooled ceramic probe is placed at the sample port 2 to collect the
93 ash deposits, which has a furnace temperature of approximately 1250 °C. More details of reactor can be
94 found in [23, 24].

95 EFR has been used in coal/biomass studies for many years and this is because it can provide
96 information on volatile release, char combustion, as well as ash deposition of the fuel combustion
97 processes in an environment that is close to that observed in an industrial furnace [23]. The time-
98 temperature history of fuel particles in the EFR is one of the key parameters for the design and operation of
99 EFR tests to achieve a similar condition as in a power plant in the study of ash deposition. This has been
100 carefully designed and operated in the experiments where a fuel particle with the residence time of
101 approximately 3 seconds has been achieved [23]. The probe that is used for collecting deposits is placed in
102 the cross section of the EFR to simulate the deposition formation on the heat exchanger which are
103 governed mainly by the cross-flow flue gas streams [17, 23]. Figure 2 shows a schematic diagram of the
104 typical structure of the deposition layers formed on the front surface of a heat exchanger tube. The initial
105 layer is typically formed due to the thermophoresis of small particles and the condensation of alkali vapour
106 compounds, such as NaCl, KCl, Na₂SO₄, and K₂SO₄ [25-33]. This initial layer is usually porous and has a
107 low thermal conductivity. The lower surface temperature of the tube enhances the growth of the initial
108 layer and as a result the surface temperature of the deposit will increase to a point that can facilitate the
109 melt and sintering to form a slag/sintered layer. The further growth of the slag/sintered layer is then
110 dictated by the inertial impaction of larger fly ash particles. The slag/sintered layer will have a higher

111 thermal conductivity compared to the initial layer because of its dense structure and the more melt formed
112 at the layer, and the grain size is also larger than that formed at the initial layer [26-29].

113 A range of scenarios of co-firing coal and biomass fuels have been experimentally investigated with
114 the Imperial College EFR, where ash depositions were collected and subsequently analysed. The
115 experimental data for the co-combustion of palm kernel expeller (PKE) with South African coal (SAC) has
116 been employed in this paper. The fuel properties, including proximate and ultimate analysis, as well as ash
117 compositions, of both coal and PKE are summarized in Table 1 [24, 34, 35]. As expected, PKE has much
118 lower values of fixed carbon and ash contents than the SAC but a higher volatile content. With regard to
119 the ash composition, SAC is mainly composed of acid oxides (silicon and aluminium) whereas PKE is
120 mainly composed of phosphorus, potassium and alkaline earth metals. Biomass with an ash rich in alkali
121 metals and chlorine have shown a tendency to accelerate the process of deposition, slagging and fouling on
122 the boiler surfaces [30-32]. The situation for phosphorus-rich biomass fuels is more complex and relatively
123 few research works and data are available. In very general terms, high potassium and high phosphorus
124 ashes tend to have a low fusion temperature and thus show a higher slagging tendency. However,
125 depending on the overall compositions of the fuel and ash, phosphorus can combine with the reactive
126 alkali/alkaline species, e.g. potassium, calcium and magnesium, to form higher melting temperature phases
127 [36, 37], and it can also influence the release of potassium during combustion, thus it can reduce the
128 overall ash depositions tendency [36, 37]. In the experiments performed at the EFR, an uncooled ceramic
129 probe is employed and placed at the sample port 2 (which has a furnace temperature of approximately
130 1250 °C) to collect the ash deposits, representing a slag/sintered layer [23, 24, 38]. Therefore, in the
131 current study, the initial layer is not modelled and the inertial impaction of particles is considered as the
132 main factor in controlling the ash deposition on the tube [8].

133 **3 Mathematical models**

134 **3.1 Combustion models**

135 In this paper, the combustion of coal and biomass is modelled in a combined Eulerian-Lagrangian
136 frame of reference where the volatile combustion is modelled in the Eulerian frame of reference and the
137 fuel/ash particles are tracked in a Lagrangian frame of reference. As stated in our previous papers, the
138 single kinetic rate model was employed for the devolatilizations of the coal and biomass, where the rate of

139 devolatilization depends on both the temperature and the volatile content of the particles [1, 39]. We have
 140 used the values of the Arrhenius rate constants, pre-exponential factor and activation energy that have been
 141 previously used and validated [1, 39, 40]. The combustion of the volatile gases was modelled using the
 142 Eddy Dissipation Model with a two-step global reaction mechanism [41]. Also it is assumed that the
 143 particle size remains constant, while the particle density reduces during the release of the volatile gases
 144 from the fuel particles [4, 39].

145 Char combustion was modelled with the intrinsic char combustion model, which assumes that the order
 146 of the surface reaction is unity and that the surface reaction rate includes the effects of both the bulk
 147 diffusion and chemical reaction rates [1]. The same model constants were employed as [41-43]. In the
 148 Smith intrinsic model, the variation of the char particle size and density is related to the fractional degree
 149 of burnout, U , in terms of the burning mode, α , as follows [7, 42]:

$$(d_p/d_{p,0}) = (1 - U)^\alpha \quad (1)$$

$$U = [1 - (m_p/m_{p,0})] \quad (2)$$

150 where d_p and m_p are the char particle size and mass, respectively, and the subscript zero refers to the
 151 initial conditions (at the start of the char combustion). For the coal, the value of α used was 0.25, and this
 152 corresponds to a decrease in both the particle size and density during combustion [7, 42]; for the biomass,
 153 it is believed that the particles would most likely maintain their original size during combustion, and
 154 therefore the value of zero was used for α , and this corresponds to a constant size but with a decreasing
 155 density of the particle during combustion [40].

156 The trajectories of the coal and biomass particles are governed by the particle momentum equation,
 157 which is a balance of the drag, gravity, and other body forces as formulated in the following equation [7,
 158 44]:

$$\frac{d\vec{v}_p}{dt} = \frac{18\mu_g C_D Re_p}{\rho_p d_p^2} (\vec{v}_g - \vec{v}_p) + \frac{\vec{g}(\rho_p - \rho_g)}{\rho_p} + \vec{F} \quad (3)$$

159 where \vec{v} , ρ , μ and d are the velocity, density, viscosity and diameter of the particles, respectively; the
 160 subscripts p and g refer to the particle and gas, respectively, C_D is the drag coefficient, and \vec{F} is other body
 161 forces, such as the thermophoretic force, virtual mass force, etc. The thermophoretic force, which is caused
 162 by the temperature gradient in the gas stream close to a solid depositing surface may be neglected when

163 modelling a heavily deposited surface such as the slag/sintered tube surface where a high surface
 164 temperature exists. The virtual mass force, which is due to the acceleration of the fluid around the particle,
 165 may also be ignored when the density of the particle is much greater than the density of the fluid.

166 In order to better resolve the particle trajectories in the boundary layer, the gas flow boundary layer
 167 has to be modelled carefully [17] and the enhanced wall treatment was used. If the near-wall mesh is fine
 168 enough to be able to resolve the fluid viscous sublayer, then the enhanced wall treatment will be identical
 169 to the traditional two-layer zonal model [45]. If a coarse mesh is used together with a wall-function, the
 170 accuracy of the near wall modelling will not significantly be reduced when the enhanced wall treatment is
 171 used, where a single wall law for the entire wall region is generated by blending the linear (laminar) and
 172 the logarithmic laws of the wall [46].

