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1 **The Absence of an Hydraulic Threshold in Small-Diameter Surcharged** 2 **Manholes**

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13 **Abstract**

14 Previous research into surcharged manholes has highlighted the existence of a threshold surcharge level
15 that separates two distinctly-different hydraulic regimes. Sharp changes in manhole energy loss and solute
16 transport characteristics occur when the surcharge depth passes through the threshold level. With respect
17 to solute transport, two scale-independent Cumulative Residence Time Distributions (CRTDs) have been
18 identified, corresponding to the below-threshold and above-threshold hydraulic regimes. However,
19 previous studies focused on large diameter manholes, in which the manhole diameter (Φ_m) was at least 4.4
20 times greater than the pipe diameter (Φ_p). This paper utilizes a validated Computational Fluid Dynamics
21 (CFD) modelling approach to explore the hydraulic behaviour and mixing processes in small-diameter
22 surcharged manholes ($1.5 < \Phi_m/\Phi_p < 4.4$). It is shown that the hydraulic threshold does not exist in the
23 small diameter manholes; instead the flow field is characterised by short-circuiting throughout the full
24 range of surcharge depths. Data generated at low surcharge levels suggests that the mixing effects in the
25 below-threshold region are not independent of surcharge level, as had previously been suggested. The
26 absence of the threshold in small-diameter manholes is explained with reference to jet theory. Several
27 previous studies that have characterised mixing and/or energy losses in surcharged manholes are revisited,
28 allowing findings to be generalised across a broader range of manhole configurations, including effects due
29 to benching and change in outlet angle.

30 **Keywords**

31 Computational Fluid Dynamics (CFD), Cumulative Residence Time Distribution (CRTD), dispersion, energy-
32 loss, manhole, mixing, sewer, solute, surcharge

33 Introduction

34 Urban drainage systems mainly comprise lengths of pipes conveying flow between nodal structures,
35 including manholes, settling tanks, storage chambers and CSOs. The predominantly dendritic nature of the
36 system lends itself to one-dimensional flow modelling, and most engineering design requirements for
37 either capacity or water quality can be satisfied using such models. However, the sudden expansions from
38 pipes into structures and contractions out of structures create complex three-dimensional flow fields, and
39 the accurate inclusion of their effect requires knowledge of headloss coefficients for capacity calculations
40 and mixing characteristics to describe water quality changes. The flow field that develops within a
41 structure governs both its energy loss and solute transport characteristics. Tracer studies therefore provide
42 an integrated measure of the effects of the complete three-dimensional flow field.

43 Manholes are used to accommodate pipe junctions, changes in gradient or direction and access for
44 maintenance. Guymer *et al.* (2005) presented results from a laboratory study describing the retention time
45 and longitudinal dispersion of a solute tracer across circular surcharged manhole structures of different
46 diameters. The manhole, of diameter Φ_m , was positioned symmetrically within a length of pipe of diameter
47 Φ_p , in this case 0.088 m. Surge (s) was defined as the level of the free surface in the manhole above
48 the pipe soffit. Four different Φ_m values were considered – 800, 600, 485 and 385 mm – corresponding to
49 Φ_m/Φ_p ratios of 9.1, 6.8, 5.5 and 4.4 respectively. Note that the 485 and 385 mm manholes were referred
50 to as 500 and 400 mm in the original paper.

51 Two different flow and solute transport regimes have been identified for this specific configuration of
52 surcharged manhole, depending upon whether the surge level is below or above a critical threshold
53 level ($s' = 0.258 \Phi_m$). The two different hydraulic regimes were originally termed pre- and post-threshold.
54 However, to avoid confusion when considering time-dependent scenarios, they will henceforth be referred
55 to as below- and above-threshold. At surge levels below the threshold, the flow field is dominated by a
56 chaotic and swirling flow which promotes mixing. The incoming jet shows asymmetry in the horizontal
57 plane, with a clear recirculatory return current in the left portion of the manhole. The vertical flow field is
58 characterised by jet dissipation and mixing throughout the flow depth. Solute trace data demonstrates that
59 the below-threshold flow regime is characterised by good mixing, i.e. significant attenuation of the peak
60 concentration. Above the threshold surge level, the vertical flow field comprises two separated flow
61 regions. A dead zone forms in the upper part of the manhole, while the majority of the incoming flow
62 passes straight through the manhole, short-circuiting the upper storage volume. The horizontal flow field is
63 almost symmetric, and the vertical flow field is characterised by limited jet dissipation and mixing. This is
64 reflected in solute traces that typically show limited mixing, short-circuiting and reduced levels of peak
65 attenuation compared with the below-threshold case.

66 One approach to modelling solute transport within complex engineered systems is to consider the system's
67 Residence Time Distribution (RTD). An RTD describes the system's response to an instantaneous upstream
68 injection, and so conceptually can only be derived from a pulse (or instantaneous injection) experiment.
69 However, responses to other injection types – including the Gaussian-like distributions considered above –
70 may be converted to an RTD via numerical manipulation or deconvolution (Levenspiel, 1972). The
71 Cumulative RTD (CRTD) represents the integral of the RTD, which is usually normalised to sum to unity.
72 CRTDs are inherently scalable. Ignoring any effects of longitudinal dispersion and assuming 'plug flow', the
73 nominal retention time, t_n , within a body of fluid may be estimated from the ratio of the storage volume (V)
74 to the flowrate (Q), ($t_n = V/Q$). Danckwerts (1958) suggested that if a model is geometrically similar to the
75 prototype the normalized CRTD ($t_{nz} = tQ/V$) should be identical for model and prototype.

76 Based primarily on CFD (Computational Fluid Dynamics)-derived CRTDs, Stovin *et al.* (2010) suggested that
77 the solute transport characteristics of the surcharged manhole described above could be modelled by just
78 two normalized CRTDs, one for below- and the other for above-threshold surcharge. Guymer and Stovin
79 (2011) used deconvolution to reanalyse a portion of the laboratory data presented by Guymer *et al.*,
80 (2005), confirming the validity of the two-CRTD model for manhole mixing. It is important to note that the
81 normalised CRTDs are scale-independent, and hence these findings are directly applicable to full-scale
82 systems.

83 Guymer *et al.* (2005) suggested that the existence of the dual hydraulic regimes could be explained with
84 reference to jet theory. Rajaratnam (1976) has presented detailed theoretical analyses of the mean flow
85 characteristics for several simple configurations of turbulent jets, together with supporting experimental
86 data. For a circular free jet discharging into a large stagnant body of fluid, the radial velocity discontinuity
87 leads to the development of a shear layer. With distance from the point of entry, the shear layer
88 increasingly invades the uniform-velocity jet on its inside and penetrates into the ambient fluid on its
89 outside. Albertson *et al.* (1950) showed that the core velocity is retained for a distance of approximately
90 $6.2 \Phi_p$, irrespective of pipe velocity, whilst the jet dissipation region on a free jet spreads at a gradient of
91 1:5 to the longitudinal axis. Where the jet discharges adjacent to a wall, a shear layer develops on the fluid
92 side, whilst a boundary layer develops on the wall side. In such conditions, Rajaratnam and Subramanya
93 (1967) experimentally confirmed that the core velocity is retained for a slightly greater length, between 6.1
94 to $7.6 \Phi_p$.

95 More complex situations of confined jets have also been analysed by Rajaratnam (1976). However, these
96 are limited to the expansion of axi-symmetric circular jets in circular ducts. There is limited previous work
97 investigating jet decay in a confined chamber (or manhole). A full understanding of the flow regime and
98 energy losses in these structures has typically required experimental analysis or three-dimensional CFD
99 modelling. Pedersen and Mark (1990) investigated the energy losses associated with sewer manholes and

100 Camino *et al.* (2012) studied vertical jets issuing into a rectangular chamber to quantify energy dissipation.
101 Their results confirm that the boundaries of the chamber produce recirculations in the surrounding fluids,
102 reducing the inflow momentum and promoting more rapid jet decay.

