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# A Data-Driven Framework for Constructing Representative Operating Cycles of High-Horsepower Tractors from Field-Road Measurements

Rundong Zhou<sup>ab</sup>, Mohammad Nasr Esfahani<sup>c</sup>, Xiaoting Deng<sup>a</sup>, Chao Su<sup>b</sup>, Zhiru Zhou<sup>b</sup>, Xiangfeng Kong<sup>b</sup>, Gracie Harte<sup>c</sup>, James Harrison<sup>c</sup>, Zhixiong Lu<sup>a</sup>

*(a. College of Engineering, Nanjing Agricultural University, Nanjing 210031, China*

*b. College of Locomotive Vehicle, Nanjing Vocational Institute of Railway Technology, Nanjing 210031, China*

*c. School of Physics, Engineering and Technology, University of York, YO10 5DD York, U.K)*

## 1<sup>st</sup> Corresponding Author

Name: Zhixiong Lu

E-mail: [1274375811@qq.com](mailto:1274375811@qq.com)

Postal address: 210031

Telephone number: 13951715780

## 2<sup>nd</sup> Corresponding Author

Name: Xiaoting Deng

F-mail: [2816344964@qq.com](mailto:2816344964@qq.com)

Postal address: 210031

Telephone number: 15895992823

**Abstract:** accurate characterization of real-world operating behaviours of high-horsepower agricultural tractors is a fundamental challenge for subsequent modelling, analysis, and system design. The highly variable and multi-modal nature of field-road operations makes it difficult to construct operating cycles that faithfully represent measured dynamics while preserving statistical distributions and temporal correlation characteristics. This study proposes a data-driven framework for constructing representative operating cycles of high-horsepower tractors based on extensive field-road measurements. The framework integrates optimized K-means clustering guided by the Davies-Bouldin Index with vehicle specific power (VSP) per unit mileage as a physically informed descriptor. A reduced-dimensional feature space is established to characterize measured tractor dynamics, enabling the extraction of representative kinematic fragments from complex operational datasets. These fragments are then synthesized into a unified operating cycle that preserves the intrinsic statistical structure of the measured data. The representativeness of the constructed cycle is quantitatively evaluated using both characteristic parameter deviation and autocorrelation function (ACF) analysis. The average deviation of key kinematic features is 5.28%, while the RMSE between the measured and synthesized ACF curves is only 0.0312, indicating strong consistency in both statistical distribution and temporal continuity. To further assess practical applicability, Cruise-based energy consumption simulations are conducted under three distinct load conditions. The predicted fuel consumption differs from experimental measurements by less than 10%, confirming that the synthesized cycle effectively reproduces real operating behaviour within acceptable measurement

38 uncertainty. The proposed framework provides a systematic approach for measurement-based  
39 representation and validation of operating profiles of high-horsepower agricultural machinery, supporting  
40 further developments in modelling, evaluation, and control of agricultural vehicle systems.  
41 **Key words:** data-driven framework, measurement-based modelling, operating cycle construction, high-  
42 horsepower tractors, field-road measurements

## 43 1. Introduction

44 Accurately representing real-world operating behaviour is a fundamental challenge in  
45 vehicle measurement and data-driven modelling. An operating cycle, commonly expressed as  
46 the temporal evolution of vehicle speed [1], serves not only as a descriptive summary of  
47 measured operation but also as a compact measurement representation of complex dynamic  
48 behaviour. The fidelity of such a representation directly affects the reliability of subsequent  
49 modelling, analysis and system evaluation. However, widely used standardized cycles, such as  
50 NEDC, FTP-75 and WLTC [2, 3], are primarily designed for certification purposes and often  
51 fail to capture the variability and statistical characteristics observed in localized real-world  
52 measurements. As a result, operating behaviours inferred from these cycles may deviate  
53 substantially from measured dynamics, limiting their effectiveness in measurement-based  
54 analysis and evaluation [4].

55 To improve the representativeness of standardized operating cycles, extensive efforts have  
56 been devoted to the development of measurement-based, data-driven cycle construction  
57 methodologies. These studies primarily aim to compensate for the inherent limitations of  
58 predefined test cycles by extracting representative operational patterns directly from measured  
59 vehicle data. In addressing the problem of operational pattern identification, numerous studies  
60 have adopted clustering and feature-space reduction techniques. Zhou et al [5] developed a  
61 UMAP-KNN-based framework for pure electric buses, enabling the extraction of representative  
62 speed-power characteristics from high-dimensional measurement data. Song et al [6] employed  
63 self-organizing map neural networks to incorporate road-gradient information into urban driving  
64 cycles, thereby improving measurement completeness under varying roadway conditions. For  
65 tracked vehicles, Hu et al [7] proposed a three-dimensional working-condition construction  
66 approach based on micro-motion segmentation and K-means clustering, effectively capturing  
67 multi-surface and multi-factor operational variability. Hull et al [8] further employed shape-  
68 based time-series clustering to construct representative operating cycles for auxiliary vehicles in  
69 South Africa, demonstrating the effectiveness of morphology-oriented pattern identification in  
70 measured operational data. Almachi et al [9] established representative urban driving cycles for  
71 Quito by combining geospatial analysis with K-means clustering, highlighting the role of  
72 spatially informed feature extraction in measurement-based cycle construction.

73 To preserve the temporal structure and stochastic characteristics of measured operation,  
74 alternative approaches have focused on micro-segmentation and probabilistic modelling. Li et al  
75 [10] constructed AMT tractor operating cycles using an improved short-stroke method  
76 combined with Markov-chain modelling, while Zhang et al [11] extended state-space  
77 representations to derive multidimensional tram driving cycles. Rueda and Daniel [12] further  
78 utilized multi-city NREL datasets to propose a generalizable micro-trip-based cycle construction

79 method for light-duty vehicles, emphasizing cross-region measurement consistency. Similar  
80 Markov-based and Monte Carlo-enhanced methodologies have been applied to electric vehicles,  
81 motorcycles and bus rapid transit systems, demonstrating improved agreement with measured  
82 operational statistics compared with conventional cycles [13-15].

83 With the availability of long-term naturalistic datasets, recent studies have further  
84 emphasized measurement representativeness and validation. Ashtari et al [16] constructed a  
85 plug-in electric vehicle driving cycle for Winnipeg using large-scale naturalistic data,  
86 illustrating the importance of long-term measurements in capturing real-world operational  
87 variability. Yang et al [17] proposed a big-data-driven framework for heavy-duty truck operating  
88 cycles and reported improved fuel-consumption prediction accuracy relative to existing standard  
89 cycles. Brady et al [18] developed an electric-vehicle driving cycle using six months of real-  
90 world data and demonstrated strong consistency with independent datasets. Bishop et al [19]  
91 quantified variability and accuracy trade-offs in realistic cycle construction through natural-  
92 driving experiments and stochastic modelling. Pouresmaeili et al [20] established a city-specific  
93 passenger-car driving cycle and calibrated emission estimates using on-board measurements,  
94 revealing substantial deviations from standardized European and American cycles.

95 Specific efforts have also been made toward reference working cycles for agricultural  
96 machinery. The development of reference working cycles for agricultural tractors utilized multi-  
97 year onboard measurements, clustering of operational states, and hidden Markov modelling to  
98 synthesize a representative tractor working cycle that better reflected real-world usage,  
99 particularly under low-load operating conditions. While this study highlights the feasibility of  
100 long-term measurement-based reference cycles for tractors, challenges remain in representing  
101 mixed field-road operation and capturing load-dependent variability using limited operational  
102 descriptors [21].

103 Overall, existing approaches for constructing vehicle operating cycles mainly rely on  
104 clustering-based methods, Markov chain models, and intelligent algorithm-assisted techniques  
105 [22-25]. Despite these advances, most existing approaches have been developed for on-road  
106 vehicles operating under relatively well-structured traffic conditions. Agricultural machinery,  
107 particularly high-horsepower tractors, remains largely underrepresented in measurement-  
108 oriented operating cycle research. In contrast to conventional road vehicles, tractors frequently  
109 operate in mixed field-road environments characterized by intermittent high-load events,  
110 frequent speed fluctuations and pronounced stochasticity. These features introduce substantial  
111 challenges for measurement-based representation, as conventional cycle-construction methods  
112 often struggle to preserve both the statistical structure and physical consistency of measured  
113 tractor dynamics. From a measurement perspective, the key challenge lies in constructing an  
114 operating cycle that can reliably represent the multidimensional characteristics of tractor  
115 operation while maintaining consistency with the underlying measured data. Existing methods

116 are not fully suited to capturing the combined effects of load variability, operational mode  
117 switching and non-stationary dynamics that are inherent to agricultural transport tasks. In recent  
118 studies, vehicle specific power (VSP) has been proven to be an effective physical descriptor for  
119 characterizing vehicle dynamic behaviours and energy consumption features, which provides a  
120 reliable basis for data-driven operating cycle construction for heavy-duty and agricultural  
121 equipment [26, 27]. Recent studies have also emphasized the importance of robust feature  
122 extraction and domain generalization for data-driven modelling under noisy and uncertain  
123 operating environments. Improving the adaptability of feature learning and clustering methods  
124 under varying field conditions is essential for ensuring the reliability of operating-cycle  
125 construction from measured tractor data. In related engineering systems, robust multi-scale  
126 feature learning and domain generalization strategies have been shown to improve data  
127 consistency and model reliability under noise disturbance and small-sample conditions [28-30],  
128 which provide useful methodological insights for operating-cycle construction under complex  
129 field-road environments.

130 To address this challenge, this study proposes a data-driven measurement framework for  
131 constructing representative operating cycles of high-horsepower tractors based on extensive  
132 field-road measurements. The proposed framework integrates optimized K-means clustering  
133 guided by the Davies-Bouldin Index (DBI) with vehicle specific power (VSP) per unit mileage  
134 as a physically informed descriptor. Measured kinematic data are first preprocessed and  
135 segmented, and a reduced-dimensional feature space is established to characterize tractor  
136 operating behaviour. Representative kinematic fragments are then identified through optimized  
137 clustering and synthesized into a unified operating cycle that preserves both the statistical  
138 distribution and the intrinsic temporal correlation of the measured data. To quantitatively  
139 evaluate the representativeness of the constructed cycle, characteristic parameter deviation and  
140 autocorrelation function (ACF) analysis are jointly employed, ensuring consistency in both  
141 kinematic features and short-term temporal continuity. Furthermore, a Cruise-based tractor  
142 model was used as an application example to assess fuel consumption under multiple load  
143 conditions. The results demonstrate that the synthesized cycle exhibits strong agreement with  
144 measured operational characteristics and effectively reproduces real-world operating behaviour,  
145 confirming its suitability as a reliable measurement-based representation of high-horsepower  
146 tractor operation.