173 The energy balance equation of the particles, which are solved along the trajectories of the particles in
 174 order to obtain the corresponding particle temperatures, is given as follows [1, 7, 11]:

$$m_p c_p \frac{dT_p}{dt} = h A_p (T_\infty - T_p) + \varepsilon_p A_p \sigma (\theta_R^4 - T_p^4) - Q_p \quad (4)$$

175 where m_p , c_p , T_p , A_p , and ε_p are the mass, specific heat, temperature, surface area and emissivity of the
 176 particles, T_∞ is the gas temperature, σ is the Stefan–Boltzmann constant, and θ_R is the radiation
 177 temperature. Q_p , which is the latent heat or the heat of reaction is determined by the following equations:

$$Q_p = \begin{cases} 0, & \text{if under the step of inert heating or cooling} \\ \frac{dm_p}{dt} h_{fg}, & \text{if under the step of devolatilization} \\ f \frac{dm_p}{dt} H_r, & \text{if under the step of char combustion} \end{cases} \quad (5)$$

178 where h_{fg} is the latent heat, f is the fraction of the heat absorbed by the particles, and H_r is the heat of
 179 reaction released by the surface reaction. The radiative heat transfer was modelled using the Discrete
 180 Ordinates model and the gas absorption coefficient was calculated with the domain based Weighted Sum
 181 of Gray Gases Model (WSGGM).

182 3.2 Revised particle impaction model

183 Due to the influence of the gas flow, not all the particles carried by the gas stream will impact on the
 184 depositing surface. The amount of ash particles that may hit a depositing surface can be estimated by the
 185 particle impaction efficiency, which is defined as the percentage of particles of given size in the projected
 186 area of deposition surface in the upstream gas flow that can impact on the deposition surface [17]. The

187 impaction efficiency is dependent on the particle Stokes number [47, 48] that is defined as follows for
188 particle impaction on a circular cylinder:

$$\text{St} = (\rho_p d_p^2 u_p) / (9\mu_g D) \quad (6)$$

189 where ρ_p , d_p , u_p , and μ_g are the particle density, particle diameter, bulk particle velocity and gas dynamic
190 viscosity, respectively, and D is the outer diameter of the deposition pipe. Particles with larger Stokes
191 number are less likely to be affected by the gas flow and more likely to impact on the surface of the
192 deposition pipe; however, particles with smaller Stokes number follow more closely to the fluid
193 streamlines and are less likely to impact on the surface [48]. Therefore, accurately predicting the gas flow
194 in the boundary layer near the deposition surface is very important for accurate predicting the particle
195 impaction efficiency, in particular for particles with a small Stokes number. However, this often requires
196 an extremely fine computer mesh close to the deposition surface in order to resolve the flow boundary
197 layer accurately and this is often prohibitively expensive computationally for modelling real combustors
198 and industrial boilers. In most cases, as seen in most publications, a reasonably coarse mesh is employed to
199 satisfy the mesh independency requirement for the bulk of the gas flow. However, with this reasonably
200 coarse mesh the particle impacting efficiency is often over-estimated [12, 17], since the trajectory of small
201 ash particles close to a deposition surface is very sensitive to details of the boundary layer flow, leading to
202 errors in the deposition rate prediction.

203 Further, during the particle tracking in CFD, the particle is usually treated as a point in the
204 computational domain and whether a particle hits a wall or not is determined by the position of the center
205 of the particle without considering the effect of the size of the particle [17, 18]. However, in reality, a
206 particle will hit the surface with a distance equal to the particle radius away from the centre of the particle
207 [17, 18]. This interception effect of the particle size on the impaction efficiency can be described using the
208 interception parameter, R , defined as follows [17, 18]:

$$R = d_p / D \quad (7)$$

209 where d_p and D are referred to the diameters of the particles and the deposition probe. Clearly the larger
210 the ratio of particle diameter and the tube diameter, the larger is the interception. In order to remove the
211 errors resulting from using a coarse computational mesh in the boundary layer and from the particle
212 interceptions, an impaction correction factor, F , may be introduced which can be defined as the ratio of the
213 real particle impaction efficiency and that predicted using a reasonably coarse computational mesh for a

214 particular particle stream, i.e. for the i th particle stream, $F_i = I_{real,i}/I_{coarse,i}$. The real impaction efficiency
 215 $I_{real,i}$ can be estimated by using a small computational domain that only contains part of the furnace that is
 216 close to the superheat tubes where extremely fine meshing may be used. The boundary conditions for flue
 217 gas and ash particle flows may be taken from the existing results obtained using the reasonably coarse
 218 computational mesh. If the interception parameter is also considered then the impaction correction factor
 219 may be calculated using the following equation:

$$F_i = (I_{fine,i} + R_i)/I_{coarse,i} \quad (8)$$

220 where $I_{fine,i}$ is the predicted particle impaction efficiency from a well resolved boundary layer, $I_{coarse,i}$ is
 221 the impaction efficiency from a reasonably coarse mesh, and R_i is the interception parameter, of the i th
 222 particle stream; This impaction correction factor can be used to correct the CFD predicted mass flux, i.e.
 223 the arrival rate of the particles to the deposition surface still using a reasonably coarse computational mesh
 224 in the CFD simulation as illustrated in Figure 3.

225 In order to validate the proposed approach for improving the accuracy of particle impaction efficiency
 226 calculations, particle impactions on a two-dimensional cylinder have been simulated and compared with
 227 the results from the direct numerical simulation (DNS) reported by Haugen and Kragset [18]. Figure 4
 228 shows the flow configuration and boundary conditions employed. A computational domain of $6D \times 12D$ is
 229 employed with the tube being placed in the centre of the domain. The diameter of the tube $D=40$ mm. Gas
 230 (viscosity= 4.6×10^{-5} Pa.s and density= 0.245 kg/m^3 , which represent a stream of hot flue gas under 1500 K
 231 [17]) enters into the domain with a given free stream velocity, U_0 . Two velocities, $U_0= 0.47 \text{ m/s}$ and 7.91
 232 m/s have been considered, which corresponding to a Reynolds number (Re_t , based on deposition tube
 233 diameter) approximately 100 and 1685, respectively. In order to obtain the impaction efficiency with a
 234 well resolved boundary layer mesh, i.e. I_{fine} , RANS simulation with a fine mesh of approximately 400
 235 nodes ($Re_t = 100$) and 1600 nodes ($Re_t = 1685$) on the tube circumference have been employed as
 236 suggested by [17, 18, 49]. Figure 5 compares the predicted particle impaction efficiencies with and without
 237 the corrections for the two cases investigated with the DNS results obtained from [18]. Reasonably coarse
 238 meshes with approximately 140 nodes and 180 nodes on the circumference of the tube for the $Re_t = 100$
 239 and $Re_t = 1685$ cases respectively have been employed. It shows that the impaction correction factor
 240 increase with larger particle Stokes number and the value of the correction factor is approaching one when
 241 the Stokes number is greater than 1. Also, it can be found that applying the revised particle impaction

242 model can substantially to decrease the errors in the predicted particle impaction efficiency when a coarse
243 mesh is used, compared with the results from the DNS.

