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Tsavdaridis, KD and Cedric, D (2011) Web buckling study of the behaviour and strength of perforated steel beams with different novel web opening shapes. *Journal of Constructional Steel Research*, 67 (10). 1605 - 1620 . ISSN: 0143-974X

<https://doi.org/10.1016/j.jcsr.2011.04.004>

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Web Buckling Study of the Behaviour and Strength of Perforated Steel Beams with Different Novel Web Opening Shapes

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1. Introduction

The use of perforated steel beams has resulted in longer span floors. Their popularity has also increased because of an architectural emphasis on exposed structures, with cellular, castellated and elongated web openings being typical in structural sections. Sections having webs penetrated by large closely spaced openings over almost the full span are now common. Although the best application of these beams appears to be for long spans which are to carry a light uniform load, some heavy-mass structures, such as bridges, have been constructed using perforated beams for the full span. Also, with greater automation, the cost of their fabrication has been reduced to the level where for certain applications they may be competitive with open-web steel joists [1].

In the last decade researchers have tried to examine the web opening shapes of perforated steel sections in order to provide a better understanding of the stress distribution in the vicinity of the web openings, and to identify those that have the best structural behaviour under certain types of loading [2,3]. The aim is to provide the maximum possible web opening area for the integration of services, whilst keeping the minimum possible self weight for different types of loading.

Abbreviations

b_e	Effective width of the strut
c	Critical opening length
d_o	Vertical web opening depth (opening diameter for circular web openings)
d_T	Depth of the stem of the tee-section
l_e	Effective length of the strut
M_{pt}	Plastic moment capacity of the top tee-section
p_c	Compressive strength of steel in BS 5950-1
p_y	Yield strength of steel in BS 5950-1
S	Web opening spacing (opening pitch)
s_o	Edge to edge spacing of the web openings (i.e. width of the web-post)
V_v	Vertical shear resistance due to web-post buckling
σ	Compressive stress acting on the strut

2. Objectives

The objective of this work is to investigate and compare, through an experimental programme of work and finite element (FE) study, the behaviour of perforated steel beams with various standard and non-standard web opening shape configurations. It is therefore necessary to review existing information on web-post buckling [4,5,6] and to examine this mode of behaviour in light perforated beams with closely spaced web openings of various shapes. This work also seeks to review the design model of circular web openings and develop design models for the novel web openings, when they are closely spaced, for the complex web-post buckling failure mode.

The sub-objectives of this research are as follows:

- To observe the structural behaviour of perforated beams with closely spaced web openings in the elastic and in the plastic range.
- To examine both the load carrying capacities and the failure modes of perforated sections.
- To examine the positions of high stress concentration points in the vicinity of the web openings.
- To correlate the experimental results with FE models, and thoroughly investigate their complex structural behaviour in terms of buckling load, stress distribution and failure mode.
- To conduct an extensive FE parametric study on web-posts and then to devise a buckling model that can be used for perforated sections with various closely spaced novel web opening shapes.

3. Experimental program

3.1 Introduction

Experimental work was conducted on seven full scale steel perforated beams with various web openings in order to validate the finite element global study. Thereafter, the appropriate conditions and input data were used in the FE study to simulate the local web-post failure for all of the web opening shapes and conduct a comprehensive parametric study.

3.1 Details of specimens

The specimen size was chosen to cover a large range of thin web beams mostly utilised as roof beams. The depth of the web openings should be low enough to prevent Vierendeel effects prior to web-post buckling failure in perforated sections with relatively thin webs. Hence, a diameter, d_o , equal to $0.7h$ was adopted. The advantage of this selection is that constant shear forces are generated and hence web-post buckling failure is expected. A widely used section UB457x152x52 of steel grade S355 was chosen. Seven beams were tested in the laboratories at City University London, all with a span of 1.7m. In all seven beams the position of the web-post is the same in relation to the reactions and so the moment-shear ratio (M/V) could easily be found. The beam is symmetrical about the mid-span centre-line. Two different sets of tests were carried out.

- The first category consists of specimens A1, A2 and A3. One is a circular cellular beam (A1) and the other two cellular beams with fillets of 25mm (A2) and 45mm (A3) introduced at the mid-depth of the web openings, as shown in **Fig. 1**. The web opening spacing was kept the same ($S = 1.3d_o$), and hence the critical web-post width decreases as the fillet radius was increased. With the advantage of profile cutting, the bigger the radius of the fillet, the larger is the depth of the final section and hence its second moment of area.
- In the second category specimens B1, B2, B3 and B4 with different web opening shapes were investigated. These consist of one circular cellular beam (B1), two perforated sections with

novel vertical elliptical web openings (B2 and B3) as well as one perforated section with novel inclined elliptical web openings (B4) (see **Fig. 1**). These web opening shapes consist of a combination of circular and straight lines. The concept was arrived at using a combination of a circular and a hexagonal web opening. The potential of these two shapes was of interest. The critical web-post width was kept the same ($s_o = 0.2d_o$), and thus the horizontal shear capacity at the web-post was investigated for each case.

Geometric details for the design of these novel web openings shapes are given in **Fig. 1** and **Table 1**. Further details of the experimental and analytical work can be obtained from Tsavdaridis et al. [7,8].

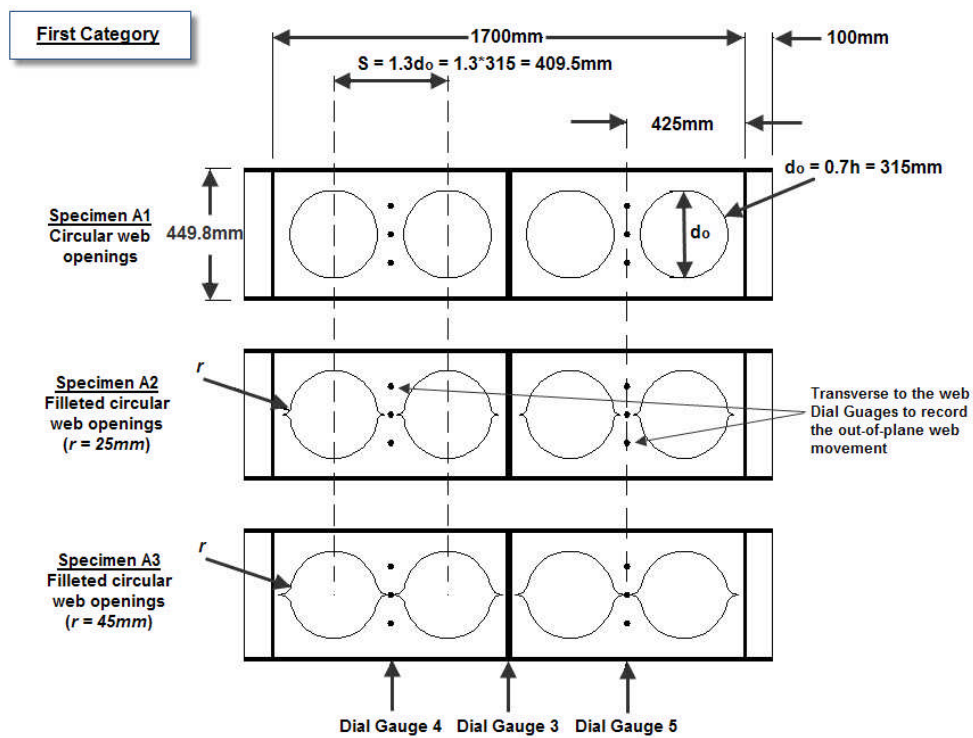


Fig. 1a: Specimen details and gauging equipment (First Category)

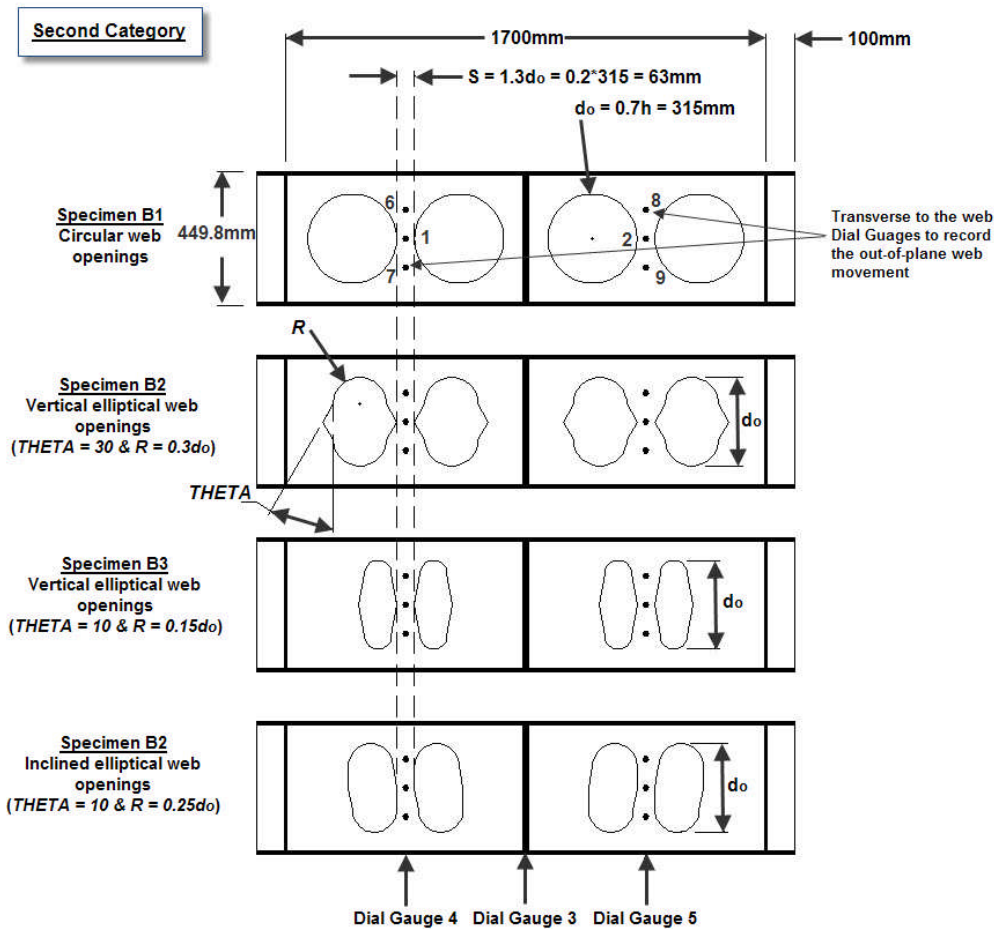


Fig. 1b: Specimen details and gauging equipment (Second Category)

Table 1

The details of experimental tests.