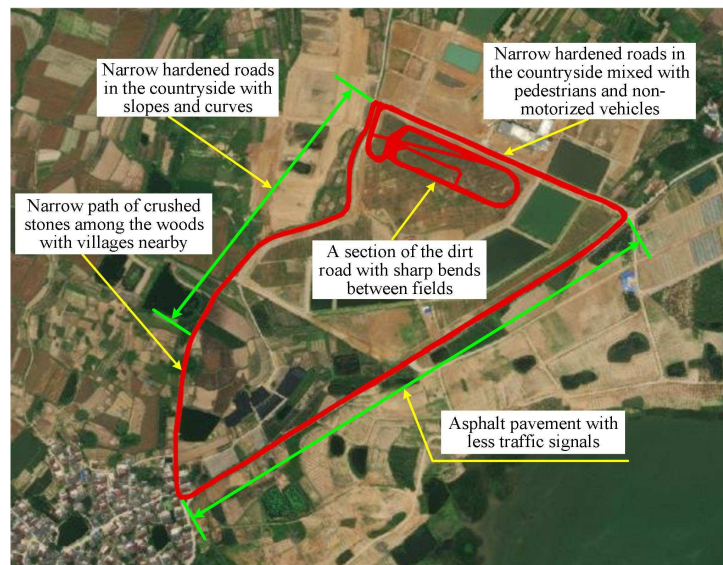
## 147 **2. Measurement data acquisition and kinematic segmentation**

### 148 **2.1 Measurement campaign design**

149 To ensure that the constructed operating cycle reflected the representative transportation  
150 characteristics of high-horsepower tractors in southern hilly regions, the measurement campaign  
151 was designed to capture a wide range of road types and real-world operational scenarios. As

152 shown in **Figure 1**, the Jiangsu Modern Agricultural Machinery Technology Demonstration Park  
153 in Baima Town, Lishui District, Nanjing, and its surrounding areas were selected as the data  
154 collection site due to their diverse terrain and typical rural transportation features.

155 The test route encompassed multiple road surface types, including asphalt and concrete  
156 straight segments ( $\geq 1$  km), S-shaped curves, gravel roads, dirt roads, forest paths, and slopes  
157 with gradients ranging from 5% to 15%. In addition, real-world operational conditions such as  
158 village crossings, narrow road encounters, and mixed-traffic situations were incorporated to  
159 ensure comprehensive coverage of tractor transportation behaviours. This combination of varied  
160 terrains and authentic agricultural road environments provided a robust data set for subsequent  
161 operating cycle construction and analysis.



162  
163 **Figure 1.** Data collection route encompassing representative road types relevant to tractor  
164 transportation operations

165 To ensure data representativeness while controlling testing duration, cost, and experimental  
166 constraints, the data collection strategy was designed based on typical tractor transportation  
167 scenarios, including agricultural supply delivery and product transfer. Three data collection  
168 methods were adopted.

169 (1) Route-based cycle testing. A predefined line-cycle test was conducted on major  
170 transportation routes to cover different road surfaces, including asphalt pavement, concrete  
171 pavement, gravel roads, and dirt roads. To reflect different operating load levels, tests were  
172 carried out under three loading conditions: 0%, 50%, and 100% of the trailer's rated capacity  
173 (i.e., empty-load, half-load, and full-load). The "multi-driver random rotation method" was used  
174 to reduce individual driving bias and improve data reliability.

175 (2) Hybrid tracking on typical segments. On representative rural road sections, such as  
176 village-to-field gravel roads and asphalt connector roads, tractors and light agricultural vehicles  
177 (e.g., tricycles and mini-trucks) were tracked using a tractor-agricultural vehicle hybrid tracking

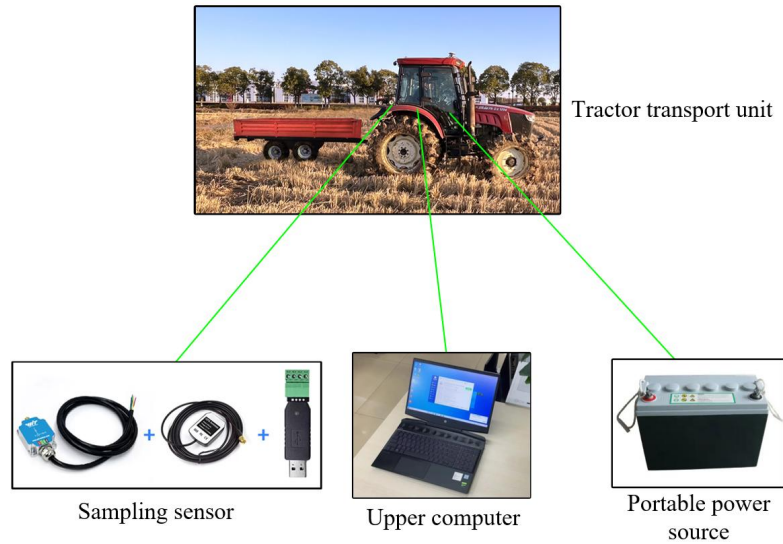
178 approach. This method was used to establish comparable reference datasets under typical field-  
179 road transport operating cycles.

180 (3) Naturalistic driver operation. During specific time periods and on narrow road sections  
181 (approximately 3.5 m wide), data were collected using a naturalistic driving approach. Drivers  
182 operated the tractor according to their own driving habits and route preferences without  
183 additional intervention, allowing the acquisition of realistic operating behaviour data.

## 184 2.2 Measurement system and sensors

185 To ensure that the collected data accurately reflected the operational characteristics of  
186 tractors in the local test environment, factors such as trailer self-weight, road surface type,  
187 terrain slope, and installation feasibility were comprehensively considered when selecting the  
188 data acquisition equipment. Accordingly, the WTGAHRS1 inertial navigation sensor (Weite  
189 Co., Ltd.) was chosen for vehicle speed measurement. This sensor integrated high-precision  
190 GPS+BeiDou dual-navigation systems, providing a speed measurement accuracy better than 0.1  
191  $\text{m s}^{-1}$ , a positioning accuracy within 2.5 m, and an output rate adjustable from 0.2 to 200 Hz.

192 In addition to real-time recording, the sensor simultaneously measured auxiliary  
193 parameters including acceleration, angular velocity, attitude angle, magnetic field strength,  
194 barometric pressure, altitude, and geographic coordinates. These supplementary signals  
195 supported data screening and subsequent feature extraction. Considering that high-horsepower  
196 tractors under mixed field-road transportation operating cycles generally operate at relatively  
197 low speeds and exhibit smoother dynamic transitions than highway vehicles, the sampling  
198 frequency was set to 1 Hz through the upper-computer interface to capture key kinematic states  
199 such as acceleration, deceleration, idling, and steady driving. Meanwhile, this frequency  
200 effectively reduced data redundancy and computational burden during subsequent feature  
201 extraction and operating cycle construction, achieving a balance between data resolution and  
202 processing efficiency. The configuration of the sampling equipment system is shown in figure 2.



203

204 **Figure 2.** Experimental data acquisition system for tractor transportation tests, consisting  
 205 of the tractor-trailer unit, a sampling sensor module (including speed, position, and dynamic  
 206 signal collectors), an upper-computer terminal for real-time data monitoring and storage, and a  
 207 portable power supply to ensure stable operation during field-road measurements.

## 208 2.3 Data preprocessing and kinematic segmentation

### 209 2.3.1 Preprocessing of measured data

210 Field-road operating data were collected during the typical autumn harvest season, which  
 211 represented the busiest and most representative transportation period for high-horsepower  
 212 tractors in southern hilly agricultural regions. During this period, tractors frequently performed  
 213 mixed field-road transport tasks, including grain hauling, field transfer, and agricultural product  
 214 transportation, resulting in high operational intensity and diverse load conditions. Compared  
 215 with field tillage operations, transport operating conditions were less sensitive to seasonal  
 216 meteorological variations, and their core kinematic characteristics, such as speed distribution,  
 217 idle proportion, acceleration frequency, and speed fluctuation patterns, remained relatively  
 218 stable across typical working periods. Therefore, the autumn harvest season provided sufficient  
 219 statistical representativeness for constructing standard tractor transport operating cycles.

220 After approximately one month of continuous measurement, more than 100 sets of  
 221 working-condition data with over 200,000 valid entries were obtained. During the data  
 222 collection and transmission process, individual missing values, outliers, and abnormal speed  
 223 fluctuations caused by road-surface interference were observed. Therefore, data preprocessing  
 224 was required to eliminate errors and improve data reliability as follows:

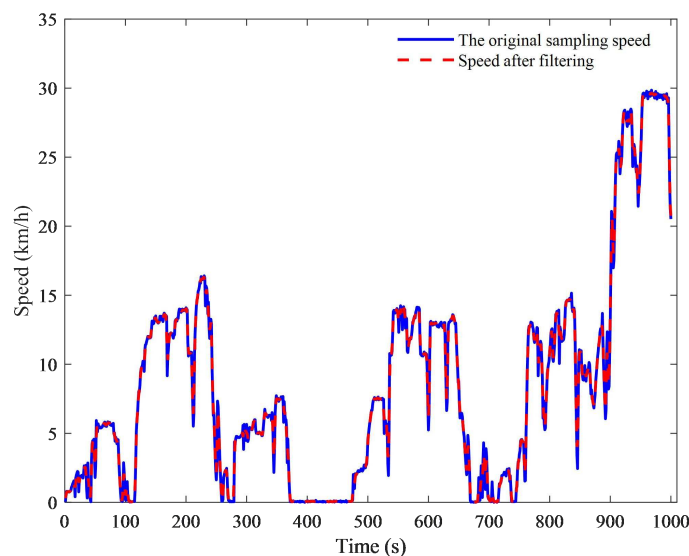
225 (1) Handling missing values: Linear interpolation was performed when the number of  
 226 consecutive missing values between two adjacent data points was fewer than 5. When the  
 227 number of missing values exceeded 5, the corresponding data segment was removed to avoid

228 excessive interpolation errors and distortion of the original driving characteristics. Since a high-  
229 precision GPS+BeiDou dual-navigation sensor was mounted on the roof, no significant missing  
230 data occurred during the test.

231 (2) Handling excessively long idle time: If the idle time exceeded 180 s, it was considered  
232 that the tractor had entered a long-term parking state rather than normal operational idling.  
233 Therefore, only 180 s of idle time was retained [31].

234 (3) Removal of abnormal speed and acceleration data: The maximum travelling speed of  
235 the tractor does not exceed 30 km h<sup>-1</sup>. Therefore, according to the national standard GB/T  
236 33195-2016, data points with speeds exceeding 30 km/h, as well as outliers with accelerations  
237 greater than 2 m s<sup>-2</sup> or decelerations less than -3.5 m s<sup>-2</sup>, were removed to ensure the accuracy  
238 and reliability of the data.