103 Data presented by Guymer *et al.* (2005) suggested that the threshold surcharge levels for the 800, 600, 485
104 and 385 mm Φ_m manholes were approximately 220, 156, 121 and 90 mm respectively. Regression through
105 the origin gives a general relationship for the threshold surcharge level, $s' = 0.258 \Phi_m$. Guymer *et al.* (2005)
106 proposed that the threshold level was linked to the jet expansion described by Albertson *et al.* (1950). If
107 the surcharge level exceeds $\Phi_m/5$, then the jet dissipation region will not break the surface. This appears to
108 lead to the development of an upper dead zone, characteristic of the above-threshold hydraulic regime.
109 Conversely, if $s < \Phi_m/5$, then the jet dissipation region will break the free surface within the manhole,
110 leading to the chaotic, well-mixed, hydraulic regime associated with below-threshold surcharge levels. The
111 small difference between the observed trend for the threshold level ($s' = 0.258 \Phi_m$) and the 1:5 gradient
112 associated with the boundary of the theoretical jet dissipation region may reflect the fact that the jet core
113 moves vertically upwards away from the manhole base due to the formation of the boundary layer.

114 In the UK, the publication 'Sewers for Adoption' (WRc, 2006) suggests that regularly-encountered manhole
115 sizings range from 1200 mm on pipes less than 375 mm diameter ($\Phi_m/\Phi_p > 3.2$) up to 1800 mm on 750-
116 900 mm pipes ($2.00 < \Phi_m/\Phi_p < 2.40$). These Φ_m/Φ_p ratios fall significantly below the minimum considered
117 in previous manhole mixing studies, i.e. $\Phi_m/\Phi_p = 4.4$ (Guymer *et al.*, 2005). Research is required to
118 understand the flow, and pollutant mixing, processes associated with these smaller-diameter manholes.

119 For $\Phi_p = 88$ mm, the theoretically-determined jet core length of $6.2 \Phi_p$ equates to a distance of
120 approximately 550 mm, suggesting that – for the larger manholes studied by Guymer *et al.* – the core
121 velocity would not be present at the outlet. However, manholes with smaller Φ_m/Φ_p ratios would retain
122 the core jet velocity at the exit to the manhole, and might therefore be expected not to develop the chaotic
123 well-mixed flow conditions even at small surcharge levels. The Guymer *et al.* (2005) study suggested that
124 the threshold was evident at all four manhole diameters, although the presence of the threshold, and the
125 occurrence of well-mixed conditions at surcharge levels below the threshold, became less evident as Φ_m/Φ_p
126 reduced, and at higher discharges. The existence of the well-mixed zone at manhole diameters $< 6.2 \Phi_p$ is
127 consistent with others' observations that the jet decay is more rapid in confined systems. However, it is
128 expected that decreasing Φ_m/Φ_p below the ratios previously considered may ultimately result in the
129 elimination of the hydraulic threshold and the disappearance of the well-mixed below-threshold condition.

130 Stovin *et al.* (2008 and 2010) have demonstrated that it is feasible to utilize validated CFD models to
131 explore the flow field and solute transport regime for this type of system. This paper will therefore utilize a
132 validated CFD modelling methodology to explore the mixing characteristics of small-diameter surcharged

133 manholes. Findings will also be related to previous investigations into the systems' energy loss
134 characteristics.

135 **Methodology**

136 **CFD modelling methodology development and validation**

137 Although it is now widely accepted that CFD has an important role to play in understanding complex flow
138 fields and solute and sediment transport processes (see e.g. Jarman *et al.*, 2008), the requirement to
139 demonstrate that the models developed are robust and fit for specific purposes remains.

140 In the present paper, the CFD modelling methodology was developed and validated with reference to two
141 complementary data sets: Guymer *et al.*, (2005) and Lau (2008). The Lau (2008) study primarily focused on
142 scale effects, and replicated the 800 mm manhole experiments in a smaller-scale laboratory model
143 ($\Phi_m = 218$ mm, $\Phi_p = 24$ mm, $\Phi_m/\Phi_p = 9.1$). As part of this study, high-resolution Particle Image Velocimetry
144 (PIV) data was collected on both horizontal and vertical planes through the manhole for two specific
145 conditions, $s = 30$ mm ($0.138 \Phi_m$) and $s = 80$ mm ($0.367 \Phi_m$). These correspond to below- and above-
146 threshold conditions respectively, and solute trace data is also available for these specific conditions. The
147 flowrate was 0.35 L/s.

148 The CFD modelling was undertaken in ANSYS Fluent versions 6 and 12. All geometry and mesh generation
149 used GAMBIT. The CFD set-up is summarised in Table 1. The configuration options listed here were guided
150 by previous studies (Lau, 2008; Stovin *et al.*, 2008), and by a number of specific parametric studies and grid-
151 independence tests, the full details of which may be found in Bennett (2012) and Bennett *et al.* (2011).
152 These studies included a detailed comparison of three alternative turbulence models (RSM, k- ϵ RNG and k- ϵ
153 Realizable), a comparison between structured and unstructured mesh generation options, an investigation
154 into whether the VOF (Volume-of-Fluid multiphase model) would improve the quality of the flow field, and
155 comparisons between options for solute transport modelling (species model versus discrete phase model).
156 The ultimate selection of options was guided primarily by model quality (as assessed using the validation
157 procedures described below), although consideration was also given to ease of mesh generation,
158 convergence speed and simulation stability.

159 The simulations used an unstructured tetrahedral mesh, with a meshing interval equivalent to $\Phi_p/8$ (3 mm
160 in the 218 mm diameter manhole). The inlet boundary was modelled as a velocity inlet, with the velocity
161 profile taken from the outlet of an independent simulation of a pipe of the same diameter. The outlet was
162 represented using a pressure-outlet, and the free surface used a fixed-lid (zero friction) wall boundary. The
163 walls were modelled using a roughness of 4×10^{-5} m (previously determined for laboratory perspex). The
164 k- ϵ Realizable turbulence model was used, combined with non-equilibrium wall functions. Checks were

165 undertaken to ensure that y^+ values were within a suitable range ($30 < y^+ < 300$). Solute transport was
 166 modelled using uncoupled particle tracking, with a discrete random walk model to represent the stochastic
 167 effects of turbulence. Each trace experiment was represented by more than 45,000 particles with density
 168 998.2 kg/m^3 (as per water) and diameter $1 \times 10^{-6} \text{ m}$. Particles were injected uniformly over the inlet
 169 surface, with the maximum number of steps set to 1×10^9 and the Time Scale Constant = 0.10. RTDs for the
 170 manhole system were generated from particle travel times monitored at the model outlet.

171 The CFD models were subjected to three levels of validation. Primary validation involved the qualitative
 172 comparison of the velocity fields on selected planes against Lau's (2008) PIV data. Two CFD models were
 173 developed, corresponding to the 218 mm manhole with $s = 30 \text{ mm}$ and $s = 80 \text{ mm}$.

174 Secondary validation involved qualitative and quantitative comparisons of solute transport predictions
 175 against Lau's (2008) data. The RTDs were convolved with measured laboratory upstream temporal
 176 concentration profiles to synthesise downstream temporal concentration profile predictions. The R_t^2
 177 parameter (Equation 1, Young *et al.*, 1980) was used to provide a measure of the goodness of fit of the
 178 predicted downstream temporal concentration profile to the measured data at the downstream site.

$$179 \quad R_t^2 = 1 - \left[\frac{\sum_{t=1}^n (c(x_2, t) - p(x_2, t))^2}{\sum_{t=1}^n (c(x_2, t))^2} \right] \quad \text{Equation 1}$$

180 The particle tracking results are directly interpretable as the system's RTD, i.e. the system's fundamental
 181 response to an instantaneous injection at its upstream end. Given that the system under consideration
 182 included both the manhole and its upstream and downstream pipe sections, the manhole RTD was
 183 determined by subtracting pipe travel times (assuming plug flow) from the total travel times. To enable
 184 manhole travel times to be compared independently of scale and specific flowrate, these travel times were
 185 normalised according to the manhole's nominal travel time ($t_n = V/Q$, where V represents manhole
 186 volume). The CRTD represents the Cumulative RTD, and t_{50} represents the median travel time.

187 The third level of validation focused specifically on a comparison of CRTDs and t_{50} values obtained following
 188 deconvolution of the original Guymer *et al.* (2005) laboratory data. The deconvolution process, together
 189 with a full analysis of the CRTDs corresponding to the 800 mm manhole, was detailed in Guymer and Stovin
 190 (2011). New CRTD data, corresponding to the three smaller diameter manholes will be presented here for
 191 the first time.