244 3.3 Particle sticking model

245 After particles reach a deposition surface, not all the arriving particles will stick to the surface. If the
246 particle, or the deposition surface, is sticky then the particle may deposit [10]. On the other hand, particles
247 with high impact energy may rebound back into the gas flow after hitting the wall [13, 15]. The sticking
248 efficiency of the impacting particles is defined as the ratio of the number of particulates depositing on the
249 surfaces to the number of the particles impacting on the surfaces [6]. There are at least two factors
250 influencing the particle sticking efficiency, namely, (i) whether a particle is sticking or not and (ii) whether
251 the particle will rebound back from the surface. In this work, the stickiness of the particles was determined
252 by the degree of melting of the particle, i.e. the molten fraction of the particles [20] calculated based on the
253 thermodynamic equilibrium of the particles. Since not all ash components can reach chemical equilibrium
254 when arriving at the deposition surface, in particular for EFR test conditions [50], the chemical
255 fractionation analysis data is employed from a stepwise leaching of the relevant fuels in order to consider
256 the short residence time of the ash particles in the reactor [50]. The chemical thermodynamics software
257 package FactSage 6.4. is employed to perform the thermodynamic equilibrium calculations based on the
258 minimization of the Gibbs free energy from the system subject to the mass balance constraints [51, 52].

259 In the thermodynamic equilibrium calculations, the gas composition N_2 , O_2 , CO_2 and H_2O are taken
260 from the CFD predictions and their amounts are dictated by the inlet air/fuel ratio. The reactants in the ash
261 are obtained from the ash analysis shown in Table 1 and the chemical fractionation data from [53, 54] for
262 both SAC and PKE as shown in Table 2. Potassium and sodium can be completely leached with chemicals
263 such as water and acetate for PKE whereas only part of sodium can be leached for SAC. Phosphorus is
264 more difficult to be leached for both PKE and SAC. The amount of the non-reactive fraction of the
265 inorganic mater in the ash particles which can reach equilibrium during combustion is difficult to
266 determine through the experiments and this amount is typically in the range 0 to 25% depending on
267 particle size, temperature and residence time, etc. [15, 50, 53, 55, 56]. Hence, in this paper, a middle value
268 of 10% was chosen [50]. For particles impacting a slagging/sintered surface, the value for the non-reactive
269 fraction is less significant in predicting the overall sticking efficiency than impacting on a new tube surface.

270 The calculations were performed for a temperature range between 1500 K and 1750 K at a temperature
 271 interval of 20 K and at atmospheric pressure. The possible products selected are the entire compound
 272 species (ideal gases and pure solids) from the ELEM, FToxid, FTsalt and FACTPS databases. The slag
 273 model chosen in the calculations was the ‘SLAGC’ with possible 2-phase immiscibility to consider the
 274 relative high amount of phosphorus in the ash [57, 58], which covers oxide liquid solutions of MgO, FeO,
 275 Na₂O, SiO₂, TiO₂, Ti₂O₃, CaO, Al₂O₃ and phosphates as Na₃(PO₄), Ca₃(PO₄)₂, Mg₃(PO₄)₂ and Fe₃(PO₄)₂,
 276 K₃PO₄ and FePO₄.

277 As mentioned earlier, whether an impacting particle actually sticks or not depends on whether the
 278 particle rebounds from the deposition surface or not, and this depends on the deformation after particle
 279 impacts on the surface and the momentum of the particles [8]. The particle energy balance model,
 280 developed by Mueller et al. [15] and Mao et al. [59], was employed to assess the excess energy, E_x , which
 281 is the excess rebounding energy particles possess after impaction and can be calculated using the following
 282 empirical formula [13, 15, 59]:

$$E_x = \frac{D^2}{4}(1 - \cos\alpha) - \frac{3D^{2.3}}{25}(1 - \cos\alpha)^{0.63} + \frac{2}{3D} - 1 \quad (9)$$

283 where D is the ratio of the maximum deformation in the particle diameter to the actual particle diameter,
 284 and α is the static contact angle of the particle. D is related to the particle Weber number, We , and the
 285 Reynolds number, Re , as follows:

$$D = (12 + We)^{0.5} \cdot [3(1 - \cos\alpha) + 4(We/Re^{0.5})]^{0.5} \quad (10)$$

$$We = (\rho_p U_n^2 d) / \sigma \quad (11)$$

$$Re = (\rho_p U_n d) / \mu_p \quad (12)$$

286 where U_n is the impact velocity component normal to the impact surface, and σ is the particle surface
 287 tension. If a particle possesses the necessary excess energy, $E_x > 0$, the particle will bounce off the surface,
 288 otherwise it will stick.

289 **4 EFR CFD model set up and results**

290 **4.1 Model set up**

291 In this paper, co-firing PKE with South Africa coal with four different co-firing rates have been
 292 investigated, namely the SAC are blended with 0, 20, 40 and 60 wt.% of the PKE. The same EFR

293 operational conditions as indicated in Figure 1 were employed for all four cases investigated. The coal
294 flow rate of 0.014 gs^{-1} , the primary air flow rate of 0.067 kgs^{-1} at $70 \text{ }^\circ\text{C}$, and the secondary air flow of
295 1.167 kgs^{-1} at $300 \text{ }^\circ\text{C}$ [8] have been used for all the cases. Only the biomass additions were different to
296 make PKE 0, 20, 40 and 60 wt.% of the fuel flow rate. Typically EFR was operated at a relatively low
297 Reynolds number of approximately 400, to acquire near laminar flow conditions in the reactor although
298 low turbulence occurs near the burner region [23]. To improve the convergence of the CFD model, the
299 Transition SST turbulence model which is applicable to low Reynolds number flows, was chosen to
300 simulate the gas flow [60, 61]. The particle size of the SAC used ranged between $1 \text{ }\mu\text{m}$ and $95 \text{ }\mu\text{m}$ with a
301 mean diameter of $50 \text{ }\mu\text{m}$ and the particle size of the PKE ranged between $105 \text{ }\mu\text{m}$ and $355 \text{ }\mu\text{m}$, with a
302 mean diameter of $130 \text{ }\mu\text{m}$ [16, 17, 20]. The temperature of the internal wall of the EFR has been specified
303 according to the experimental conditions as indicated in Figure 1. The adiabatic condition has been
304 employed at the surface of the deposition probe to consider its uncooled condition. The commercially
305 available CFD software package ANSYS Fluent version 15.0 has been employed to perform the basic
306 calculations of the coal and biomass combustion incorporating the in-house developed User Defined
307 Functions and Memories in order to model the ash deposition process with the revised particle impaction
308 model and the particle sticking model.

309 Six different meshes consisting of 0.5, 0.7, 1.1, 1.6, 2.0, and 3.3 million hexahedral cells have been
310 employed to investigate the effect of mesh on the CFD solutions. The mesh is refined in the vicinity of the
311 probe and the injection region, as well as in the burner region. Both the typical gas flow properties, such as
312 the distributions of the gas temperature and gas velocity and the arrival rate of fly ash particles that impact
313 the probe surface were examined. It was found that independent solutions for the gas temperature and
314 velocity in the bulk of the EFR may be obtained using a mesh with no less than 0.7M cells which may be
315 regarded as a reasonable coarse mesh. However, mesh independency was not achieved for the predicted
316 arrival rate of fly ash particles until a mesh with a cell number of over 1.6M was used, as shown in Figure
317 6. It can be seen from Figure 6, if the 0.7M cell mesh is employed, then the arrival rate of the impacting
318 particles would be overestimated by approximately 40% compared to the results obtained using 1.6M cells.
319 This is because a higher impaction efficiency is predicted for all the particles as shown in Figure 6. It
320 should be noted that for industrial boilers, the gas flow is highly turbulent ($\text{Re} \geq 100000$) and a very thin

321 flow boundary layer will be formed, thus achieving a grid independent solution for particle impaction is
322 usually very difficult in a 3D geometry [17].

