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Numerical Simulation of Powder Flow during Spreading in Additive

Manufacturing

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7 Abstract: Additive manufacturing (AM) has attracted increasing attention in a wide range of applications, due to its ability for rapid manufacturing of complex shapes directly from a 8 9 Computer-Aided Design (3D CAD) output. One of the manufacturing methods is based on 10 powder processing, where a thin bed is formed to which an energy beam is applied to sinter and 11 melt the powder. A major bottleneck in this method is associated with powder spreading, as its 12 dynamics is sensitive to powder properties, machine design and operation conditions, such as speed of spreading. The effects of gap height and blade spreading speed on the evolving shear 13 14 band and mass flow rate through the gap have been simulated by Discrete Element Method, using 15 the most realistic physical and mechanical properties of the particles. It is shown that the particle velocity in the powder heap in front of the blade could well be described by a universal curve 16 17 given by the Gauss error function. The mass flow rate through the gap increases linearly with the gap height. There exist two flow regimes with the increase of the blade spreading speed. Initially, 18 19 the mass flow rate has a linear dependence on the blade speed, but eventually approaches an 20 asymptotic value, implying a limit beyond which the mass flow rate cannot be further increased. 21 This has an important implication on the speed of spreading.

Keywords: Additive Manufacturing; Particle Spreading; Sintering; Discrete Element Method;
Shear band; Mass flow rate.

1 1. Introduction

2 Additive manufacturing (AM) is paving the way towards the next industrial revolution [1-3]. AM creates three-dimensional (3D) objects by stepwise layer-by-layer approaches which 3 are controlled by a digital model. This unique feature allows the production of complex or 4 5 customized parts directly from the Computer-Aided Design (3D CAD) output, without the need for expensive tooling such as punches, dies or casting moulds and the assembling of multiple 6 components [4-12]. It also reduces the need for many conventional processing steps and 7 8 eliminates most of the constraints that hinder optimal design, creativity and ease of manufacturing of complex parts. Therefore, AM is going through an exponential growth and is 9 being widely used as a novel production technology for the design and manufacturing of 10 high-performance components in the aerospace, biology, medical and energy applications [5, 7]. 11 12 For example, complex fuel injector nozzles in aerospace technology which previously required 13 assembly of multiple parts can now be directly fabricated by AM with lightweight engineered 14 structures, resulting in significant cost savings.

Among several manufacturing methods, the powder-based process has attracted increasing attention due to its flexibility [11, 13]. In this process, typically 10-50 µm particles are spread over a work surface made of particles, which are already partially sintered/melted, by a roller or blade, resulting in a particle layer with a thickness of a few particle diameters. Then a laser or electron beam is shone onto it to melt the specified area. Once a cross-section is scanned, the work surface is lowered and the process is repeated until the completion of the production. The powder spreading process has an important effect on the quality of the powder bed for subsequent sintering/melting, which in turn influences the characteristics and quality of the final product [4, 5]. Poor powder flow during spreading causes variations in the solid volume fraction of the particle layer and roughness of its surface, leading to weak bonding between layers or formation of cavities, producing inferior products with poor mechanical performance [14-16]. Lack of sufficient understanding of the powder dynamics, such as the factors which influence the powder flow rate through the spreader gap, hinders further refinement of this technology and introduction of new materials.

8 Recently, the Discrete Element Method (DEM) has been used to describe the mechanical 9 behaviour of the powder spreading system in additive manufacturing [17-21]. Parteli and Pöschel [17] and Haeri et al. [18] studied the effect of spreading speed and gap height on the bed quality 10 11 in the roller spreading system, and showed that high spreader translational velocity could lead to 12 low bed quality. Chen et al. [19] used spherical particles with tuned mechanical properties and 13 found that the effects of the blade spreading speed and gap height on the dynamic repose angle of 14 the heap in front of the blade were negligible. Haeri [20] explored the effect of the blade 15 geometry on the spreading performance, and found that the bed quality could be improved by 16 using a super-elliptic edge profile for the spreader. These works also showed that the powder spreading process was very sensitive to powder properties (including particle size distribution, 17 18 interfacial surface energy, sliding friction coefficient and so on) and operation conditions 19 (including spreading speed and gap height). Nan et al. [21] characterised all the relevant physical 20 and mechanical properties of gas-atomised 316L stainless steel particles and used them in numerical simulations of powder spreading by a blade. They showed that transient jamming 21

occurred in narrow spreader gaps and was manifested by the formation of empty patches over the 1 work surface. They also analysed the frequency and period of jamming and established their 2 3 relationships with particle properties, gap height and spread speed. The factors which have not been analysed in detail so far are the powder flow rate through the spreader gap as influenced by 4 5 the spreader speed, as well as the shear zone in the heap in front of the blade which is responsible 6 for controlling of the powder flow rate. They are directly related to the issues during powder 7 spreading, such as the criterion for the maximum spreading throughput, and the empty patches 8 caused by the particle jamming for small spreading gaps.