239 A moving-average filter with a window size of 3 samples (equivalent to 3 s at a sampling  
240 frequency of 1 Hz) was applied to suppress high-frequency noise induced by road-surface  
241 irregularities. As shown in figure 3, the filtered speed profile preserves the trend of the original  
242 data while effectively removing high-frequency fluctuations, providing a more accurate  
243 representation of the tractor's actual operating speed.



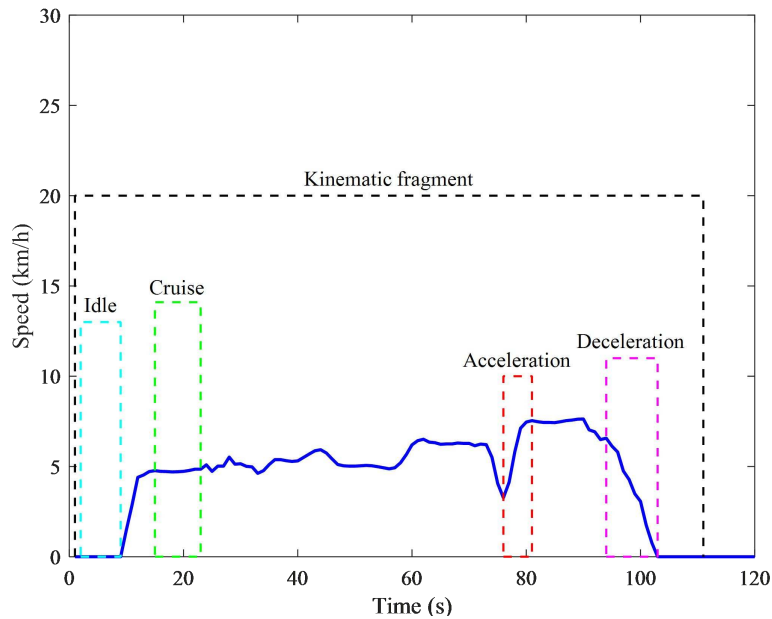
244

245

**Figure 3.** Comparison of raw speed data and moving-average filtered speed data

### 246 2.3.2 Kinematic segmentation

247 An idle segment is defined as the period from vehicle stop to the subsequent restart, while  
248 a kinematic segment refers to the driving process between two consecutive idle states. As shown  
249 in figure 4, a typical kinematic fragment contains four motion states, namely the acceleration  
250 state, the deceleration state, the uniform state and the idle state. Considering the operational  
251 efficiency and mechanical reliability of tractors, while taking into account the health of drivers  
252 and environmental protection requirements, the total duration of the operating conditions  
253 constructed in this study is 2,400 seconds.



254  
255 **Figure 4.** Kinematic fragment of tractor

256 In the construction of tractor transportation operating cycles, the speed-time curve is  
257 usually composed of several segments with specific kinematic characteristics. Due to  
258 differences in factors such as traffic environment, road conditions and time dimension,  
259 kinematic fragments show significant heterogeneity. The presence of short-term abnormal data  
260 and invalid fragments in the kinematic segment obtained from the initial division may lead to  
261 deviations in the calculation of characteristic parameters. To this end, a three-level kinematic  
262 fragment data screening mechanism for tractor was established:

263 (1) Integrity screening: When the interval between adjacent valid data points exceeds 180  
264 seconds or the data loss rate exceeds 10%, it is determined as an incomplete fragment and  
265 eliminated.

266 (2) Duration screening: Eliminate abnormal segments with a duration of less than 30  
267 seconds or more than 600 seconds.

268 (3) Motion state screening: According to the standard definition of GB/T 38146.2-2019,  
269 strictly distinguish: For the segments that do not meet the above definitions under acceleration  
270 conditions ( $a \geq 0.15 \text{ m s}^{-2}$ ), deceleration conditions ( $a \leq -0.15 \text{ m s}^{-2}$ ), uniform conditions  
271 ( $|a| < 0.15 \text{ m s}^{-2}$  and  $v \geq 0.5 \text{ km h}^{-1}$ ), and idle conditions ( $a < 0.15 \text{ m s}^{-2}$  and  $v < 0.5 \text{ km h}^{-1}$ ),  
272 secondary filtering is carried out.

273 After three levels of screening, the invalid fragments in the original data set were  
274 effectively eliminated, and finally 349 effective kinematic fragments of tractor were obtained.

## 275 2.4 Selection and calculation of kinematic fragment characteristic parameters

### 276 2.4.1 Selection of characteristic parameters

277 The construction of tractor operating conditions is essentially a feature-driven pattern

278 recognition process. Representative kinematic fragments are selected from a classified fragment  
 279 library based on optimized criteria, and a complete operating cycle is formed through time-  
 280 series recombination. To mathematically describe each fragment and analyse its dynamic  
 281 behaviour, a multidimensional characteristic parameter system must be established.

282 Analysis of the measured field-road transport data indicates that tractor operations exhibit  
 283 distinct dynamic features: a start-stop frequency of 5-8 events per minute, a running-speed  
 284 standard deviation exceeding  $2.5 \text{ m s}^{-1}$ , speed fluctuation ranges approaching 40% of the  
 285 nominal speed, and peak acceleration-rate changes above  $0.8 \text{ m s}^{-2}$  during obstacle-avoidance  
 286 maneuvers. Considering these characteristics: low-frequency, high-amplitude velocity  
 287 fluctuations and frequent start-stop transitions, a total of 20 characteristic parameters selected in  
 288 this study are shown in **table 1** and divided into two major categories:

289 (1) Motion-state descriptive parameters.

290 These include average speed (with and without idle periods), average acceleration,  
 291 maximum speed, speed standard deviation, maximum acceleration, maximum deceleration, and  
 292 other indicators used to describe the dynamic behaviour within each fragment.

293 (2) Statistical distribution parameters.

294 These parameters characterize the temporal distribution of operating states, including idle-  
 295 time proportion, speed-interval proportions, acceleration distribution, and related indicators.  
 296 They capture the statistical patterns of fragment behaviour and serve as key metrics for  
 297 validating the representativeness and effectiveness of the constructed tractor transportation  
 298 operating cycle.

299 **Table 1.** Kinematic fragment parameters of tractor

Number	Characteristic parameters	Symbol	Unit
1	The duration of the kinematic fragment	$T_p$	s
2	Acceleration time	$T_a$	s
3	Deceleration time	$T_d$	s
4	Uniform time	$T_c$	s
5	Idle time	$T_i$	s
6	Average speed (including idle speed)	$V_m$	$\text{km h}^{-1}$
7	Average driving speed (excluding idle speed)	$V_{mr}$	$\text{km h}^{-1}$
8	Maximum speed	$V_{\max}$	$\text{km h}^{-1}$

Number	Characteristic parameters	Symbol	Unit
9	Standard deviation of velocity	$V_{sd}$	km h <sup>-1</sup>
10	Average acceleration	$A_m$	m s <sup>-2</sup>
11	Average deceleration	$D_m$	m s <sup>-2</sup>
12	Standard deviation of acceleration	$A_{sd}$	m s <sup>-2</sup>
13	Deceleration standard deviation	$D_{sd}$	m s <sup>-2</sup>
14	Acceleration time ratio	$P_a$	%
15	Deceleration time ratio	$P_d$	%
16	Uniform time ratio	$P_c$	%
17	Idle time ratio	$P_i$	%
18	The proportion of 0-10 km h <sup>-1</sup>	$P_{0-10}$	%
19	The proportion of 10-20 km h <sup>-1</sup>	$P_{10-20}$	%
20	The proportion of 20-30 km h <sup>-1</sup>	$P_{20-30}$	%

#### 300 2.4.2 Calculation of characteristic parameters

301 Based on the selection results in Section 2.4.1, the 20 characteristic parameters selected in  
302 this study are defined as follows:

303 (1) The duration of the kinematic fragment  $T_p$ , acceleration time  $T_a$ , deceleration time  
304  $T_d$ , uniform speed time  $T_c$ , idle time  $T_i$ .

$$305 T_p = \text{The duration of the kinematic fragment} \quad (1)$$

$$306 T_a = \text{The total duration of } A_i \geq 0.15 \text{ in this kinematic fragment} \quad (2)$$

$$307 T_d = \text{The total duration of } D_i \leq -0.15 \text{ in this kinematic fragment} \quad (3)$$

$$308 T_c = \text{The total duration of } abs(A_i) < 0.15 \text{ and } V_i \geq 0.5 \text{ in this kinematic fragment} \quad (4)$$

$$309 T_i = \text{The total duration of } abs(A_i) < 0.15 \text{ and } V_i \leq 0.5 \text{ in this kinematic fragment} \quad (5)$$

310 (2) Average speed (including idle speed)  $V_m$ , average driving speed (excluding idle speed)

311  $V_{mr}$ , maximum speed  $V_{max}$ , speed standard deviation  $V_{sd}$ .

312 
$$V_m = \frac{S}{T_p} \quad (6)$$

313 
$$V_{mr} = \frac{S}{T - T_i} \quad (7)$$

314 
$$V_{\max} = \max\{V_i \sim V_n\} \quad (8)$$

315 
$$V_{sd} = \sqrt{\frac{1}{n-1} \sum_{i=1}^n (V_i - V_m)^2}, i = 1, 2, \dots, n \quad (9)$$

316 Where,  $S$  represents the running distance of this kinematic segment,  $V_i$  indicates the  
 317 velocity at the  $i$ -th second, in km/h, and  $n$  represents the total number of data points in this  
 318 kinematic segment.

319 (3) Average acceleration  $A_m$ , average deceleration  $D_m$ , standard deviation of acceleration  
 320  $A_{sd}$ , standard deviation of deceleration  $D_{sd}$ .

321 
$$A_m = \frac{\text{sum}\{A_i \mid A_i \geq 0.15, i = 1, 2, \dots, n-1\}}{T_a} \quad (10)$$

322 
$$D_m = \frac{\text{sum}\{A_i \mid A_i \leq -0.15, i = 1, 2, \dots, n-1\}}{T_d} \quad (11)$$

323 
$$A_{sd} = \sqrt{\frac{1}{n-1} \sum_{i=1}^n (A_i - A_m)^2}, i = 1, 2, \dots, n \quad (12)$$

324 
$$D_{sd} = \sqrt{\frac{1}{n-1} \sum_{i=1}^n (D_i - D_m)^2}, i = 1, 2, \dots, n \quad (13)$$

325 Where,  $A_i$  and  $D_i$  represent the acceleration and deceleration of the segment in the  $i$ -th  
 326 second, respectively, in  $\text{m} \cdot \text{s}^{-2}$ . The specific calculation formula is shown in formula (14).

327 
$$A_i(D_i) = \begin{cases} 0 & i = 1 \\ \left( \frac{V_{i+1} - V_{i-1}}{2} \right) / 3.6 & i = 2, 3, \dots, n-1 \\ 0 & i = n \end{cases} \quad (14)$$

328 (4) Acceleration time ratio  $P_a$ , deceleration time ratio  $P_d$ , constant speed time ratio  $P_c$ ,  
 329 idle time ratio  $P_i$ .