323 In this particular case of the Imperial EFR, the particle impaction results obtained from, using the 1.6M
324 grid was taken as I_{fine} since they are mesh independent solutions for the particle impaction, and the
325 particle impaction results from the 0.7M grid was taken as I_{coarse} since they are mesh independent
326 solutions of the gas flows. For the case investigated, the interception parameter, R , is in the range 2.9×10^{-5}
327 to 7.5×10^{-3} which give an overall effect on the particle collection efficiency in the range of 7-8% and
328 therefore cannot be ignored.

329 4.2 Predicted impaction efficiency and sticking efficiency

330 The modelling of the particle impaction and sticking is critical for predicting ash deposition formation
331 and growth. As discussed in Section 3.2, both the improper resolving of the gas flow within the boundary
332 layer near the deposition surface and particle interception can affect the accuracy of the predicted particle
333 impaction in CFD modelling. Figure 7 shows the calculated impaction correction factor F_i and particle
334 impaction efficiency as a function of the Stokes numbers of the coal and biomass particles employed in the
335 EFR simulation for the four co-firing cases investigated. It can be found that the correction factor increases
336 with an increase in the particle Stokes number. The value is approaching one when the Stokes number is
337 greater than one because the particles are mainly driven by their inertia and the imperfection in the
338 prediction of the boundary flow field is of secondary importance [17, 18]. However, particles with small
339 Stokes number have a small value of the correction factor and this indicates that the numerical related
340 errors are large when a coarse mesh was used and this is because the particle trajectories are strongly
341 affected by flow field within the boundary layer which needs to be accurately modelled [17-19]. The
342 predicted impaction efficiencies for particles with a similar Stokes number are similar for different levels
343 of the PKE addition, in particular for low co-firing rates. This is because the air and fuel flow rates are
344 almost the same for the four cases and this results in a similar flow distribution, including in the boundary
345 layer flow.

346 Further, the figure shows that the Stokes number of most of the fuel particles is less than 1 and the
347 maximum Stokes number is approximately 1.5, and this indicates the significance of the impaction
348 correction factor in predicting the ash deposition rate. Figure 8 compares the results of the predicted

349 overall impaction efficiencies both with and without applying the particle impaction correction. It can be
350 observed that for all the cases, there is 30-50% over prediction in the overall impaction efficiency and this
351 may be resulted if a coarse mesh is employed and no correction is made. Also the figure shows that on
352 increasing the co-firing rates from 0-60%, the overall impaction efficiency increases from about 3.1% to
353 4.6%. This is due to the fact that the biomass particles have a higher value for the Stokes number than that
354 of the coal particles due to the much larger particle size, and this results in a higher impaction efficiency
355 than that of the SAC particles.

356 Figure 9 shows the predicted overall sticking efficiency (defined as the ratio of the overall mass flow
357 rate of the deposited particles to the overall mass flow rate of the impacting particles) taking into
358 consideration both the stickiness of the ash particles and the deposition surface for the four cases
359 investigated. The figure shows that the overall sticking efficiency for the 60% PKE co-firing ratio is
360 approximately 33%, which is the lowest of the four cases investigated. The overall sticking efficiencies are
361 similar for the 20% and 40% ratio cases which lie in the range from 53% to 56%. This is because the
362 molten fraction increases on adding 20% and 40% PKE co-firing ratio due to the larger amount of molten
363 phases formed than in the case of 0% PKE co-firing ratio and then the sticking efficiency increases.
364 However, the sticking efficiency decreases on adding 60% PKE. This is due to the formation of high
365 temperature solid phases (such as $\text{Ca}_3(\text{PO}_4)_2$, $\text{Mg}_3\text{P}_2\text{O}_8$, AlPO_4 , and KAlSi_2O_6) with increasing the alkaline
366 earth metals (calcium and magnesium) by further adding the PKE.

367 It was noted that the sticking efficiencies for particles were almost the same for all Stokes numbers,
368 with only a slightly increase with an increase in the Stokes number. This is because the particles were at
369 the cooling stage at the end of the char combustion, the small particles cool earlier and more quickly and
370 thus have a lower temperature and are less sticky than those of the larger particles [14].

371 4.3 Predicted ash deposition

372 In the experiments reported in [23, 24], the deposition efficiency, which was calculated for each EFR
373 run based on the mass percentage of the fuel ash that impacts on the projected surface area of the probe
374 that was retained in the collected deposit, was employed to evaluate the deposition propensity. Figure 10
375 shows a quantitative comparison between the computed deposits (with and without the revised particle
376 impaction model) and the experimental data in terms of the deposition efficiency. In general, the predicted
377 deposition efficiency without the revised particle impaction model is much higher than both the

378 experimental data and the results obtained with the revised particle impaction model. The predicted
379 deposition efficiency with the revised particle impaction model varies between 6.1% to 9.5% for the four
380 cases investigated, which is in good agreement with the experimental data. Also the calculation results
381 show that the deposition efficiency at the 60% PKE co-firing ratio is lower than that at 40% PKE co-firing
382 ratio and the deposition efficiency is the highest for a co-firing ratio of 40% of PKE. The reason for this
383 modelling outcome is due to the decreasing trend of the overall sticking efficiency at 60% PKE co-firing
384 ratio. However, in the experimental data, the deposition efficiency with a value of 8.5% at 60% PKE co-
385 firing ratio is still slightly higher than that at 40% PKE co-firing ratio which is 8.4%. The repeatability
386 error for the measurement of the deposition efficiency can be up to 4.7% [24]. Therefore, the small
387 disagreement in predicted the deposition efficiency between is acceptable [24, 62]. No obvious increase of
388 deposition efficiency for co-combustion of PKE with South African Coal was found comparing with pure
389 coal comustion both in the experimental results and the simulation results which is controlled by both the
390 particle impaction and sticking, although higher sintering degree of the deposits was shown in the
391 experiments. This is because the viscosity of the deposit was decreased with the decrease in SiO₂ and
392 increase in MgO, CaO and P₂O₅ in the ash composition by adding PKE [24].

Comment [a1]: ??

393 **5 Conclusions**

394 An ash deposition model based on modelling the particle impaction and sticking has been developed
395 for the modelling of ash deposition for co-combustion of coal and palm kernel. A revised particle
396 impaction modelling approach is proposed in order to minimize the numerical related errors and avoid
397 using an excessive mesh size. A particle impaction correction factor that takes account of both the effect of
398 particle interception and errors in the particle impaction prediction when a coarse computational mesh is
399 employed. The particle sticking is predicted based on the molten fraction of the particle obtained from the
400 chemical equilibrium calculation using the chemical fractionation data of fuels in order to consider the
401 short residence time of the fly ash particles.