192 **Identification of the threshold in small-diameter manholes**

193 The validated CFD modelling methodology was utilised to replicate and extend the range of Φ_m/Φ_p ratios
 194 considered by Guymer *et al.* (2005) ($\Phi_p = 88 \text{ mm}$), and in particular to explore whether the hydraulic

195 threshold (and its associated sudden change in mixing characteristics) would persist in the smaller diameter
196 manholes more regularly encountered in practice. In addition to the four previously-considered
197 configurations ($\Phi_m/\Phi_p = 9.1, 6.8, 5.5$ and 4.4), manholes with diameters 308, 220, 176 and 132 mm (Φ_m/Φ_p
198 = 3.5, 2.5, 2.0 and 1.5) were also modelled. Each was modelled at a minimum of 9 different surcharge
199 depths ($s = 10$ to 272 mm), with additional depths included where required to define the threshold level
200 more accurately. 87 different combinations of manhole diameter and surcharge level were considered in
201 total. As it has previously been shown that the threshold level is independent of flowrate, all simulations
202 were undertaken at 1 l/s. However, for the largest manhole, additional simulations at 4 l/s were included
203 to confirm this.

204 Identification of the presence or absence of the threshold surcharge level involved the visual inspection of
205 flow fields and normalised CRTDs and an assessment of normalised t_{50} travel times plotted against
206 surcharge level.

207 **Results**

208 **CFD modelling methodology development and validation**

209 Flow fields

210 Fig. 1 compares the simulated flow field (right-hand column) with the laboratory PIV flow field data (left-
211 hand column) collected by Lau (2008) for a 218 mm diameter manhole ($\Phi_p = 24$ mm). The data is
212 presented for two validation cases: below-threshold ($s = 30$ mm) and above-threshold ($s = 80$ mm).
213 Qualitatively it may be seen that the simulations effectively replicate the two contrasting flow regimes, the
214 asymmetric below-threshold condition characterised by full mixing in the vertical plane, and the more-
215 symmetric high surcharge condition characterised by short-circuiting. There are minor discrepancies, most
216 notably surrounding the jet. In the low-surcharge case the jet core appears to extend too far into the
217 manhole in the CFD simulation (see Fig. 1b), whereas in the high-surcharge case the jet core is a little
218 weaker in the CFD simulation (Fig. 1f) than was observed in practice (Fig. 1e).

219 Solute traces

220 Figs. 2a and 2b compare the CFD-derived CRTDs with their laboratory counterparts for the two flow
221 regimes. In this case the CRTDs are for the complete system, including the pipe sections upstream and
222 downstream of the manhole. In Figs. 2c and 2d the CFD-derived RTDs have been convolved with the
223 measured upstream temporal concentration profiles to generate downstream profiles that are directly
224 comparable with the laboratory data. Figs. 2a and 2c include output from larger and smaller mesh
225 densities than that used for the bulk of the simulations; they confirm that the chosen density was
226 essentially grid-independent for the purposes of this study.

227 It may be seen that there is good agreement between the simulated solute transport characteristics and
228 those observed in the laboratory. R_t^2 values for the simulated traces shown in Figs. 2c (3 mm mesh) and 2d
229 are 0.983 and 0.994 respectively.

230 Threshold identification

231 The third level of validation focused on comparisons of the CFD results (CRTDs and t_{50} travel times) with the
232 original Guymer *et al.* (2005) laboratory data for manholes with $\Phi_p = 88$ mm and Φ_m of 800, 600, 485 and
233 385 mm. The deconvolution process used to derive CRTDs from the laboratory trace data has been
234 described elsewhere (Guymer and Stovin, 2011), although data from the two smallest manholes has not
235 been presented previously. It is important to note that there are some limitations to the laboratory data
236 set (concerns about mass balance, incomplete mixing, incomplete/unrepresentative sampling of dye traces
237 over the pipe cross-section, calibration errors, etc). Nonetheless, confirmation that the CFD model is able
238 to replicate key trends observed in the laboratory over a range of higher Φ_m/Φ_p ratios provides confidence
239 in its application to explore the processes acting in smaller diameter manholes.

240 Fig. 3a presents normalised CRTDs for the 800 mm laboratory manhole. This figure clearly shows the
241 characteristic steep rise associated with short-circuiting effects in the high surcharge conditions. There is a
242 very clear change from the highest below-threshold surcharge level (210 mm, $s/\Phi_m = 0.263$) to the lowest
243 above-threshold surcharge level (238 mm, $s/\Phi_m = 0.298$). The below-threshold CRTDs do not collapse onto
244 a single curve, as has previously been suggested. Instead, systematic variations are observed with mixing
245 becoming increasingly well-developed as surcharge level increases. Particularly in the above-threshold
246 cases, it may be seen that full mass recovery on the CRTDs was not observed, likely as a result of premature
247 truncation of the tail of the downstream trace.

248 Fig. 3b presents the modelled data for the same configuration, and confirms that similar trends are
249 reproduced. The sharp transition between the two hydraulic regimes appears to occur at a similar
250 surcharge level (in this case $0.20 < s/\Phi_m < 0.27$). The same trends in the below-threshold CRTDs are
251 apparent, although the systematic variation in below-threshold CRTDs appears to be greater in the CFD
252 data than was observed in the laboratory data.

253 Figs. 3a and 3b suggest that the t_{50} parameter may be used as an indicator of the systematic variations in
254 hydraulic regime as a function of surcharge. t_{50} values increase with surcharge up to the threshold level, s' ,
255 and fall to a low, constant, value for surcharge levels $> s'$.

256 In Figs. 3c and 3d the variations in t_{50} values as a function of surcharge level for all four manhole diameters
257 are compared. Both the laboratory and the modelled data sets provide strong indications of the existence
258 of the hydraulic threshold (i.e. a sharp drop in t_{50} values at the threshold surcharge level), and both show
259 that mixing effects increase approximately linearly with surcharge in the below-threshold region. The

260 laboratory data appears to suggest that the threshold is spread over a range of levels ($0.2 < s/\Phi_m < 0.258$),
261 whereas the CFD data suggests a sharper transition at close to $s/\Phi_m = 0.2$.

262 The laboratory data also supports the suggestions in the CFD data that, as the manhole diameter reduces,
263 the maximum value of the normalised t_{50} decreases and the s/Φ_m ratio associated with the threshold
264 appears to reduce. The CFD data appears to overestimate mixing in the high below-threshold region. This
265 may be due to laboratory cut-off techniques; discrete-phase particle tracking ensures 100% mass recovery
266 whereas this was not consistently achieved in the laboratory data.

267 Both laboratory and CFD data sets show that the normalised t_{50} in the above-threshold region increases as
268 manhole diameter reduces. This is to be expected, as the short-circuiting jet flow bypasses a greater
269 volume of inactive flow in the larger diameter manholes than in the smaller ones.

270 Validation Summary

271 Overall the validation process has demonstrated that the CFD modelling methodology is fit for purpose.
272 The simulated flow fields capture key elements of these complex, jet-driven, confined, flows in two
273 contrasting hydraulic regimes. The particle tracking model is capable of replicating observed downstream
274 temporal concentration profiles with a high degree of accuracy. Lastly, it has been demonstrated that CFD-
275 derived t_{50} values are consistent with those generated in the laboratory, and that they provide a robust
276 indication of hydraulic regime that can be used with confidence to explore the existence of the hydraulic
277 threshold in small-diameter manholes.

278 **Identification of the threshold in small-diameter manholes**

279 Flow fields

280 The left-hand column of Fig. 4 shows the simulated flow field on the central horizontal plane in four out of
281 the eight simulated manholes, for flows with a surcharge level of $0.13 \Phi_m$. Note that the actual manhole
282 diameter decreases from top to bottom; the pipe size is constant. The flow fields shown correspond to
283 surcharge levels which are 'expected' to be below the hydraulic threshold, s' . Figs. 4a ($\Phi_m = 800$ mm) and
284 4c ($\Phi_m = 600$ mm) clearly show the asymmetric flow field associated with the below-threshold low-
285 surcharge flow regime. The 308 mm diameter manhole (Fig. 4e) shows a slight deviation of the jet.
286 However, in contrast to the larger diameter manholes, the high velocity core passes directly to the manhole
287 outlet. The smaller diameter manholes (for example, the 220 mm manhole shown in Fig. 4g) show no
288 obvious signs of the asymmetric jet. This suggests that, at Φ_m/Φ_p ratios below 3.5, the jet is fully restrained
289 by the geometry of the manhole, and the highly-asymmetric (well mixed) below-threshold flow regime
290 cannot be established. The flow field is more comparable with that observed in high-surcharge conditions
291 (above-threshold) in larger-diameter manholes.

