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A GPU-based coupled SPH-DEM method for particle-fluid flow with free surfaces

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ABSTRACT

Particle-fluid flows with free-surfaces are commonly encountered in many industrial processes, such as wet ball milling, slurry transport and mixing. Accurate prediction of particle behaviors in these systems is critical to establish fundamental understandings of the processes, however the presence of the free-surface makes modelling them a challenge for most traditional, continuum, multi-phase methodologies. Coupling of smoothed particle hydrodynamics and discrete element method (SPH-DEM) has the potential to be an effective numerical method to achieve this goal. However, practical application of this method remains challenging due to high computational demands. In this work, a general purposed SPH-DEM model that runs entirely on a Graphic Processing Unit (GPU) is developed to accelerate the simulation. Fluid-solid coupling is based on local averaging techniques and, to accelerate neighbor searching, a dual-grid searching approach is adapted to a GPU architecture to tackle the size difference in the searching area between SPH and DEM. Simulation results compare well with experimental results on dam-breaking of a free-surface flow and particle-fluid flow both qualitatively and quantitatively, confirming the validity of the developed model. More than 10 million fluid particles can be simulated on a single GPU using double-precision floating point operations. A linear scalability of calculation time with the number of particles is obtained for both single-phase and two-phase flows. Practical application of the developed model is demonstrated by simulations of an agitated tubular reactor and a rotating drum, showing its capability in handling complex engineering problems involving both free-surfaces and particle-fluid interactions.

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1. Introduction

Understanding particle behaviors in free-surface flows is crucial to many natural phenomena and industrial processes, such as debris flow [1], wet ball milling [2, 3], slurry transport [4], mixing and separation in chemical and mineral processing [5–7] and additive manufacturing [8]. Despite wide popularity, application of traditional grid-based methods, such as coupled computational fluid dynamic (CFD) and discrete element method (DEM), to handle particles in free-surface flows is still challenging due to the presence of free-surfaces, especially splashing and fragmentations [9-11]. Additional detection algorithms are required to track the free surfaces for grid-based Eulerian methods, such as volume of fluid [9], marker-and-cell [10] and level-set method [11]. In addition, numerical diffusion may arise due to advection terms. On the other hand, grid-based Lagrangian methods face problems of mesh distortion, which requires expensive mesh re-generation. The presence of splashing and fragmentations requires a numerical method which can handle the discrete nature of the free-surface flows. SPH, as a meshless method, shows strong potentials in this regard

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as it discretize the fluid into a set of particles, thus allowing the dynamics of the free-surface flows to be readily captured. Since the pioneer work of Gingold and Monaghan [12], SPH has been widely used to model free-surface flows in many fields [13]. It has been recently extended to deal with solid particles in free-surface flows by coupling SPH with DEM [7, 14–17]. Nevertheless, application of this method to engineering problems remains limited due to associated high computational cost. This is especially the case when complex moving boundaries are present. Thus, a high performance implementation of the SPH-DEM method is necessary for applications in engineering practice.

Coupling of SPH and DEM enables a unified Lagrangian particlebased method, well suited for applications where free-surfaces and strong fluid-particle interactions coexist. Approaches for coupling the momentum transfer between the fluid and particles can be classified into two groups: resolved and unresolved methods. In the resolved method the SPH particles are significantly smaller than the solid particles and the flow around the solid particles is explicitly resolved. In the unresolved method the SPH particles are of a comparable size of the solid particles and empirical force correlations are used to capture the momentum transport between phases. For resolved simulation, a variety of methods have been developed to enable no-slip conditions at solid surfaces. Potapov et al. [18] coupled SPH with DEM to simulate

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shear flow of neutrally buoyant particles, where no-slip boundary condition is enabled by placing SPH particles inside large solid particles [19]. Densities of the interior SPH particles are updated in the same manner as the fluid SPH particles while moving the interior SPH particles with the solid particles. This approach was verified with experimental results on the drag force and size of the wake for fluid flow around a circular cylinder. Canelas et al. [20] coupled SPH with a so-called distributed contact DEM, where the solid body is represented by a set of small particles with fixed relative position. The interaction between fluid and solid particles are calculated in a manner similar to the methodology of dynamic boundary conditions [21]. The interactions among solid bodies are calculated through the small constituent particles by means of soft-sphere collision model. This model shows good comparison with experiments on tracking blocks subjected to a dambreak wave [20] and on a debris flow [22]. Ren et al. [23] also reported a similar method to study stability of 2D blocks on a slope due to wave-structure interaction. Since the fluid resolution has to be sufficiently small to resolve flow structure around the particle, high computational demand is thus inevitable when dealing with a large number of solid particles. The requirement of a larger size of solid particles than the fluid particles further limits their range of applicability, especially considering a wide size distribution of solid particles is commonly encountered in practical systems.