Test	Specimen	UB section	Type	Specifications	Web-Post Width (mm)	Opening Area (mm ²)
First Category	A1	457x152x52	Circular	-----	94.5	77931
	A2	457x152x52	Filleted Circular	$r = 25\text{mm}$ of fillets	48	78361
	A3	457x152x52	Filleted Circular	$r = 45\text{mm}$ of fillets	15	79129
Second Category	B1	457x152x52	Circular	-----	63	77931
	B2	457x152x52	Vertical Elliptical	$THETA = 30$ and $R = 0.3d_o$	63	56452
	B3	457x152x52	Vertical Elliptical	$THETA = 10$ and $R = 0.15d_o$	63	32138
	B4	457x152x52	Inclined Elliptical	$THETA = 10$ and $R = 0.25d_o$	63	45383

3.2 Coupon tests

To determine the mechanical strength of the steel I-sections used for these tests, tensile coupon tests were carried out to the specifications and guidelines in **BS EN 10002-1** [9]. Coupons were taken from two un-yielded locations (one flange and one web) of the four steel beams tested. The yield (f_y equal to 375.3MPa and 359.7MPa for web and flange/stiffener, respectively) and the

ultimate stresses (f_{ult} . equal to 492.7MPa and 480.9MPa for web and flange/stiffener, respectively) were determined by averaging the results obtained from all the tensile coupon tests.

3.3 Test set-up

To ensure load distribution from supports and load and to prevent failure by lateral torsional buckling (LTB) and buckling due to compression, bearing stiffeners were utilised at supports and mid-span together with a loading plate (**Fig. 2**). The thickness of the stiffeners at supports is similar to the thickness of the flange (i.e. $t_f = 10.9mm$), while the thickness of the stiffeners at mid-span is $20mm$. The loading plate was designed to distribute the load, especially in the case in which the web opening edge is very close to the load centre-line. The thickness of the loading plate is $10mm$. The apparatus for the testing program was arranged in such a way that a bending moment was applied at the mid-span web post of the beams without inducing significant axial forces at the edges of the web openings. Data acquisition equipment was used to monitor the tests and alter the loading increments for more precise results. Also, it provided a means of identifying the onset of web-post buckling during the test.

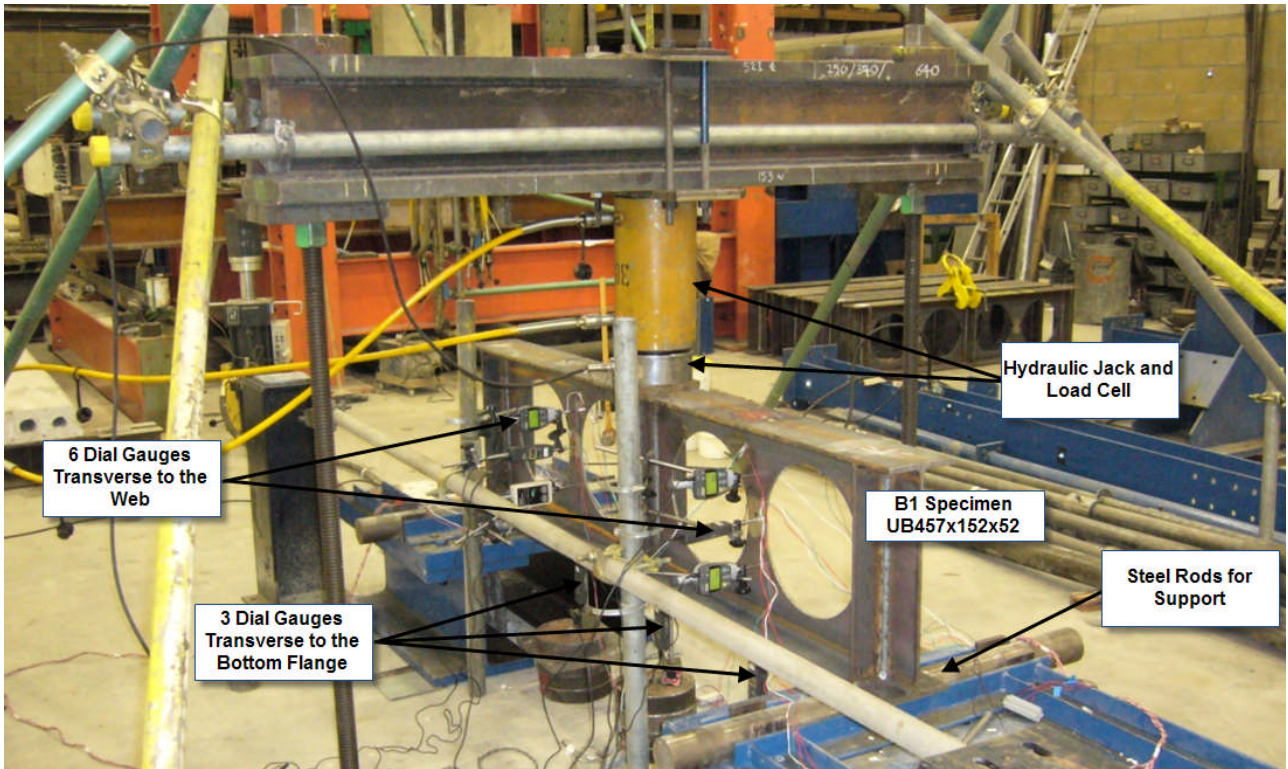


Fig. 2: Test set-up arrangement

3.4 Test procedure

The simply supported beam was loaded at mid-span through a hydraulic jack. The load application is conducted by controlled deformation with a low displacement rate. The test procedure followed was incrementally monotonic loading to failure, followed by full unloading. Loading was continued after the ultimate load was reached in order to examine the post-elastic failure behaviour and the strain hardening slope.

The specimens were heavily gauged, as shown in **Fig. 1**, in an attempt to identify a primary mode of failure i.e. either a ‘buckling load’ for the web-posts, the formation of a yield mechanism (plastic hinges), or local buckling of the flange.

In the following description of the test results, critical load, P_{cr} , is the load on beam of first visible web-post buckling or rupture, which is higher than the yield load, P_y . The ultimate load

carrying capacity, $P_{ult.}$ is the maximum load value recorded in the test, usually followed by a sudden drop of load shortly afterwards.

4 FEA study

4.1 Introduction

Global finite element analyses on seven full scale steel perforated beams with various web openings were conducted and compared with the experimental results. The aim of this global study was to validate the FE model using the experiments in order for it to be used for the parametric study, as well as to compare the results of nominal steel properties and the actual ones taken from coupon tests.

4.2 Model description

A skeleton model based on beam geometry was initially developed for each of the test specimens. Analysis of the web-post using the finite element method (FEM) was based on ANSYS v11.0 [10]. Similar FE analyses can be found in literature [11]. It should be noted that although the specimens used in the experimental work had visible imperfections in their geometry, perfect models were used for modelling purposes. SHELL181 plastic elements were used for this work, with an average maximum size of 20mm in the vicinity of the web openings, as determined from a convergence study [7]. For specimens A2 and A3, in order to provide a better stress distribution in the vicinity of the fillets at the mid-depth of the web, much smaller elements were incorporated as such elements are more likely to be distorted. The material properties of the FE models were the same for all specimens. Linear elastic properties of the material were taken as 200GPa and 0.3 for Young's Modulus and Poisson's Ratio, respectively. For the plastic properties of the steel material, the average values from coupon tests were used (FEM 1 in **Fig. 4**). Additionally, a nominal yield strength value ($S355, f_y=355MPa$) was used for comparison (FEM 2 in **Fig. 4**). The Tangent Modulus value was assumed to be 580MPa for both the Multi-linear Isotropic Model (MISO) with

true branches (i.e. f_{ult} is used) and the Bi-linear Isotropic Model (BISO). The Newton-Raphson solution method was used for the analyses. For determining the post-elastic curve, the arc-length method was activated to help avoid bifurcation points and track unloading. Initial imperfections with maximum amplitude of $t_w/200 = 7.6/200 = 0.038$ were also taken into account.

4.3 Model verification

The above model was primarily calibrated against experimental work found in the literature [1] and extensively presented in Tsavdaridis et al. [7,8]. This work led to the proposed FEA model being used to calculate the ultimate strength and load versus deformation curves for all seven perforated beams subjected to buckling load.

5 Experimental and FE results

5.1 Discussion of the results

Each specimen is discussed separately. The order of yielding, mode of failure and comparison of actual to calculated FE performance is reported and a summary of the results provides overall comparison. All measurements from dial gauges are shown in **Fig. 3** as they are useful when studying the web-post buckling as shown in **Fig. 4** which depicts the highly distorted web-post on every specimen after failure. Relative predictions from the FEM models are also presented together with the schematic representation of Von-Mises stresses distribution (**Fig. 4**). The Von Mises yield criterion has been adopted to define the yielding point of the steel.

First category includes perforated sections with circular web openings with and without fillets:

Specimen A1

Web-post buckling movement of the web showed very small values up to the yield load, where the tee-sections were symmetrically displaced in opposite directions. The first visible yield was located in the “left” half-span of the beam and at a load of 257kN (i.e. P_{cr}). From the dial gauge

readings it was seen that both sides experience high yielding and web-post buckling by the end of the test. The ultimate load, $P_{ult.}$, carried by the specimen was 289kN, whilst the beam behaves plastically with constant strain hardening to a deflection of almost 65mm.

FE models significantly overestimated the experimental values and the initiation of buckling and the ultimate load carrying capacity are 26% and 17% higher. After reaching the ultimate load carrying capacity there is a sudden loss of stiffness in the FE model, something which was observed in the tested beam too, but with less effect. Von-Mises stresses in FE model show the yielding of the compression area; the yielding of top portion of the flange and the yielding of the web extending into the flange, representing the secondary effects of shear.

Specimen A2

Very small values of web-post buckling movement were seen up to the yield load, where the tee-sections move symmetrically in opposite directions. The first visible yield was located in the “left” half-span of the beam and at a load of 267kN (i.e. P_{cr}). From the dial gauge readings it was seen that both sides experience high yielding and web-post buckling at the end of the test. Following full yielding at 295kN, which is the ultimate load, $P_{ult.}$, a sudden drop of load was observed. However, as the specimen was loaded further, it behaved plastically to a deflection of almost 50mm.