292 CRTDs

293 The normalised CRTDs for the full range of surcharge levels are presented for these same configurations in
294 the right-hand column of Fig. 4. The 800 mm manhole CRTDs presented in Fig. 4b show clear groupings
295 corresponding to the previously-identified flow regimes. At surcharge levels above the threshold, the
296 CRTDs collapse onto a single curve. However, below the threshold a systematic variation in CRTD shape is
297 observed, with the short-circuiting indicated by the steep initial rise in the CRTD increasing as surcharge
298 decreases. The smallest amount of short-circuiting is evident when the surcharge level is just below the
299 threshold. This is indicative of increased mixing within the manhole due to the highly asymmetric jet.
300 Replicating data presented in Fig. 4b, Fig. 5 confirms that the normalised CRTDs are independent of
301 flowrate.

302 At very low surcharges (10-20 mm), which have not been investigated in previous studies, a third grouping
303 is present. Its characteristic CRTD shape is similar to that of the above-threshold condition.

304 The 600 mm manhole CRTDs (Fig. 4d) show similar behaviour to the 800 mm manhole data. There is a
305 sharp discontinuity (i.e. the threshold) between the CRTDs corresponding to $s/\Phi_m = 0.18$ and $s/\Phi_m = 0.20$,
306 and the below-threshold CRTDs show a systematic decrease in short-circuiting with increased surcharge.

307 In contrast, the CRTDs corresponding to the 308 and 220 mm manholes do not show any evidence of
308 below-threshold, well-mixed, hydraulic regimes. Both the flow field plots and the CRTDs suggest that, at
309 Φ_m/Φ_p ratios of 3.5 and below, the well-mixed, below-threshold flow regime is not established; instead the
310 flow field is characterised by short-circuiting throughout the full range of surcharge levels. The gradient of
311 the CRTDs is less steep in the larger manholes; this reflects the reduction in inactive manhole volume as
312 Φ_m/Φ_p decreases.

313 Normalised t_{50} travel times

314 Fig. 6a shows normalised t_{50} against s/Φ_m . The 800 ($Q = 1$ and 4 l/s), 600, 485 and 385 mm manholes all
315 exhibit a sharp step in normalised t_{50} travel times, confirming the existence of the hydraulic threshold. The
316 308 and 220 mm manholes do not exhibit a large step in t_{50} times, indicating the lack of an hydraulic
317 threshold. Fig. 6a shows that for the manholes that do exhibit a hydraulic threshold, the region below-
318 threshold is transitional in nature.

319 In Fig. 6b, the Y-axis scale has been normalised according to the equivalent travel time assuming the
320 manhole was replaced by a pipe of the same length. The above-threshold conditions induce mixing
321 equivalent to a pipe. In the below-threshold regime, as the manhole volume available for mixing decreases
322 (Φ_m/Φ_p decreases), the normalised t_{50} values tend towards those expected for a pipe (i.e. plug flow).

323 Results for two different flowrates ($Q = 1$ l/s; $Q = 4$ l/s) are shown for the 800 mm manhole in Figs. 6a and
324 6b. These are nearly identical for all surcharge levels, confirming that the threshold location is independent
325 of flow rate.

326 It has been hypothesised that a link exists between the Albertson *et al.* (1950) 1:5 jet expansion gradient
327 and the ratio of the manhole diameter to the threshold level. This value (i.e. $s' = 0.2 \Phi_m$) is indicated in
328 Figs. 6a and 6b. The manholes with a Φ_m/Φ_p ratio greater than 5 (485, 600 and 800 mm diameter
329 manholes) show an hydraulic threshold close to that of the Albertson *et al.* (1950) jet expansion limit.
330 Below this limit, the 385 mm manhole ($\Phi_m/\Phi_p = 4.4$) also shows a step, indicative of a change in hydraulic
331 regime. However, as the Φ_m/Φ_p ratio decreases, the surcharge required to cause a change in hydraulic
332 regime also decreases. This deviation away from the idealised model may reflect the increasingly
333 significant effects of confinement as the manhole diameter decreases.

334 Discussion

335 Variation in CRTD shape in the below-threshold region in large-diameter manholes

336 Guymer and Stovin (2011) identified consistent, systematic variations in CRTD shape in the below-threshold
337 region. As with the current CFD-derived data, mixing appeared to increase (less steep CRTD) with
338 increasing surcharge. The effects were less marked than the current data set, leading to the conclusion
339 that the below-threshold region could be reasonably approximated using a single normalised CRTD. The
340 current data set suggests, however, that an improved model of mixing for large-diameter, low-surcharge
341 manholes should probably aim to account for these systematic variations.

342 Variations occur because the free surface places a vertical constraint on the jet expansion. As the
343 surcharge level increases, the jet has more freedom to expand vertically, which also leads to enhanced
344 transverse mixing, and more complete, chaotic mixing throughout the full manhole volume.

345 Comparison with the threshold surcharge level proposed by Guymer *et al.* (2005)

346 For manholes with $\Phi_m/\Phi_p \geq 4.4$, it has been shown that the threshold surcharge level, s' , corresponds to
347 $0.2 \Phi_m$. It has also been shown that this ratio is consistent with the theoretical model of jet expansion
348 proposed by Albertson *et al.* (1950). However, the data originally presented by Guymer *et al.*, 2005,
349 corresponded to $s' = 0.258 \Phi_m$. There are certain limitations to the Guymer *et al.* (2005) data. The data
350 was analysed using fitted ADE and ADZ models, which did not fit the measured data particularly well in
351 many cases. The authors acknowledged scatter in their data set and qualified their findings with the
352 suggestion that there may be a transition region rather than a distinct threshold. Equally, the discrepancies
353 may reflect minor deficiencies in the CFD model's representation of the system, particularly the fixed lid
354 free surface approximation and/or the turbulence model.

355 **Conditions required to establish a well-mixed hydraulic regime in a surcharged manhole**

356 Fig. 7 provides a schematic representation of the hydraulic conditions that occur in circular, un-benched,
357 straight-through, surcharged manholes. Manhole A represents a system with $\Phi_m/\Phi_p \geq 4.4$, such as those
358 considered in the previous (Guymer *et al.*, 2005) study; Manhole B is representative of the smaller-
359 diameter manholes that have been considered for the first time in the present study. Flow patterns are
360 characterised by short-circuiting unless two conditions apply, $\Phi_m/\Phi_p \geq 4.4$ and $s < \Phi_m/5$. This zone is
361 highlighted with diagonal hatched shading in Fig. 7.

362 These two constraints are discussed below:

363 **Manhole diameter constraint: $\Phi_m/\Phi_p \geq 4.4$** – The manhole diameter must be sufficiently large such that
364 the incoming jet core can fully dissipate before it reaches the manhole outlet. For small diameter manholes
365 (e.g. Manhole B), the jet tends to bridge the distance across the manhole, and minimal mixing (horizontal
366 or vertical) is observed. In the present study, manholes with Φ_m of 308 mm and below ($\Phi_m/\Phi_p \leq 3.5$)
367 showed no evidence of well-mixed behaviour at any surcharge levels, whereas manholes with Φ_m of 385
368 mm and above ($\Phi_m/\Phi_p \geq 4.4$) did exhibit the two distinct hydraulic regimes previously reported. Given the
369 weakness of the threshold effects observed in the 385 mm manhole, $\Phi_m/\Phi_p = 4.4$ is taken as the lower limit
370 for the development of well-mixed conditions below a threshold surcharge level.