9 In this work, the particle flow in the blade spreading process is simulated by Discrete 10 Element Method. The particle heap comprises gas-atomised stainless 316L steel particles. It is 11 subjected to a translational motion by a vertical blade with a gap allowing a thin particle layer to 12 be spread. The effects of the gap height and blade spreading speed on the shear zone of the 13 particle heap in front of the blade and the resulting mass flow rate through the gap are analysed. 14 This provides a benchmark for the maximum speed of the spreader, an influential factor 15 controlling the production throughput.

16 **2. Method**

Gas-atomised 316L stainless steel particles, provided by Sandvik Osprey Ltd, Neath, UK, are modelled in this work. The characterisation of their physical and mechanical properties have been done by Nan et al. [21], including the size and shape distributions, hardness and Young's modulus, interfacial surface energy, coefficients of restitution and sliding friction. The particles have a size distribution in the range 15-55 μ m, for which the number based D₁₀, D₅₀ and D₉₀ are 20 μm, 32μm and 45 μm, respectively. To describe the dynamics of particle flow in the spreading
 process, the particles are modelled as discrete entities and their motions are tracked individually
 by solving Newton's laws of motion [22-24], for which the EDEMTM software package provided
 by DEM Solutions, Edinburgh, UK, is used.

5 2.1. Discrete Element Method

According to the DEM, originally proposed by Cundall and Strack [22], the movement of an
individual particle is described by the translational and rotational motions:

8 $m_i \frac{d\mathbf{v}_i}{dt} = \sum \mathbf{F}_{c,i} + m_i \mathbf{g}$ (1)

$$\frac{\mathrm{d}(\mathbf{I}_{i} \cdot \boldsymbol{\omega}_{i})}{\mathrm{d}t} = \mathbf{R}_{i} \cdot \sum \mathbf{M}_{\mathrm{c},i}$$
(2)

(3)

10 where m_i , I_i , v_i and ω_i are the mass, moment of inertia, translational velocity and angular velocity, 11 respectively; $\mathbf{F}_{c,i}$ is the contact force, originating from its interaction with neighbouring particles or 12 walls; $\mathbf{M}_{c,i}$ is the contact torque, arising from the tangential and normal contact forces; \mathbf{R}_i is the 13 rotation matrix from the global to the local coordinate system in which the calculation of the 14 rotation expressed by Eq. (2) is accomplished.

As introduced by Favier et al. [25], the non-spherical particles are described by the overlapping multi-sphere model, as shown in Fig. 1. Thus, the interactions between any two non-spherical particles can be simplified as that of spherical particles. In this work, the elastic contact force is described by Hertz-Mindlin contact model [23], and the adhesive interaction is accounted for by JKR theory [26]:

 $F_{\rm JKR} = 4\sqrt{\pi\gamma E^*} a^{3/2}$

21 where γ is the surface energy; E* is the equivalent Young's modulus; a is the contact radius.

More features and further information of the simulation method are given by Nan et al. [21] and
 not shown here for brevity.

Fig. 1. Particle shapes used in DEM simulations; each size and shape class are housed in a box with coloured
 borders, as shape varies with size; six blue boxes (15-25 μm), eight purple boxes (25-35 μm), five red boxes
 (35-45 μm), and five green boxes (45-55 μm).

6 2.2. Simulation Conditions

7 The simulation system comprises a spreading blade and a base, as shown in Fig. 2, where D represents the characteristic size D₉₀ by number. The dimensions of the simulation domain in the 8 9 spreading and lateral directions are 400D and 10D, respectively. The front and rear boundaries 10 (i.e. in the Y direction) are treated as periodic boundaries for particle flow. The base with the 11 same length and width as of the simulation domain is made up of clumped cylinders with axes along the Y direction, where the cylinder diameter is equal to D and the distance between adjacent 12 13 cylinder centres is equal to 0.5D. In this arrangement, the bulk sliding of the particles on the base is mitigated by the fully-rough wall. The blade with the same width of the base has a thickness of 14 15 4D in the spreading direction, and its height is much larger than that of the initial particle bed.