FE analyses diverged at a higher load than that in the test, whereas their maximum vertical deflection is much lower. Analytically, web-post buckling was observed prior to yielding, similar to that observed in the test, with the ultimate load carrying capacity being only 10% higher in FE model. A sudden drop with loss of strength was observed, with the beam behaving plastically with some strain hardening. Both sides of the beam deflected symmetrically but with different magnitude. The yielding of the top flange and that of the web extending into the flange were seen in

the Von-Mises stresses contour plot. Slightly higher stresses were concentrated at the fillets, but they were not the cause of failure.

Specimen A3

From the first load steps, web movement was recorded and a web-post buckling profile was observed from the transverse web displacement readings. At a load of approximately 80kN, a second buckling mode occurred and the web moved in the opposite direction. Slight web-post movement was recorded followed by rupture of the “right” web-post at a load of 164kN (i.e. P_{cr}). A lower strain was observed from the rectangular strain gauges at this side. This was followed by a sudden rupture after two load steps at the “left” web-post and a load of 156kN. Plastic behaviour was followed by strain hardening to a deflection of almost 5mm. After both web-posts had ruptured, an increase of strain hardening was obtained, with an ultimate load carrying capacity, $P_{ult.}$, of 186kN. This occurred as the top and bottom tee-sections moved together in-plane and so provide some stiffness. Large deflection finally resulted in the decrease of the load carrying capacity and then total failure.

Modelling the structural behaviour of such a specimen was found to be challenging especially for the very narrow web-post. Failure of the model was taken as being directly after rupture of the steel web-post. It was found that the strength of the FE models is dramatically higher than the tested specimen. Finally, the top and bottom tee-sections moved together in-plane providing some stiffness, before high beam deflections and final failure occurs.

Second category includes perforated sections with circular and novel elliptical web openings:

Specimen B1

The recorded web-post movement showed very small values up to the yield load, when the tee-sections moved symmetrically in opposite directions. First visible yield was located in the “right”

half-span of the beam at a load of 220kN (i.e. P_{cr}). Dial gauge readings showed that both sides experience high yielding and web-post buckling at the end of the test. As the specimen was loaded, the yielding portion distorted considerably. The ultimate load carried, $P_{ult.}$, was 256kN, while perfectly plastic behaviour was observed followed by strain hardening to a deflection of 27mm. Von-Mises stresses in FE model showed the yielding of the compression area and the yielding of top portion of the flange and the yielding of the web extending into the flange. This represents well the secondary effects of shear.

The results of the FE models are in good agreement with the experimental values. It can easily be seen that both web-posts were highly stressed, but with only one side highly distorted before divergence of the FE model. The final load was taken to be 275kN, slightly higher than the experimental value. A comparison between Specimen A1 and B1 showed how the web-post width affects the load carrying capacity of cellular beams.

Specimen B2

The first visible web-post movement was observed at a load of 387kN (i.e. P_{cr}) and the first failure mode occurred at a load of 402kN (i.e. $P_{ult.}$). Deformation of the highly distorted web-post continued until a significant sudden fall in and a deflection of almost 50mm. Transverse web dial gauges confirmed that only the “right” side experienced web-post buckling up to the end of the test. The experiment was stopped when the “right” web-post had dramatically distorted.

Web movement was recorded from the beginning of the analysis, similar to that in the tested beam. Due to the very steep post-elastic curve, as observed in the experiments too, FE models diverged after reaching the ultimate load carrying capacity. The model was unable to redistribute the stresses, with the high stress concentration at the narrow web-post with the sharp corner edges causing sudden failure. This web-post shape, with such closely spaced web openings, caused

locally stressed areas as seen from the Von-Mises stresses. Both sides of the beam experienced buckling, however only one side was highly distorted at the end of the test.

Specimen B3

The first visible web-post movement was observed at the “left” side and at a load of 387kN (i.e. P_{cr}). Following the ultimate load carrying capacity of 416kN, $P_{ult.}$, the distortion continued as load was gradually falling off. The transverse web dial gauges confirmed that only the “right” side experienced web-post buckling by the end of the test. The experiment was stopped when the “left” web-post had dramatically distorted at a maximum mid-span deflection of 27mm. It is worth mentioning that in this particular specimen, stresses at both web-posts moved to reasonable values.

The FE analyses showed similar behaviour in both the elastic and plastic regions. After reaching the ultimate load carrying capacity, the web became highly distorted whilst the beam was gradually losing strength. Some strain hardening caused a double curvature at the plastic region. In contrast to Specimen B2 and despite the fact that the web-post width is identical ($s_o = 0.2d_o = 63mm$), very low stresses were observed at the narrower part of it. Horizontal shear of the web-post is not a critical factor when such web openings are utilised. High Von-Mises stresses (i.e. plastic hinges) were concentrated at the top and bottom tee-sections. Again, the yielding of the flange and web was shown.

Specimen B4

Web-post buckling movement of both web-posts showed very small values up to the yield load, with the transverse displacement at the “right” web-post being slightly higher. The first visible yield was located in the “left” half-span of the beam and at a load of 280kN (i.e. P_{cr}). From dial gauges readings it was seen that only the “left” side experienced high yielding and web-post buckling at the end of the test. On the “left” side there was high distortion, while the “right” half span no web-post buckling was observed up to a load of 327kN. The ultimate load, $P_{ult.}$, carried by

the specimen was 346kN, with a deflection of almost 40mm. Again, in this particular specimen, stresses at both web-posts moved to a reasonable range.

Specimen B4 is very similar to the Specimen B3 and the web-post width is the same ($s_o = 0.2d_o = 63mm$). Analytically, there was a significant strain hardening in the plastic region, which made the ultimate load carrying capacity much higher than that in the tested beam. After reached the ultimate load level and similar to the test, there was a gradual loss of strength. Similar to Specimen B3, high Von-Mises stresses (i.e. plastic hinges) were concentrated at the top and bottom tee-sections. The yielding of top portion of the flange and web extending into the flange were shown. Finally, it should be mentioned that when such inclined web opening were considered, the results did not differ much and the web-post buckling curvature appeared normal.

5.2 Summary of the results

Table 2 summarizes all test and FE results for the yielding load, buckling (or critical) load and ultimate loads. It is difficult to make a direct comparison of all the specimens as the new web opening configurations of the perforated sections include more than one parameter. However, some general conclusions can be drawn.

In the first category it was easily seen that when beams with circular web openings with 45mm of fillet were investigated, a very high stress concentration was observed at the notch point of the fillet. In the particular case, web-post rupture was obtained at a relatively low load level before any web-post buckling was observed. Conversely, when perforated beams with circular web openings with 25mm of fillet were examined, very similar results to the regular cellular beams (A1) were found. Although the minimum web-post width of Specimen A2 is only half the web-post width of the cellular beam, web and flange stresses, as well as vertical deflections and transverse web displacements, are very close. Moreover, the capacity of Specimen A2 was slightly higher, whereas the stiffness and the post-elastic strength were similar. Hence, in providing a lighter beam (A1), the

overall outcome can be better. In the second category, the most important comparison is between Specimen B1 and B2 as the latter has almost 70% of the cellular's web opening area, whilst its capacity is 1.6 times the capacity of B1. Comparing Specimens B2 and B3, it can be concluded that the critical opening length, c , and not the web opening area, affects the beam's capacity in this case. Highly distorted web-posts are shown in **Fig. 4**.

Fig. 5 shows the variation of ratio of the test loads to FEM loads plotted against the web opening areas (left), size of web opening and the web-post widths (right). These results suggest that a satisfactory prediction with low variability has been provided by the FE method, mainly at yield load level. P_y is very well estimated by FEM for all cases with ratios close to unity and independently from the web-post width or the web openings area and shape. For higher loads, test to FE ratios are independent of the web-post area but dependent on the web-post width.

It is well known that the linear buckling prediction of thin shell is purely theoretical and it should be reduced in order to account for the influence of geometric imperfection and other defects. Initially, buckling starts at local imperfections with the formation of outer and inner waves. Geometric imperfections caused by manufacturing are the main cause of significant differences between critical buckling loads calculated using FEA and experimental buckling loads. Even very small imperfections can cause a substantial drop in the buckling load. Regarding these particular specimens, a number of manufacturing imperfections were found, especially in the perforated beams with the novel web opening shapes, due to lack of manufacturing capabilities. Other factors responsible for the deviation of the experimental and FE results are the nominal dimensions, instrumentation errors and residual stresses.