371 Albertson *et al.* (1950) showed that for a free jet, the core velocity is retained for a distance of $6.2 \Phi_p$,
372 irrespective of pipe velocity. For $\Phi_p = 88$ mm, this suggests that the core velocity would not be present at
373 the outlet for manholes with $\Phi_m \geq 550$ mm. However, the persistence of well-mixed conditions in the
374 region ($4.4 < \Phi_m/\Phi_p < 6.2$) suggests either that the tip of the jet needs to penetrate some distance into the
375 outlet pipe to induce short-circuiting effects, or that the geometrical constraints imposed by the manhole
376 cause the jet to dissipate over a shorter distance than it would within an idealised semi-infinite volume.

377 **Surcharge depth constraint: $s < \Phi_m/5$** – For manholes with $\Phi_m/\Phi_p \geq 4.4$ (e.g. Manhole A), the surcharge
378 level must also be sufficiently shallow to enable the dissipating jet to promote mixing throughout the
379 manhole's entire surcharge depth. It is evident that when s is high, the head of water in the overlying
380 surcharge volume will tend to dampen the mixing effects associated with the diffusing jet, and that the
381 characteristic above-threshold short-circuiting flows will develop. On the other hand, as s reduces, a critical
382 depth will be reached at which the momentum associated with the diffusing jet as it impacts upon the
383 manhole's downstream face will be sufficient to promote full mixing throughout the entire surcharge
384 depth. Our observations suggest that the threshold level (s') is approximately equal to $\Phi_m/5$, consistent
385 with the Albertson *et al.* (1950) theoretical model of jet expansion at 1:5.

386 **Links between mixing and energy loss characteristics**

387 Given that the transition between below-threshold well-mixed conditions and above-threshold short-
388 circuiting conditions in large diameter manholes results from a fundamental change in the system's
389 hydraulic regime, comparable changes in energy loss characteristics should also be evident. The energy
390 loss owing to the manhole, ΔH , is defined as the difference in pressure head between the extrapolated
391 upstream and downstream hydraulic grade lines at the manhole axis:

$$392 \quad \Delta H = K \frac{u^2}{2g} \quad \text{Equation 2}$$

393 where K is the energy loss coefficient, u is mean pipe velocity and g is gravitational acceleration.

394 Data presented by Lau *et al.* (2008) for laboratory manholes with $\Phi_m/\Phi_p = 9.1$ showed strong evidence of
395 the links between flow regime, energy loss and solute transport, with K values falling sharply from near-
396 constant values of approximately 1.0 below-threshold to around 0.45 for above-threshold surcharge
397 depths. Energy loss data presented by Arao and Kusuda (1999) ($\Phi_m/\Phi_p = 3.6$) showed similar trends. High
398 K values are associated with higher levels of energy dissipation in well-mixed flows; low K values reflect the
399 reduced levels of energy dissipation associated with short-circuiting flow patterns.

400 The present study has introduced manholes with smaller Φ_m/Φ_p ratios than either of these studies. Fig. 6c
401 presents K values for the current data set, calculated from the CFD pressure data. Consistent with the
402 solute transport data (Fig. 6a) and with the previous studies, the data for the four largest manholes shows
403 clear evidence of an hydraulic threshold. Also consistent with the solute transport data, the energy loss
404 data suggests that the threshold does not arise in the two smaller diameter manholes, where short-
405 circuiting tends to occur throughout the range of surcharge depths. The 308 mm manhole shows some
406 evidence of transitional behaviour in both Figs. 6a and 6c. Its mixing and energy loss characteristics are
407 similar to the larger manholes at low surcharge depths, but become more comparable with the 220 mm
408 manhole at higher surcharge depths.

409 **Implications for more complex manhole configurations**

410 Both the normalised t_{50} and the energy loss coefficient K are dimensionless (scale-independent)
411 parameters. The observations presented here are therefore applicable to full-scale systems (Lau *et al.*,
412 2008; Stovin *et al.*, 2010). However, it is recognised that the specific manhole configuration considered
413 here is far simpler than many that are encountered in practice, which may include, for example, multiple
414 inlets, angled outlets, benching or a drop in invert. Several of these configurations have been considered in
415 previous laboratory studies, and selected findings will now be reviewed and interpreted with reference to
416 jet theory and the underlying hydraulics.

417 In the unbenched manhole systems described earlier in this paper, well-mixed conditions were observed
418 only when the incoming jet was unconfined and able to dissipate fully within the surcharge depth. It
419 follows that the confining effects of benching will tend to prevent well-mixed conditions from ever
420 developing and lead to short-circuiting flows throughout the range of surcharge depths. In contrast, the
421 placement of the outlet at an angle from the inlet centreline will tend to enhance mixing. The impact of
422 the jet on the downstream manhole wall is expected to lead to chaotic and well-mixed conditions at lower
423 Φ_m/Φ_p ratios and/or at greater surcharge depths than with a straight-through outlet.

424 Saiyudthong (2003) investigated the effects of varying outlet angle in benched and unbenched surcharged
425 manholes on solute transport and head loss, using a laboratory system with $\Phi_p = 88$ mm and $\Phi_m = 388$ mm.
426 The outlet angles were 0° (i.e. straight-through), 30° , 60° , and 90° and flow rates were 1 l/s, 2 l/s, 4 l/s, 6 l/s
427 and 8 l/s. Interestingly, with $\Phi_m/\Phi_p = 4.4$, this study lies just at the boundary of the conditions required for
428 the hydraulic threshold that have been proposed here. Mixing characteristics were reported in terms of
429 fitted ADE model mean travel times, and it should be noted that this approach has subsequently been
430 shown to poorly represent this type of system, and may lead to significant loss of discrimination (Stovin *et*
431 *al.*, 2010). The data suggests that at an outlet angle of 0° , the system behaved as a ‘small diameter’
432 manhole, i.e. there was little or no evidence of well-mixed flow conditions. However, the travel time data
433 shows some evidence of the threshold (i.e. enhanced mixing at low levels of s) for outlet angles of 30° and
434 60° in unbenched manholes. With benching, there was no evidence of the threshold, or of well-mixed flow
435 conditions, over the full range of outlet angles.

436 Sonnenwald *et al.* (2011) used deconvolution to recover CRTDs from a portion of Saiyudthong’s original
437 laboratory data (1 l/s, unbenched). Their analysis confirmed that whilst well-mixed conditions seemed not
438 to occur at any surcharge depths for the 0° outlet angle configuration, they were increasingly evident as the
439 outlet angle increased. The data also suggested that, as outlet angle increased, the threshold surcharge
440 depth increased. For an outlet angle of 90° , the threshold surcharge depth approaches Φ_m , compared with
441 approximately $0.2 \Phi_m$ for a straight-through configuration.

442 The cases above provide qualitative support to hypotheses derived from an understanding of the system’s
443 fundamental hydraulics. Benching will tend to confine the jet, resulting in reduced mixing and minimal
444 energy losses, whilst an angled outlet will tend to increase mixing and energy losses.

445 For more complex systems, even a descriptive assessment becomes problematic. Manholes are
446 characterised by complex, interacting, hydrodynamic processes. Considerable research effort continues to
447 be invested in empirical laboratory studies aimed at quantifying and modelling the integrated effects of a
448 wide range of configuration variables. For example, Arao *et al.* (2011) build upon extensive previous
449 laboratory-based experience to focus on drop manholes subjected to two inflows and four alternative
450 inflow/outflow diameter and elevation combinations. The derived relationships for K incorporate seven

451 separate non-dimensional parameters, one of which relates manhole diameter to the downstream pipe
452 diameter. The current findings complement this work in three respects: firstly, it has been shown that
453 significant thresholds exist in both the Φ_m/Φ_p ratio and s/Φ_m . It does not appear that these critical
454 thresholds have been accounted for in any existing empirical relationships; secondly, we have shown that
455 these relationships are linked to the inlet jet, which suggests that a dimensionless group based on inlet,
456 rather than outlet, pipe diameter may prove to be more robust; and thirdly, CFD-based modelling tools
457 have been shown to provide an effective alternative to physical laboratory tests. Given the infinite range of
458 possible manhole configurations, it may be impractical to generate empirical relationships to accurately
459 capture all possibilities. In complex cases, where accurate understanding is critical, specific studies will
460 always be required. As computing power improves, it becomes feasible to anticipate the use of hybrid
461 1D/3D sewer network models that utilise simple hydraulics for the connecting pipes but expand out to 3D
462 hydrodynamic CFD analysis for complex hydraulic structures at the nodes.

