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Dellar, O. and Jones, B. orcid.org/0000-0002-7465-1389 (2018) Efficient frequency response computation for loworder modelling of spatially distributed systems. *International Journal of Control*, 93 (3). pp. 366-376. ISSN 0020-7179

<https://doi.org/10.1080/00207179.2018.1468927>

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To cite this article: O. J. Dellar & B. Ll. Jones (2018): Efficient frequency response computation for low-order modelling of spatially distributed systems, International Journal of Control, DOI: [10.1080/00207179.2018.1468927](https://doi.org/10.1080/00207179.2018.1468927)

To link to this article: <https://doi.org/10.1080/00207179.2018.1468927>



Accepted author version posted online: 24 Apr 2018.



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Publisher: Taylor & Francis



Journal: *International Journal of Control*

DOI: <https://doi.org/10.1080/00207179.2018.1468927>

MANUSCRIPT SUBMITTED TO INTERNATIONAL JOURNAL
OF CONTROL

**Efficient frequency response computation for low-order modelling of
spatially distributed systems**

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ARTICLE HISTORY

Compiled April 23, 2018

ABSTRACT

Motivated by the challenges of designing feedback controllers for spatially distributed systems, this paper presents a computationally efficient approach to obtaining the point-wise frequency response of such systems, from which low-order models can be easily identified. This is achieved by sequentially combining the individual frequency responses of the constituent lower-order subsystems in a way that exploits the interconnectivity arising from spatial discretisation. Importantly, this approach extends to the singular subsystems that naturally arise upon spatial discretisation of systems governed by partial differential-algebraic equations, with fluid flows being a prime example.

The main result of this paper is a proof that the computational complexity associated with forming the overall frequency response is minimised if the smallest subsystems are first merged into larger subsystems, before combining the frequency responses of the latter. This reduces the complexity by several orders of magnitude; a result that is demonstrated upon the numerical example of a spatially discretised two-dimensional wave-diffusion equation.

By avoiding the necessity to construct, store, or manipulate large-scale system matrices, the modelling approach presented in this paper is well conditioned and computationally tractable for spatially distributed systems consisting of enormous numbers of subsystems. It therefore bypasses many of the problems associated with conventional model reduction techniques.

KEYWORDS

Low-order modelling, large scale systems, spatially distributed systems, frequency response, computational efficiency, flow control.

1. Introduction

The interconnection of numerous low-order subsystems can result in large-scale systems that display a dynamical wealth far in excess of their constituent parts. Examples include power fluctuations within distributed power grids (Zhong & Hornik, 2013), string instabilities in traffic systems (Swaroop & Hedrick, 1996) and congestion of the internet (Low et al., 2002). Such systems are also obtained upon spatial discretisation of systems governed by partial differential equations (PDEs) and partial differential algebraic equations (PDAEs). These include the flexing of beams, propagation of sound waves, and heat transfer (Curtain & Morris, 2009), and the motion of fluid flows (Aamo & Krstić, 2003).

The focus of this paper is on obtaining low-order models of such systems for the purpose of feedback controller design. Fluid flows are of particular interest since the control of these could lead to significant economic and environmental benefits (el Hak, 2000; Bradley, 2000; Kim, 2011). However, control of fluid flows remains difficult since the underlying plant dynamics are, in many cases, governed by the incompressible Navier-Stokes equations; a coupled set of nonlinear PDAEs in which the algebraic constraints arise not only from incompressibility, but also from the imposition of boundary conditions (Jones et al., 2015). Assumptions of linearity are justifiable in many cases, thereby simplifying the model to one that is linear and time-invariant (LTI), albeit still infinite-dimensional and algebraically constrained.

Methods based on infinite-dimensional linear systems theory (Curtain & Zwart, 1995) have been developed, whereby spatially distributed systems governed by PDEs can, in some cases, be transformed into irrational transfer functions, which can then be approximated for the purposes of controller design (Curtain & Morris, 2009). However, it is not clear to what extent this methodology can extend to PDAEs, particularly those in more than one spatial dimension and where the geometry of the domain is complex.

If a further approximation is made based upon spatial discretisation, the model simplifies to one that is LTI, finite-dimensional and singular (Dai, 1989), with the following state-space representation.

$$\mathbf{E}\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{u}(t), \quad (1a)$$

$$\mathbf{y}(t) = \mathbf{C}\mathbf{x}(t) + \mathbf{D}\mathbf{u}(t), \quad (1b)$$

where $\mathbf{u}(t) \in \mathbb{R}^q$, $\mathbf{y}(t) \in \mathbb{R}^p$, $\mathbf{x}(t) \in \mathbb{R}^n$ are the input, output and state vectors, respectively, and $\mathbf{E} \in \mathbb{R}^{n \times n}$, $\mathbf{A} \in \mathbb{R}^{n \times n}$, $\mathbf{B} \in \mathbb{R}^{n \times q}$, $\mathbf{C} \in \mathbb{R}^{p \times n}$, $\mathbf{D} \in \mathbb{R}^{p \times q}$ are state-space matrices. However, the state dimension will be prohibitively large from the point of view of direct synthesis of a model-based controller, hence necessitating some form of model-order reduction. For flows in simple geometries, such as plane channel flow, further assumptions can be made to reduce the dimensionality of the system (Aamo & Krstić, 2003), but for flows in complex geometries, such as over the rear face of a road vehicle, such techniques are not readily applicable.

A number of further assumptions are made in relation to (1) that whilst motivated in particular by flow control applications are nevertheless still representative of a wider class of problems. These are listed below:

- A1. The pair (\mathbf{E}, \mathbf{A}) is regular (Dai, 1989), so as to ensure existence of the transfer function of (1).
- A2. The state dimension is too large and/or the spatial domain is too complex to explicitly construct the full-scale system matrices in (1).

- A3. The majority of subsystems arising from the spatial discretisation have no control input or measured output and the number of sensors and actuators is low. Centralised, rather than distributed controllers are therefore of relevance.
- A4. The subsystems arising from discretisation may each possess different dynamics.
- A5. Controllers based upon low-order approximations of (1) should come with *a priori* guarantees of closed-loop stability/performance when applied to (1).

The second assumption immediately discounts the majority of existing methods based upon balanced truncation or Krylov approximation of large scale systems (Antoulas, 2005; Stykel, 2004). On the other hand, data-driven methods, such as system identification, proper orthogonal decomposition (POD) and balanced proper orthogonal decomposition (BPOD) (Holmes et al., 1996; Rowley, 2005; Willcox & Peraire, 2002; Weller et al., 2009; Mathelin et al., 2010; Akhtar et al., 2012) are not applicable here owing to the fact that they yield models that are close to the high-dimensional system in an open-loop sense and thus may not capture the dynamics of importance from a closed-loop perspective (Curtain & Morris, 2009; Jones & Kerrigan, 2010) nor reflect the closed-loop objectives (Bewley et al., 2016). The other problem with data-driven methods is the need to generate the data in the first place. Physical experiments are typically expensive in terms of cost, whilst simulations can be expensive in terms of time. Other notable works on the control of spatially distributed system include that of D’Andrea & Dullerud (2003) where distributed controllers were designed for large-scale systems consisting of identical sub-systems each with sensing and actuation capabilities; a separate problem to that considered in this paper.

The techniques developed in this paper stem from the viewpoint that if the governing PDAE is known, then the dynamic information within this should be exploited, rather than discarded in favour of a data driven approach. Hence, the aim of this work is to develop a computationally tractable approach to obtaining low-dimensional models of PDAE systems suitable for feedback control design. Furthermore, this approach should avoid the construction of extremely large state-space matrices, or the need to generate data from experiments. It is inspired by two key works, the first of which is that of Dahan et al. (2012) who designed feedback controllers for bluff body drag reduction. Despite Dahan’s simulation model employing in excess of $\mathcal{O}(10^4)$ states, open-loop harmonic excitation revealed a frequency response almost identical to that of a *second-order* system owing to the fact that the vast majority of states had negligible influence upon the input-output response of the system. Obtaining the frequency response of a large-scale system is thus central to exposing its low-order nature, or otherwise. The second key work is that of Baramov et al. (2004), who constructed the frequency response of a 2D plane channel flow system in a point-wise fashion using the Redheffer Star Product (Doyle et al., 1991). More recent work by Jones (2014) has shown how low-order models can be fitted to point-wise-in-frequency data in such a way as to enable the derivation of an upper bound on the ν -gap (Vinnicombe, 2000) between the low-order fitted model and the full-scale system. This subsequently enables the design of low-order robust feedback controllers with *a priori* guarantees of robust stability and performance when connected to the full-scale system.

In the work of Baramov et al. (2004), the frequency response of the 2D flow system was obtained by defining wall-normal subsystems of the flow and then sequentially combining these in the stream-wise direction. Implicit in this approach was the definition of subsystems and the fashion in which to combine them, and the choices made were intuitive and no-doubt guided by the simple (rectangular) geometry of the domain. However, for flows over more complex geometries, (e.g. bluff-body flows), the

choice of subdomains over which to define subsystems and the subsequent order in which to combine these, is far from intuitive. If the aim is to obtain the overall frequency response with the least computational cost, which choices are best? This paper addresses these issues, with particular focus on the first; the optimal choice of subdomain. The key results are that for a PDAE in two spatial dimensions, an optimal scale at which to define a subsystem exists (Proposition 3.1) and that upon combining these subsystems in a prescribed sequence, the resulting complexity of computing the system frequency response is reduced to $\mathcal{O}(\varrho^2)$ (Theorem 3.2), where ϱ is the mesh density, as compared to $\mathcal{O}(\varrho^4)$ and $\mathcal{O}(\varrho^6)$, for the extremes of smallest and largest subsystems, respectively. In practical terms this means that the time taken to compute the frequency response of a spatially distributed system can be shortened by several orders-of-magnitude through optimal choice of domain decomposition.