On the other hand, phase interaction in the unresolved simulation is handled by empirical force correlations. Both one-way and two-way coupled SPH-DEM methods have been reported. For example, Komoroczi et al. [24] combined SPH with DEM by treating either DEM particles as SPH particles or treating SPH particles as DEM particles. Cleary et al. [4, 5] treated the solid bed as a dynamic porous media by averaging the velocities and porosities of the DEM particles, through which the SPH fluid is able to flow. This one-way coupled method has been applied to predict slurry transport in SAG mills [4] and to model slurry flow on a double deck vibrating banana screen [5] in mineral processing. Sun et al. [6] reported a two-way coupled SPH-DEM method in which a boundary model based on variational approach is explicitly included in the momentum equation. A unified boundary representation is enabled for both phases, eliminating the redundant wall particles for the fluid phase. Similarly, Robinson et al. [17] developed a coupled SPH-DEM method based on the locally averaged Navier-Stokes equations. The model showed good agreement with sedimentation test cases with increasing complexity, namely, settling of single sphere, constant block of particles and multiple particles. The fully coupled SPH-DEM method is gaining popularity in different engineering applications, including fluid-particle-structure interaction problem with free-surface flow [14, 15] and landslide generated surge waves [16] and particle separation due to density difference in waste recycling [7]. To date, however, most of these studies are concerned with 2D simulation [14-16] or limited to a very small scale [7, 17], therefore, developing a general SPH-DEM model capable of accelerating simulation is of great importance for practical problems in engineering applications.

Computational efficiency of numerical methods is closely related to the progress of hardware architecture. GPU-based parallelism has been increasingly applied to speedup simulations of particle-based numerical methods due to high memory bandwidth and instruction throughput offered by GPU processors, such as Molecular Dynamics [25, 26], Lattice Boltzmann Method [27-29], SPH [30, 31] and DEM [32–36]. GPU-based implementations show orders of magnitude faster than their serial counterparts, making it very attractive for schemes that can make use of the single instruction multiple data architecture. Due to the locality of particle interactions, both SPH and DEM schemes require identification of neighboring particles. In pure SPH models the interaction length is greater than twice the particle diameter, where in pure DEM simulations there is no long distance force and the interaction search area is approximately the solid particle diameter. In coupled SPH-DEM systems, the interactions between different particle types leads to increased complexity. To accelerate neighbor searching on GPU, different algorithms have been reported, including the *k-d* tree method [37] and spatial subdivision approaches, such as the bounding volume hierarchy method [38, 39] and the uniform grid method [35]. The *k-d* tree structure needs to be reconstructed at every time step, thus not suitable for discrete methods while the spatial subdivision approaches are limited by the fact that the grid size needs to be large enough to host the largest particles. Application of these searching methods mentioned above to SPH-DEM is thus not straightforward, especially on a GPU platform. A general purpose SPH-DEM program also needs to be efficient at GPU memory management. To the best of our knowledge, coupling of SPH and DEM methods that run entirely on a GPU platform has yet to be reported.

Starting from the theoretical background, this work will address the development of a general purposed GPU-based SPH-DEM method in detail. A dual-grid searching approach is incorporated to handle the difference in the particle interacting range between SPH and DEM. Model validation and performance analysis of the GPU-based model will be evaluated for both the single phase and the particle-fluid flows, respectively. Practical application of the GPU-based model to chemical engineering will be illustrated by the simulating free-surface flows in an agitated tubular reactor and particle-fluid flows in a rotating drum. The paper is organized as follows: the model formulation is first presented in Section 2 followed by a detailed description of the GPU implementation, including the neighbor searching method, memory management and program flow. Then, the validation, engineering application and performance of the developed model in handling both single-phase flow and particle-fluid flow with free-surfaces are addressed separately in Section 3.

2. Model description and GPU implementation

To achieve unresolved simulation, different approaches to couple DEM to SPH have been reported [6, 14–17], which are essentially solving governing equations of the conventional Two Fluid Model (TFM) [40] in the framework of SPH, which are given as,

$$\frac{\partial(\epsilon\rho_{\rm f})}{\partial t} + \nabla \cdot (\epsilon\rho_{\rm f} \mathbf{u}) = \mathbf{0} \tag{1}$$

$$\frac{\partial(\epsilon\rho_{\rm f}\mathbf{u})}{\partial t} + \nabla \cdot (\epsilon\rho_{\rm f}\mathbf{u}\mathbf{u}) = -\nabla P - S_{\rm p} + \nabla \cdot (\epsilon\boldsymbol{\tau}_{\rm f}) + \epsilon\rho_{\rm f}\mathbf{g}$$
(2)

with $\rho_{\rm f}$ the fluid density, *P* pressure acting on the fluid phases, $\tau_{\rm f}$ the viscous stress tensor and **g** the acceleration due to gravity. ε is the volume fraction of fluid in each cell. $S_{\rm p}$ is the source term due to the rate of momentum exchange between the fluid phase and the solid phase. The coupling strategy adopted here is similar to that proposed by Robinson et al. [17]. In this section, an overview of the SPH and the DEM methods used to discretize the governing equations and the strategy of phase coupling are provided. A detailed description of the force models used in DEM can be found elsewhere [41–43]. For ease of reading, the fluid particles are labeled as particle *a* and *b* while the solid particles are labeled as particle *i*.