402 The deposition efficiencies of co-firing SAC with PKE of four different ratios in the EFR have been
403 calculated using the model and the results obtained have been compared with the experimental data
404 obtained in the EFR. Reasonably good agreement was obtained and it demonstrated that the proposed
405 model can reduce the numerical related errors in the ash deposition prediction using a reasonable coarse

406 computational mesh adaqit for the combustion process simulation. The results suggested that the overall
407 impaction efficiency of the particles increase with the increase in the co-firing ratio of PKE, whilst the
408 overall stickiness efficiency may depend on the relative amount of high- and low- melting point
409 compounds that are formed and this is dictated by the ash composition of the fuels.

410 It is noted that the Stokes numbers of most of the ash particles are less than one and thus a significance
411 correction to the predicted ash deposition rate may be required and this depends on the number of
412 computational meshes employed. When the particle Stokes number approaches or is greater than one then
413 the particles are mainly driven by their inertia and the impact correction factor approaches one and
414 therefore using a relatively coarse mesh may be acceptable.

415 **Acknowledgment**

416 The authors wish to thank Prof. J. Williamson (Imperial College, London) for the helpful discussions
417 on the deposition behaviours in the co-firing experiments, Prof. R. Weber and Dr A.M. Beckmann
418 (Clausthal University of Technology) for their very helpful advice on modelling particle impaction. Also X.
419 Yang would like to acknowledge the China Scholarship Council and the University of Sheffield and
420 University of Leeds, for funding his research studies. Support from the RCUK is also acknowledged.

421 **References**

- 422 [1] Black S, Szuhánszki J, Pranzitelli A, Ma L, Stanger PJ, Ingham DB, et al. Effects of firing coal and biomass
423 under oxy-fuel conditions in a power plant boiler using CFD modelling. *Fuel* 2013;113:780-6.
424 [2] Rubiera F, Pis J, Pevida C. Raw materials, selection, preparation and characterization. In: Puigjaner L,
425 editors. *Syngas from Waste*, London; Springer Verlag; 2011, p. 11-22.
426 [3] Lundmark D, Mueller C, Backman R, Zevenhoven M, Skrifvars BJ, Hupa M. CFD based ash deposition
427 prediction in a BFBC firing mixtures of peat and forest residue. *J Energ Resour-ASME* 2010;132:031003/1-/8.
428 [4] Taha TJ, Stam AF, Stam K, Brem G. CFD modeling of ash deposition for co-combustion of MBM with coal
429 in a tangentially fired utility boiler. *Fuel Process Technol* 2013;114:126-34.
430 [5] Wang H, Harb JN. Modeling of ash deposition in large-scale combustion facilities burning pulverized coal.
431 *Prog Energ Combust* 1997;23:267-82.
432 [6] Akbar S. Numerical simulation of deposit formation in coal-fired utility boilers with biomass co-combustion
433 [dissertation]. University of Stuttgart; 2011.
434 [7] Degereji MU, Ingham DB, Ma L, Pourkashanian M, Williams A. Prediction of ash slagging propensity in a
435 pulverized coal combustion furnace. *Fuel* 2012;101:171-8.
436 [8] Garba MU, Ingham DB, Ma L, Degereji MU, Pourkashanian M, Williams A. Modelling of deposit
437 formation and sintering for the co-combustion of coal with biomass. *Fuel* 2013;113:863-72.
438 [9] Huang LY, Norman JS, Pourkashanian M, Williams A. Prediction of ash deposition on superheater tubes
439 from pulverized coal combustion. *Fuel* 1996;75:271-9.
440 [10] Kær SK, Rosendahl LA, Baxter LL. Extending the capability of CFD codes to assess ash related problems
441 in biomass fired boilers. *Prepr Pap - Am Chem Soc, Div Fuel Chem* 2003;49:97-108.
442 [11] Lee FCC, Lockwood FC. Modelling ash deposition in pulverized coal-fired applications. *Prog Energ*
443 *Combust* 1999;25:117-32.

- 444 [12] Losurdo M, Spliethoff H, Kiel J. Ash deposition modeling using a visco-elastic approach. *Fuel*
445 2012;102:145-55.
- 446 [13] Ma L, Pourkashanian M, Williams A, Jones J. A numerical model for predicting biomass particle
447 depositions in a pf furnace. In: Proc ASME Turbo Expo 2006, Barcelona, Spain; 2006, p. 333-42.
- 448 [14] Ai W, Kuhlman JM. Simulation of coal ash particle deposition experiments. *Energy Fuels* 2011;25:708-18.
- 449 [15] Mueller C, Selenius M, Theis M, Skrifvars B-J, Backman R, Hupa M, et al. Deposition behaviour of
450 molten alkali-rich fly ashes—development of a submodel for CFD applications. *Proc Combust Inst*
451 2005;30:2991-8.
- 452 [16] Mueller C, Skrifvars B-J, Backman R, Hupa M. Ash deposition prediction in biomass fired fluidised bed
453 boilers - combination of CFD and advanced fuel analysis. *Prog Comput Fluid Dyn* 2003;3:112-20.
- 454 [17] Weber R, Schaffel-Mancini N, Mancini M, Kupka T. Fly ash deposition modelling: Requirements for
455 accurate predictions of particle impaction on tubes using RANS-based computational fluid dynamics. *Fuel*
456 2013;108:586-96.
- 457 [18] Haugen NEL, Kragset S. Particle impaction on a cylinder in a crossflow as function of Stokes and
458 Reynolds numbers. *J Fluid Mech* 2010;661:239-61.
- 459 [19] Haugen NEL, Kragset S, Bugge M, Warnecke R, Weghaus M. MSWI super heater tube bundle: Particle
460 impaction efficiency and size distribution. *Fuel Process Technol* 2013;106:416-22.
- 461 [20] Wieland C, Kreutzkam B, Balan G, Spliethoff H. Evaluation, comparison and validation of deposition
462 criteria for numerical simulation of slagging. *Appl Energ* 2012;93:184-92.
- 463 [21] Brach RM, Dunn PF. A mathematical model of the impact and adhesion of microspheres. *Aerosol Sci Tech*
464 1992;16:51-64.
- 465 [22] Johnson KL, Kendall K, Roberts AD. Surface energy and the contact of elastic solids. *Proc R Soc London*
466 A: Math Phys Sci 1971;324:301-13.
- 467 [23] Hutchings IS, West SS, Williamson J. An assessment of coal-ash slagging propensity using an entrained
468 flow reactor. In: Baxter L, DeSollar R, editors. *Proceeding of a conference on Applications of Advanced*
469 *Technology to Ash-Related Problems in Boiler. Proceedings of the Engineering Foundation Conference.*
470 *Waterville Valley, New Hampshire. Plenum Press New York; 1995 vol. 1621. p. 201–22.*
- 471 [24] Wigley F, Williamson J, Malmgren A, Riley G. Ash deposition at higher levels of coal replacement by
472 biomass. *Fuel Process Technol* 2007;88:1148-54.
- 473 [25] Zhan Z, Bool LE, Fry A, Fan W, Xu M, Yu D, et al. Novel temperature-controlled ash deposition probe
474 system and its application to oxy-coal combustion with 50% inlet O₂. *Energy Fuels* 2013;28:146-54.
- 475 [26] Zhou H, Zhou B, Dong K, Ding J, Cen K. Research on the slagging characteristics of easy to slagging coal
476 in a pilot scale furnace. *Fuel* 2013;109:608-15.
- 477 [27] Zhou H, Zhou B, Li L, Zhang H. Experimental measurement of the effective thermal conductivity of ash
478 deposit for high sodium coal (Zhun Dong coal) in a 300 kw test furnace. *Energy Fuels* 2013;27:7008-22.
- 479 [28] Zhou H, Zhou B, Li L, Zhang H. Investigation of the influence of the furnace temperature on slagging
480 deposit characteristics using a digital image technique. *Energy Fuels* 2014;28:5756-65.
- 481 [29] Zhou H, Zhou B, Zhang H, Li L, Cen K. Investigation of slagging characteristics in a 300 kw test furnace:
482 Effect of deposition surface temperature. *Ind Eng Chem Res* 2014;53:7233-46.
- 483 [30] Wang X, Xu Z, Wei B, Zhang L, Tan H, Yang T, et al. The ash deposition mechanism in boilers burning
484 Zhundong coal with high contents of sodium and calcium: A study from ash evaporating to condensing. *Appl*
485 *Therm Eng* 2015;80:150-9.
- 486 [31] Niu Y, Zhu Y, Tan H, Hui S, Jing Z, Xu W. Investigations on biomass slagging in utility boiler: Criterion
487 numbers and slagging growth mechanisms. *Fuel Process Technol* 2014;128:499-508.
- 488 [32] Niu Y, Du W, Tan H, Xu W, Liu Y, Xiong Y, et al. Further study on biomass ash characteristics at elevated
489 ashing temperatures: The evolution of K, Cl, S and the ash fusion characteristics. *Bioresour Technol*
490 2013;129:642-5.
- 491 [33] Wang G, Pinto T, Costa M. Investigation on ash deposit formation during the co-firing of coal with
492 agricultural residues in a large-scale laboratory furnace. *Fuel* 2014;117, Part A:269-77.
- 493 [34] Lu G, Yan Y, Cornwell S, Whitehouse M, Riley G. Impact of co-firing coal and biomass on flame
494 characteristics and stability. *Fuel* 2008;87:1133-40.
- 495 [35] Smart JP, O’Nions P, Riley GS. Radiation and convective heat transfer, and burnout in oxy-coal
496 combustion. *Fuel* 2010;89:2468-76.
- 497 [36] Grimm A. Experimental studies of ash transformation processes in combustion of phosphorus-rich biomass
498 fuels [dissertation]. Luleå University of Technology; 2012.
- 499 [37] Lindström E, Sandström M, Boström D, Öhman M. Slagging characteristics during combustion of cereal
500 grains rich in phosphorus. *Energy Fuels* 2007;21:710-7.
- 501 [38] Baptiste F. Numerical and experimental characterization of drop tube furnace and ash deposition analysis
502 [dissertation]. RWTH Aachen University; 2013.