The remainder of the paper is organised as follows. Section 2 presents an overview of the approach employed for modelling spatially distributed systems, detailing the steps required to obtain the point-wise frequency response of a system governed by PDAEs. In Section 3, the domain decomposition optimisation is described, and the main results of the paper presented. The modelling approach is demonstrated in Section 4 upon a 2D wave-diffusion system. The frequency response obtained using the approach outlined in Section 3 is compared to that obtained via direct construction of the full-scale system model, and relevant computational properties are discussed. Finally, conclusions and future work are presented in Section 5.

2. Preliminaries on spatially distributed systems

This paper considers linear/linearised PDAEs of not more than $d = 2$ spatial dimensions, spatially discretised using three-point finite-differences, but such assumptions are not restrictive. Hence, consider a two-dimensional and linear PDAE on spatial domain Ω with domain boundary $\partial\Omega$, discretised in space upon a cartesian grid consisting of nodes characterised by indices (i, j) in the x and y directions, respectively. Each node is thus a low-dimensional descriptor state-space models of the form:

$$\mathbf{E}_{i,j} \frac{d}{dt} \mathbf{x}_{i,j}(t) = \mathbf{A}_{i,j} \mathbf{x}_{i,j}(t) + \mathbf{B}_{i,j} \boldsymbol{\xi}_{i,j}(t) + \mathbf{B}_{u_{i,j}} \mathbf{u}_{i,j}(t), \quad (2a)$$

$$\mathbf{z}_{i,j}(t) = \mathbf{C}_{i,j} \mathbf{x}_{i,j}(t), \quad (2b)$$

$$\mathbf{y}_{i,j}(t) = \mathbf{C}_{y_{i,j}} \mathbf{x}_{i,j}(t), \quad (2c)$$

where $t \in \mathbb{R}^+$ is time and $\mathbf{x}_{i,j}(t) \in \mathbb{R}^n$ is the node's state vector, (assuming each subsystem possesses the same number of states). Note the use of bold fonts for system variables and regular fonts for spatial directions. Inputs in the form of state information from neighbouring nodes are represented by $\boldsymbol{\xi}_{i,j}(t) \in \mathbb{R}^{2dn}$, where:

$$\boldsymbol{\xi}_{i,j}(t) := [\mathbf{x}_{i,j-1}^\top(t) \quad \mathbf{x}_{i+1,j}^\top(t) \quad \mathbf{x}_{i,j+1}^\top(t) \quad \mathbf{x}_{i-1,j}^\top(t)]^\top \in \mathbb{R}^{4n},$$

in the 2D case. The vector of control inputs is denoted $\mathbf{u}_{i,j}(t) \in \mathbb{R}^{q_{i,j}}$, $\mathbf{y}_{i,j}(t) \in \mathbb{R}^{p_{i,j}}$ is a vector of measured outputs and $\mathbf{z}_{i,j}(t) \in \mathbb{R}^{2dn}$ is the output of the node's state vector to neighbouring nodes. The system matrices are $\mathbf{E}_{i,j}, \mathbf{A}_{i,j} \in \mathbb{R}^{n \times n}$, $\mathbf{B}_{i,j} \in \mathbb{R}^{n \times 2dn}$, $\mathbf{B}_{u_{i,j}} \in \mathbb{R}^{n \times q_{i,j}}$, $\mathbf{C}_{i,j} := [\mathbf{I} \quad \mathbf{I} \quad \cdots \quad \mathbf{I}]^\top \in \mathbb{R}^{2dn \times d}$ and $\mathbf{C}_{y_{i,j}} \in \mathbb{R}^{p_{i,j} \times n}$. Note that

the majority of subsystems will not possess control inputs or measured outputs, only nodes corresponding to the physical locations of actuation or sensing will have these.

Taking Laplace transforms of (2) with initial condition $\mathbf{x}_{i,j}(0) = 0$ yields:

$$\begin{bmatrix} \tilde{\mathbf{z}}_{i,j}(s) \\ \tilde{\mathbf{y}}_{i,j}(s) \end{bmatrix} = \underbrace{\begin{bmatrix} \mathbf{C}_{i,j} \\ \mathbf{C}_{y_{i,j}} \end{bmatrix} (s\mathbf{E}_{i,j} - \mathbf{A}_{i,j})^{-1} [\mathbf{B}_{i,j} \quad \mathbf{B}_{u_{i,j}}]}_{\mathbf{P}_{i,j}(s)} \begin{bmatrix} \tilde{\boldsymbol{\xi}}_{i,j}(s) \\ \tilde{\mathbf{u}}_{i,j}(s) \end{bmatrix}, \quad (3)$$

where $\mathbf{P}_{i,j}(s) \in \mathcal{R}^{(2dn+p_{i,j}) \times (2dn+q_{i,j})}$, \mathcal{R} is the space of real-rational transfer function matrices, $\tilde{\cdot}$ denotes a Laplace transformed quantity, and $s \in \mathbb{C}$. Assuming the pair $(\mathbf{E}_{i,j}, \mathbf{A}_{i,j})$ is regular (Dai, 1989) ensures the existence of $\mathbf{P}_{i,j}(s)$.

Being defined at the nodal level, the $\mathbf{P}_{i,j}$ are the smallest definable subsystems and are henceforth referred to as ‘atoms’. The interconnection of neighbouring atoms is shown in Figure 1. For a given frequency $\omega \in \mathbb{R}$, the frequency response of each atom $\mathbf{P}_{i,j}(i\omega) \in \mathbb{C}^{(2dn+p_{i,j}) \times (2dn+q_{i,j})}$ can be computed in a point-wise fashion. The complexity associated with this computation is dominated by the *LU* factorisation of $(i\omega\mathbf{E}_{i,j} - \mathbf{A}_{i,j})$ which, in big-O notation, is $\mathcal{O}(n^3)$ flops (Golub & Loan, 1996), where one flop is defined as a single addition, subtraction, multiplication or division between two floating point numbers (Golub & Loan, 1996). Given the small state dimension of each atom, the point-wise evaluation of an atom’s frequency response is nevertheless extremely cheap. The remaining challenge is to combine the (point-wise) frequency response of each atom in an efficient fashion to yield the overall (point-wise) frequency response of the underlying large-scale system. As in Baramov et al. (2004), this can then be accomplished using the Redheffer star product (RHSP).

2.1. Redheffer star product

The RHSP describes the interconnection of two multi-input multi-output (MIMO) LTI systems (Doyle et al., 1991). The block diagram of the RHSP between two arbitrary systems $\mathbf{Q}(s)$ and $\mathbf{M}(s)$ is depicted in Figure 2.

Assuming $\mathbf{Q}(s)$ and $\mathbf{M}(s)$ are compatibly partitioned as follows:

$$\begin{bmatrix} \tilde{\mathbf{y}}_{\mathbf{Q}_1}(s) \\ \tilde{\mathbf{y}}_{\mathbf{Q}_2}(s) \end{bmatrix} = \underbrace{\begin{bmatrix} \mathbf{Q}_{11}(s) & \mathbf{Q}_{12}(s) \\ \mathbf{Q}_{21}(s) & \mathbf{Q}_{22}(s) \end{bmatrix}}_{\mathbf{Q}(s)} \begin{bmatrix} \tilde{\mathbf{u}}_{\mathbf{Q}_1}(s) \\ \tilde{\mathbf{u}}_{\mathbf{Q}_2}(s) \end{bmatrix}, \quad (4a)$$

$$\begin{bmatrix} \tilde{\mathbf{y}}_{\mathbf{M}_1}(s) \\ \tilde{\mathbf{y}}_{\mathbf{M}_2}(s) \end{bmatrix} = \underbrace{\begin{bmatrix} \mathbf{M}_{11}(s) & \mathbf{M}_{12}(s) \\ \mathbf{M}_{21}(s) & \mathbf{M}_{22}(s) \end{bmatrix}}_{\mathbf{M}(s)} \begin{bmatrix} \tilde{\mathbf{u}}_{\mathbf{M}_1}(s) \\ \tilde{\mathbf{u}}_{\mathbf{M}_2}(s) \end{bmatrix}, \quad (4b)$$

where $\mathbf{Q}(s) \in \mathcal{R}^{(a_{\mathbf{Q}_1}+a_{\mathbf{Q}_2}) \times (b_{\mathbf{Q}_1}+b_{\mathbf{Q}_2})}$, $\mathbf{M}(s) \in \mathcal{R}^{(a_{\mathbf{M}_1}+a_{\mathbf{M}_2}) \times (b_{\mathbf{M}_1}+b_{\mathbf{M}_2})}$, $a_{\mathbf{Q}_2} = b_{\mathbf{M}_1}$, and $a_{\mathbf{M}_1} = b_{\mathbf{Q}_2}$, the Redheffer star product $\cdot \star \cdot : \mathcal{R}^{(a_{\mathbf{Q}_1}+a_{\mathbf{Q}_2}) \times (b_{\mathbf{Q}_1}+b_{\mathbf{Q}_2})} \times$