330 
$$P_a = \frac{T_a}{T_p} \quad (15)$$

331 
$$P_d = \frac{T_d}{T_p} \quad (16)$$

332 
$$P_c = \frac{T_c}{T_p} \quad (17)$$

333 
$$P_i = \frac{T_i}{T_p} \quad (18)$$

334 (5) The proportion of 0-10 km h<sup>-1</sup>  $P_{0-10}$ , that of 10-20 km h<sup>-1</sup>  $P_{10-20}$ , and that of 20-30 km  
 335 h<sup>-1</sup>  $P_{20-30}$ .

336 
$$P_{0-10} = \frac{T_{0-10}}{T_p} \quad (19)$$

337 
$$P_{10-20} = \frac{T_{10-20}}{T_p} \quad (20)$$

338 
$$P_{20-30} = \frac{T_{20-30}}{T_p} \quad (21)$$

339 Where,  $T_{0-10}$ ,  $T_{10-20}$ , and  $T_{20-30}$  respectively represent the total number of velocities  
 340 belonging to [0, 10), [10, 20), [20, 30] in this segment.

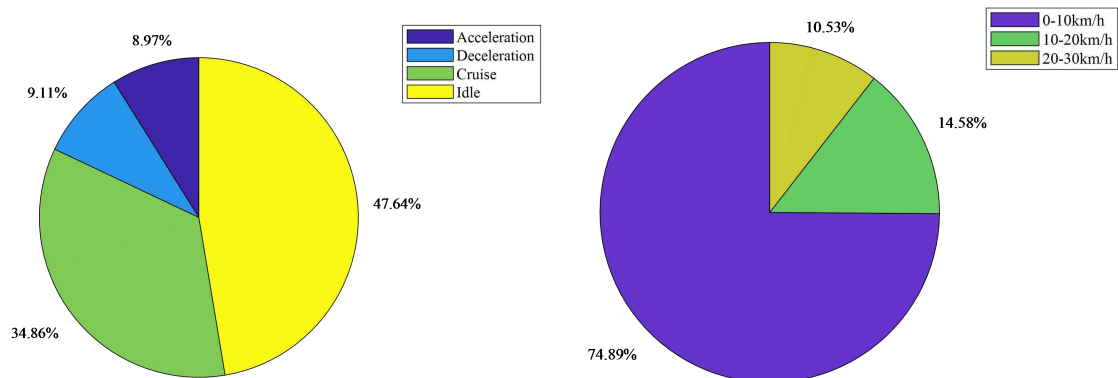
341 Based on the above formulas, the 349×20-dimensional kinematic fragment characteristic  
 342 parameter value matrix Z of the tractor transportation condition was obtained, as shown in table  
 343 2.

344 **Table 2.** Kinematic Fragment Characteristic Parameter Matrix Z

Number	$T_p$	$T_a$	$T_d$	$T_c$	$T_i$	$V_m$	$V_{mr}$	$V_{max}$	...	$P_{10-20}$	$P_{20-30}$
1	105	10	14	77	5	2.86	2.92	5.77	...	0	0
2	165	22	32	100	12	9.97	10.61	16.32	...	0.66	0
3	102	7	9	78	9	4.91	5.32	7.63	...	0	0
4	298	19	33	144	103	5.94	9.01	14.09	...	0.35	0
5	317	59	69	182	8	14.57	14.90	29.64	...	0.31	0.3
6	182	24	28	115	16	10.70	11.66	16.95	...	0.62	0
7	567	58	69	317	124	7.39	9.43	23.63	...	0.39	0.02
8	66	6	10	32	19	6.00	8.20	10.77	...	0.35	0
...	...	...	...	...	...	...	...	...	...	...	...
348	639	61	60	367	152	15.07	19.29	30	...	0.18	0.44
349	338	38	43	69	189	6.97	14.77	29.6	...	0.09	0.18

345 Meanwhile, the overall operational characteristics of the processed test sample data were  
 346 calculated, and the corresponding statistical pie chart. Figure 5 shows that the overall  
 347 characteristics of the tractor's operating conditions are as follows: the proportion of idle speed  
 348 and constant speed is relatively high, and the proportion of the medium and low speed range of

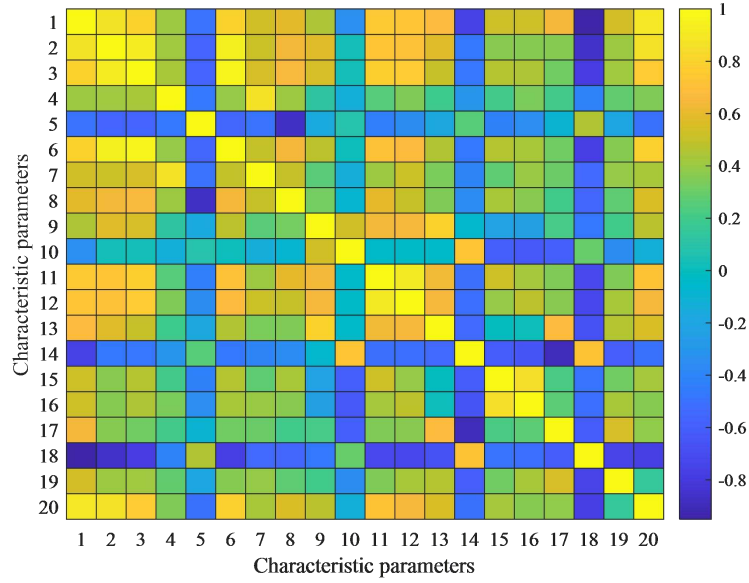
349 0-10 km h<sup>-1</sup> is also large. This reflects the transportation operation characteristics of the tractor,  
 350 such as long waiting time during the transfer operation and continuous driving at a stable speed.



351 **Figure 5.** Overall velocity characteristic distribution of the test data

### 352 3. Measurement-based synthesis of operating cycles

353 As shown in **figure 6**, based on the heat map analysis of the Pearson correlation coefficient  
 354 matrix, the interaction relationships among 20 characteristic parameters can be obtained. The  
 355 diagonal elements of this matrix represent the autocorrelation of parameters (coefficient  $r=1$ ),  
 356 while the off-diagonal elements quantify the linear correlation strength between parameter pairs.  
 357 **A significant negative correlation was shown among them.** Among time-domain parameters: the  
 358 average speed ( $V_m$ ) shows a strong positive correlation with the maximum travelling speed  
 359 ( $V_{max}$ ) ( $r=0.9083$ ). The statistical distribution parameters show modular characteristics: the  
 360 proportion of idle time ( $P_i$ ) and the standard deviation of speed ( $V_{sd}$ ) form a negative  
 361 correlation cluster ( $r=-0.4451$ ); The dynamic characteristic parameters have independent  
 362 information dimensions: the correlation coefficient between the standard deviation of  
 363 deceleration ( $D_{sd}$ ) and most other characteristic parameters **is** less than 0.35. Therefore, it is  
 364 necessary to use principal component analysis to merge the characteristic parameters with large  
 365 redundancy while retaining the independent parameters.



366

367

**Figure 6.** Heat map of the Pearson correlation coefficient matrix

368

### 3.1 Principal component analysis

369

#### 3.1.1 PCA-based feature matrix construction

370

The sample contained 349 kinematic fragment data, and there are 20 characteristic parameters used to evaluate the sample data. PCA [32, 33] was employed to reduce redundancy among the 20 characteristic parameters while preserving the principal variance information of the original data set. Therefore, 349×20-dimensional matrix  $A$  can be obtained:

371

372

373

$$A = \begin{bmatrix} a_{11} & a_{12} & \cdots & a_{1,20} \\ a_{21} & a_{22} & \cdots & a_{2,20} \\ \vdots & \vdots & \ddots & \vdots \\ a_{349,1} & a_{349,2} & \cdots & a_{349,20} \end{bmatrix} \quad (22)$$

374

375

#### 3.1.2 Principal component selection based on variance contribution

376

Based on PCA, the original 20-dimensional characteristic parameter matrix was reduced while preserving the dominant variance information of the datasets. The variance contribution rate and cumulative contribution rate of each principal component were then calculated, and the results are presented in table 3.

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**Table 3.** Principal component variance contribution rate and cumulative contribution rate

Principal component	Principal component variance	Variance contribution rate	Cumulative contribution rate
1	10.326	51.629%	51.629%
2	3.013	15.064%	66.693%
3	2.017	10.087%	76.780%
4	1.394	6.969%	83.749%
5	0.869	4.347%	88.096%
6	0.765	3.822%	91.918%

7	0.693	3.467%	95.385%
8	0.287	1.436%	96.821%
9	0.167	0.835%	97.656%
10	0.117	0.586%	98.242%
11	0.106	0.532%	98.774%
12	0.096	0.481%	99.255%
13	0.077	0.385%	99.640%
14	0.038	0.191%	99.831%
15	0.016	0.083%	99.914%
16	0.012	0.061%	99.975%
17	4.844E-3	0.024%	99.999%
18	6.444E-5	3.222E-4	99.999E-4
19	1.012E-5	5.063E-5	99.999E-5
20	3.489E-32	1.721E-31	100%

381 It can be seen from table 3 that the first 7 principal components can explain 95.385% of the  
382 information of the original variable. To balance the information retention degree and the model  
383 complexity, they were selected as the effective feature subsets. Therefore, in order to further  
384 conduct cluster analysis, the principal component score matrix of the original sample data is  
385 shown in table 4.

386 **Table 4.** The principal component score matrix of the original sample

Number	PCA1	PCA2	PCA3	PCA4	PCA5	PCA6	PCA7
1	-1.460	-3.297	1.258	0.709	-0.408	-0.976	-1.772
2	2.988	-3.397	0.923	1.360	0.972	1.853	0.613
3	-1.008	-2.742	0.848	1.950	-1.451	-0.986	-1.538
4	1.435	-0.404	0.572	2.960	0.936	0.714	-0.609
5	6.679	-1.412	-0.121	0.1783	1.700	0.233	-2.016
6	3.335	-2.886	0.653	2.075	0.245	1.826	0.379
7	4.449	1.317	2.0127	1.489	1.672	2.357	-2.937
8	0.460	-2.918	-0.814	2.482	-0.400	0.598	0.318
...	...	...	...	...	...	...	...
348	5.983	3.201	2.675	-1.65	-0.006	0.128	-1.457
349	1.716	1.798	-1.075	-0.898	1.618	-0.143	-0.170

### 387 3.2 Clustering of kinematic fragments using DBI-optimised K-means

#### 388 3.2.1 K-means clustering framework

389 In this study, the K-means clustering method [34] is used to study the test data of tractor  
390 transportation operating cycles. The core idea of the method is to assign each sample point to  
391 the category where the nearest mean (centroid) is located. The specific calculation steps are as  
392 follows:

- 393 (1) Initialize the clustering centres and randomly select  $n$  data points as the initial

394 clustering centres.