463 The similarities between Figs. 6a and 6c suggest that it might be feasible to use solute transport data to
464 estimate energy losses, for example within real sewer systems. Alternatively, it should also be feasible to
465 model a system's solute transport characteristics from known energy loss data. Further research into this
466 aspect is recommended. Whilst the t_{50} parameter has been adopted here to characterise the hydraulic
467 regime, there is likely to be value in considering a range of other parameters associated with the CRTD
468 (Stovin *et al.*, 2008). For example, whilst t_{50} indicates the median residence time, t_{16} and t_{84} provide
469 information about the system's dispersion characteristics.

470 **Conclusions**

471 A CFD modelling methodology has been developed to enable the flow fields and solute transport
472 characteristics of small-diameter surcharged manholes ($\Phi_m/\Phi_p < 4.4$) to be studied. Validation comprised
473 three stages: direct validation of flow fields against laboratory PIV measurements; indirect validation of
474 CRTDs against laboratory solute trace tests; and the use of t_{50} values to identify the hydraulic threshold
475 within a range of surcharge depths.

476 The hydraulic threshold previously identified for surcharged manholes is only evident in systems with large
477 Φ_m/Φ_p ratios. In smaller-diameter manholes the incoming jet will tend to bridge the distance across the
478 manhole such that short-circuiting effects dominate, irrespective of surcharge level. The critical Φ_m/Φ_p
479 ratio has been found to be $\Phi_m/\Phi_p = 4.4$. Many manholes found in practice have Φ_m/Φ_p ratios of less than
480 4.4; their mixing characteristics can be assumed to be equivalent to a pipe.

481 In manholes with high Φ_m/Φ_p ratios, a low-surcharge, well-mixed, condition develops at surcharge levels
482 below approximately $0.2 \Phi_m$. In contrast to previous findings, a systematic variation in CRTD shape has
483 been observed, with the greatest mixing occurring at surcharge levels just below the hydraulic threshold.

484 Furthermore, at very low surcharges (10-20 mm), which have not been investigated in previous studies, a
485 third hydraulic regime is present. Its characteristic CRTD shape is similar to that of the above-threshold
486 condition.

487 Consistent with previous laboratory findings, data from the current CFD study has confirmed that the
488 hydraulic threshold is also evident in the energy loss coefficients.

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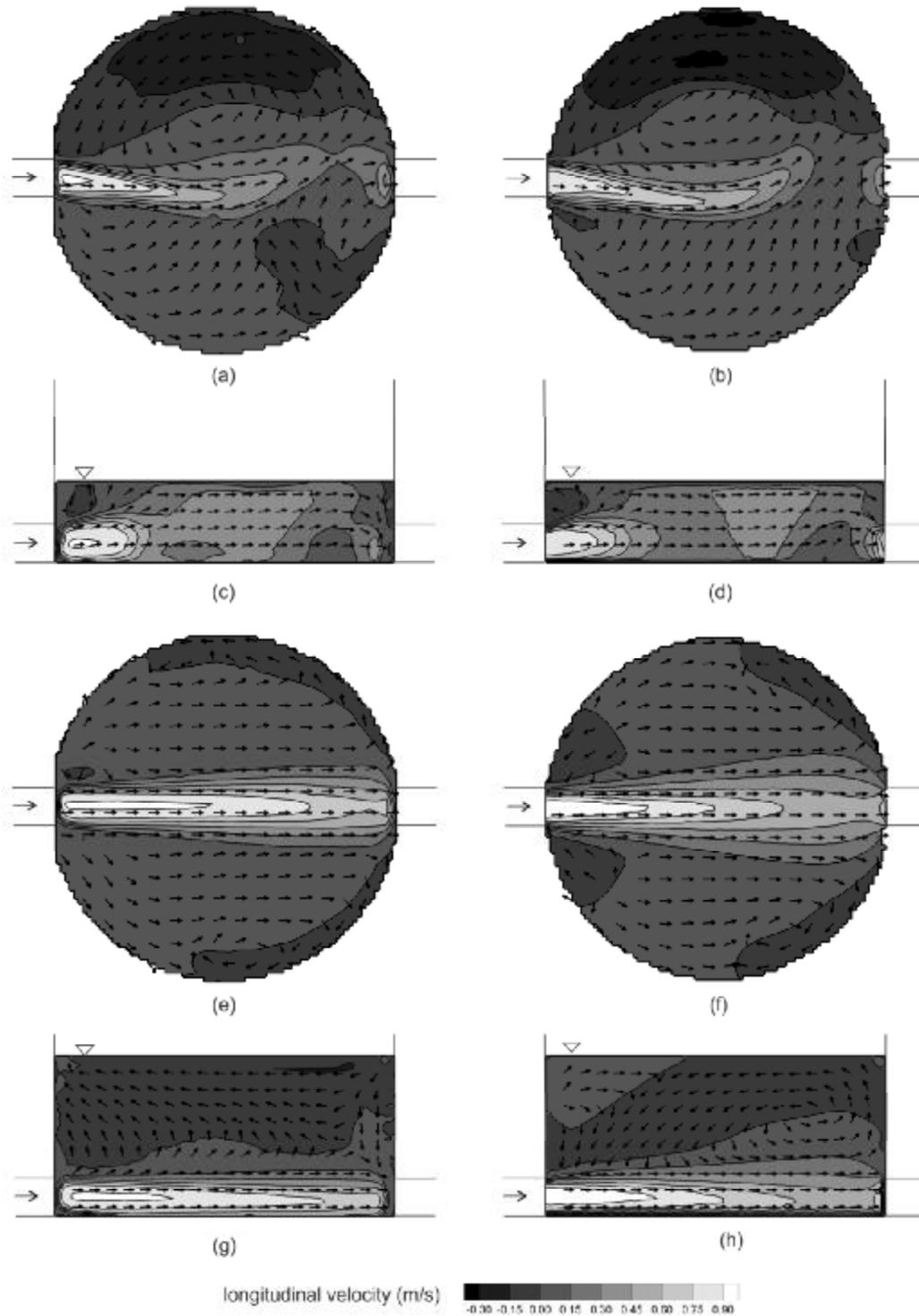


Figure 1 – Flow fields corresponding to the central horizontal plane (a, b, e and f) and the central vertical plane (c, d, g and h) for a 218 mm manhole ($Q = 0.35$ L/s) for PIV data (Lau (2008), left) and the CFD model ($k-\epsilon$ Realizable turbulence model, right). Contours are coloured by longitudinal velocity, vectors indicate flow direction. Plots (a) to (d) correspond to the below-threshold validation case, whereas plots (e) to (h) correspond to the above-threshold case.

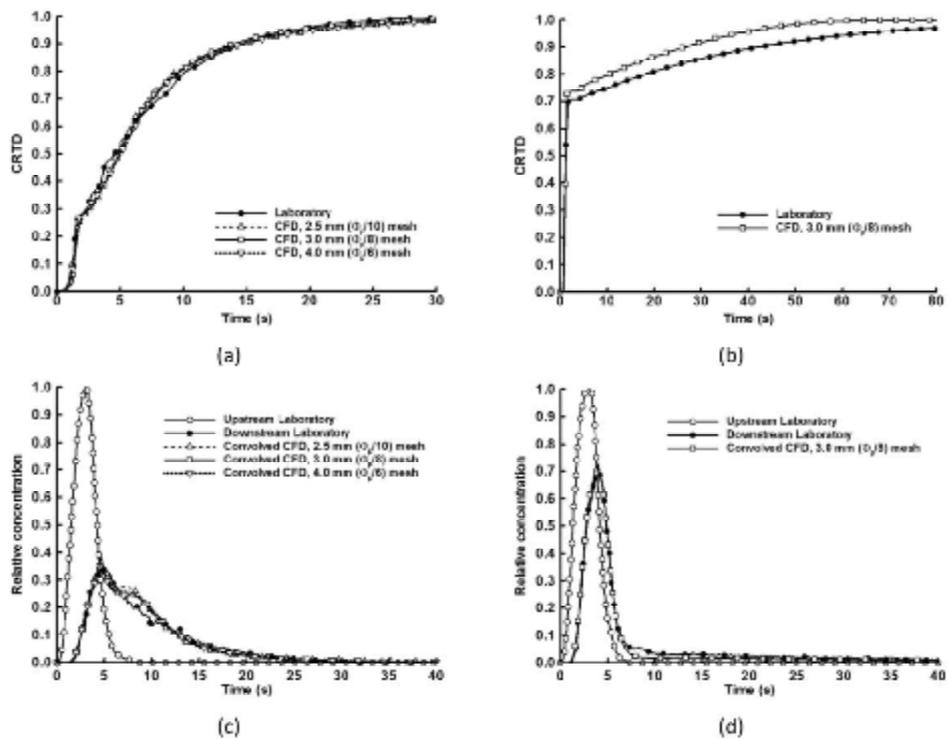


Figure 2 – CRTDs and downstream predictions compared with the laboratory results of Lau (2008). (a) and (b) show the CRTDs for below- and above-threshold conditions respectively. Laboratory CRTDs have been derived using deconvolution from laboratory trace data. (c) and (d) compare measured downstream concentration profiles with profiles synthesised by convolving the measured upstream trace with the CFD-derived CRTDs shown in (a) and (b).