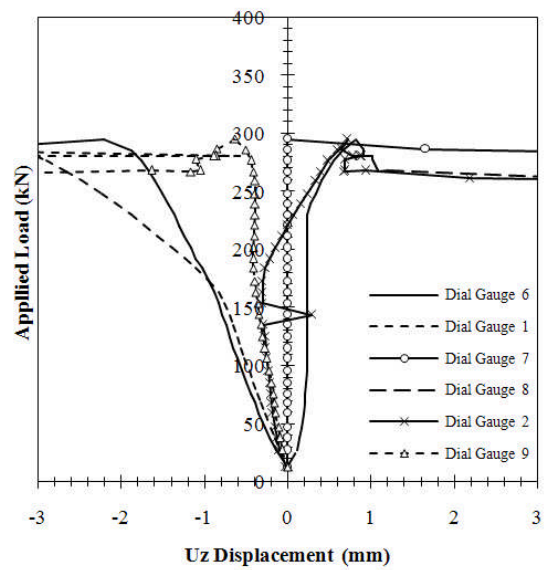
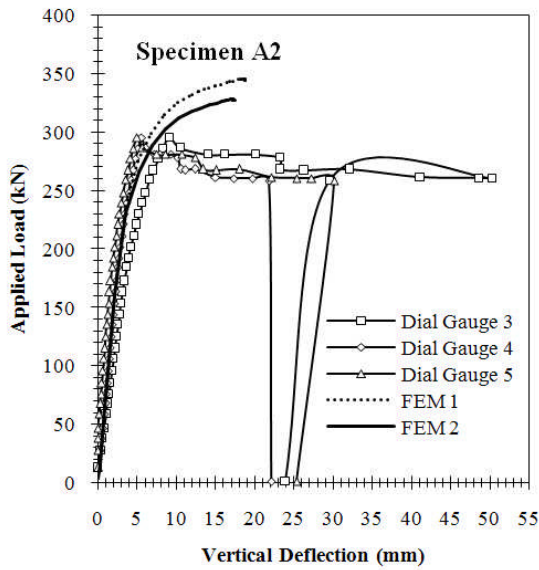
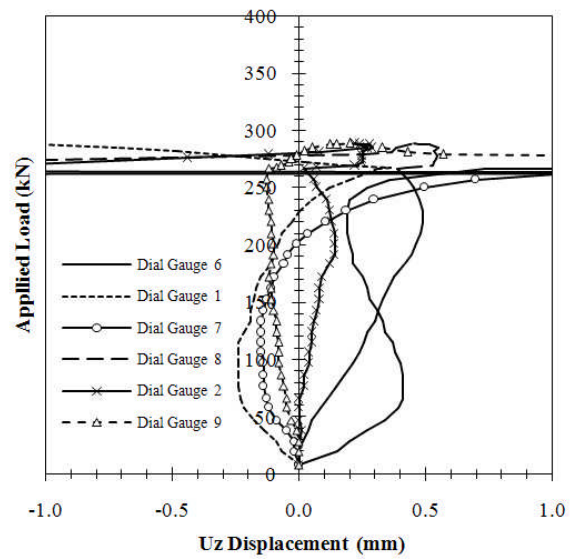
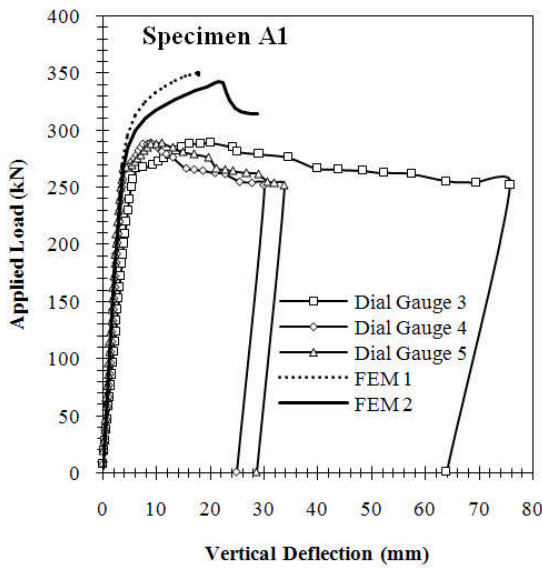
Table 2

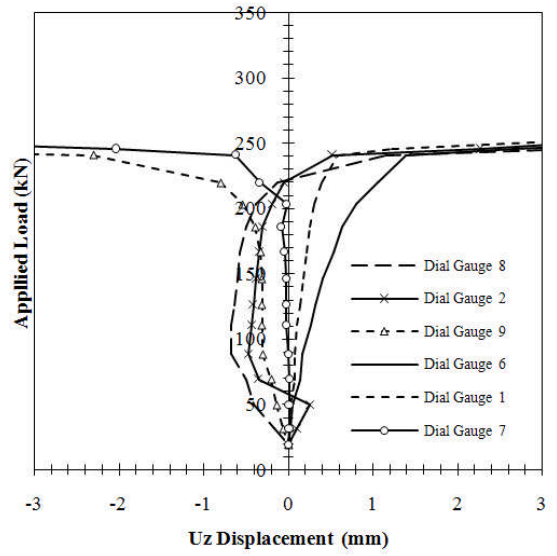
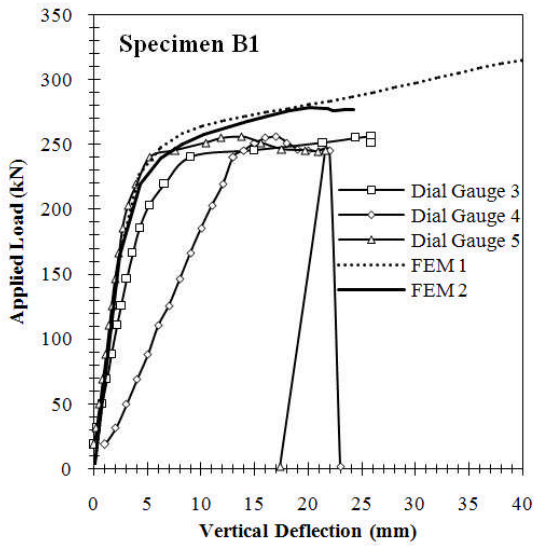
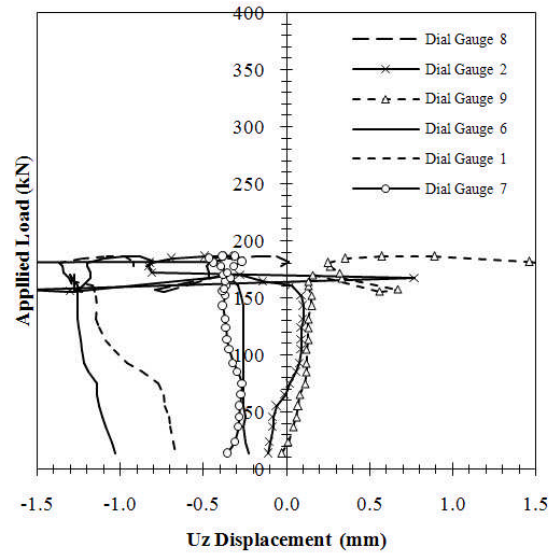
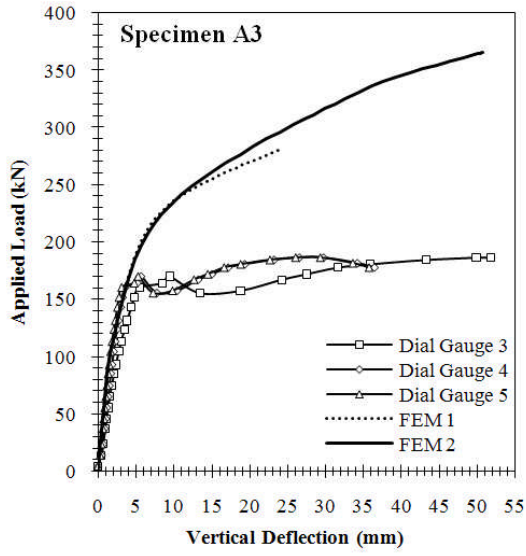
Summary of experimental and FEM results.

Spec.	P_y		P_{cr}		$P_{ult.}$		Max. δ @ P_{cr} (mm)	*Primary Failure Mode	**Strut-Tie Angles (degrees)
	FE (kN)	TEST (kN)	FE (kN)	TEST (kN)	FE (kN)	TEST (kN)			
A1	265	250	327	266.5	342	288.7	5.69	WPB	132/-44
A2	204	184.5	300	298	325	298	7.09	WPB	129/-44
A3	148	131.5	274	160.5	365	186.5	8.54	Mid-Post Rupture	---
B1	204	185.5	204	203	275	255	6.55	WPB	124/-50
B2	281	276	371	375	420	402.4	10.15	WPB (1 side)	135/-43
B3	362	354	310	370	426	415	6.92	WPB (1 side)	117/-54
B4	270	250	330	290	422	350.6	5.99	WPB (1 side)	127/-46

*WPB: Web-Post Buckling

**Strut-Tie Angles to the Horizontal Mid-depth Web-post Centre-line (Top Tee/Bottom Tee)





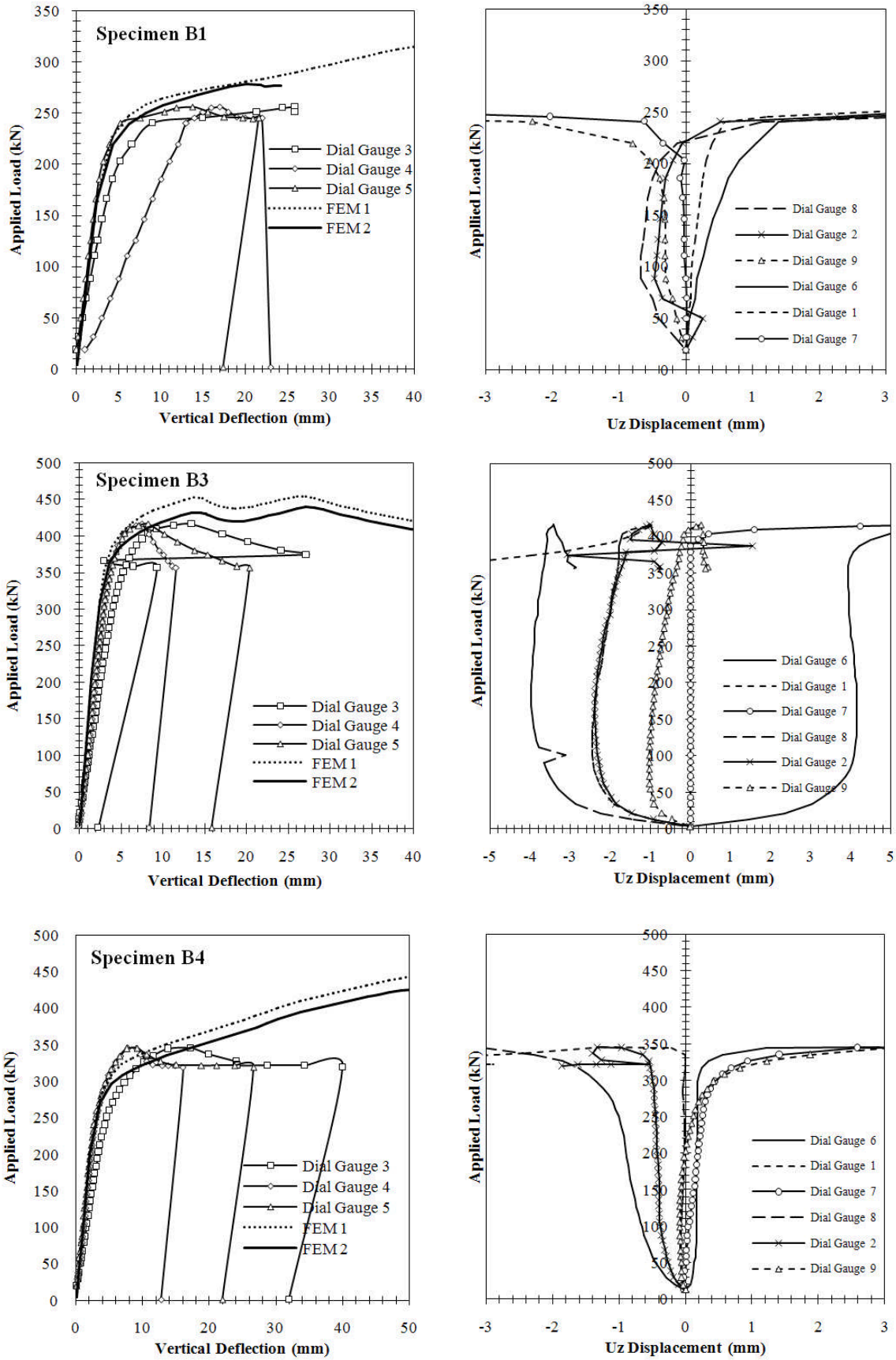
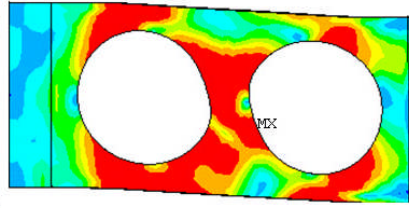
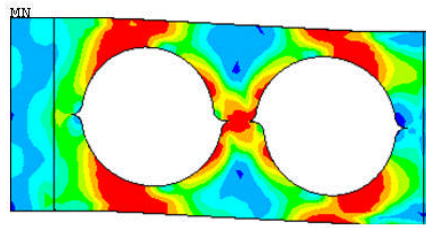