503 [39] Badzioch S, Hawksley PGW. Kinetics of thermal decomposition of pulverized coal particles. *Ind Eng*
504 *Chem Process Des Dev* 1970;9:521-30.

505 [40] Ma L, Gharebaghi M, Porter R, Pourkashanian M, Jones JM, Williams A. Modelling methods for co-fired
506 pulverised fuel furnaces. *Fuel* 2009;88:2448-54.

507 [41] Magnussen BF, Hjertager BH. On mathematical modeling of turbulent combustion with special emphasis
508 on soot formation and combustion. *Proc Combust Inst* 1977;16:719-29.

509 [42] Backreedy RI, Fletcher LM, Ma L, Pourkashanian M, Williams A. Modelling pulverised coal combustion
510 using a detailed coal combustion model. *Combust Sci Technol* 2006;178:763-87.

511 [43] Ma L, Jones JM, Pourkashanian M, Williams A. Modelling the combustion of pulverized biomass in an
512 industrial combustion test furnace. *Fuel* 2007;86:1959-65.

513 [44] Chen L, Ghoniem AF. Development of a three-dimensional computational slag flow model for coal
514 combustion and gasification. *Fuel* 2013;113:357-66.

515 [45] Chen HC, Patel VC. Near-wall turbulence models for complex flows including separation. *AIAA J*
516 1988;26:641-8.

517 [46] Kader BA. Temperature and concentration profiles in fully turbulent boundary layers. *Int J Heat Mass Tran*
518 1981;24:1541-4.

519 [47] Baxter LL. Ash deposit formation and deposit properties. A comprehensive summary of research conducted
520 at Sandia's combustion research facility [Technical Report]. Sandia National Labs; 2000 Aug.

521 [48] Barker B, Casaday B, Shankara P, Ameri A, Bons J. Coal ash deposition on nozzle guide vanes—Part II:
522 computational modeling. *J Turbomach* 2013;135:011015.

523 [49] Bouhairie S, Chu VH. Two-dimensional simulation of unsteady heat transfer from a circular cylinder in
524 crossflow. *J Fluid Mech* 2007;570:177-215.

525 [50] Nutalapati D, Gupta R, Moghtaderi B, Wall TF. Assessing slagging and fouling during biomass
526 combustion: A thermodynamic approach allowing for alkali/ash reactions. *Fuel Process Technol* 2007;88:1044-
527 52.

528 [51] Bale CW, Bélisle E, Chartrand P, Decterov SA, Eriksson G, Hack K, et al. FactSage thermochemical
529 software and databases — recent developments. *Calphad* 2009;33:295-311.

530 [52] Fryda L, Sobrino C, Cieplik M, van de Kamp WL. Study on ash deposition under oxyfuel combustion of
531 coal/biomass blends. *Fuel* 2010;89:1889-902.

532 [53] Tortosa Masiá AA, Buhre BJP, Gupta RP, Wall TF. Characterising ash of biomass and waste. *Fuel Process*
533 *Technol* 2007;88:1071-81.

534 [54] Aho M, Ferrer E. Importance of coal ash composition in protecting the boiler against chlorine deposition
535 during combustion of chlorine-rich biomass. *Fuel* 2005;84:201-12.

536 [55] Tortosa Masiá A. Characterisation and prediction of deposits in biomass co-combustion [dissertation]. Delft
537 University of Technology; 2010.

538 [56] Plaza P, Griffiths AJ, Syred N, Rees-Gralton T. Use of a predictive model for the impact of cofiring
539 coal/biomass blends on slagging and fouling propensity. *Energy Fuels* 2009;23:3437-45.

540 [57] Yakaboylu O, Harinck J, Gerton Smit KG, de Jong W. Supercritical water gasification of manure: A
541 thermodynamic equilibrium modeling approach. *Biomass Bioenerg* 2013;59:253-63.

542 [58] Zhang L, Ninomiya Y. Transformation of phosphorus during combustion of coal and sewage sludge and its
543 contributions to PM10. *Proc Combust Inst* 2007;31:2847-54.

544 [59] Mao T, Kuhn DCS, Tran H. Spread and rebound of liquid droplets upon impact on flat surfaces. *AIChE J*
545 1997;43:2169-79.

546 [60] Ansys, 15.0 Theory Guide, 2013.

547 [61] Zhang Z, Kleinstreuer C, Hyun S. Size-change and deposition of conventional and composite cigarette
548 smoke particles during inhalation in a subject-specific airway model. *J Aerosol Sci* 2012;46:34-52.

549 [62] Wigley F, Williamson J, Riley G. The effect of mineral additions on coal ash deposition. *Fuel Process*
550 *Technol* 2007;88:1010-6.

Table 1. Coal and PKE properties used in the calculations [24, 34, 35].