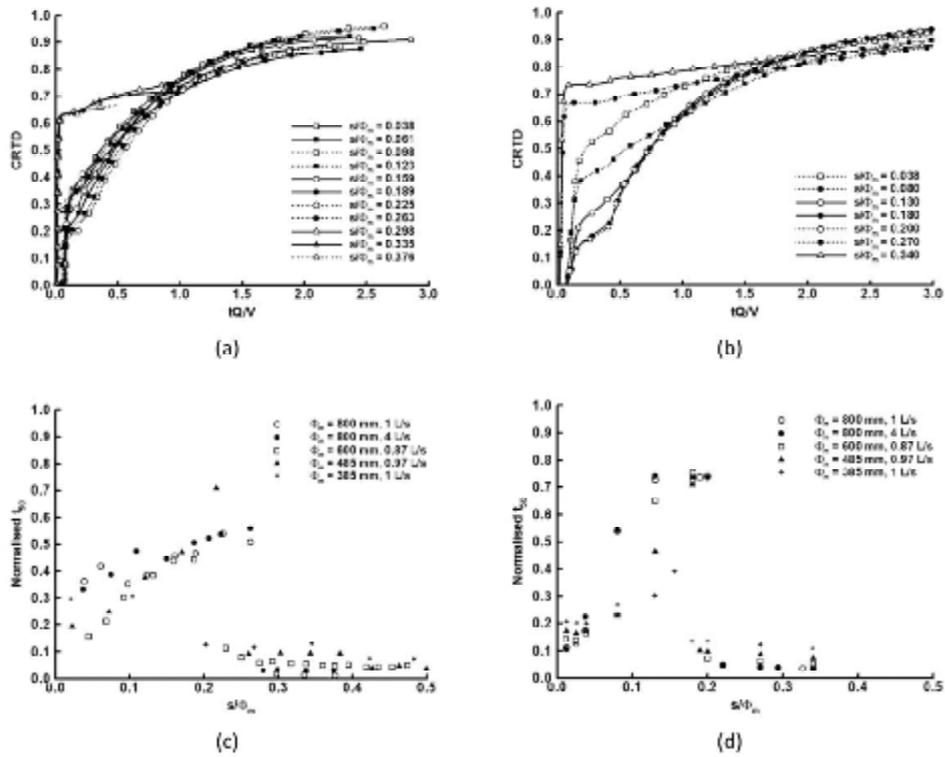


Figure 3 – Threshold identification. (a) and (c) present laboratory data (see Guymer *et al.* (2005) and Guymer and Stovin (2011)); (b) and (d) show the current CFD simulation results. (a) and (b) show CRTDs for the 800 mm diameter manhole with $Q = 1$ L/s; (c) and (d) show t_{50} travel times for the four different Φ_m/Φ_p ratios considered.

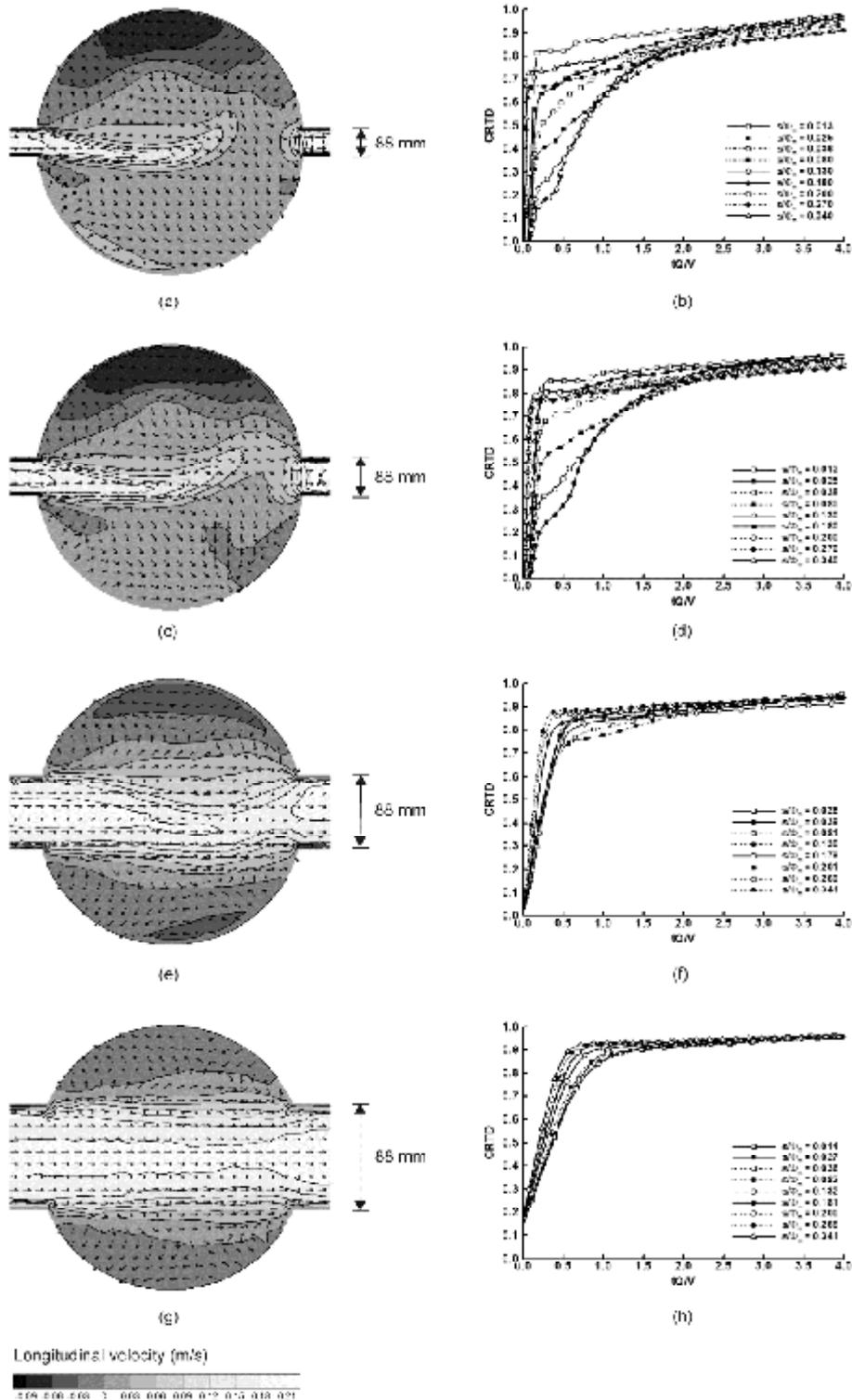


Figure 4 – Simulated flow fields (central horizontal plane, $s = 0.13 \Phi_m$, left) and CRTDs for a range of manhole diameters and surcharge depths (right). In all cases $\Phi_p = 88$ mm and $Q = 1$ L/s. (a) and (b) correspond to the 800 mm manhole ($\Phi_m/\Phi_p = 9.1$), (c) and (d) the 600 mm manhole ($\Phi_m/\Phi_p = 6.8$), (e) and (f) the 308 mm manhole ($\Phi_m/\Phi_p = 3.5$) and (g) and (h) the 220 mm manhole ($\Phi_m/\Phi_p = 2.5$).