Fig. 3: Comparison of experimental and FEM extracted results

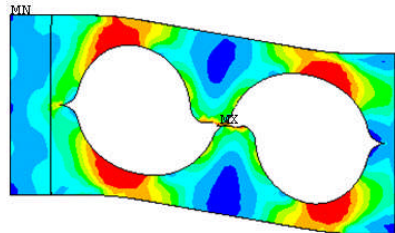
Specimen A1



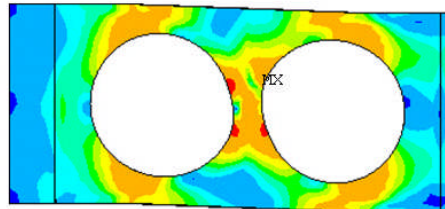
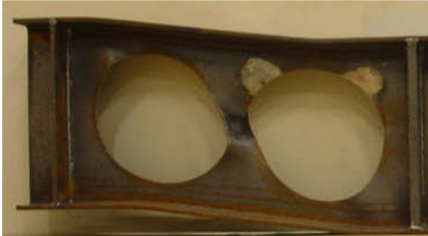
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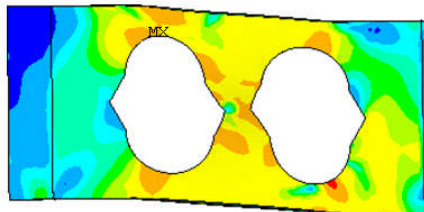
Specimen A3



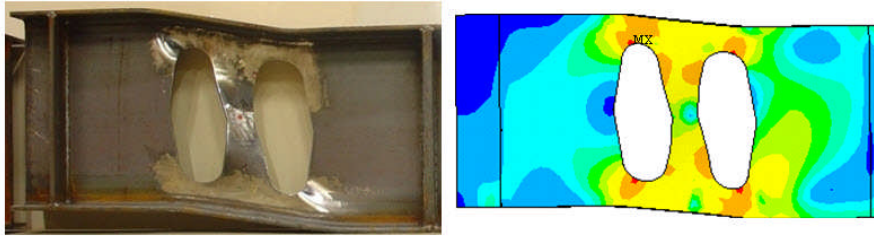
Specimen B1



Specimen B2



Specimen B3



Specimen B4

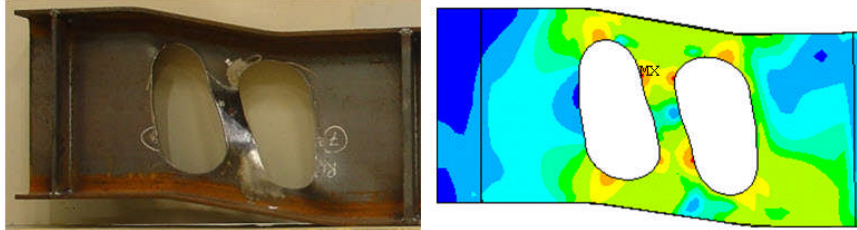


Fig. 4: Distorted web-posts after testing and Von-Mises stresses at failure are shown by FE contour plots

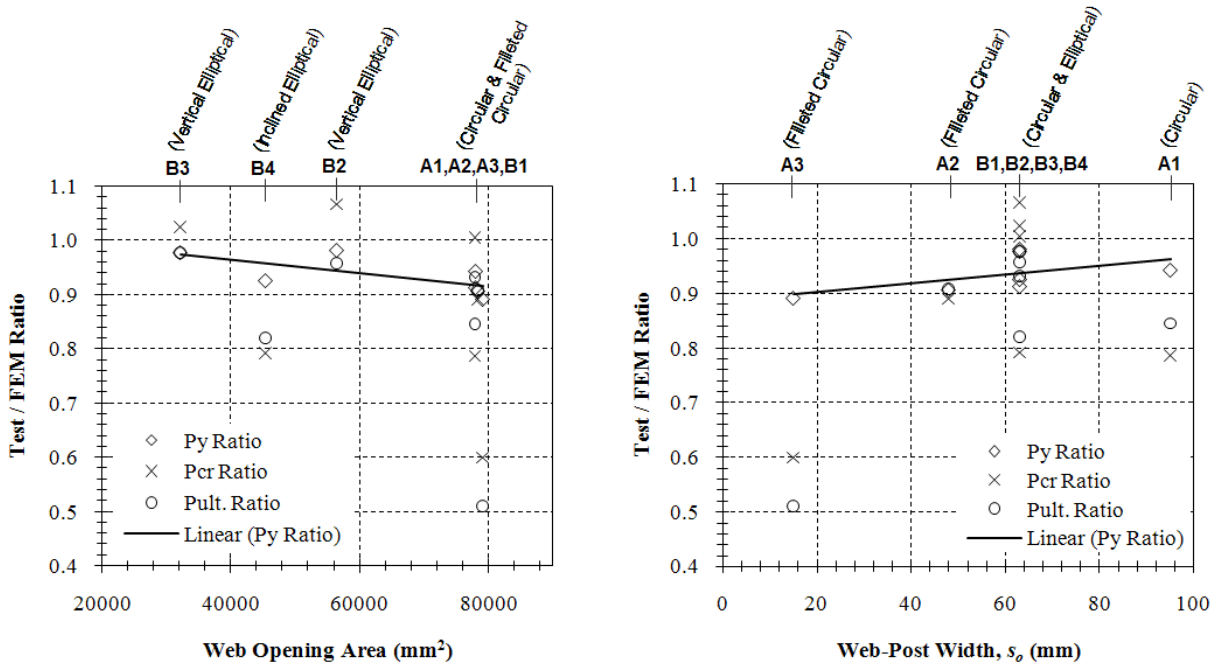


Fig. 5: Variations of test loads to FEM loads

6 Parametric FE study

6.1 Introduction

The ultimate strength of the web-post is governed by one of two modes: (i) Flexural failure caused by the development of a plastic hinge in the web-post, or (ii) Buckling failure of the web-

post. The mode of failure is dependent on the geometry of the web-post. The FEA was carried out for a range of parameters, including the web slenderness. It included material non-linearity (i.e. plastic behaviour) and geometrical non-linearity (buckling). The model used in FEA considered of a short section of beam between the centre-line of adjacent web openings. The main objective was to revise the design model of circular web openings and devise design models for each of the other five novel web openings.

6.2 FE model

The FE model used in this study was based on the previous examined UB457x152x52 deep section with an opening diameter/depth, d_o , equal to 315mm, giving a beam opening/depth ratio of 0.7. Only the web thickness and spacing of the web openings were varied to assess the effect on web-post capacity. A parametric study was conducted to investigate the effect of S/d_o ratio on the resistance of the web-post. Eight S/d_o ratios in the range 1.1 to 1.8 were studied with five web thicknesses of 3.9mm, 5.0mm, 6.0mm, 7.6mm (nominal thickness of this section) and 10.5mm (maximum possible thickness of this section), giving d_o/t_w ratios which varied from 30 to 80.8. The model was extended to examine the web openings presented in **Fig. 1**.

Four-node shell elements (SHELL181) with 6 degrees of freedom at each node were used to model the web-post. Depending on the S/d_o ratio and the web opening shape, different FE meshes were used to model the web-posts with the element size kept the same. The magnitude of the applied load was varied manually for each case in order to achieve convergence in the initial linear stage of the analysis. The load was applied in 100 sub-steps, the size of which could be automatically varied by the software in order to achieve convergence as the solution approached the failure load. The boundary conditions used in the model are shown in **Fig. 6** and **Table 3**. The web-flange connection was assumed to be pinned as a safe lower bound, independent of the flange size and type of weld. In practice, some degree of fixity would exist, which increases the web buckling resistance.

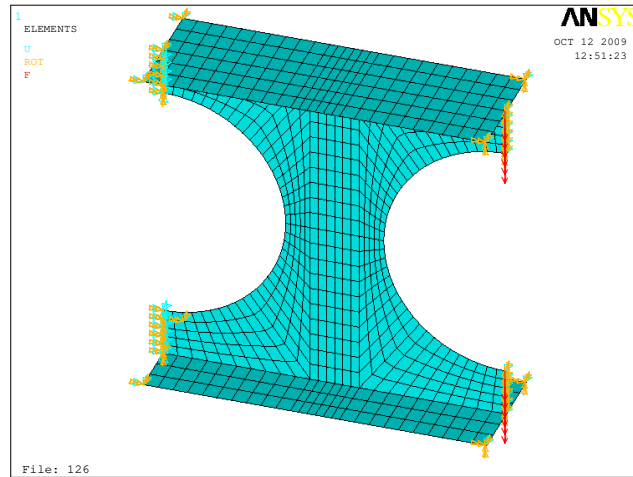


Fig. 6: Boundary Conditions and Loading applied to the FE Model

Table 3
Applied boundary conditions used in the FE model.

Location	UX	UY	UZ	ROT X	ROT Y	ROT Z
Flange (LHS)	FIXED	FREE	FIXED	FIXED	FREE	FIXED
Web (LHS)	FIXED	FIXED	FIXED	FIXED	FIXED	FREE
Flange (RHS)	FREE	FREE	FIXED	FIXED	FIXED	FIXED
Web (RHS)	FREE	LOAD	FIXED	FREE	FIXED	FIXED

LHS/RHS: Left Hand Side/Right Hand Side

A bi-linear representation of the stress-strain curve was used. The initial elastic modulus, E , was taken as 200kN/mm^2 with a reduction to 2kN/mm^2 ($E/100$) on reaching the yield point specified as 355N/mm^2 . The material model used the Von-Mises yield criterion with kinematic hardening which is suitable for most metals, including steel.