395 (2) For each sample  $x_i = (x_{i1}, x_{i2}, \dots, x_{im})$ , calculate its distance to all centroids  
396  $c_n = (c_{n1}, c_{n2}, \dots, c_{nm})$  (usually using Euclidean distance), and assign it to the cluster  
397 corresponding to the nearest centroid. The calculation formula is as follows:

$$398 \quad d(x_i, c_n) = \sqrt{\sum_{j=1}^m (x_{ij} - c_{nj})^2} \quad (23)$$

399 Where, m represents the feature dimension;

400 (3) Update the cluster centre: For all samples in each cluster, calculate the mean of each  
401 feature dimension and take this mean as the new centroid.

$$402 \quad c_n^{(new)} = \frac{1}{/S_n/} \sum_{x_i \in S_n} x_i \quad (24)$$

403 Where,  $/S_n/$  represents the sample size of the i-th cluster;

404 (4) Iterate until convergence. Repeat steps 2 and 3 until one of the following conditions is  
405 met: the centroid change is less than the preset threshold (e.g.,  $\|c_n^{(new)} - c_n^{(old)}\| < \epsilon$ ,  $\epsilon$  is the set  
406 threshold), and the maximum number of iterations is reached or the cluster allocation result no  
407 longer changes.

### 408 3.2.2 Determination of optimal cluster number using DBI

409 To determine the optimal number of clusters, this study uses the DBI to evaluate the  
410 clustering results. The clustering quality is assessed by calculating the ratio of the intra-cluster  
411 distance to the inter-cluster distance. The smaller the DBI value, the better the clustering effect  
412 [35, 36]. Its calculation method is as shown in equation (25):

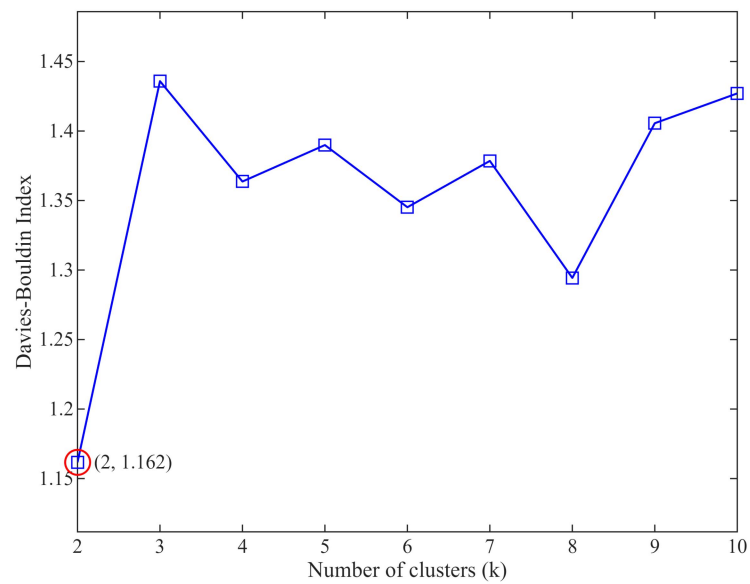
$$413 \quad DBI = \frac{1}{k} \sum_{i=1}^k \max \frac{S(\mu_i) + S(\mu_j)}{M(\mu_i, \mu_j)}, \quad i \neq j \quad (25)$$

414 Where, k represents the number of clusters, and  $S(\mu_i)$  is the average Euclidean distance  
415 from all points in cluster  $i$  to its clustering centre  $\mu_i$ .  $S(\mu_j)$  is the average Euclidean distance  
416 from all points in cluster  $j$  to its clustering centre  $\mu_j$ .  $M(\mu_i, \mu_j)$  represents the Euclidean  
417 distance between cluster centre  $\mu_i$  and cluster centre  $\mu_j$ .

418 As shown in figure 7, the DBI of the sample reaches its minimum value of 1.162 when  $k=2$ ,  
419 and then increases with the increase of k, indicating that the clustering quality is optimal at this  
420 point. To further improve the robustness of cluster number determination, the Silhouette  
421 coefficient was additionally introduced as an auxiliary validation index. As shown in figure 8,  
422 the Silhouette analysis [37] also indicates that the optimal number of clusters is  $k=2$ , where the  
423 Silhouette coefficient reaches its maximum value of 0.493. The consistency of these two

424 independent validity indices confirms the robustness and reliability of the selected clustering  
 425 result. Therefore, the kinematic fragments of the tractor are classified into two representative  
 426 categories.

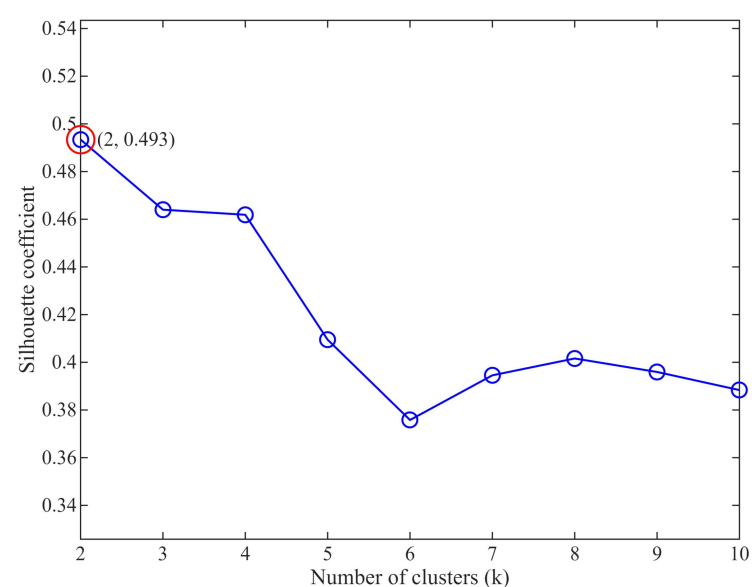
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428

**Figure 7. DBI versus number of clusters**

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430

**Figure 8. Silhouette coefficient versus number of clusters**

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The characteristic values of various fragment samples are shown in table 5. The sample numbers of the two types of fragments are 178 and 171 respectively, and the sampling distribution is relatively balanced. This indicates that both types of scenes are important in actual transportation, and the characteristic parameters between the categories are significantly different. The characteristics of category 1 are as follows: a relatively high average driving speed (14.563 km h<sup>-1</sup>), a high proportion of constant speed (43%), strong short-term acceleration ability (20-30 km h<sup>-1</sup> accounting for 21%), and a moderate proportion of idle speed (34%). These features are in line with the traffic conditions of county roads or rural roads and are

439 suitable for short-distance transportation scenarios in urban and rural areas. When in this  
 440 category, the tractor can maintain stable driving on sections with fewer traffic signals. However,  
 441 due to the interference of intersections and pedestrians, it still needs to stop intermittently,  
 442 which makes the speed of 0-10 km h<sup>-1</sup> account for 55.8%. The overall characteristics indicate  
 443 medium and low-speed mixed transportation scenarios.

444 Category 2 is mainly characterized by: the main speed is low (0-10 km/h accounts for  
 445 94.1%), with occasional demands for medium and high speeds (5.6% for medium speed and  
 446 0.3% for high speed). The proportion of idle speed is relatively high (61.3%), and the  
 447 acceleration and deceleration behaviours are similar (6.1% for acceleration and 6.5% for  
 448 deceleration). These characteristics fully reflect the frequent start-stop and loading-unloading  
 449 operation requirements of the field-road transportation operating cycles and the stability control  
 450 requirements when towing heavy equipment, which precisely match the scenarios of internal  
 451 field transportation and rural road transportation. Meanwhile, the average driving speed of  
 452 category 2 (5.429 km h<sup>-1</sup>) is slightly higher than the actual working conditions in the field  
 453 (usually less than 5 km/h), but the proportion of low speed and idle speed can still effectively  
 454 characterize the typical features of the transportation operation scenarios of high-horsepower  
 455 tractor.

456 **Table 5.** The eigenvalues of various optimized fragment samples

Category	$V_m$	$V_{mr}$	$P_a$	$P_d$	$P_c$	$P_i$	...	$P_{0-10}$	$P_{10-20}$	$P_{20-30}$	Number of fragments
1	9.928	14.563	0.116	0.114	0.430	0.340	...	0.558	0.232	0.210	178
2	2.342	5.429	0.061	0.065	0.261	0.613	...	0.941	0.056	0.003	171

### 457 3.3 Synthesis of representative operating cycle based on VSP-informed selection criteria

458 To achieve a representative measurement-based operating cycle, a synthesis strategy is  
 459 proposed by jointly considering eigenvalue-based feature deviation and VSP per unit mileage.  
 460 The overall procedure of operating-cycle synthesis is illustrated in figure 9. To balance  
 461 measurement representativeness, computational efficiency and practical applicability, the total  
 462 duration of the synthesized operating cycle is set to 2400 s. This duration is sufficient to capture  
 463 the dominant operating modes observed in the field-road measurement data set while avoiding  
 464 excessive redundancy in the reconstructed cycle.

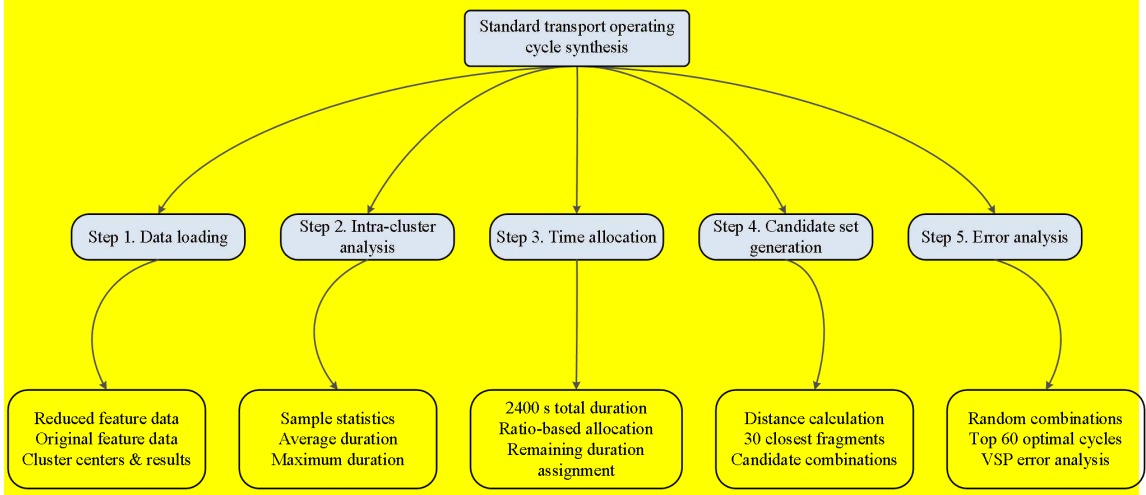


Figure 9. Stepwise workflow of operating cycle synthesis

The duration allocation for each category is derived from formula (26):

$$T_t = \frac{t_t}{T} \times 2400 \quad (t = 1, 2) \quad (26)$$

Where,  $T_t$  represents the time occupied by the  $t$ -th type kinematic fragment in the final synthetic operating condition curve;  $t_t$  represents the total duration of all kinematic segments of Class  $t$ , and  $T$  indicates the total duration of all kinematic segments.