Ash composition (wt.%)	Proximate analysis (wt.%(ar))			Ultimate analysis (wt.%(daf))	
	SAC	PKE		SAC	PKE
SiO ₂	54.1	15.1	Volatiles	26.4	71.6
Al ₂ O ₃	33.5	3.2	Fixed carbon	60.2	28.4
Fe ₂ O ₃	3.1	5.3	Ash	12.1	4.2
CaO	4.1	10.7	GCV(MJ/kg)	27.3	18.7
MgO	1.3	12.0			
K ₂ O	0.7	9.7	C	70.0	44.2
Na ₂ O	0.1	0.3	H	3.9	7.0
TiO ₂	1.7	0.1	O	7.3	46.2
MnO	0.0	1.0	N	1.7	2.6
P ₂ O ₅	1.1	42.7	S	0.6	0.5

Table 2. Chemical fractionation (percentage) of ash components leached from fuels [53, 54].

	SiO ₂	Al ₂ O ₃	Fe ₂ O ₃	CaO	MgO	K ₂ O	Na ₂ O	TiO ₂	P ₂ O ₅
SAC	0	0	0	70	0	2	40	0	0
PKE	1	0	0	5	3	100	100	0	3

Figure Captions

Figure 1. A schematic diagram of the geometry of the EFR based [23, 24].

Figure 2. Schematic diagram of the main formation of the different deposition layers on the front surface of the cylindrical probe.

Figure 3. The methodology for the revised particle impaction model.

Figure 4. The flow configuration and boundary conditions of the 2D computational domain.

Figure 5. The impaction correction factor and comparisons of the predicted particle impaction efficiency using a coarse mesh and the DNS, with and without particle impact correction when (a): $Re_t=100$ and (b): $Re_t=1685$ as a function of the Stokes number.

Figure 6. The arrival rate of fly ash particles that impact the probe surface as a function of the number of cells and the impaction efficiency of particles as a function of the particle Stokes number for 0.7M and 1.6M.

Figure 7. The impaction correction factor and impaction efficiency of particles as a function of the particle Stokes number.

Figure 8. The overall impaction efficiency for SAC and for different levels of PKE with and without the revised particle impaction model.

Figure 9. The overall sticking efficiency for SAC and for different levels of PKE.

Figure 10. A comparison between the computed and the experimental data of the deposition efficiency for the SAC with different levels of PKE.

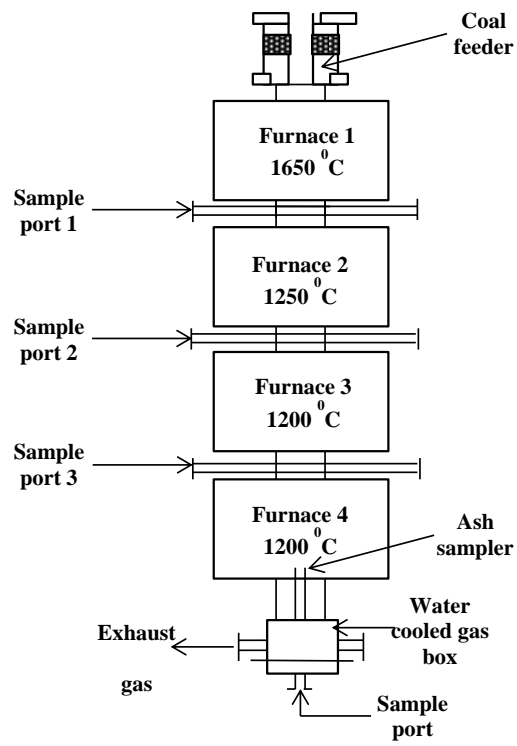


Figure 1. A schematic diagram of the geometry of the EFR based [23, 24].

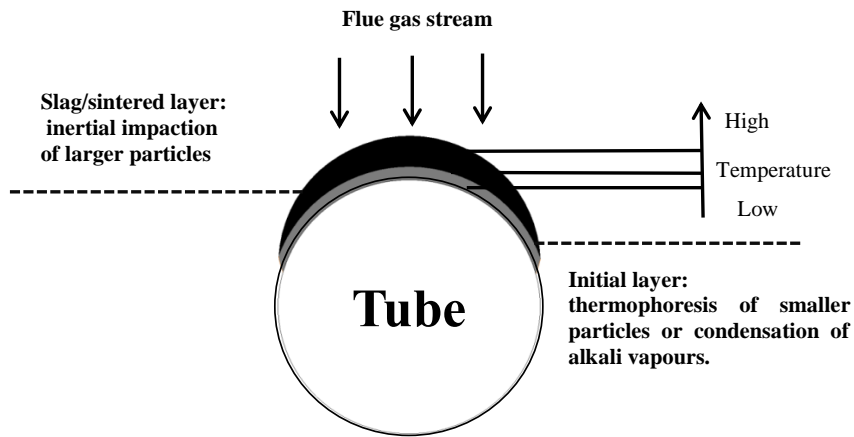


Figure 2. Schematic diagram of the main formation of the different deposition layers on the front surface of the cylindrical probe.

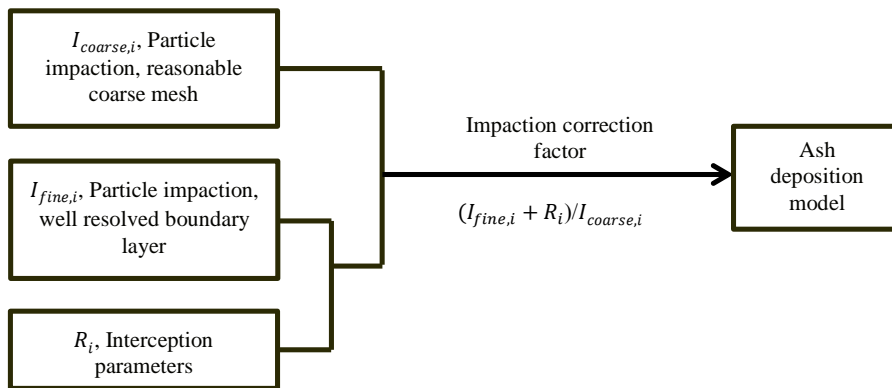


Figure 3. The methodology for the revised particle impaction model.

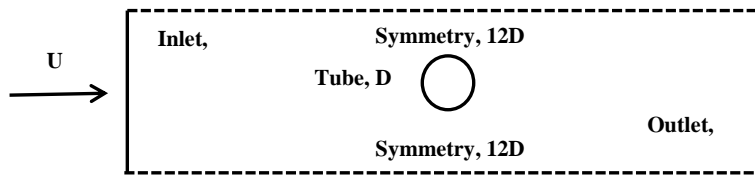


Figure 4. The flow configuration and boundary conditions of the 2D computational domain.