$\mathcal{R}^{(a_{M_1}+a_{M_2}) \times (b_{M_1}+b_{M_2})} \rightarrow \mathcal{R}^{(a_{Q_1}+a_{M_2}) \times (b_{Q_1}+b_{M_2})}$ is defined as (Foias & Frazho, 1984):

$$\begin{aligned} \mathbf{R}(s) &= \mathbf{Q}(s) \star \mathbf{M}(s) \\ &:= \begin{bmatrix} \mathbf{Q}_{11} + \mathbf{Q}_{12}\mathbf{M}_{11}(\mathbf{I} - \mathbf{Q}_{22}\mathbf{M}_{11})^{-1}\mathbf{Q}_{21} & \mathbf{Q}_{12}\mathbf{M}_{11}(\mathbf{I} - \mathbf{Q}_{22}\mathbf{M}_{11})^{-1}\mathbf{Q}_{22}\mathbf{M}_{12} + \mathbf{Q}_{12}\mathbf{M}_{12} \\ \mathbf{M}_{21}(\mathbf{I} - \mathbf{Q}_{22}\mathbf{M}_{11})^{-1}\mathbf{Q}_{21} & \mathbf{M}_{21}(\mathbf{I} - \mathbf{Q}_{22}\mathbf{M}_{11})^{-1}\mathbf{Q}_{22}\mathbf{M}_{12} + \mathbf{M}_{22} \end{bmatrix}, \end{aligned} \quad (5)$$

which yields the overall interconnection:

$$\begin{bmatrix} \tilde{\mathbf{y}}_{Q_1}(s) \\ \tilde{\mathbf{y}}_{M_2}(s) \end{bmatrix} = \mathbf{R}(s) \begin{bmatrix} \tilde{\mathbf{u}}_{Q_1}(s) \\ \tilde{\mathbf{u}}_{M_2}(s) \end{bmatrix}. \quad (6)$$

The overall frequency response of the PDAE system, denoted $\mathbf{G}(i\omega)$, from control inputs $\tilde{\mathbf{u}}(s)$ to measured outputs $\tilde{\mathbf{y}}(s)$, is thus obtained by ‘chaining’ the individual atoms’ frequency responses $\mathbf{P}_{i,j}(i\omega)$ together with the RHSP:

$$\mathbf{G}(i\omega) = \mathbf{P}_{1,1}(i\omega) \star \mathbf{P}_{2,1}(i\omega) \star \cdots \star \mathbf{P}_{i,j}(i\omega) \star \cdots \star \mathbf{P}_{n_i,n_j}(i\omega) \quad (7)$$

The method of subsystem chaining is shown in Figure 3 and connects atoms in a ‘snake-like’ pattern. This pattern is heuristic and it remains to be proven whether or not it leads to the lowest computational cost. However, it is guided by the complexity associated with the Redheffer star product operation (5). The complexity of this is dominated by the factoring of the system operators describing the mapping from interconnecting inputs and outputs. By connecting the atoms in such a fashion, the number of unconnected interconnections of the chained system remains low in comparison to other sequences as the chaining commences, thus preventing the factored matrices in (5) from becoming excessively large.

This procedure is completed for a number of frequencies of interest, before a low-order transfer function is fitted to the frequency response using, for example, least squares regression (Hansen et al., 2012). This process lends itself to the theory developed by Jones (2014), where an upper bound on the ν -gap (Vinnicombe, 1993, 2000) between the low-order approximation and the high-order system can be computed. This subsequently informs the design of low-order \mathcal{H}_∞ controllers with guarantees concerning the robust stabilisation of the large-scale system.

2.2. Computational complexity

The subsystem chaining method can be applied to complex geometries by embedding such geometries within a bounding rectangular (in the 2D case) computational domain. Upon spatial discretisation of the computational domain, nodal subsystems that lie outside the physical (complex) geometry can be modelled as static systems with zero direct feedthrough. When the subsystem frequency responses are chained together, all nodes in the bounding rectangular domain are included (and thus the following proofs of complexity hold), but only nodes within the complex (physical) subdomain contribute to the system frequency response. This ultimately yields the frequency response of the spatially distributed system within the physical domain.

As such, in the following analysis we assume, without loss of generality, a rectangular domain $\Omega := [-\ell_x/2, \ell_x/2] \times [-\ell_y/2, \ell_y/2]$. A uniform mesh density $\varrho \in \mathbb{N}$ (nodes per unit length) is used in both directions, resulting in $n_x \in \mathbb{N}$ and $n_y \in \mathbb{N}$ nodes in the x

and y directions, respectively. Assuming the snake-like chaining of subsystems shown in Figure 3, the following lemma establishes the complexity of forming the overall frequency response as a function of mesh density.

Lemma 2.1. *The computational complexity $\mathcal{C} \in \mathbb{R}^+$ of obtaining $\mathbf{G}(i\omega)$ is $\mathcal{O}(\varrho^4)$.*

Proof. Noting the cost of typical matrix operations as presented in Table 1, summing all the operations required to construct all $n_{\text{total}} = n_x n_y$ atoms using (3) and evaluate (7) yields the overall complexity:

$$\mathcal{C} = \mathcal{O}\left(n_x^3 n_y (1+n)n^2 + 2n_x^2 n_y (1+6n)n^2 + 52n_x n_y n^3\right), \quad (8)$$

where $n \in \mathbb{N}$ is the state dimension of each subsystem.

Since $n_x = \varrho l_x$ and $n_y = \varrho l_y$, for given ϱ , l_x , and l_y , (8) can be written:

$$\mathcal{C} = c_1 l_x^3 l_y (1+n)n^2 \varrho^4 + 2c_2 l_x^2 l_y (1+6n)n^2 \varrho^3 + 52c_3 l_x l_y n^3 \varrho^2, \quad (9)$$

where $c_1, \dots, c_3 \in \mathbb{R}^+$ are unknown constants of $\mathcal{O}(1)$. Hence the complexity is $\mathcal{O}(\varrho^4)$. \square

3. Optimisation by domain decomposition

The complexity of the above modelling approach can be reduced further by employing a domain decomposition optimisation. The computational mesh is first split into $n_\Omega = n_{\Omega_x} n_{\Omega_y} \in \mathbb{N}$ subdomains, defined here as computational ‘molecules’, with $n_{\Omega_x} \in \mathbb{N}$ and $n_{\Omega_y} \in \mathbb{N}$ molecules in the x and y directions, respectively. First, the frequency response of each molecule, $\mathbf{P}_{\Omega_{i,j}}(i\omega)$, is constructed by chaining together the individual atoms within the molecule, using the subsystem chaining method according to:

$$\mathbf{P}_{\Omega_{i,j}}(i\omega) = \mathbf{P}_{1,1}(i\omega) \star \mathbf{P}_{2,1}(i\omega) \star \dots \star \mathbf{P}_{n_i, n_j}(i\omega). \quad (10)$$

As shown in Figure 4, the overall PDAE system frequency response $\mathbf{G}(i\omega)$ is then obtained by connecting each of the molecule frequency responses together:

$$\mathbf{G}(i\omega) = \mathbf{P}_{\Omega_{1,1}}(i\omega) \star \mathbf{P}_{\Omega_{2,1}}(i\omega) \star \dots \star \mathbf{P}_{\Omega_{n_{\Omega_x}, n_{\Omega_y}}}(i\omega). \quad (11)$$

For the two spatial dimension case, we now aim to prove that the the overall computational complexity of evaluating $\mathbf{G}(i\omega)$ can be reduced by orders of magnitude by optimising the domain decomposition. To begin with, the following proposition establishes existence and uniqueness of values n_{Ω_x} and n_{Ω_y} that minimise the complexity:

Proposition 3.1. *Assume mesh density $\varrho \gg n, l_x, l_y, c_k$, where $c_k \in \{\mathbb{R}_+ | c_k \sim \mathcal{O}(1)\}$ and $k \in \mathbb{N}$. Then there exist unique values $n_{\Omega_x} = n_{\Omega_x}^*$ and $n_{\Omega_y} = n_{\Omega_y}^*$ that minimise the complexity of evaluating (10) and (11).*

Proof. Summing the complexity of all the computational operations required to eval-

uate (10) for each molecule, followed by (11) yields the overall complexity:

$$\begin{aligned}
\mathcal{C} = & \mathcal{O}\left(n_x^3 n_{\Omega_x}^{-2} n_y (1+2n)n^2 + 2n_x n_{\Omega_x}^{-1} n_y (1+6n)n^2 + 32n_x n_y n^3 + 4n_x n_y n^2 \right. \\
& + 8n_x n_y n + 16\left(n_x n_{\Omega_x}^{-1} + n_y n_{\Omega_y}^{-1}\right)^3 n_{\Omega_x} n_{\Omega_y} n^3 \\
& + 5n_{\Omega_x} n_{\Omega_y} \left(n_x n_{\Omega_x}^{-1} + n_y n_{\Omega_y}^{-1}\right)^2 \left(n_x - n_x n_{\Omega_x}^{-1}\right) n^3 \\
& + n_{\Omega_x} n_{\Omega_y} \left(n_x n_{\Omega_x}^{-1} + n_y n_{\Omega_y}^{-1}\right)^2 n^2 + n_{\Omega_x} n_{\Omega_y} \left(n_x - n_x n_{\Omega_x}^{-1}\right)^2 n^2 \\
& \left. + n_{\Omega_x} n_{\Omega_y} \left(n_x n_{\Omega_x}^{-1} + n_y n_{\Omega_y}^{-1}\right) \left(n_x - n_x n_{\Omega_x}^{-1}\right) n^2\right). \tag{12}
\end{aligned}$$