The initial particle bed is prepared by using the poured packing method, where approximately 16,000 particles are generated. These particles have a size distribution in the range 15-55 μm, and are classified into four main size classes based on the equivalent-circle diameter of the projected area. The number frequency for the size classes of 15-25 μm, 25-35 μm, 35-45 μm and 45-55μm are 29.6%, 40.8%, 23.9% and 5.7% [21], respectively. For each size class, 5-10 particles are randomly selected for simulating their shapes, with a total of 24 types of shape used in the simulation, as shown in Fig. 1. The physical and mechanical properties of particles, as

1	needed for numerical simulations are given in Table 1. They are characterised by special methods
2	and have been described in the previous work [21]. The integration time step is 10^{-8} s. At the start
3	of the simulation, the blade is lifted vertically to the specified position, forming a vertical gap δ
4	between the spreading blade and layering base. It is then moved along the X direction with a
5	constant velocity U, by which the particles are spread. To evaluate the effect of spreading
6	conditions on the particle flow in the spreading process, the gap δ is varied from 1.5D to 4.0D, and
7	the blade speed U is varied from 0.01 m/s to 0.16 m/s.
8	Fig. 2. Schematic of particle spreading process for simulation [21].
9	Table 1. Particle properties in the simulation.
10	3. Simulation Results
11	3.1. Shear band
12	As the particle bed is subjected to the horizontal motion of the blade, a shear band is locally
13	formed around the blade. Here, the region 'in front of the blade' is shown in Fig. 3. Its dimensions
14	are 10D in the Y direction, 2.5D in length, in the X direction, and 6D in height, in the Z direction.
15	It is divided into 11 bins in the Z direction with a height equal to D, and the adjacent bins overlap
16	in height by 50%. This is done to get a smooth velocity profile. As the blade moves in the X
17	direction, only the component of particle velocity in this direction u_x is considered, and it is
18	normalised by the blade speed U.
19	Fig. 3. Illustration of the heap region 'in front of the blade'.
20	The profile of u_x along the vertical direction is shown in Fig. 4, where the blade speed is 0.04
21	m/s. The particle velocity u_x increases with the bin centre position H and reaches a plateau when
22	H is larger than H ₁ = δ +D. This trend suggests that the effect of blade shearing on the particles

mainly takes place in the zone with H $\leq \delta$ +D. For a small gap height (i.e. $\delta/D=1.5$), the shear 1 band centre position for u_x/U=0.5 is very close to the base, and the shear zone is not fully 2 3 developed near the base, resulting in a large particle velocity at H/D=0.5. With the increase of the 4 gap height, the profiles move to the right, and the velocity of particles closest to the base decreases significantly, due to further developing of the shear zone near the base. When the gap 5 6 height is increased to $\delta/D=4.0$, the shear band is fully developed, resulting in a sigmoidal shape 7 profile of particle velocity. 8 Fig. 4. Particle velocity profiles in the region 'in front of the blade' for different gap heights for blade speed of 9 0.04 m/s. 10 Profiles of u_x for other blade speeds are all similar to the ones shown in Fig. 4. By rescaling 11 of the bin centre position H in terms of $(H-h_c)/h_w$, where h_c is the shear band centre position in the 12 vertical direction with respect to the base and h_w is a linear function of the shear band width, w, 13 acting as a scale factor. These profiles (i.e. totally 32 cases) all collapse onto a universal curve, as 14 shown in Fig. 5. This curve is well fitted by the Gauss error function which is denoted by "erf": $\frac{u_x}{U} = 0.5[1 + erf(\frac{H - h_c}{h})]$ 15 (4) Thus, the profile of particle velocity could be completely described by the centre position and 16 widths of the shear band. 17 18 Fig. 5. Universal curve for the particle velocity profile in the region 'in front of the blade'. 19 The variation of the shear band centre position h_c with the blade speed is shown in Fig. 6. 20 With the increase of blade speed, the shear band centre position moves to a lower vertical 21 position. The shear band centre position is more sensitive to the blade speed for larger gap heights. 22 With the increase of gap height, the shear band centre position moves to higher vertical positions, 8

which agrees well with the results shown in Fig. 4. The difference between the shear band
 centre positions for different gap heights decreases with the blade speed. This suggests that the
 shear band centre position is less affected by the gap height at large blade speed.