VSP [38] provides a more physically meaningful representation of vehicle operating load by jointly considering speed, acceleration, slope, and resistance factors. Its unit is  $\text{kW t}^{-1}$ . According to the national standard “Technical Conditions for the Operation Safety of Motor Vehicles” (GB7258-2017), the specific power of the tractor transport unit must be greater than or equal to  $4.0 \text{ kW t}^{-1}$ . The calculation formula for VSP is:

$$\text{VSP} = v \times [a(1 + \varepsilon) + g \times \text{grade} + C_R g] + \frac{\rho_a C_D A_s (v_w + v)^2 \times v}{2M} \quad (27)$$

Where,  $v$  represents the speed of the tractor,  $\text{m s}^{-1}$ ,  $a$  is the acceleration of the tractor,  $\text{m s}^{-2}$ ,  $\varepsilon$  is the mass factor, 0 for heavy-duty vehicles,  $g$  is the acceleration due to gravity,  $9.8 \text{ m s}^{-2}$ ,  $\text{grade}$  is the slope, 0.06,  $C_R$  is the rotational resistance coefficient of the tire, 0.03,  $\rho_a$  is the air density,  $1.2 \text{ kg m}^{-3}$ ,  $C_D$  is the wind resistance coefficient, 0.7.  $A_s$  represents the windward area,  $6.2 \text{ m}^2$ .  $v_w$  represents the windward speed of the tractor,  $\text{m s}^{-1}$ . Since the maximum transport speed of the tractor in this study does not exceed  $30 \text{ km/h}$  and aerodynamic resistance contributes much less to the total tractive demand than rolling resistance, grade resistance, and inertial resistance,  $v_w = 0$ .  $M$  represents the total mass of the tractor and transport unit,  $12,375 \text{ kg}$ .

Based on formula (28), 60 candidate operating condition curves that meet the requirements of operating condition time length and the maximum VSP constraint are screened out.

$$\begin{cases}
E = \sum_{n=1}^{20} \left( \frac{|\bar{F}_n^{syn} - \bar{F}_n^{real}|}{\bar{F}_n^{real}} \times 100 \right) \\
C_{60} \in \arg \text{top} E_{60} \\
s.t. \\
\sum_{i=1}^{N_1} T_{1i} + \sum_{j=1}^{N_2} T_{2j} = 2400 \pm 20 \\
VSP(C_{60})_{\max} \geq 4
\end{cases} \quad (28)$$

489 Where,  $E$  represents the sum of the average relative errors of all feature dimensions,  $\bar{F}_n^{syn}$   
490 is the average value of the synthetic working condition on the n-th feature, and  $\bar{F}_n^{real}$  is the  
491 overall average value of the original data on the n-th feature.  $\sum_{i=1}^{N_1} T_{1i}$  is the total time of the first  
492 type of kinematic segments ( $N_1$ ) in the synthetic condition, and  $\sum_{j=1}^{N_2} T_{2j}$  is the total time of the  
493 second type of kinematic segments ( $N_2$ ) in the synthetic condition. The sum of the two satisfies  
494 the constraint of a total duration of  $2400 \pm 20$  s.  $C_{60}$  represents 60 candidate working conditions,  
495 and  $\arg \text{top} E_{60}$  indicates the top 60 combinations with the smallest error in taking the  
496 eigenvalues.  
497

498 In this study, VSP per unit mileage was selected as the physical descriptor because it  
499 provides a more comprehensive representation of tractor operating load than purely kinematic or  
500 torque-based indicators. Previous studies on heavy-duty vehicles and agricultural machinery  
501 have demonstrated that VSP-based descriptors are more effective than purely kinematic  
502 indicators for reflecting real driving load and energy consumption characteristics [26, 27].  
503

504 Acceleration mainly describes local speed variation, but it cannot independently reflect the  
505 effects of road grade, rolling resistance, aerodynamic resistance, and vehicle mass. Torque-  
506 related indicators can directly characterize load demand; however, reliable torque measurements  
507 are difficult to obtain continuously during long-duration field-road tests without additional  
508 driveline instrumentation.  
509

510 By contrast, VSP integrates vehicle speed, acceleration, road grade, rolling resistance,  
511 aerodynamic resistance, and total vehicle mass, thereby providing a physically interpretable  
512 measure of power demand under mixed field-road transportation operating cycles. In addition,  
513 normalizing VSP by mileage allows kinematic fragments with different durations and distances  
514 to be compared on a consistent basis.  
515

516 Thus, this study calculates the VSP per mileage called  $P_L$  [ $\text{kW} (\text{t} \cdot \text{km})^{-1}$ ] of the tractor from

514 the 60 candidate operating condition curves with the smallest **feature deviation errors**, as the  
 515 reference for the final operating condition selection, as shown in formula (29).

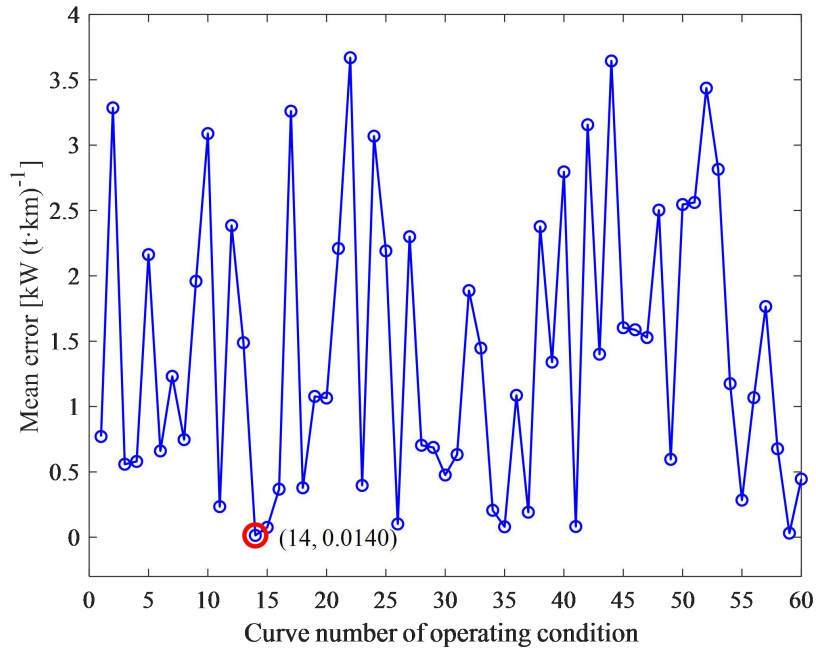
$$516 \quad \begin{cases} P_L(c_n) = \frac{\sum_{i=1}^N VSP}{L} = \frac{\sum_{i=1}^N [v(a + 0.882) + 0.0002104v^3]}{L} \\ c_{best} = \underset{c_n \in C_{60}}{\operatorname{argmin}} |P_L(c_n) - P_L(\text{mean})| \end{cases} \quad (29)$$

517 Where,  $L$  represents the total mileage of the candidate working conditions, km.  $N$   
 518 represents the number of data points for each candidate working condition.  $C_{best}$  represents the  
 519 final selected working condition;  $P_L(\text{mean})$  represents the VSP per mileage of the overall  
 520 sample data; and  $P_L(c_n)$  represents the VSP per mileage of the candidate working conditions.

521 **The final calculated VSP per mileage of the overall sample data** was 893.0871[kW (t·km)<sup>-1</sup>],  
 522 and the specific parameter statistics of the 60 candidate working conditions are shown in  
 523 **table 6**. **Figure 10 shows that the absolute deviation between Candidate Cycle No. 14 and the**  
 524 **overall sample data was the smallest, with a value of 0.0140.**

525 **Table 6.** Statistics of specific power per mileage under candidate working conditions

Candidate working conditions	Maximum VSP (kW t <sup>-1</sup> )	The sum of VSP (kW t <sup>-1</sup> )	Total mileage (km)	$P_L(c_n)$ [kW (t·km) <sup>-1</sup> ]	Absolute deviation [kW (t·km) <sup>-1</sup> ]
1	8.9029	5545.5829	6.2041	893.8578	0.7707
2	10.9778	3684.3602	4.1103	896.3726	3.2855
3	9.9732	4103.0857	4.5914	893.6459	0.5588
4	11.3015	3540.3505	3.9616	893.6668	0.5797
5	10.5915	3340.9823	3.7319	895.2497	2.1626
6	9.0792	3395.2378	3.8045	892.4268	0.6603
7	8.6534	4712.2987	5.2837	891.8558	1.2313
8	9.9615	2708.3171	3.0300	893.8340	0.7469
9	9.5190	5164.3565	5.7953	891.1284	1.9587
10	9.9615	3669.9984	4.1236	889.9986	3.0885
11	8.7290	3765.3498	4.2150	893.3214	0.2343
12	7.8478	3898.3363	4.3767	890.7022	2.3849
13	8.0569	3670.4435	4.1167	891.5985	1.4886
14	10.1289	4912.1454	5.5001	893.0871	0.0140
...	...	...	...	...	...
59	10.0880	3766.1043	4.2171	893.0555	0.0316
60	8.7518	3496.4774	3.9170	892.6417	0.4454