The optimal values of n_{Ω_x} and n_{Ω_y} which achieve the complexity reduction discussed above are those which satisfy:

$$\begin{aligned}
\frac{\partial \mathcal{C}}{\partial n_{\Omega_x}} = 0 = & -2c_1 \ell_x^3 \ell_y (1+n) n^2 n_{\Omega_x}^{-3} \varrho^4 - 48c_6 \ell_x^2 \ell_y n^3 n_{\Omega_x}^{-2} \varrho^3 \\
& - 32c_6 \ell_x^3 n_{\Omega_y} n^3 n_{\Omega_x}^{-3} \varrho^3 + 16c_6 \ell_y^3 n^3 n_{\Omega_y}^{-2} \varrho^3 + 10c_7 \ell_x^2 \ell_y n^3 n_{\Omega_x}^{-2} \varrho^3 \\
& - 5c_7 \ell_x^3 n_{\Omega_y} n^3 n_{\Omega_x}^{-2} \varrho^3 + 5c_7 \ell_x \ell_y^2 n^3 n_{\Omega_y}^{-1} \varrho^3 - c_8 n_{\Omega_x} \ell_x^2 n^2 n_{\Omega_x}^{-2} \varrho^2 \\
& + 10c_7 \ell_x^3 n_{\Omega_y} n^3 n_{\Omega_x}^{-3} \varrho^3 + c_8 \ell_y^2 n^2 n_{\Omega_y}^{-1} \varrho^2 + c_9 \ell_x^2 n_{\Omega_y} n^2 n_{\Omega_x}^{-2} \varrho^2 \\
& + c_9 \ell_x \ell_y n^2 \varrho^2 - c_{10} \ell_x^2 n_{\Omega_y} n^2 n_{\Omega_x}^{-2} \varrho^2 + c_{10} \ell_x^2 n_{\Omega_y} n^2 \varrho^2 \\
& - 2c_2 \ell_x \ell_y (1+6n) n^2 n_{\Omega_x}^{-2} \varrho^2, \tag{13a}
\end{aligned}$$

$$\begin{aligned}
\frac{\partial \mathcal{C}}{\partial n_{\Omega_y}} = 0 = & -48c_6 \ell_x \ell_y^2 n^3 n_{\Omega_y}^{-2} \varrho^3 + 16c_6 \ell_x^3 n^3 n_{\Omega_x}^{-2} \varrho^3 + 5c_7 \ell_x^3 n^3 n_{\Omega_x}^{-1} \varrho^3 \\
& - 32c_6 \ell_y^3 n_{\Omega_x} n^3 n_{\Omega_y}^{-3} \varrho^3 + 5c_7 \ell_x \ell_y^2 n^3 n_{\Omega_y}^{-2} \varrho^3 + c_8 \ell_x^2 n^2 n_{\Omega_x}^{-1} \varrho^2 \\
& - 5c_7 \ell_x \ell_y^2 n_{\Omega_x} n^3 n_{\Omega_y}^{-2} \varrho^3 - 5c_7 \ell_x^3 n^3 n_{\Omega_x}^{-2} \varrho^3 - c_8 n_{\Omega_x} \ell_x^2 n^2 n_{\Omega_y}^{-2} \varrho^2 \\
& + c_9 \ell_x^2 n^2 \varrho^2 - c_9 \ell_x^2 n^2 n_{\Omega_x}^{-1} \varrho^2 + c_{10} \ell_x^2 n^2 n_{\Omega_x}^{-1} \varrho^2 \\
& - 2c_{10} \ell_x^2 n^2 \varrho^2 + c_{10} \ell_x^2 n_{\Omega_x} n^2 \varrho^2, \tag{13b}
\end{aligned}$$

which can be written as the following depressed cubic equations:

$$an_{\Omega_x}^3 + bn_{\Omega_x} + c = 0, \tag{14a}$$

$$dn_{\Omega_y}^3 + en_{\Omega_y} + f = 0, \tag{14b}$$

where:

$$\begin{aligned}
a := & 16c_6 \ell_y^3 n^3 n_{\Omega_y}^{-2} \varrho^3 + 5c_7 \ell_x \ell_y^2 n^3 n_{\Omega_y}^{-1} \varrho^3 + c_8 \ell_y^2 n^2 n_{\Omega_y}^{-1} \varrho^2 + c_9 \ell_x \ell_y n^2 \varrho^2 \\
& + c_{10} \ell_x^2 n_{\Omega_y} n^2 \varrho^2, \tag{15a}
\end{aligned}$$

$$\begin{aligned}
b := & 10c_7 \ell_x^2 \ell_y n^3 \varrho^3 - 48c_6 \ell_x^2 \ell_y n^3 \varrho^3 - 5c_7 \ell_x^3 n_{\Omega_y} n^3 \varrho^3 - c_8 n_{\Omega_x} \ell_x^2 n^2 \varrho^2 \\
& + c_9 \ell_x^2 n_{\Omega_y} n^2 \varrho^2 - c_{10} \ell_x^2 n_{\Omega_y} n^2 \varrho^2 - 2c_2 \ell_x \ell_y (1+6n) n^2 \varrho^2, \tag{15b}
\end{aligned}$$

$$c := 10c_7 \ell_x^3 n_{\Omega_y} n^3 \varrho^3 - 32c_6 \ell_x^3 n_{\Omega_y} n^3 \varrho^3, \tag{15c}$$

$$d := 16c_6\ell_x^3 n^3 n_{\Omega_x}^{-2} \varrho^3 + 5c_7\ell_x^3 n^3 n_{\Omega_x}^{-1} \varrho^3 - 5c_7\ell_x^3 n^3 n_{\Omega_x}^{-2} \varrho^3 + c_8\ell_x^2 n^2 n_{\Omega_x}^{-1} \varrho^2 + c_9\ell_x^2 n^2 \varrho^2 - c_9\ell_x^2 n^2 n_{\Omega_x}^{-1} \varrho^2 - 2c_{10}\ell_x^2 n^2 \varrho^2 + c_{10}\ell_x^2 n^2 n_{\Omega_x}^{-1} \varrho^2 + c_{10}\ell_x^2 n_{\Omega_x} n^2 \varrho^2, \quad (16a)$$

$$e := 5c_7\ell_x\ell_y^2 n^3 \varrho^3 - 48c_6\ell_x\ell_y^2 n^3 \varrho^3 - 5c_7\ell_x\ell_y^2 n_{\Omega_x} n^3 \varrho^3 - c_8 n_{\Omega_x} \ell_y^2 n^2 \varrho^2, \quad (16b)$$

$$f := -32c_6\ell_y^3 n_{\Omega_x} n^3 \varrho^3. \quad (16c)$$

For $\mathcal{C} > 0$, and for all $\varrho, n_{\Omega_x}, n_{\Omega_y} > 0$, the coefficients a and d must be positive. In this case, sufficient conditions for the existence of global minimisers $n_{\Omega_x}^*, n_{\Omega_y}^*$ to (12) in the intervals $n_{\Omega_x}, n_{\Omega_y} > 0$ are solutions to (14) with three distinct real roots. In this case the roots sum to zero, meaning that at most, two roots will lie in the interval of interest $n_{\Omega_{x,y}} > 0$. With $a, d > 0$, the root with largest real part corresponds to a minimiser of \mathcal{C} and is the only minimiser in the interval of interest. With respect to (14a), existence of three distinct real roots is ensured if the following condition on the discriminant of the polynomial holds:

$$-4ab^3 - 27c^2 > 0. \quad (17)$$

A necessary condition for (17) is $b < 0$. Referring to (15b), we can ensure $b < 0$ by selecting n_{Ω_y} sufficiently large. For $n_{\Omega_y} \rightarrow \varrho$, to leading order the coefficients (15) are:

$$a \sim \mathcal{O}(+n_{\Omega_y} \varrho^2), \quad b \sim \mathcal{O}(-n_{\Omega_y} \varrho^3), \quad c \sim \mathcal{O}(\pm n_{\Omega_y} \varrho^3), \quad (18)$$

thus satisfying (17). A similar argument applies to the coefficients of (14b). For $n_{\Omega_x} \rightarrow \varrho$, to leading order the coefficients (16) are:

$$d \sim \mathcal{O}(+n_{\Omega_x} \varrho^2), \quad e \sim \mathcal{O}(-n_{\Omega_x} \varrho^3), \quad f \sim \mathcal{O}(-n_{\Omega_x} \varrho^3), \quad (19)$$

again satisfying the requirement that the discriminant of (14b) be positive. \square

We next establish how the complexity of evaluating the frequency response (11) scales with mesh density for the case when the number of molecules are optimal. This could be achieved by solving (14) and substituting the solutions into (12). However, a more tractable solution is afforded by instead considering the following problem. For a fixed number of molecules within the domain, let mesh density ϱ be the optimisation variable. As this varies, so too will the size (state dimension) of the molecules. Assume the number of molecules in all directions are simultaneously optimal for a particular (optimal) value of the mesh density. Then what mesh density is required to achieve this and what is the resulting complexity? The following theorem establishes this main result.