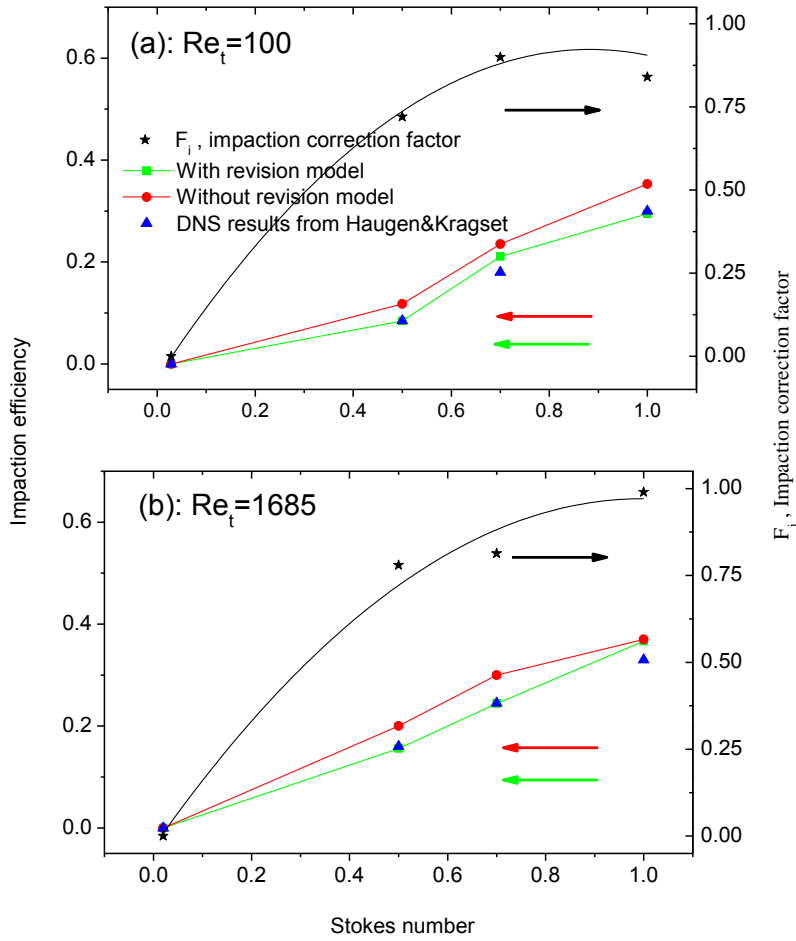


Figure 5. The impact correction factor and comparisons of the predicted particle impact efficiency using a coarse mesh and the DNS, with and without particle impact correction when (a): $Re_t=100$ and (b): $Re_t=1685$ as a function of the Stokes number.

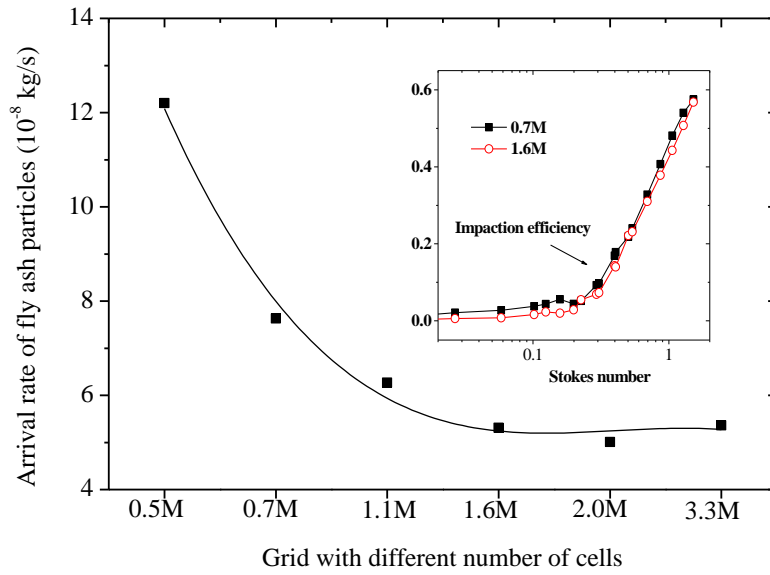


Figure 6. The arrival rate of fly ash particles that impact the probe surface as a function of the number of cells and the impact efficiency of particles as a function of the particle Stokes number for 0.7M and 1.6M.

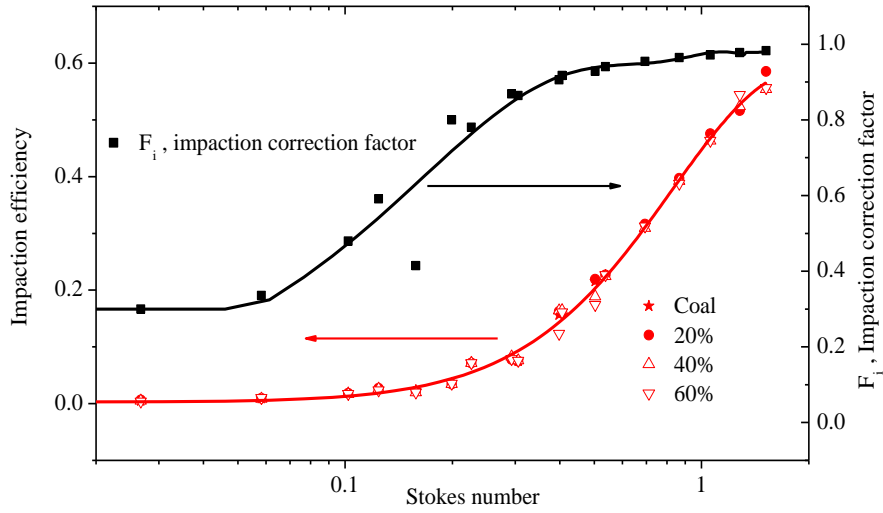


Figure 7. The impactation correction factor and impactation efficiency of particles as a function of the particle Stokes number.

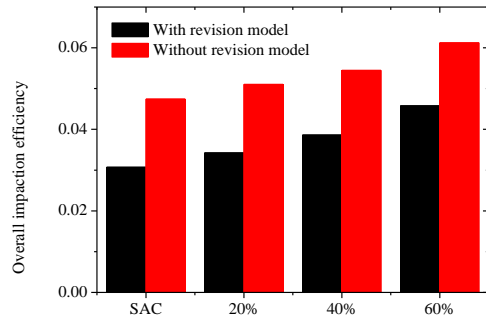


Figure 8. The overall impacton efficiency for SAC and for different levels of PKE with and without the revised particle impaction model.

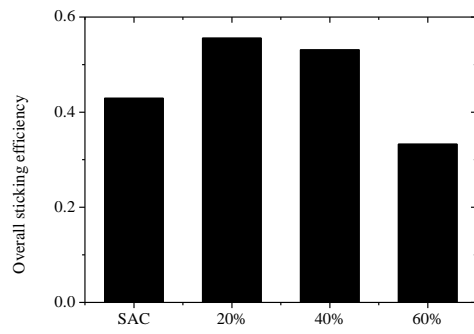


Figure 9. The overall sticking efficiency for SAC and for different levels of PKE.

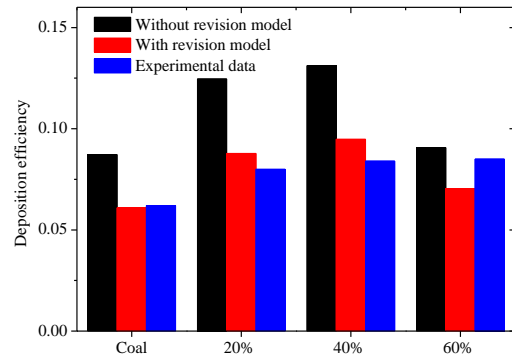


Figure 10. A comparison between the computed and the experimental data of the deposition efficiency for the SAC with different levels of PKE.