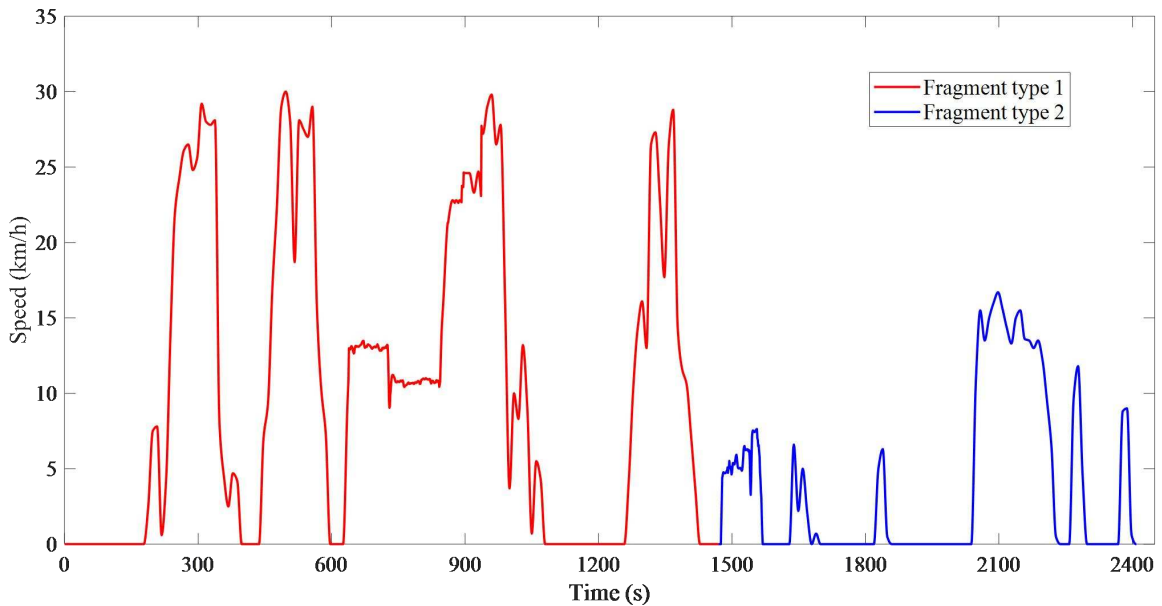


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**Figure 10.** Absolute deviation of unit mileage specific power of candidate operating cycles from the overall sample mean



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**Figure 11.** The constructed tractor's transportation working condition

The final synthesized operating cycle is shown in figure 11, with a total duration of 2,409 seconds. Among them, the first type of kinematic fragment accounts for 61.711%, and the second type of kinematic fragment accounts for 38.289%. The tractor speed exhibits significant variations during transportation operations. When the tractor is transporting within the field, the speed curve shows frequent start-stop and low-speed fluctuations (5-10 km h<sup>-1</sup>). When the tractor is transporting on the road, the speed fluctuates periodically in the medium to high-speed range (20-30 km h<sup>-1</sup>), corresponding to the deceleration-acceleration operation when avoiding obstacles or pedestrians, and occasionally there are brief medium-speed stable segments. When

539 the tractor is engaged in short-distance transportation between urban and rural areas, the overall  
 540 speed is relatively high, but there are also sharp speed drops caused by sudden deceleration at  
 541 intersections or pedestrian interference. These speed variations reflect the tractor's switching  
 542 between different operation links during transportation, different road conditions (such as flat  
 543 roads, slopes, curves, etc.), different load situations (such as empty, half-load, full load), and  
 544 adjustments in driving strategies.

#### 545 4. Validation of the constructed operating cycle

##### 546 4.1 Error inspection of characteristics

547 The characteristic parameter error test of the constructed tractor transportation operation  
 548 conditions was carried out. Twelve representative characteristic parameters, including average  
 549 speed, average travelling speed, maximum speed, standard deviation of speed, average  
 550 acceleration, and average deceleration, which are critical for evaluating the representativeness  
 551 of the constructed operating cycle, are selected for error analysis. The results are shown in table  
 552 7. The characteristic parameters of the constructed tractor transportation cycle show good  
 553 agreement with the measured data. The relative errors between the selected characteristic  
 554 parameters and the overall sample data remain below 10%, and the overall average error is  
 555 5.28%. This indicates that the constructed tractor transportation operating cycle can accurately  
 556 reflect the actual transportation situation in terms of characteristic parameters and have high  
 557 practical value.

558 **Table 7.** The errors between the constructed working condition and the overall sample

Characteristic parameters	$V_m$	$V_{mr}$	$V_{max}$	$V_{sd}$	$A_m$	$D_m$	$A_{sd}$	$D_{sd}$	$P_a$	$P_d$	$P_c$	$P_i$
Constructed working condition	6.813	10.59	30.000	6.418	0.267	-0.274	0.081	0.072	0.086	0.077	0.322	0.455
Overall sample	6.211	11.060	30.000	6.169	0.260	-0.257	0.077	0.069	0.081	0.083	0.349	0.476
Relative error	9.69%	5.27%	0	4.04%	2.69%	6.61%	5.19%	4.35%	6.17%	7.23%	7.74%	4.41%
Average error	5.28%											

##### 559 4.2 Temporal correlation validation based on speed ACF

560 To further verify whether the synthesized driving cycle preserves the temporal dependency  
 561 of the measured operating conditions, the speed autocorrelation function (ACF) was introduced  
 562 for comparison. The ACF reflects the temporal continuity and dynamic persistence of speed

563 fluctuations, which is essential for evaluating the representativeness of micro-trip sequencing.

564 The ACF [39] at lag  $e$  is defined as:

$$565 \quad R(e) = \frac{\sum_{t=1}^{N-e} (v_t - \bar{v})(v_{t+e} - \bar{v})}{\sum_{t=1}^N (v_t - \bar{v})^2} \quad (30)$$

566 Where,  $R(e)$  is the autocorrelation coefficient at lag  $e$ ,  $v(t)$  is the vehicle speed at time  $t$ ,  $\bar{v}$  is  
567 the mean speed, and  $N$  is the total number of samples.

568 The ACF describes the temporal dependency of vehicle speed. A slower decay indicates  
569 stronger continuity and smoother speed transitions, while a faster decay reflects more frequent  
570 fluctuations and transient changes.

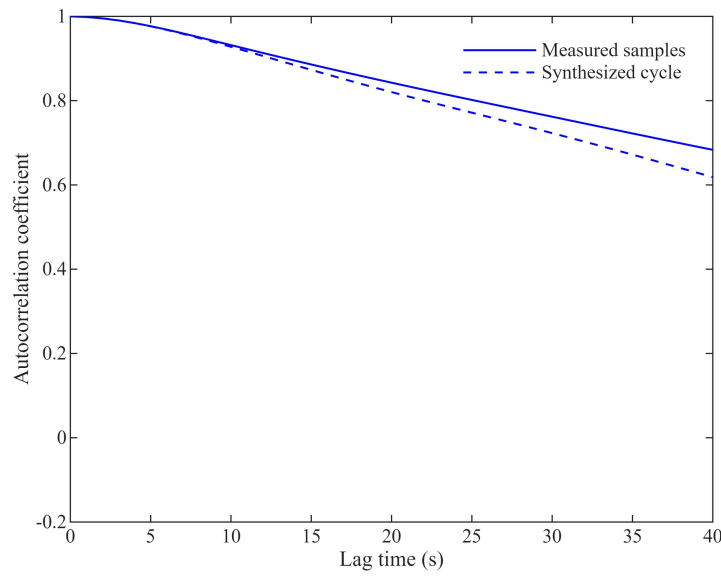
571 Because the measured dataset is much longer than the synthesized cycle, the measured  
572 speed data were divided into multiple sliding windows with the same duration as the  
573 synthesized cycle. To ensure objective comparison, windows with the closest speed-related  
574 characteristics (mean speed, speed variation, idle ratio, acceleration/deceleration ratio, and  
575 fluctuation intensity) were selected as reference samples, and their averaged ACF was used for  
576 comparison.

577 The lag range of 0-40 s was selected because it captures the typical short-term speed  
578 transitions of tractor transport operations, such as stop-start, deceleration, and re-acceleration.  
579 Beyond this range, the ACF gradually approaches zero and is more affected by random  
580 disturbances, making it less representative for temporal-correlation evaluation.

581 To quantitatively evaluate the similarity between the two ACF curves, the root mean  
582 square error (RMSE) was calculated as:

$$583 \quad \text{RMSE} = \sqrt{\frac{1}{M} \sum_{i=1}^M (R_m(i) - R_s(i))^2} \quad (31)$$

584 Where,  $R_m(i)$  and  $R_s(i)$  represent the ACF values of the measured samples and the  
585 synthesized cycle, respectively, and  $M$  is the number of lag points.



586

587

**Figure 12.** Comparison of ACF curves between measured samples and synthesized cycle

588

As shown in figure 12, the ACF curves of the synthesized cycle and the measured samples

589

exhibit highly similar attenuation trends within the lag range of 0-40 s. Both curves decrease

590

gradually with increasing lag time, indicating that the influence of the current speed state on

591

future speed states weakens progressively. The RMSE between the two ACF curves was only

592

0.0312, which demonstrates that the synthesized cycle successfully preserves the short-term

593

temporal dependence of the measured operating conditions.

594

This result further confirms that the proposed DBI-Kmeans and VSP-based driving cycle

595

construction method not only reproduces the macroscopic statistical distribution characteristics,

596

but also maintains the intrinsic microscopic temporal continuity of actual tractor transport

597

operations.

598

#### 4.3 Application-based validation using energy consumption simulation

599

The main component parameters of the test tractor transport unit are shown in table 8. To

600

verify the fuel consumption characteristics of the synthesized operating cycle, a tractor

601

simulation model was built based on Cruise software, and the mechanical and signal

602

connections are completed. As shown in figure 13, the overall model includes the tractor body,

603

the two-axle trailer, the braking system, the transmission system and the suspension system, etc.

604

In the simulation, the trailer loads are respectively set to empty, half-load and full-load states.

605

To be closer to the actual operation data, the simulation step size is set to 0.01 seconds. As

606

shown in figure 14, the simulation operating cycle is set to the constructed tractor condition road

607

spectrum.