Theorem 3.2. *Given n_{Ω_x} and n_{Ω_y} such that $n_{\Omega_x} = n_{\Omega_x}^*$, $n_{\Omega_y} = n_{\Omega_y}^*$ for $\varrho = \varrho^*$, the overall complexity of evaluating (10) and (11) is $\mathcal{O}(\varrho^2)$.*

Proof. The complexity of evaluating (10) for each molecule, followed by (11), is given by (12). Since $n_x = \varrho\ell_x$ and $n_y = \varrho\ell_y$, then (12) is a polynomial in ϱ :

$$\mathcal{C} = \varrho^2 (a\varrho^2 + b\varrho + c), \quad (20)$$

subject to the constraint that $\mathcal{C} \geq 0 \forall \varrho \geq 0$, where:

$$a := c_1 \ell_x^3 \ell_y (1+n) n^2 n_{\Omega_x}^{-2}, \quad (21a)$$

$$\begin{aligned} b := & 48c_6 \ell_x^2 \ell_y n^3 n_{\Omega_x}^{-1} + 48c_6 \ell_x \ell_y^2 n^3 n_{\Omega_y}^{-1} + 16c_6 \ell_x^3 n_{\Omega_y} n^3 n_{\Omega_x}^{-2} + 10c_7 \ell_x^2 \ell_y n^3 \\ & + 16c_6 \ell_y^3 n_{\Omega_x} n^3 n_{\Omega_y}^{-2} - 10c_7 \ell_x^2 \ell_y n^3 n_{\Omega_x}^{-1} - 5c_7 \ell_x \ell_y^2 n^3 n_{\Omega_y}^{-1} + 5c_7 \ell_x^3 n_{\Omega_y} n^3 n_{\Omega_x}^{-1} \\ & + 5c_7 \ell_x \ell_y^2 n_{\Omega_x} n^3 n_{\Omega_y}^{-1} - 5c_7 \ell_x^3 n_{\Omega_y} n^3 n_{\Omega_x}^{-2}, \end{aligned} \quad (21b)$$

$$\begin{aligned} c := & c_8 n_{\Omega_y} \ell_x^2 n^2 n_{\Omega_x}^{-1} + 2c_8 \ell_x \ell_y n^2 + c_8 n_{\Omega_x} \ell_y^2 n^2 n_{\Omega_y}^{-1} + c_9 \ell_x^2 n_{\Omega_y} n^2 - c_9 \ell_x \ell_y n^2 \\ & - c_9 \ell_x^2 n_{\Omega_y} n^2 n_{\Omega_x}^{-1} + c_9 \ell_x \ell_y n_{\Omega_x} n^2 + c_{10} \ell_x^2 n_{\Omega_y} n^2 n_{\Omega_x}^{-1} - 2c_{10} \ell_x^2 n_{\Omega_y} n^2 \\ & + 32c_3 \ell_x \ell_y n^3 + c_{10} \ell_x^2 n_{\Omega_x} n_{\Omega_y} n^2 + 2c_2 \ell_x \ell_y (1+6n) n^2 n_{\Omega_x}^{-1} + 4c_4 \ell_x \ell_y n^2 \\ & + 8c_5 \ell_x \ell_y n, \end{aligned} \quad (21c)$$

and $c_1, \dots, c_{10} \in \mathbb{R}^+$ are unknown constants.

The minimum complexity is achieved when:

$$\frac{\partial \mathcal{C}}{\partial \varrho} = \varrho (4a\varrho^2 + 3b\varrho + 2c) = 0, \quad (22)$$

which has solutions:

$$\varrho_1 = -\frac{3b}{8a} + \frac{\sqrt{9b^2 - 32ac}}{8a}, \quad (23a)$$

$$\varrho_2 = -\frac{3b}{8a} - \frac{\sqrt{9b^2 - 32ac}}{8a}, \quad (23b)$$

$$\varrho_3 = 0. \quad (23c)$$

Since:

$$\frac{\partial^2 \mathcal{C}}{\partial \varrho^2} = 12a\varrho^2 + 6b\varrho + 2c, \quad (24)$$

and \mathcal{C} must be increasing when $\varrho = 0$,

$$\begin{aligned} \frac{\partial^2 \mathcal{C}(0)}{\partial \varrho^2} &= 2c > 0 \\ \therefore c &> 0. \end{aligned} \quad (25)$$

Given $a, c > 0$, for the non-trivial roots of (22) to be physical, i.e. $\varrho_1, \varrho_2 > 0$, the following must be true:

$$b < 0. \quad (26)$$

Analysis of (24), evaluated at the non-trivial roots, shows that the local minimum lies at $\varrho^* = \varrho_1$.

Substituting ϱ^* into (20) yields:

$$\begin{aligned} \mathcal{C}(\varrho^*) &= \frac{9b^2c}{32a^2} - \frac{c^2}{4a} - \frac{27b^4}{512a^3} + \frac{9b^3\sqrt{9b^2-32ac}}{512a^3} - \frac{2bc\sqrt{9b^2-32ac}}{32a^2} \\ &= c\varrho^{*2} + \frac{9b^3}{64a^2}\varrho^* + \frac{c}{4} \left(\frac{c}{a} + \frac{b\sqrt{9b^2-32ac}}{8a^2} \right) \\ &= \check{a}\varrho^{*2} + \check{b}\varrho^* + \check{c}, \end{aligned} \quad (27)$$

where:

$$\check{a} := c, \quad (28a)$$

$$\check{b} := \frac{9b^3}{64a^2}, \quad (28b)$$

$$\check{c} := \frac{c}{4} \left(\frac{c}{a} + \frac{b\sqrt{9b^2-32ac}}{8a^2} \right). \quad (28c)$$

Therefore for $\varrho = \varrho^*$, the complexity (20) reduces to (27), which is $\mathcal{O}(\varrho^2)$. \square

Hence, by chaining together molecules of optimal size, rather than atoms, the total complexity of evaluating the frequency response of the overall system reduces from $\mathcal{O}(\varrho^4)$ to $\mathcal{O}(\varrho^2)$ using the subsystem chaining method.

4. Application to 2D wave-diffusion equation

The effectiveness of the approach outlined above is next demonstrated upon a 2D wave-diffusion system, whose dynamics are described according to:

$$\frac{\partial^2 \varphi(x, y, t)}{\partial t^2} = c^2 \left(1 + k \frac{\partial}{\partial t} \right) \varphi(x, y, t), \quad \forall (x, y, t) \in \Omega \times [0, t_f], \quad (29)$$

where $\varphi(\cdot, \cdot, \cdot) : \Omega \times [0, t_f] \rightarrow \mathbb{R}$ is the height of the surface, $c, k \in \mathbb{R}^+$ are constants that dictate the wave propagation speed and rate of diffusion, respectively, $\Omega := [-2, 2] \times [-1, 1] \subset \mathbb{R}^2$ is a rectangular spatial domain with boundary $\partial\Omega$, $t_f \in \mathbb{R}^+$ is the endpoint of the time interval, and $(x, y) \in \Omega$ is a point in the domain. Periodic boundary conditions were assumed on all boundaries, i.e.:

$$\varphi(2, y, t) = \varphi(-2, y, t), \quad \forall (y, t) \in [-1, 1] \times [0, t_f], \quad (30a)$$

$$\varphi(x, 1, t) = \varphi(x, -1, t), \quad \forall (x, t) \in [-2, 2] \times [0, t_f], \quad (30b)$$

actuation takes the form of direct control of the surface height at $(x, y) = (1.5, 0)$:

$$u(t) = \varphi(1.5, 0, t), \quad (31)$$

and sensing is a measurement of the surface height in the centre of the domain:

$$y_{\text{sens}}(t) = \varphi(0, 0, t). \quad (32)$$

4.1. Full-order model

In order to derive a full-order model, the spatial domain was first discretised on a uniform computational mesh with mesh density $\varrho \in \mathbb{N}$ in both the x and y directions. As such, there were $n_x = 4(\varrho - 1) + 1$ and $n_y = 2(\varrho - 1) + 1$ computational nodes in the x and y directions, respectively, yielding a total of $n_{\text{total}} = n_x n_y$ nodes.

Second-order spatial derivatives in both directions were approximated using second-order accurate centred finite-differences, yielding the following differentiation matrices:

$$\mathcal{D}_{2,x} := \frac{1}{\delta_x^2} \begin{bmatrix} -2 & 1 & 0 & \cdots & 0 \\ 1 & -2 & 1 & & \\ 0 & 1 & -2 & 1 & \\ & & & \ddots & \\ \vdots & & & & 1 & -2 & 1 & 0 \\ 0 & \cdots & & & 1 & -2 & 1 & 0 \\ 0 & & \cdots & & 0 & 1 & -2 \end{bmatrix} \in \mathbb{R}^{n_x \times n_x}, \quad (33)$$

with a similar definition for $\mathcal{D}_{2,y}$. In each case, $\delta_x = \delta_y = \delta$ is the (uniform) mesh spacing in both the x and y directions. Periodic boundary conditions were implemented by setting the $(1, n_x)$ and $(n_x, 1)$ elements of $\mathcal{D}_{2,x}$ equal to one, and similarly for $\mathcal{D}_{2,y}$.