6.3 Design model

The design model was developed based on a ‘strut’ analogy in which the stability of the web was checked using buckling curves to **BS 5950-1**[12]. Compressive and tensile forces act across the web-post on opposite diagonals, as illustrated in **Fig. 7**. Failure occurred when a local web buckle formed adjacent to the web opening as shown by the shaded areas in **Fig. 7**. The compressive stress acting on the strut was calculated using the force in the upper tee-section or half the applied vertical

shear force for a symmetrically placed opening. The effective width of the web-post resisting compression was taken as the half the total width of the web-post (i.e. $b_e = s_o/2$) as it is shown in **Fig. 8**. This assumption was only used to evaluate the compressive strength from **BS** and it was based on empirical observations from literature [5]. The compressive stress acting on the strut is as follows:

$$\sigma = \frac{V_v}{2(s_o/2)t_w} = \frac{V_v}{s_o t_w} \quad (1)$$

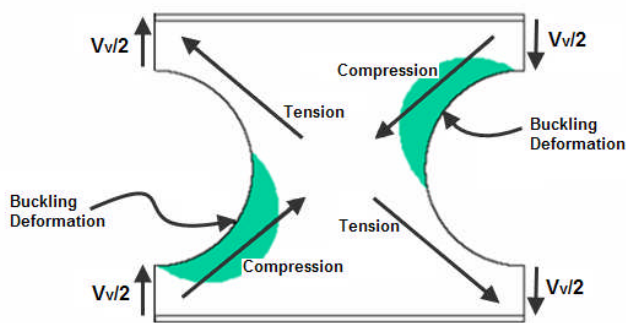


Fig. 7: Typical web-post behaviour

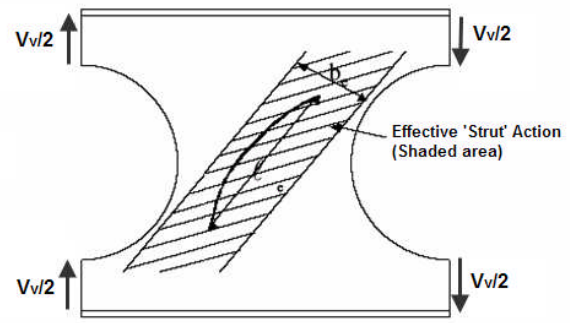


Fig. 8: Strut model of web-post buckling

6.4 Results of the FE analysis

The vertical shear capacities obtained from the FE analyses of the web-posts are graphically represented in **Fig. 9**. Improved design formulas have been developed for perforated sections with circular web openings (A1 & B1) which cover the range of $1.1 \leq S/d_o \leq 1.8$ (**Fig. 9**). As a result of the non-linear FE parametric study on 225 models, similar formulas have been proposed for perforated sections with the aforementioned novel web openings. Design formulas for perforated sections with vertical elliptical web openings (B2) cover the range of $0.931 \leq S/d_o \leq 1.631$ are shown in **Fig. 9** for further comparison.

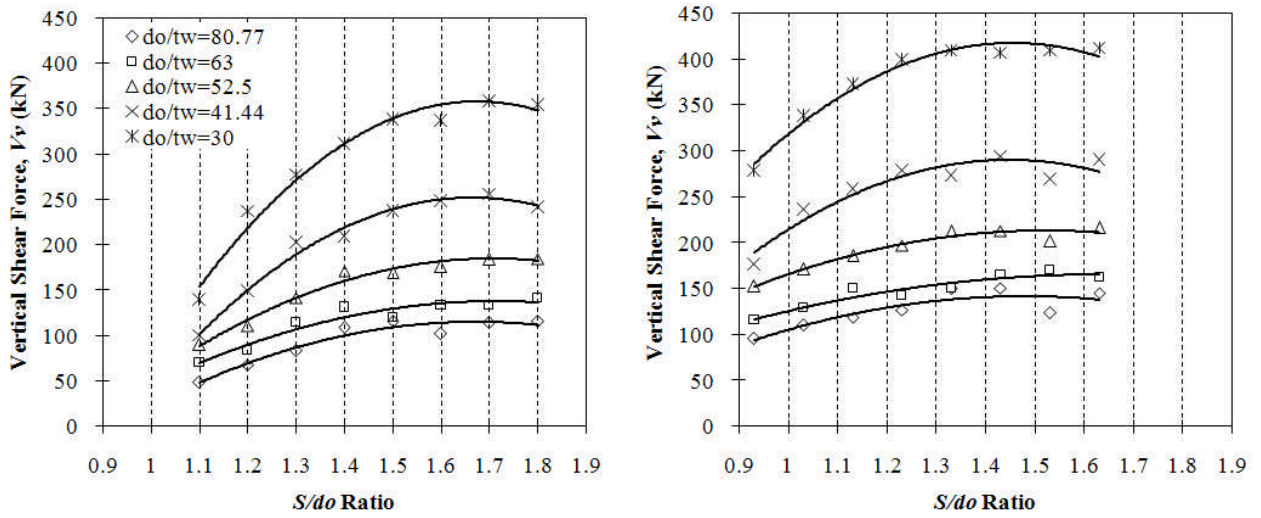


Fig. 9: Vertical applied shear force (kN) according to FE model (left: A1 & B1 and right: B2)

6.5 Design equations

6.5.1 Basic design model for web-post between regularly spaced web openings

The design model derived from the finite element analyses was based on a simple strut model. The strut was considered as acting diagonally across the member (**Fig. 8**). For perforated sections with circular web openings; the effective length, l_e , of the strut was calculated as the diagonal distance across the web-post using an effective length factor of 0.5 in the literature [5], as follows:

$$l_e = 0.5\sqrt{s_o^2 + d_o^2} \quad (2)$$

The slenderness, λ , of the web-post as a strut was then calculated as:

$$\lambda = \frac{l_e\sqrt{12}}{t_w} \quad (3)$$

This value of slenderness was used to obtain the compressive strength, p_c , from buckling curve “c” of **BS 5950-1:2000** [12], which was appropriate for fabricated sections with maximum thickness less than 40mm. Then the vertical shear resistance, V_v , due to web-post buckling was calculated as follows:

$$V_v = 2b_e p_c t_w = p_c s_o t_w \quad (4)$$

6.5.2 Lower bound to web-post buckling

The FE analyses have shown that yielding rather than buckling occurs for stockier webs. This means that web-post buckling does not occur and in some cases, the Vierendeel bending capacity will be the design check that governs rather than web-post buckling.

Fig. 11 shows the compressive strength calculated for perforated sections with circular web openings (A1 & B1), using the established design model for all d_o/t_w ratios plotted against the failure stress obtained from the finite element analyses. The stress on the strut corresponding to the limiting vertical shear force determined from the maximum Vierendeel moment capacity of the tee-section is also plotted (dashed lines). The limiting vertical shear force resulting from Vierendeel bending of the top tee-section was calculated by converting the circular web opening into an equivalent rectangular web opening with a width of $0.5d_o$ and depth d_o (**Fig. 10**). Regarding the novel web openings, suggested values for converting them to equivalent shapes were found from the Vierendeel study on perforated beams with isolated web openings [2,7,8] and briefly presented in **Table 4**. The limiting vertical shear force in the top tee-section is given as follows:

$$\frac{V_v}{2} = \frac{M_{pt}}{(\text{effective opening width}) d_o} \quad (5)$$

Where M_{pt} is the plastic moment capacity of the top tee-section and calculated as follows:

$$M_{pt (plastic)} = A_{tee}/2 \left(\frac{\bar{x}_1}{2} + \bar{x}_2 \right) p_y \quad (6)$$

$$A_{tee} = (bt_f) + (d_T t_w) \quad (7)$$

The centroids for equal areas were estimated from:

$$\bar{x}_1 = \frac{A_{tee}}{2b} \quad (8)$$

$$\bar{x}_2 = \frac{b(T-\bar{x}_1)^2/2 + d_T t_w [(T-\bar{x}_1) + d_T/2]}{A_{tee}/2} \quad (9)$$

$$T = t_f - \bar{x}_1 \quad (10)$$

However, when d_T/t_w for the web of the tee-section exceeds the limit 9ϵ for a compact section (**BS 5950-1:2000**), the moment capacity, M_{pt} , was calculated using elastic properties of the section.

The FE analyses show that yielding rather than buckling occurs for stockier webs. This means that web-post buckling does not occur and the Vierendeel bending capacity is the governed design check rather than web-post buckling. For example, **Fig.11** shows that the Vierendeel moment check is critical for d_o/t_w equal to 41.44 and 30 for S/d_o larger than 1.3 and 1.2, respectively. The webs governed by Vierendeel bending capacity for the above web opening shapes are summarised in **Table 4**.