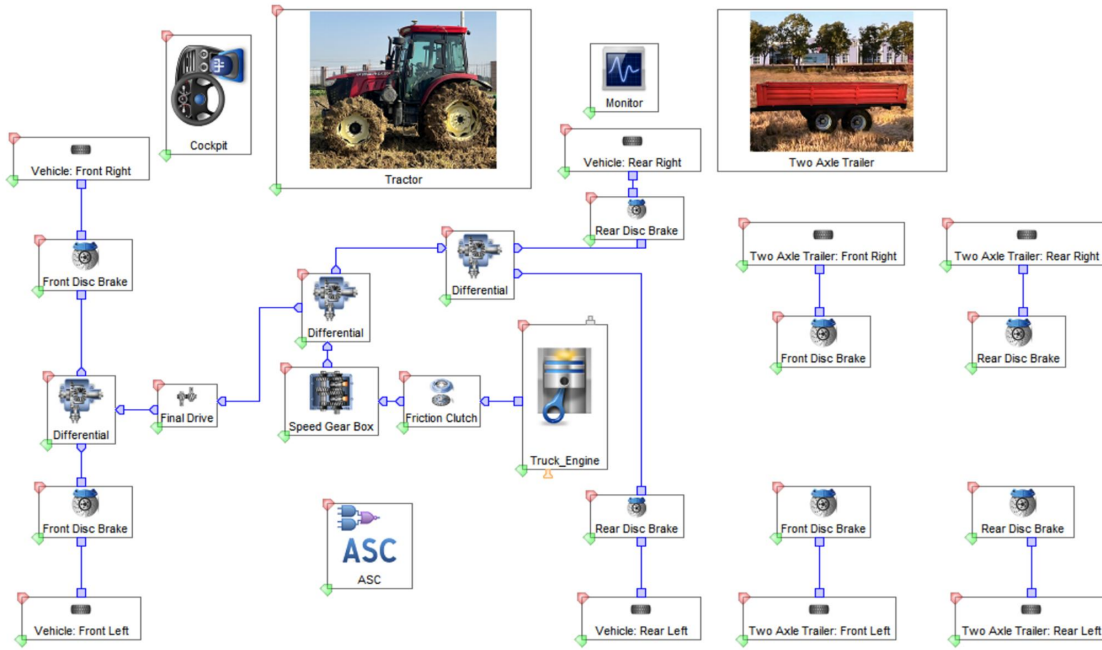
608

**Table 8.** Parameters of tractor's transport unit

Parameter	Technical specifications	Explanation
Tractor model	Wode Aolong DX1204	Four-wheel drive (4WD) agricultural tractor
Engine	Inline four-cylinder	Displacement: 4.5 L

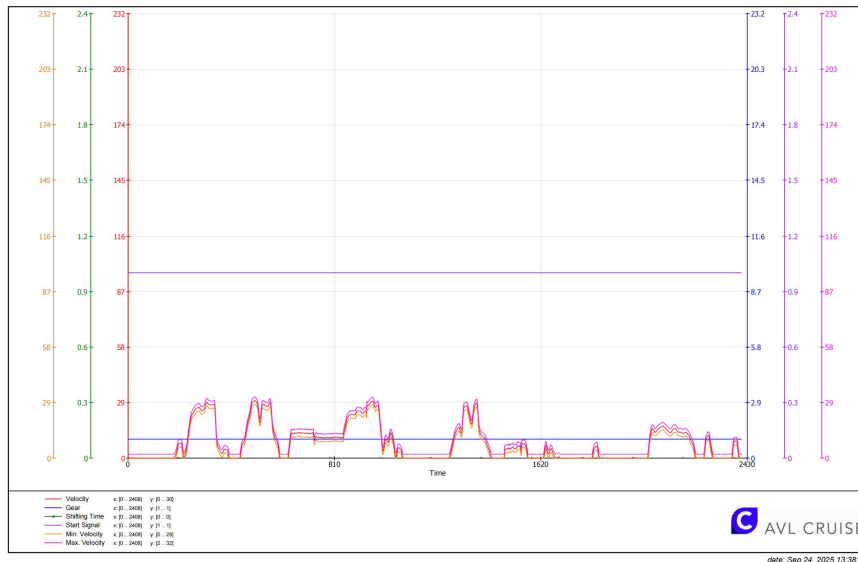
Parameter	Technical specifications	Explanation
	turbocharged diesel engine	
Rated power (kW)	88.2	Rated speed: 2300 r min <sup>-1</sup>
Fuel tank capacity (L)	150	The battery life is approximately 8 to 10 hours (Medium-load operation)
Overall dimension (mm)	4552×2078×2987	A cab and standard tire configuration are included
Wheelbase (mm)	2330	The minimum turning radius is 5500 mm
Minimum usage weight (kg)	4375	Counterweight blocks and standard configuration are included
Maximum traction force (kN)	≥38	The maximum load capacity of the matching trailer is 8 to 10 tons (including its own weight)
Maximum lifting force (kN)	≥25	Compatible with hydraulic farm tools
Trailer model	Woli 12101	Self-unloading trailer
Vehicle body dimension (mm)	3800×2000×500	Double-axis type

609



610

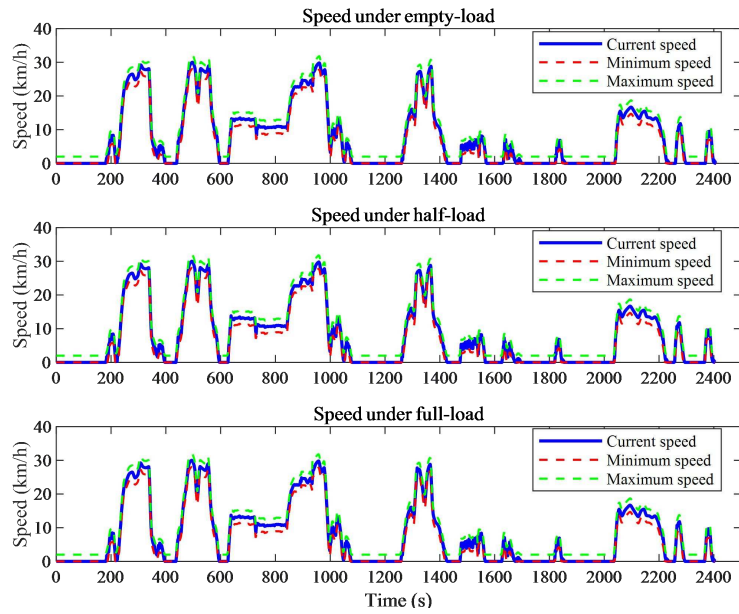
Figure 13. Cruise simulation model of tractor transport unit



611  
612

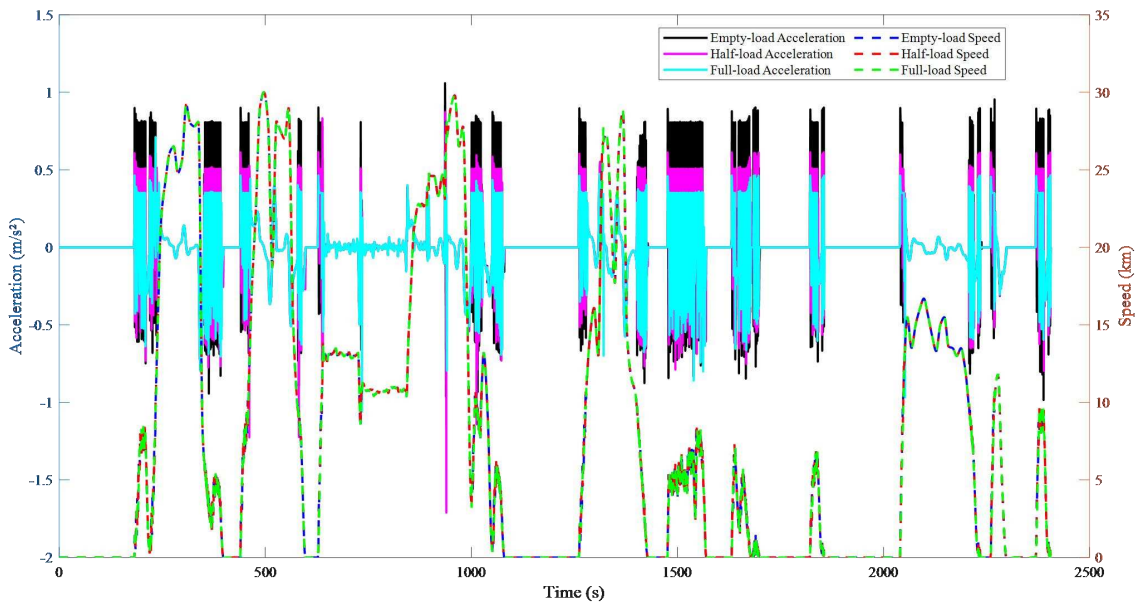
**Figure 14.** The setting of the Cruise simulation **operating cycle**

613 The tractor travelled a total distance of 5.286 kilometres. As shown in **figure 15**, when the  
 614 tractor is **empty**, half, and **full-load** respectively, **the real-time operating speed of the tractor**  
 615 **remained within the upper and lower deviation bounds, and the speed tracking performance**  
 616 **showed good agreement.** **Figure 16** shows that the trend of acceleration is consistent with that of  
 617 velocity. Under **the empty-load** condition, due to the tractor's light weight and small inertia, its  
 618 acceleration response is more sensitive compared to when it is half-load **or full-load**. This  
 619 accurately reflects the transportation operation characteristics of tractors. Meanwhile, the fuel  
 620 consumption characteristic data of the tractor under three working conditions are obtained and  
 621 compared respectively with the test data. As shown in **table 9**, the fuel consumption per 100  
 622 kilometres of the tractor under **empty-load**, half-load and full-load conditions has an error of  
 623 less than 10% compared with the actual tested fuel consumption of the tractor, indicating that  
 624 the constructed tractor transportation **operating cycle** truly reflects the actual transportation  
 625 operation characteristics of high-horsepower tractor.



626  
627

**Figure 15.** The real-time speed tracking situation of the tractor during operation



628  
629

**Figure 16.** The acceleration response of tractors under different loads

630

**Table 9.** Comparison between simulation and test results of tractor fuel consumption

Load	Empty-load	Half-load	Full-load
Simulated fuel consumption (L·100 km <sup>-1</sup> )	21.65	27.09	33.30
Test fuel consumption (L·100 km <sup>-1</sup> )	23.22	29.31	36.84
Error (%)	6.76	7.57	9.61

631

## 5 Conclusion

632

In summary, this study proposes a measurement-oriented, data-driven framework for

633

constructing representative transportation operating cycles of high-horsepower tractors by

634 integrating DBI-guided K-means clustering with VSP per mileage. The main findings can be  
635 summarized as follows:

636 (1) The synthesized operating cycle effectively captures the coupled speed-acceleration  
637 characteristics of tractor operation under mixed field-road conditions. Compared with  
638 conventional cycle-construction methods relying on empirically selected cluster numbers, the  
639 proposed framework provides a more objective and measurement-based representation of  
640 complex and highly variable tractor operating behaviours.

641 (2) The constructed cycle demonstrates high fidelity in both statistical distribution and  
642 temporal correlation characteristics. The average deviation of representative kinematic features  
643 relative to the full measured dataset is 5.28%, while the RMSE between the ACF curves of the  
644 measured samples and the synthesized cycle is only 0.0312. These results confirm that the  
645 proposed cycle preserves not only the global statistical characteristics but also the intrinsic  
646 short-term temporal continuity of real operating conditions.

647 (3) Cruise-based validation further verifies the practical applicability of the constructed  
648 cycle in energy-consumption analysis. Under empty-load, half-load, and full-load conditions,  
649 the discrepancies between simulated and measured fuel consumption remain within 10% (6.76%,  
650 7.57%, and 9.61%, respectively), demonstrating that the synthesized cycle can effectively  
651 reproduce real-world operational behaviour within acceptable measurement uncertainty.

652 (4) The developed operating-cycle database provides a standardized measurement basis for  
653 the evaluation and comparison of tractor operating behaviour. It addresses the current gap in  
654 measurement-based transportation cycle research for high-horsepower tractors and supports  
655 further applications such as energy-consumption assessment, powertrain optimization, and  
656 control strategy development in agricultural machinery systems.

#### 657 **Data availability statement**

658 The data will be available to access if needed.

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