The Laplacian operator was approximated as (Trefethen, 2000):

$$\nabla^2 \approx \mathcal{L} := \mathbf{I}_{n_x} \otimes \mathcal{D}_{2,y} + \mathcal{D}_{2,x} \otimes \mathbf{I}_{n_y} \in \mathbb{R}^{n_{\text{total}} \times n_{\text{total}}}, \quad (34)$$

where \otimes denotes the Kronecker product. The resulting semi-discrete approximation of (29) is thus as follows:

$$\underbrace{\begin{bmatrix} \mathbf{E}_{11} & 0 \\ 0 & \mathbf{I}_{n_{\text{total}}} \end{bmatrix}}_{\mathbf{E}} \frac{d}{dt} \underbrace{\begin{bmatrix} \varphi(t) \\ \dot{\varphi}(t) \end{bmatrix}}_{\mathbf{x}(t)} = \underbrace{\begin{bmatrix} \mathbf{A}_{11} & \mathbf{A}_{12} \\ c^2 \mathcal{L} & c^2 k \mathcal{L} \end{bmatrix}}_{\mathbf{A}} \mathbf{x}(t) + \mathbf{B}u(t), \quad (35a)$$

$$y_{\text{sens}}(t) = \mathbf{C}\mathbf{x}(t), \quad (35b)$$

where $\varphi(t) \in \mathbb{R}^{n_{\text{total}}}$ is a vector of the values of $\varphi(x, y, t)$ at the nodes and $\mathbf{x}(t) \in \mathbb{R}^{2n_{\text{total}}}$ is the state vector. The matrices $\mathbf{E}_{11}, \mathbf{A}_{12} \in \mathbb{R}^{n_{\text{total}} \times n_{\text{total}}}$ are identity matrices, except for the element on the diagonal corresponding to the $\varphi(t)$ state in the actuation location being set equal to 0. Similarly, the matrices $\mathbf{A}_{11} \in \mathbb{R}^{n_{\text{total}} \times n_{\text{total}}}$ and $\mathbf{B} \in \mathbb{R}^{2n_{\text{total}} \times 1}$ are matrices of zeros except for the element on the diagonal (respectively vertical) corresponding to the $\varphi(t)$ state in the actuation location, which is set equal to 1 (respectively -1). Lastly, $\mathbf{C} \in \mathbb{R}^{1 \times 2n_{\text{total}}}$ is a vector of zeros except for the column corresponding to the $\varphi(t)$ state at the sensor location, which is set to 1. Note the algebraic constraint imposed by the actuation (31) causes \mathbf{E} in (35) to be singular.

Whilst this full-order model is a dynamically correct representation of the wave-diffusion system, the system matrices quickly become large and ill-conditioned as the computational mesh density is increased. The frequency response of (35) can be computed for frequencies of interest $\omega \in \mathbb{R}$ as:

$$\mathbf{G}(i\omega) = \mathbf{C}(i\omega\mathbf{E} - \mathbf{A})^{-1}\mathbf{B}. \quad (36)$$

Inverting the matrix $(i\omega\mathbf{E} - \mathbf{A}) \in \mathbb{C}^{n_{\text{total}} \times n_{\text{total}}}$ here is particularly costly as $n_{\text{total}} = n_x \times n_y = \rho\ell_x \times \rho\ell_y$, and so the complexity of this operation is $\mathcal{O}(\rho^6)$. Whilst this is tractable for the simple wave-diffusion equation on a relatively coarse computational mesh, and serves as a means of providing a benchmark model, it is not a practical approach in general.

4.2. Obtaining frequency response using the subsystem chaining method

Discretising (29) on the same computational mesh as the full-order model, again using second-order accurate centred finite-differences yields the following state-space representation for each atom within the domain ($1 \leq i \leq n_x$, $1 \leq j \leq n_y$):

$$\frac{d}{dt} \begin{bmatrix} \varphi_{i,j}(t) \\ \dot{\varphi}_{i,j}(t) \end{bmatrix} = \underbrace{\begin{bmatrix} 0 & 1 \\ -\frac{4c^2}{\delta^2} & -\frac{4c^2k}{\delta^2} \end{bmatrix}}_{\mathbf{A}_{i,j}} \underbrace{\begin{bmatrix} \varphi_{i,j}(t) \\ \dot{\varphi}_{i,j}(t) \end{bmatrix}}_{\mathbf{x}_{i,j}(t)} + \underbrace{\begin{bmatrix} \check{\mathbf{B}}_{i,j} & \check{\mathbf{B}}_{i,j} & \check{\mathbf{B}}_{i,j} & \check{\mathbf{B}}_{i,j} \end{bmatrix}}_{\mathbf{B}_{i,j}} \boldsymbol{\xi}_{i,j}(t), \quad (37a)$$

$$\mathbf{z}_{i,j}(t) = \underbrace{\begin{bmatrix} \mathbf{I} & \mathbf{I} & \mathbf{I} & \mathbf{I} \end{bmatrix}^\top}_{\mathbf{C}_{i,j}} \mathbf{x}_{i,j}(t), \quad (37b)$$

where $\varphi_{i,j}(t) \in \mathbb{R}$ is the value of $\varphi(x, y, t)$ at the location of the subsystem, $\boldsymbol{\xi}_{i,j}(t) \in \mathbb{R}^8$ and $\mathbf{z}_{i,j}(t) \in \mathbb{R}^8$ are defined as in Section 2,

$$\check{\mathbf{B}}_{i,j} := \begin{bmatrix} 0 & 0 \\ \frac{c^2}{\delta^2} & \frac{c^2k}{\delta^2} \end{bmatrix},$$

$\mathbf{A}_{i,j} \in \mathbb{R}^{2 \times 2}$, $\mathbf{B}_{i,j} \in \mathbb{R}^{2 \times 8}$, and $\mathbf{C}_{i,j} \in \mathbb{R}^{8 \times 2}$. Each atom has the frequency response:

$$\mathbf{P}_{i,j}(i\omega) = \mathbf{C}_{i,j} (i\omega\mathbf{I} - \mathbf{A}_{i,j})^{-1} \mathbf{B}_{i,j} \in \mathbb{C}^{8 \times 8}. \quad (38)$$

The atoms describing the nodes with actuation and sensing are altered as required (these are defined in Appendix A).

The overall system frequency response $\mathbf{G}(i\omega)$ was obtained from (7), both with and without the use of domain decomposition optimisation. The results are presented in Figure 5 for mesh density $\rho = 25$ and parameters $c = 0.5$ and $k = 0.1$. For these parameters, the optimal molecule size was $(n_{\Omega_x}^*, n_{\Omega_y}^*) = (25, 13)$. The frequency response obtained from the full-order model (35) is also plotted for comparison. As expected and despite the differences in the numerical implementations of the various methods, the subsystem chaining method produces identical frequency response data to those obtained from the full-order model, both with and without the domain decomposition optimisation.

From such frequency response data, one could then identify a low-order model and use this as the basis for designing a low-order controller.

4.3. Numerical aspects of subsystem chaining method

Using the various methods described above, point-wise frequency responses were computed on increasingly fine computational meshes in order to highlight the numerical

benefits of the subsystem chaining method, in terms of complexity reduction, memory usage and conditioning. Firstly, Figure 6 shows the wall-clock time T required to compute¹ $\mathbf{G}(i\omega)$ for a single frequency ($\omega = 1 \text{ rad s}^{-1}$), as a function of mesh density ϱ . It should be noted that the frequency response could not be computed from a full-order state-space system model for mesh resolutions as high as those considered for the subsystem chaining method as the memory requirements quickly became too high.

Since the complexity is $\mathcal{O}(\varrho^\beta)$ for some $\beta \in \mathbb{R}^+$, then for large ϱ we assume the following relationship between wall-clock time and complexity:

$$\begin{aligned} T &= \alpha \varrho^\beta, \\ \therefore \log_{10} T &= \log_{10} \alpha + \beta \log_{10} \varrho, \end{aligned}$$

where $\alpha \in \mathbb{R}^+$ is some unknown constant. The gradients of the plots in Figure 6 confirm that the complexity of computing the frequency response of a 2D system is:

- $\mathcal{O}(\varrho^6)$ directly from the full-order state-space system,
- $\mathcal{O}(\varrho^4)$ using the subsystem chaining method upon atoms,
- $\mathcal{O}(\varrho^2)$ using the subsystem chaining method upon molecules of optimal size,

as predicted in Sections 2.2–4.1.

Memory requirements and numerical conditioning were analysed by considering the memory $\mathcal{M} \in \mathbb{N}$ (bytes) required to store the largest matrix constructed, and the condition number $\kappa \in \mathbb{R}^+$ (Golub & Loan, 1996) of the most ill-conditioned matrix that required factoring, during computation. The values of \mathcal{M} and κ are plotted for increasing values of ϱ in Figures 7(a) and 7(b), respectively.

Both \mathcal{M} and κ are several orders of magnitude smaller for all ϱ considered when using the subsystem chaining method with domain decomposition optimisation than computing the frequency response from the full order state-space system, indicating favourable computational/numerical properties of the approach.