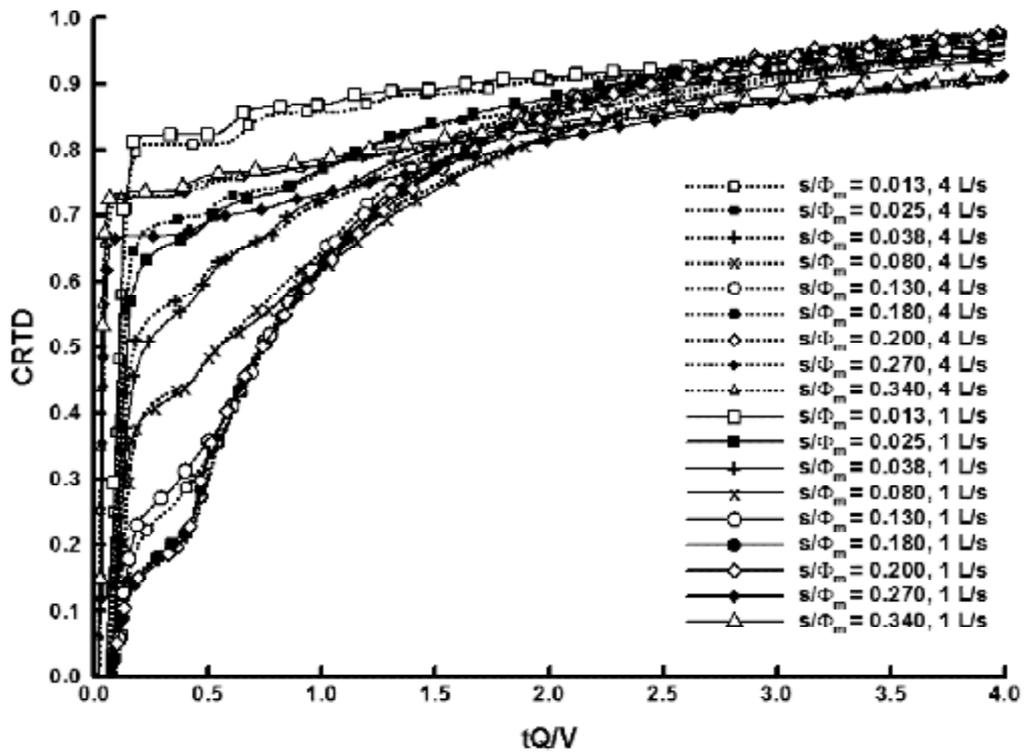
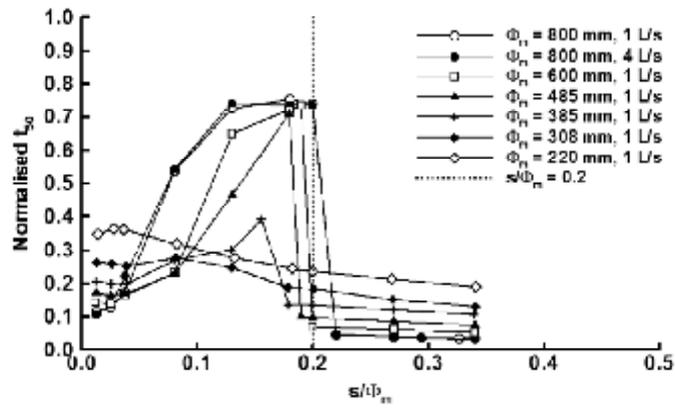
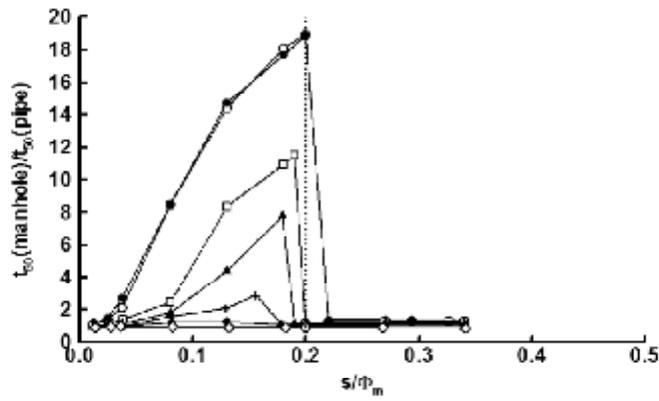


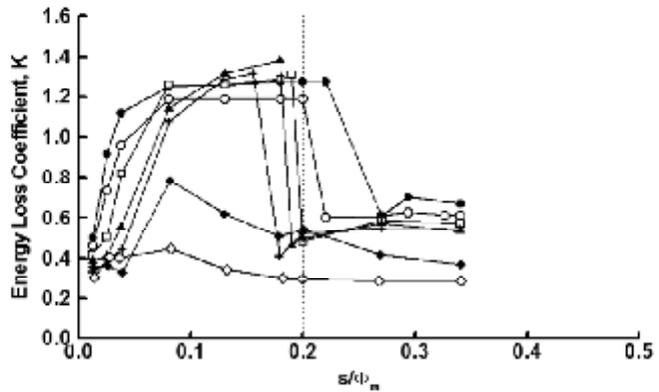
Figure 5 – The effect of flowrate on normalised CFD-derived CRTDs.



(a)



(b)



(c)

Figure 6 – Travel times and energy loss coefficients as a function of s/Φ_m . The dotted line indicates $s/\Phi_m = 0.2$, which equates to the upper limit of the jet dissipation zone according to Albertson *et al.*, (1950).
 (a) Normalised t_{50} travel times; (b) t_{50} travel times normalised with respect to an equivalent pipe length; (c) Energy loss coefficient, K .

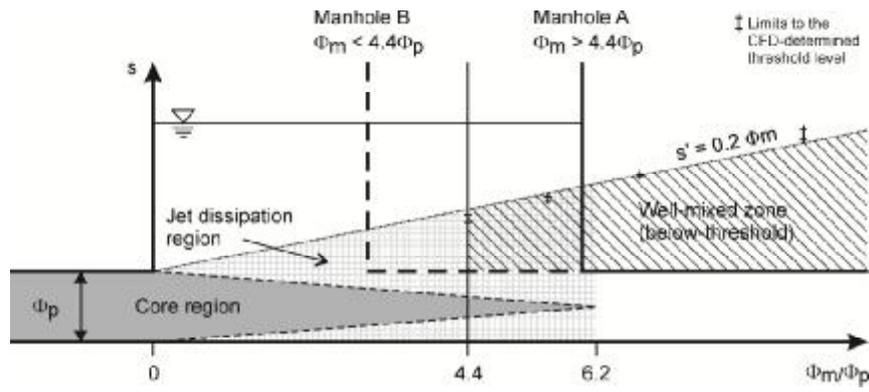


Figure 7 – Hydraulic conditions associated with well-mixed flow in surcharged circular manholes. Jet dissipation associated with an ideal discharge into a semi-infinite volume (after Albertson *et al.*, 1950) is also indicated.

Table 1 Validated set-up parameters for meshing, solving and modelling solute transport

Parameter	Setting/Value
Mesh:	
Meshing Strategy	Unstructured, fixed-lid
Face Mesh	Tri-pave
Volume Mesh	Tgrid
Reference Face	Inlet
Inlet boundary	Velocity-inlet
Outlet boundary	Pressure-outlet
Top boundary	Wall (zero friction)
Wall boundary	Wall (Perspex)
Mesh interval	3 mm ($\Phi_p/8$)
y^+ values	$30 < y^+ < 300$
Solver:	
Pressure	PRESTO
Momentum terms	QUICK
Velocity-pressure coupling	SIMPLEC
Turbulence Model	k- ϵ Realizable
Gravity	Yes
Boundary wall roughness	0 m, 4×10^{-5} m
Near-wall treatment	Non-equilibrium wall functions
Residuals	1×10^{-5}
Solute:	
Model	Uncoupled particle tracking
Discrete random walk	Yes
No. of particles	>45,000
Particle density	998.2 kg/m ³ (as per water)
Particle size	1×10^{-6} (uniform)
Injection	from surface (inlet)
Max no. of steps	1×10^9 (maximum)
Time Scale Constant	0.10