Here, the positions where $u_x/U=0.05$ and $u_x/U=0.95$ are regarded as the lower and upper 4 boundaries of the shear band. Thus, the shear band width, w, is calculated from the distance 5 between these two extreme positions, and is equal to 2.5h_w, due to 0.5(1+erf(-1.25))=0.05 and 6 7 0.5(1+erf(1.25))=0.95 in Eq. (4). The variation of the shear band width w with the blade speed 8 is shown in Fig. 7. For small blade speeds, the shear band width fluctuates with the increase of 9 gap height. For example, when the blade speed is between 0.04 m/s and 0.08 m/s, the shear band 10 width is around 3.0-3.5D, which is also about 4.0-4.7D_v, as D=1.35D_v. As the blade speed is 11 increased to 0.10 m/s and beyond, the shear band width increases almost linearly with the gap 12 height.

Fig. 6. Variation of the shear band centre position h_c with respect to the spread layer surface with the blade
 speed for different gap heights.

Fig. 7. Variation of the shear band width w with the gap height for different blade speeds.

15

16

3.2. Mass flow rate through the gap

As the particle bed is sheared by the blade, particles with smaller u_x than the blade are deposited onto the base through the gap. The mass flow rate is unsteady, due to the discontinuous nature of particle flow in narrow openings. When the gap height is small (i.e. $\delta/D=1.5$), the transient mass flow rate could even be zero, due to the particle jamming, which has been discussed in detail by Nan et al. [21]. Here, the mass flow rate is time-averaged and normalised by dividing it by ρ WD, where ρ is particle density, W is the width of the base in the lateral 1 direction (Y direction), and D is D₉₀ of particle size number distribution.

2 Variations of the mass flow rate with the blade speed are shown in Fig. 8(a). At low blade 3 speeds, the mass flow rate increases linearly with the blade speed, and its slope increases with the gap height. At large blade speeds, there is a change in the trend and the mass flow rate increases 4 to an asymptotic value, which depends on the gap height. This trend suggests that the spreading 5 6 throughput is limited at large spreading speeds. There are therefore two flow regimes, one 7 dependent and another independent of the blade speed. It appears that the blade speed 8 corresponding to the flow regime transition is not affected by the gap height. This will be probed 9 in more detail below. The variation of mass flow rate with the gap height is shown in Fig. 8(b), for which a linear relationship with the gap height is observed. The slope depends on the blade 10 11 speed. Thus, the following functional form is proposed for the mass flow rate:

12

$$Q = \rho f_{\rm U} W(\delta - \delta_{\rm c}) \tag{5}$$

13 where δ_c is the critical gap height at which the mass flow rate is zero, namely the intercept on the 14 abscissa in Fig. 8(b). δ_c accounts for the boundary effects of the blade front tip, resulting in an 15 effective gap height of δ - δ_c . The dependency of the slope of the line on the blade speed is 16 expressed by f_U for the linear flow regime.

17

Fig. 8. Variation of mass flow rate with (a) blade speed and (b) gap height.

The variation of f_U with the blade speed is shown in Fig. 9(a). At low blade speeds U, f_U increases linearly with U with a slope of about 0.55. At large blade speeds, f_U increases gradually to a plateau around 0.043. Fig. 9(a) also shows there exists a critical blade speed, U_c, above which the mass flow rate is almost independent of the blade speed. It is around 0.1 m/s for the particle

1	system here. The critical gap height δ_c as a function of the blade speed is shown in Fig. 9(b). Its
2	value is around D (i.e. $1.35D_V$, where D_V is the arithmetic mean volume equivalent diameter) for
3	the speed range 0.04-0.1 m/s, which is commonly used. In the case here, the critical gap height is
4	caused by one side wall moving and the other being stationary, whilst the two side walls in the
5	wedge-shaped hoppers are stationary [27-30]. Thus, the critical gap size in this work is smaller
6	than that in hopper flow. Fig. 9 also suggests that even in the ideal case where the particles are
7	uniformly spread onto the base without jamming, the gap height should be at least 2D to cover
8	the base without any empty patches.
9	Fig. 9. Variation of (a) f_U and (b) critical gap height with blade speed.
10	The critical blade speed could also be examined by comparing the gravity inertial timescale
11	tg and spreading inertial timescale ts, which are defined as
12	$t_g = \sqrt{D / g} \tag{6}$
13	$t_{s} = D/U $ (7)
14	The gravity inertial timescale tg is the time taken by a particle with zero initial velocity to fall a
15	distance of D/2 under gravity acceleration g, whilst the spreading inertial timescale t_s is the time
16	taken by the blade with velocity U to pass over a static particle with D. If the spreading inertial
17	timescale is much less than the gravity inertial timescale, i.e. $t_s=t_g/k$ (k>>1), the gravity effect on
18	the spreading process could be limited, and the particles do not have enough time to fall down to
19	fill the dilated zone caused by the blade shearing. This could also be observed from the vertical
20	velocity of particles in front of the blade in Fig. 3, as shown in Fig. 10. When the blade speed is
21	0.1 m/s or greater, the falling velocity of particles (i.e. $-u_z$) does not show any further increase