Of course, one could improve on the performance of the simplest approach (computing the frequency response from the full-order state-space system) by using sparse matrices and methods (Davis, 2006), for which numerical tests revealed a complexity of $\sim \mathcal{O}(\varrho^{2.4})$. This is still more expensive than the best approach above, and suffers from large condition numbers and the difficulty of forming the full-scale system matrices in the first place.

5. Conclusions

In this paper, a computationally tractable approach to obtaining low-order linear models for the purpose of feedback control of systems governed by spatially discretised PDAEs was presented. The low-order nature of such systems is exposed by the frequency response, and so a method was developed for computing this by combining the individual frequency responses of numerous interconnected subsystems. This avoided the need to construct, store, or operate upon the extremely large full scale system matrices that arise as a result of spatial discretisation.

For the two spatial dimension case, it was shown that the complexity of the method

¹All computations were carried out using IEEE standard 754 double-precision floating point arithmetic on a 3.40GHz Intel Core i7 (quad core) machine with relative machine precision $\varepsilon = 2.2 \times 10^{-16}$.

was $\mathcal{O}(\varrho^4)$ when combining the smallest subsystems (atoms). However, the key result of the paper established that this complexity reduced to $\mathcal{O}(\varrho^2)$ by first forming larger subsystems (molecules) of optimal size, before combining these together.

The modelling approach was demonstrated by application to a 2D wave-diffusion system, and the resulting frequency response was in exact agreement with that obtained from a full-order state-space model. Results were presented which showed the computational complexity scaled as predicted, and that the proposed modelling approach was far better conditioned and required less memory than methods relying upon large-scale system matrices.

The method of combining subsystems assumed a snake-like sequence that was motivated by heuristic argument, and so it remains to be proven that this is indeed the optimal sequence. Further future work involves applying these techniques for developing controllers for flow-control problems.

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Appendix A. 2D wave-diffusion equation actuation and sensing subsystems

A.1. Actuation

The actuation node has a descriptor state-space representation:

$$\underbrace{\begin{bmatrix} 0 & 0 \\ 0 & 1 \end{bmatrix}}_{\mathbf{E}_{\text{act}}} \frac{d}{dt} \begin{bmatrix} \varphi_{\text{act}}(t) \\ \dot{\varphi}_{\text{act}}(t) \end{bmatrix} = \underbrace{\begin{bmatrix} -1 & 0 \\ -\frac{4c^2}{\delta^2} & -\frac{4c^2k}{\delta^2} \end{bmatrix}}_{\mathbf{A}_{\text{act}}} \underbrace{\begin{bmatrix} \varphi_{\text{act}}(t) \\ \dot{\varphi}_{\text{act}}(t) \end{bmatrix}}_{\mathbf{x}_{\text{act}}(t)} + \underbrace{\begin{bmatrix} \check{\mathbf{B}}_{i,j} & \check{\mathbf{B}}_{i,j} & \check{\mathbf{B}}_{i,j} & \check{\mathbf{B}}_{i,j} \\ 1 \\ 0 \end{bmatrix}}_{\mathbf{B}_{\text{act}}} \begin{bmatrix} \boldsymbol{\xi}^{\text{sens}}(t) \\ u(t) \end{bmatrix}, \quad (\text{A1a})$$

$$\mathbf{z}_{\text{act}}(t) = \underbrace{\begin{bmatrix} \mathbf{I} & \mathbf{I} & \mathbf{I} & \mathbf{I} \end{bmatrix}^{\top}}_{\mathbf{C}_{\text{act}}} \mathbf{x}_{\text{act}}(t), \quad (\text{A1b})$$

with corresponding frequency response:

$$\mathbf{G}_{\text{act}}(i\omega) = \mathbf{C}_{\text{act}} (i\omega \mathbf{E}_{\text{act}} - \mathbf{A}_{\text{act}})^{-1} \mathbf{B}_{\text{act}} \in \mathbb{C}^{8 \times 9}. \quad (\text{A2})$$

A.2. Sensing

The sensing node has a state-space representation:

$$\frac{d}{dt} \begin{bmatrix} \varphi_{\text{sens}}(t) \\ \dot{\varphi}_{\text{sens}}(t) \end{bmatrix} = \underbrace{\begin{bmatrix} 0 & 1 \\ -\frac{4c^2}{\delta^2} & -\frac{4c^2k}{\delta^2} \end{bmatrix}}_{\mathbf{A}_{\text{sens}}} \underbrace{\begin{bmatrix} \varphi_{\text{sens}}(t) \\ \dot{\varphi}_{\text{sens}}(t) \end{bmatrix}}_{\mathbf{x}_{\text{sens}}(t)} + \underbrace{[\check{\mathbf{B}}_{i,j} \quad \check{\mathbf{B}}_{i,j} \quad \check{\mathbf{B}}_{i,j} \quad \check{\mathbf{B}}_{i,j}]}_{\mathbf{B}_{\text{sens}}} \boldsymbol{\xi}_{\text{sens}}(t), \quad (\text{A3a})$$

$$\begin{bmatrix} \mathbf{z}_{\text{sens}}(t) \\ y_{\text{sens}} \end{bmatrix} = \underbrace{\begin{bmatrix} \mathbf{I} & \mathbf{I} & \mathbf{I} & \mathbf{I} & \begin{bmatrix} 1 \\ 0 \end{bmatrix} \end{bmatrix}^{\top}}_{\mathbf{C}_{\text{sens}}} \mathbf{x}_{\text{sens}}(t), \quad (\text{A3b})$$

with corresponding frequency response:

$$\mathbf{G}_{\text{sens}}(i\omega) = \mathbf{C}_{\text{sens}} (i\omega \mathbf{I} - \mathbf{A}_{\text{sens}})^{-1} \mathbf{B}_{\text{sens}} \in \mathbb{C}^{9 \times 8}. \quad (\text{A4})$$

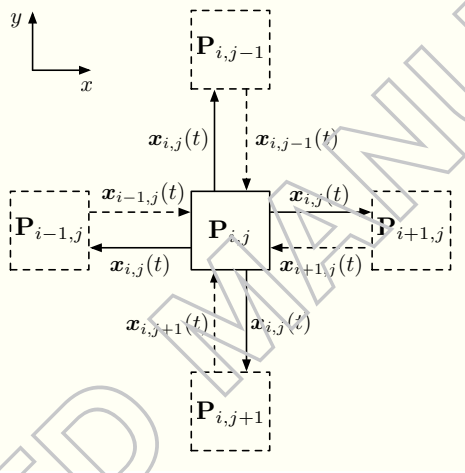


Figure 1. Interconnection of neighbouring nodal subsystems (atoms) after spatial discretisation.

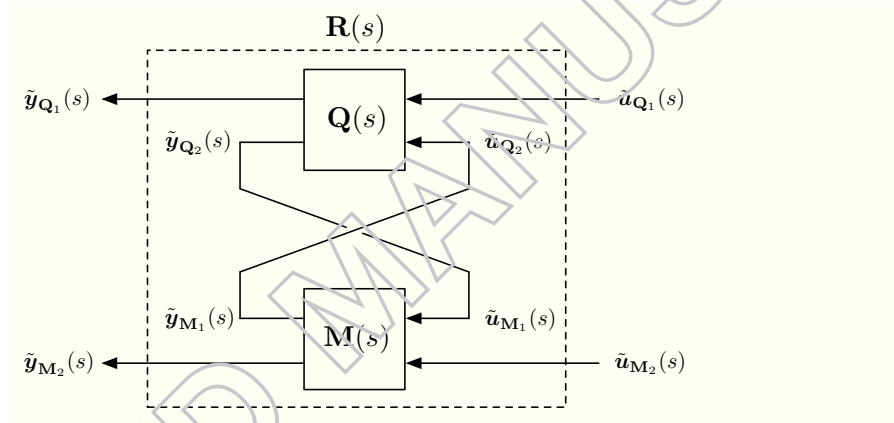


Figure 2. Redheffer star product between two systems.

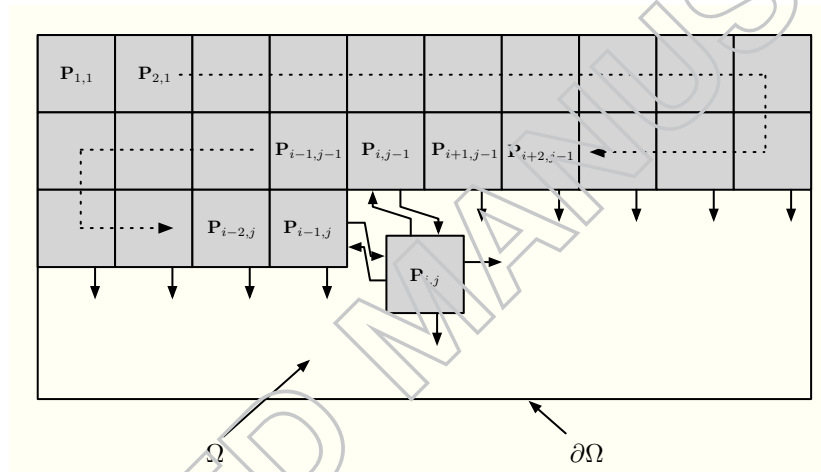


Figure 3. Connecting atoms together to obtain overall discretised PDAE system frequency response.