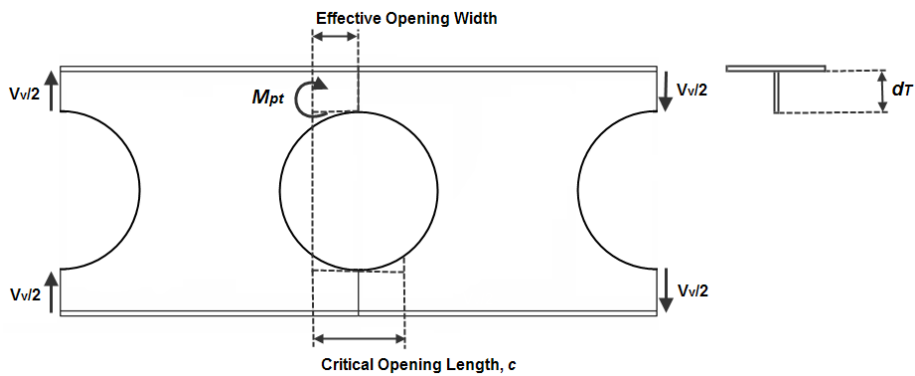


Fig. 10: Effective opening width and critical opening length of circular web opening

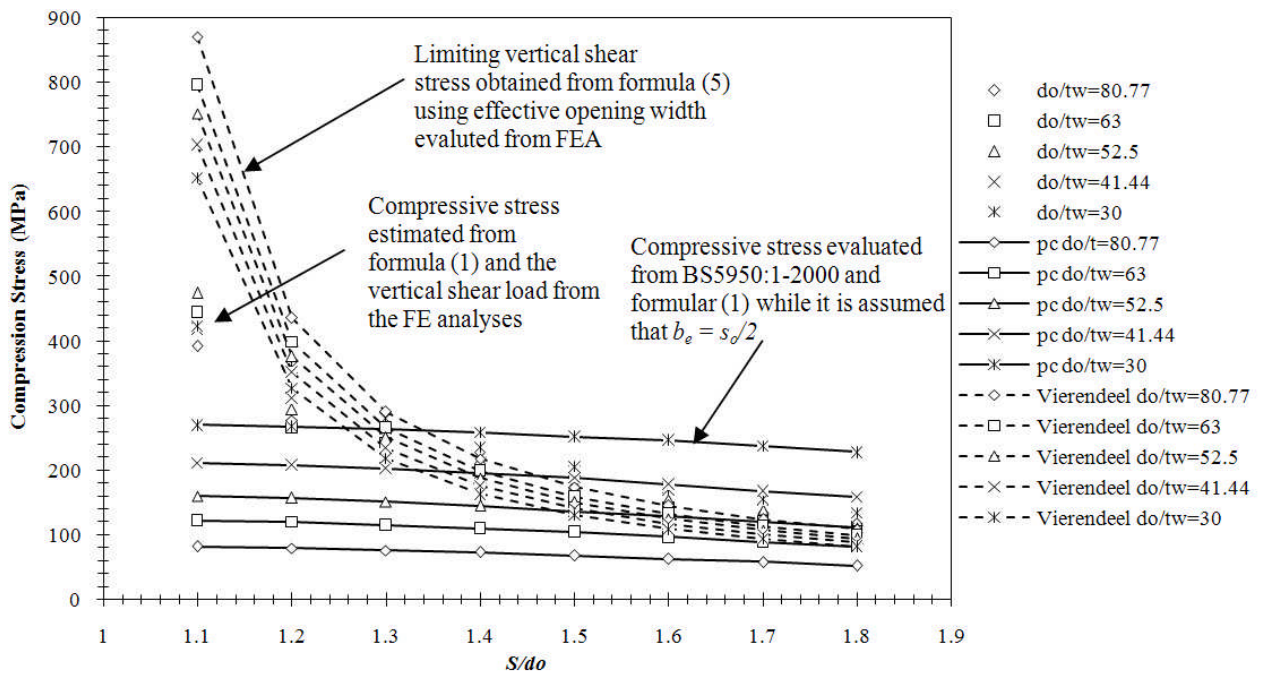


Fig. 11: Compressive strength of the ‘strut’ compared with FE failure stress (A1 & B1)

Table 4
Results from Vierendeel study.

Specimen	Critical open. length	Vierendeel capacity should be checked for:	
		d_o/t_w	S/d_o
A1	$0.5d_o$	= 41.44	> 1.3
		= 30	> 1.2
		= 52.5	> 1.6
A2	$0.5d_o$	= 41.44	> 1.3
		= 30	> 1.2
		= 52.5	> 1.6
A3	$0.5d_o$	= 41.44	> 1.3
		= 30	> 1.2
		= 41.44	> 1.3
B1	$0.5d_o$	= 30	> 1.2
		= 41.44	> 1.231
		= 30	> 1.131
B2	$0.36d_o$	No Critical	No Critical
B3	$0.2d_o$	= 41.44	> 1.05
		= 30	> 0.85

6.5.3 Design curve

The ratios of the vertical shear capacities evaluated by using the **BS** and the effective width of the web-post assumption over the vertical shear capacities estimated by FE analyses are given in **Table 5**. There is conservatism up to 79% for narrow and thin web-posts, but the level of conservatism is reduced for wider web-posts with greater web thicknesses. The shaded area indicates where the Vierendeel bending rather than the web-post buckling controls according to **Fig. 11**. The un-conservative stresses resulting from the basic design model are leading to safe design for this web slenderness, as the Vierendeel bending resistance is checked separately. **Table 5** is only representing the examined model with circular web openings (A1 & B1). The solid line in **Fig. 12** demonstrates the minimum compressive strength for a web-post of any web thickness within the examined range and a certain web opening spacing. This is an empirical design curve, evaluating the compressive stress at a web-post considering both the FE results and the theoretical evaluations based on plastic moment capacity of the top tee-section.

Table 5
Ratio of vertical shear capacities (A1 & B1).

S/d_o	Width, s_o (mm)	t_w (mm)				
		3.9	5.0	6.0	7.6	10.5
		d_o/t_w				
		80.77	63	52.5	41.44	30
1.1	31.5	0.21	0.27	0.34	0.50	0.64
1.2	63	0.29	0.45	0.53	0.67	0.75
1.3	94.5	0.34	0.47	0.60	0.72	0.94
1.4	126	0.33	0.51	0.64	0.90	1.10
1.5	157.5	0.36	0.69	0.76	0.95	1.23
1.6	189	0.45	0.68	0.83	1.03	1.46
1.7	220.5	0.44	0.73	0.86	1.10	1.53
1.8	252	0.44	0.73	0.91	1.25	1.70

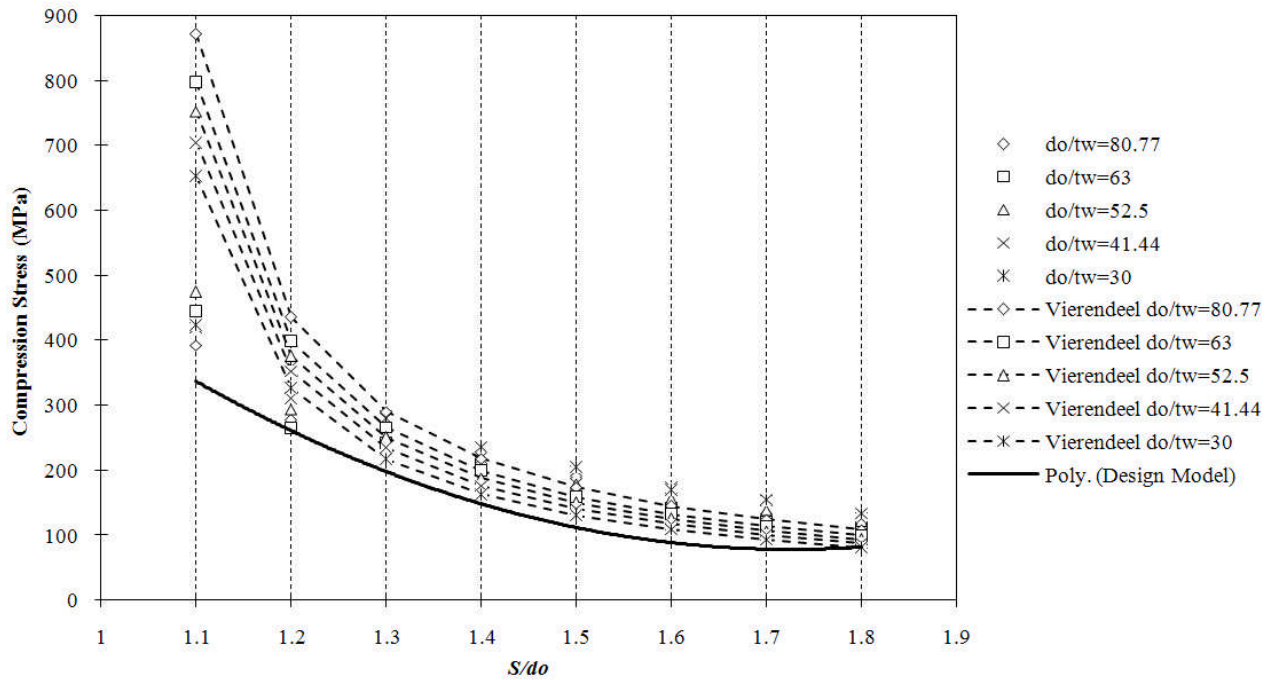


Fig. 12: Empirical design curve evaluating the compressive strength of the ‘strut’ (A1 & B1)

An empirical design formula was developed (11) and shown below while the constant factors for all examined FE models are summarised in **Table 6**. It is inevitable that by using this method some overestimation was considered for certain models. Conservative but also economic results for web-posts under buckling and shear provisions were achieved.

$$V_v = -C_1 \left(\frac{S}{d_o}\right)^2 + C_2 \left(\frac{S}{d_o}\right) - C_3 \quad (11)$$

Table 6

Design proposed factors for perforated section with novel web openings.

t_w (mm)	d_o/t_w	Factor	A1	A2	A3	B1	B2	B3	B4
3.9	80.77	C_1	160.2	181.1	57.25	207.3	160.2	63.8	143.2
		C_2	474	633.4	240	692.4	474	129.8	327.2
		C_3	208.2	422.4	160.4	462	208.2	52.63	77.12
5.0	63	C_1	96.46	245	180.3	180.8	96.46	17.66	9.958
		C_2	316.8	802.8	657.9	619.7	316.8	99.06	99.88
		C_3	94.19	556.2	465.5	392.6	94.19	95.2	44.59
6.0	52.5	C_1	172.4	320.3	379.4	262.8	172.4	227.9	81.88
		C_2	525	1104	1312	896.5	525	509.2	285.8
		C_3	186.6	766.8	950.5	580	186.6	36.82	22.31
7.6	41.44	C_1	379.6	517	420.4	471.7	379.6	279.5	202.2
		C_2	1099	1768	1505	1569	1099	630.4	620.4
		C_3	505.6	1255	1097	1053	505.6	47.62	164.3
10.5	30	C_1	477.8	683.3	847.3	617.5	477.8	498.8	454.1
		C_2	1390	2285	2885	2066	1390	1070	1147
		C_3	594	1560	2118	1371	594	125.5	333.2

6.5.4 Widely spaced web openings

For widely spaced web openings of any shape, the shear resistance can be obtained with $S/d_o = 2.0$ being used as a cut-off limit in this analysis. For instance, in this limiting case when the effective length factor is equal to 0.5 (i.e. circular web openings) the following approach is considered:

$$b_e = 0.5s_o$$

$$l_e = 0.5\sqrt{d_o^2 + d_o^2} = 0.5\sqrt{2}d_o \approx 0.7d_o \quad (12)$$

$$V_v = 2b_e p_c t_w = p_c s_o t_w \leq 0.6p_y \quad (13)$$

In all cases, the shear resistance cannot exceed that given by a stress of $0.6p_y$ acting on the web-post. This also applies to isolated circular web openings.