with the increase of U, indicating that no more additional particles could move to the dilated zone
as the blade speed is increased. Therefore, by comparing t_s and t_g, a critical blade speed could
be defined as:

4

$$U_c = k\sqrt{gD}$$
 (8)

where k is the ratio of the two timescales (>>1). The variation of the critical blade speed with k is
shown in Table 2. In this work, k=5 gives the same critical blade speed (i.e. 0.1 m/s) as given in
the description of Figs. 8 and 9. Therefore, the critical blade speed is well described by Eq. (8).
Fig. 10. Profiles of u_z in the heap in front of the blade for different blade speeds for the gap height of δ/D=3.0.

Table 2. Variation of critical blade speed with k.

9

10 **4. Conclusions**

11 The particle spreading process in additive manufacturing has been analysed by DEM 12 simulations, using realistic physical and mechanical properties of particles as measured for single 13 particles in the previous work. The effect of the spreading process on the particle flow has been 14 analysed in terms of the shear band and mass flow rate through the gap height. The main results 15 from the present study are summarised as follows:

16 1) The velocity profiles of particles in the heap in front of the blade follow a universal curve
given by the Gauss error function. With the increase of blade speed or the decrease of the gap
height, the shear band centre moves to lower vertical positions.

19 2) The mass flow rate through the gap increases linearly with the gap height. There is a 20 critical size of about D below which there is no flow. The dependence of the mass flow rate on 21 the blade speed is more complex. There are two different flow regimes. The mass flow rate is 22 initially linear but approaches an asymptotic value at large blade speeds. The transition point

- 1 from linear to the asymptotic trend is independent of the gap height. The critical blade spreading
- 2 speed, above which the mass flow rate is independent on the blade spreading speed, is 0.1 m/s for
- 3 the system studied here.
- 4 3) The critical blade spreading speed could be examined by comparing the gravity inertial
- 5 timescale and spreading inertial timescale, and it is about $5(gD)^{0.5}$.

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Table 1. I article properties in the simulation.				
Parameters	Value			
Particle diameter, D (mm)	0.045			
Particle density, ρ (kg/m ³)	7980			
Young's modulus, E (GPa)	2.1			
Poisson ratio, v	0.3			
Friction coefficient, µ	0.5			
Restitution coefficient, e	0.64			
Surface energy, γ (mJ/m ²)	1.4			

Table 1. Particle properties in the simulation.

Table 2. Variation of critical blade speed with k.

k	3	4	5	6	7
U _c (m/s)	0.063	0.084	0.105	0.126	0.147



Fig. 1. Particle shapes used in DEM simulations; each size and shape class are housed in a box with coloured borders, as shape varies with size; six blue boxes (15-25 μ m), eight purple boxes (25-35 μ m), five red boxes

(35-45 μm), and five green boxes (45-55 μm).





Fig. 3. Illustration of the heap region 'in front of the blade'.



Fig. 4. Particle velocity profiles in the region 'in front of the blade' for different gap heights for blade speed of
 0.04 m/s.



Fig. 5. Universal curve for the particle velocity profile in the region 'in front of the blade'.



Fig. 6. Variation of the shear band centre position h_c with respect to the spread layer surface with the blade
 speed for different gap heights.



Fig. 7. Variation of the shear band width w with the gap height for different blade speeds.



Fig. 8. Variation of mass flow rate with (a) blade speed and (b) gap height.



Fig. 9. Variation of (a) f_{U} and (b) critical gap height with blade speed.



2 Fig. 10. Profiles of u_z in the heap in front of the blade for different blade speeds for the gap height of $\delta/D=3.0$.