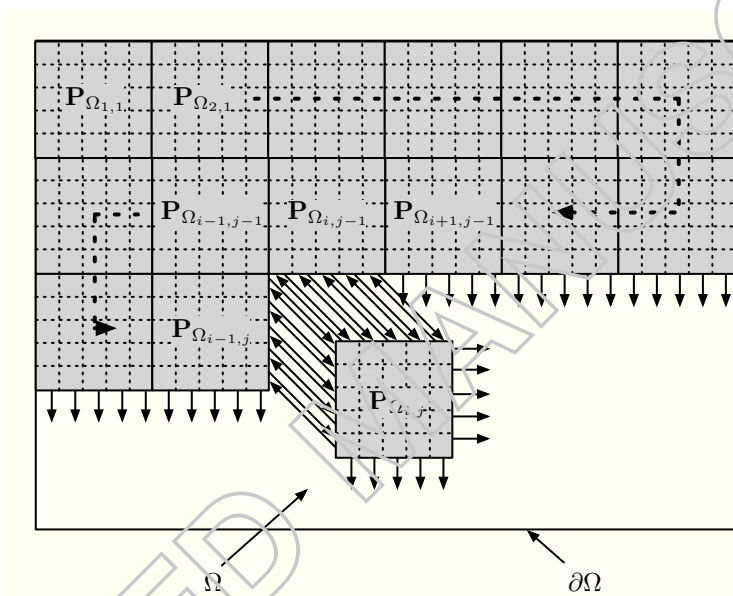


Figure 4. Connecting molecules together via domain decomposition to obtain the overall system frequency response. The point-wise frequency response of each molecule $\mathbf{P}_{\Omega_{i,j}}(i\omega)$ is first computed by chaining constituent atoms together. Subsequent chaining of molecules (as shown) using the Redheffer star product then yields the point-wise frequency of the overall system.

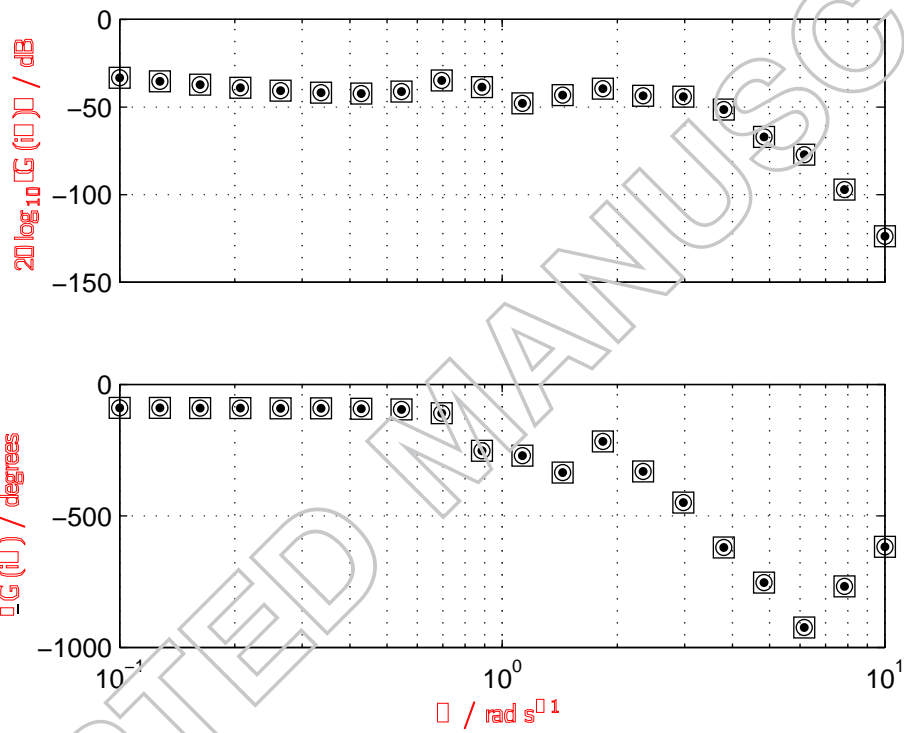


Figure 5. Frequency response of 2D wave-diffusion system: from full-order system (\square), using subsystem chaining method both with (\circ) and without (\bullet) domain decomposition optimisation.

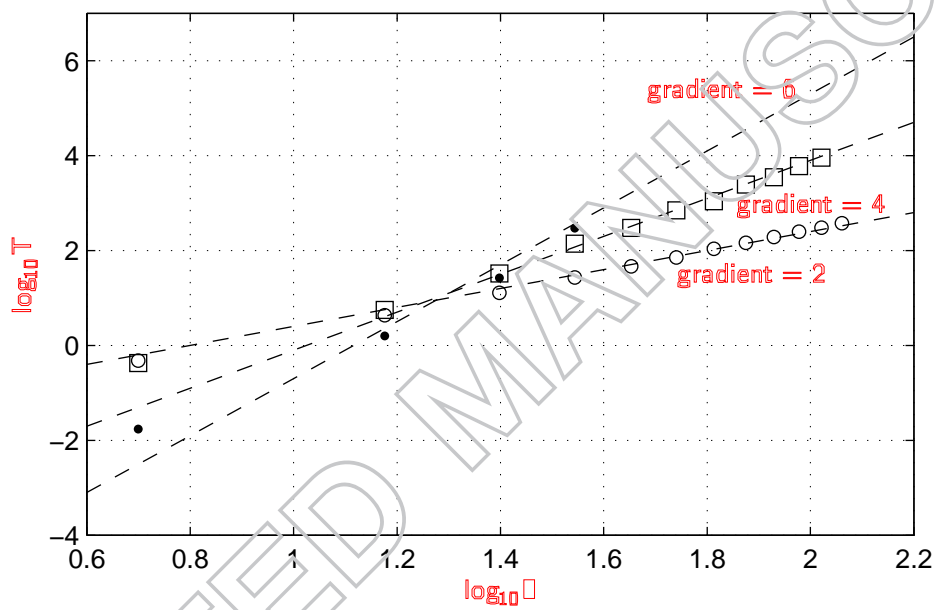


Figure 6. log of wall-clock time T (seconds) required to obtain $\mathbf{G}(i\omega)$ for a single frequency as a function of $\log_{10} n$: from full-order state-space system (\bullet), using subsystem chaining method both with (\circ) and without (\square) domain decomposition optimisation.

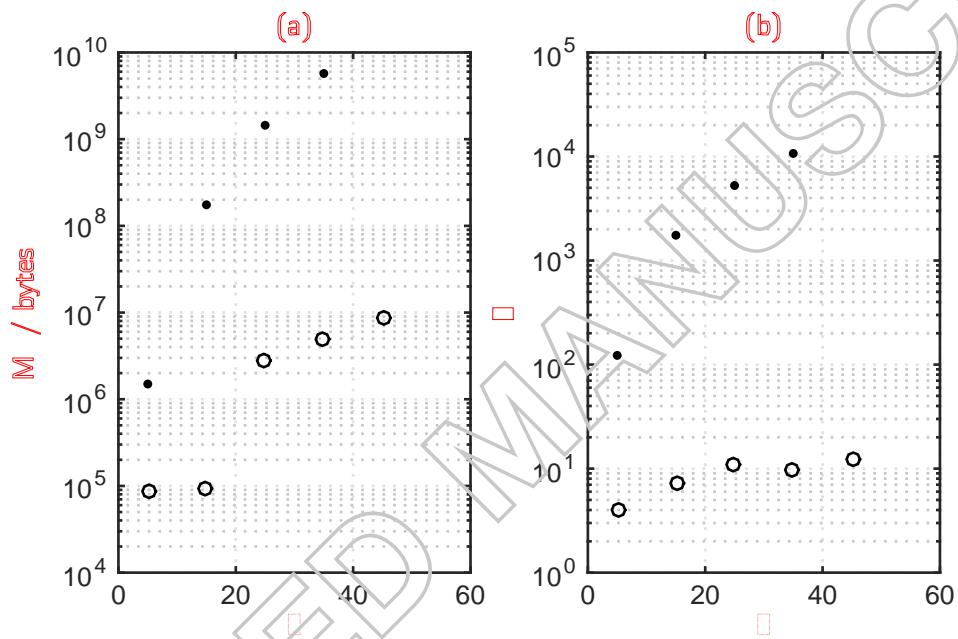


Figure 7. (a) memory \mathcal{M} required to store largest matrix constructed during computation of frequency response; (b) condition number κ of most ill-conditioned matrix which required inversion during computation of frequency response: from full-order state-space system (\bullet), using subsystem chaining method with domain decomposition optimisation (\circ).

Table 1. Computational complexity of matrix operations.

matrix operation	big-O complexity (flops)
addition, $\mathbf{A} + \mathbf{B}$, where $\mathbf{A}, \mathbf{B} \in \mathbb{C}^{a \times b}$	$\mathcal{O}(ab)$
multiplication, \mathbf{AB} , where $\mathbf{A} \in \mathbb{C}^{a \times b}$, $\mathbf{B} \in \mathbb{C}^{b \times c}$	$\mathcal{O}(abc)$
inversion, \mathbf{A}^{-1} , where $\mathbf{A} \in \mathbb{C}^{a \times a}$	$\mathcal{O}(a^3)$