6.6 Position of plastic hinges and effective widths of the web openings

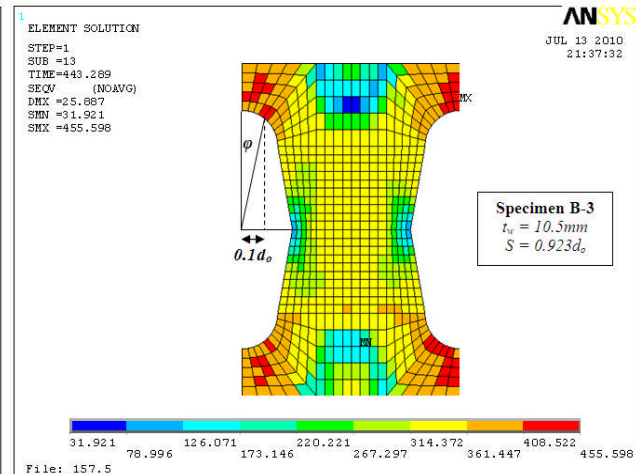
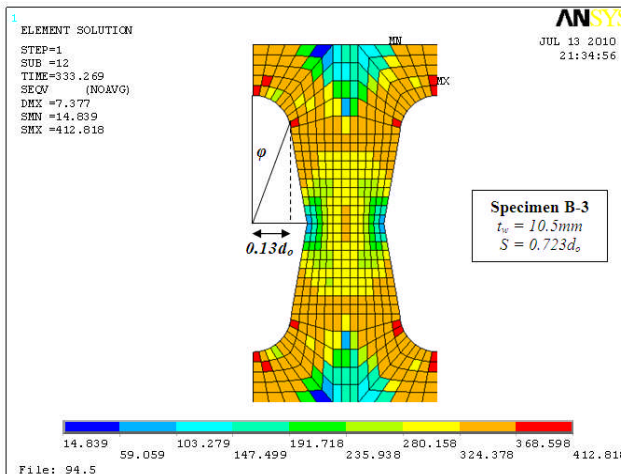
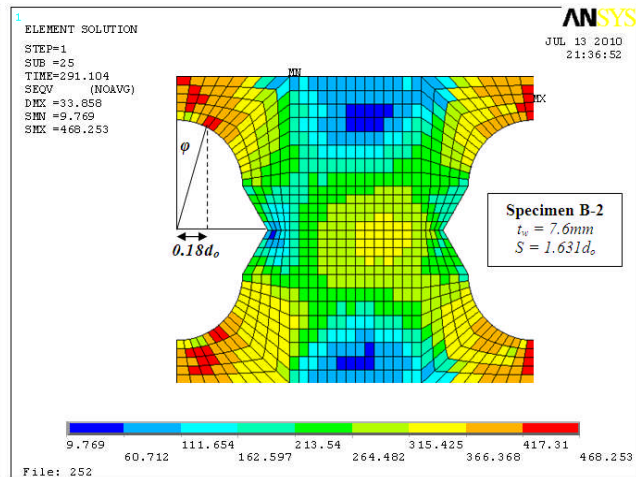
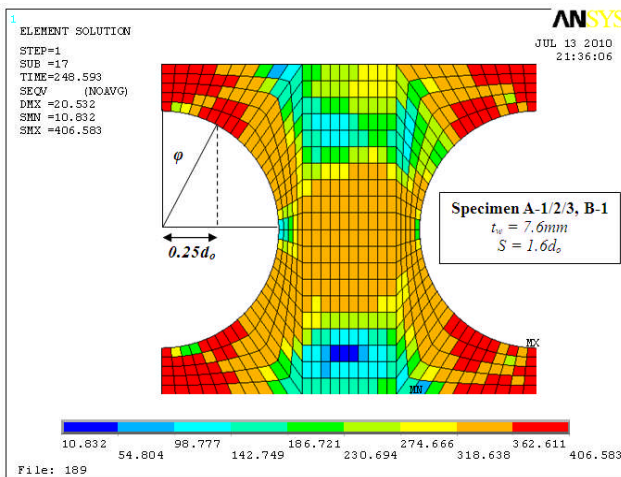
Table 7 and **Fig. 13** illustrate the position of the plastic hinges and so the effective opening widths for circular and elliptical web opening shapes. The positions of plastic hinges are estimated from models with widely spaced web openings. It is assumed that the hinge positions adopted for the novel elliptical web openings are non-unique while represent only the particular web opening shape with an angle (θ) and a radius (R). A comprehensive FE study on numerous perforated

sections with isolated vertical and inclined elliptical web openings has been carried out and the position of the plastic hinges has been carefully examined [7,8].

Table 7
Summary of equivalent effective widths for novel web openings.

Specimen	ϕ range due to mobilization (degrees)	under Vierendeel action		under Web-Post Buckling action		Max. used ϕ (degrees)
		ϕ (degrees)	Effective open. width	ϕ (degrees)	Effective open. width	
A1/2/3 & B1	20-29	28	$0.25d_o$	28	$0.25d_o$	28
B2	13-22	22	$0.18d_o$	17	$0.14d_o$	22
B3	12-22	12	$0.1d_o$	12-14	$0.1-0.13d_o$	12
B4	15-30	21	$0.18d_o$	17-45	$0.15-0.28d_o$	21

d_o : Web opening depth



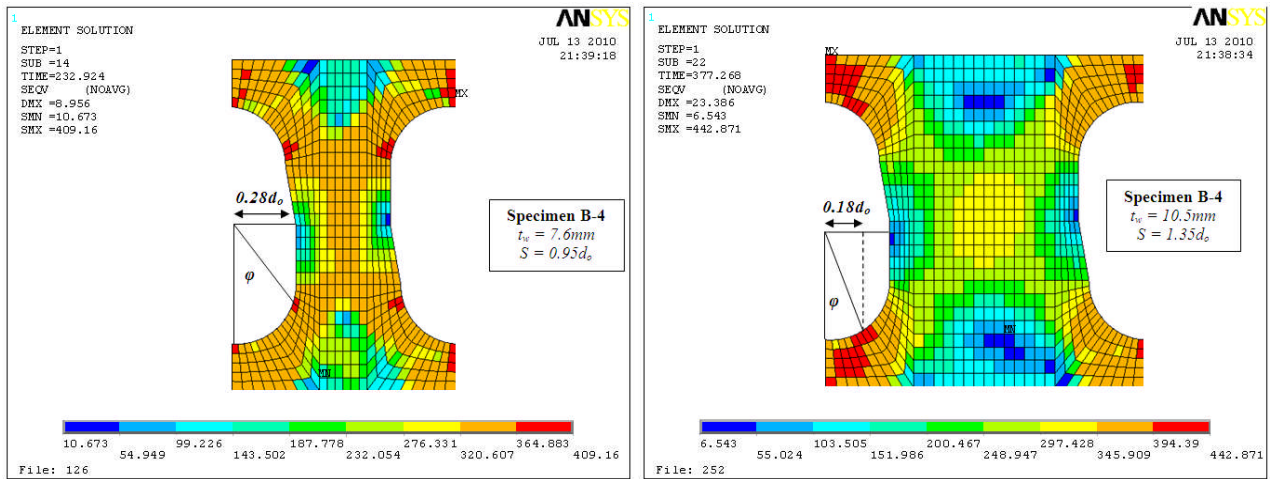


Fig. 13: Stress concentration points, ‘strut’ models and equivalent effective opening widths

Conclusions

The tests described provide considerable information on the plastic behaviour of perforated beams with various web opening shapes. Even with a limited experimental programme involving a single standard beam configuration, it has been possible to draw a few conclusions. Failure of the specimens occurs under the combined action of shear and moment. High deformation is observed as the web-posts are subjected to high shear, accompanied by heavy distortion of the web opening. Web-post buckling occurred at loads slightly in excess of that at which deformations were first observed. No attempt has been made to present a complete analysis of the tests regarding equilibrium and stability problems and it is believed that a considerable number of further tests will be required to justify plastic beam action of these sections. The work reported here demonstrates that behaviour of perforated beams with different web opening shapes is similar.

A theoretical finite element analysis was carried out to acquire a more complete view of the web-post buckling study and the complex failure mode for perforated sections with various web opening shapes and different spacing. The conclusions from the parametric FE work conducted on 225 models are:

- Specimens A1, A2, A3, B1, B2 and B4 are governed by Vierendeel bending capacity when the web thickness is between $7.6\text{mm} < t_w < 10.5\text{mm}$ and the web-post width is between $63\text{mm} < s_o < 252\text{mm}$. In general, when novel elliptical web openings are considered, the critical openings length is narrower and hence the Vierendeel capacity is high.
- Between the FE estimations and the theoretical evaluations for web-post buckling, a range of conservatism from 50% to 90% was found. Higher conservatism is obtained when perforated beams with circular and elliptical web openings are considered, with lower conservatism for circular web openings with fillets.
- From the design equations it can be seen that in general the vertical shear capacities increase as the web-post width is increased, and conversely slightly decreased when they are subjected to high Vierendeel bending forces. However, when narrow elliptical web openings are considered, the capacity only gradually increases as Vierendeel bending is not critical.
- Specimen A2 and A3 behave similarly beyond a certain web-post width, where there is better stress distribution and cut-off 'shear' failure is avoided. Hence, for those models, the existence of the fillets at the mid-height of the web opening with different radiuses, does not affect the vertical shear capacity of the web-posts.

In more detail, regarding the horizontal shear stresses can be concluded that:

- From the FE analysis, due to better stress distribution, the maximum shear stresses are lower in Specimens B3 and B4 for any web opening spacing and Specimen B2 when the web opening spacing is greater than $1.131d_o$. The above stresses were found to be lower than the horizontal shear stress resistance.
- Specimen A2 and A3 present very high shear stresses at mid-height web-post due to the existence of the fillet and especially for web opening spacing $1.3d_o$ and $1.2d_o$. Also, Specimen B1 presents high shear stresses at mid-height web-post.

- When relatively wide web opening spacing is considered, the maximum shear stresses move from the mid-height of the web-post towards the centroid of the axial forces, closer to the flanges.

Acknowledgments

This report has been prepared as a result of research carried out at City University London for the completeness of a Philosophy Doctorate degree. The experimental work described in this report was entirely carried out at City University laboratories. The authors would like to thank ASD Westok Ltd. group for the supply of the steel perforated specimens and the approval of the experimental structural arrangement and geometrical configurations of the specimens.

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