2.1. Fluid phase: SPH

The methodology behind SPH is based on the theory of integral interpolants, the interpolated value of a function $A(\mathbf{r})$ at position \mathbf{r} is expressed as,

$$A(\mathbf{r}) = \int A(\mathbf{r}')W(|\mathbf{r}-\mathbf{r}'|,h)d\mathbf{r}'$$
(3)

where the kernel function $W(|\mathbf{r} - \mathbf{r}'|, h)$ tends to delta function when the interpolation domain is infinitely small. The size of the interpolation domain is characterized by a smoothing length *h*. In SPH, fluid is discretized into individual particles with each carrying a set of associated properties, such as density, pressure and momentum. The fluid particles follow the flow due to pressure gradient, viscous shear and body force while acting as interpolation points for their neighbors. The integral interpolant at the position of the particle is approximated by,

$$\mathbf{A}(\mathbf{r}_{a}) = \sum_{b} A_{b} \frac{m_{b}}{\rho_{b}} W(|\mathbf{r}_{ab}|, h)$$
(4)

with m_b and ρ_b the mass and density of particle b, $|\mathbf{r}_{ab}|$ being the distance between two fluid particles. The summation is taken over all particles within the support domain of particle a.

The kernel function $W(|\mathbf{r}_{ab}|, h)$ must obey a number of mathematic constraints, including positivity, monotonically decreasing, compact support and normalization. In this study, the Wendland kernel function is used since it can achieve a good balance between numerical accuracy and computational cost [44].

$$W(|\mathbf{r}_{ab}|, h) = \alpha_D \left(1 - \frac{q}{2}\right)^4 (1 + 2q) \ 0 \le q \le 2$$
(5)

where α_D is $7/8\pi\hbar^3$ in 3D. $q = |\mathbf{r}_{ab}|/h$. In practice, the kernel function vanishes when the particle separation is greater than 2h to achieve a compact support domain.

2.1.1. Continuity equation

Applying the SPH particle approximation, the continuity equation of Eq. (1) can be written as,

$$\frac{d\overline{\rho}_{a}}{dt} = \sum_{b} m_{b} \mathbf{v}_{ab} \nabla_{a} W_{ab}(|\mathbf{r}_{ab}|, h)$$
(6)

with $\nabla_{\mathbf{a}} W(|\mathbf{r}_{\mathbf{a}b}|, h)$ the gradient of the kernel function at the position of particle *a*, and $\mathbf{v}_{\mathbf{a}b} = \mathbf{v}_{\mathbf{a}} - \mathbf{v}_{\mathbf{b}}$ is the velocity vector. $\overline{\rho}$ denotes the superficial fluid density defined as $\overline{\rho}_a = \varepsilon \rho_{\mathbf{f}}$, with ε the volume fraction of fluid and $\rho_{\mathbf{f}}$ the intrinsic fluid density.

2.1.2. Equation of state

In the weakly compressible SPH schemes [45], fluid particles are driven by local pressure gradient. Pressure is expressed as a function of the fluid density, giving a quasi-incompressible equation of state as follows:

$$P = B\left(\left(\frac{\overline{\rho}_a}{\varepsilon\rho_0}\right)^{\gamma} - 1\right) \tag{7}$$

with $\gamma = 7$ and ρ_0 reference density of the fluid. *B* is a pressure constant determines the speed of sound by $c_s^2 = \gamma B/\rho_0$. The density fluctuation in fluid flow is proportional to M^2 where *M* is the Mach number. By limiting the Mach number, the flow can be considered practically incompressible. To limit the density variation within 1%, the coefficient *B* can be calculated as,

$$B = \frac{100\rho_0}{\gamma} v_{\text{max}}^2 \tag{8}$$

2.1.3. Momentum equation

The momentum equation of Eq. (2) in SPH form is given by,

$$\frac{d\mathbf{v}_{a}}{dt} = -\sum_{b} m_{b} \left[\frac{P_{a}}{\overline{\rho}_{a}^{2}} + \frac{P_{b}}{\overline{\rho}_{b}^{2}} + R_{ab} \right] \nabla_{a} W_{ab}(|\mathbf{r}_{ab}|, h) + \Pi_{ab} + S_{a} + \mathbf{g}$$
(9)

with *P* the pressure due to fluid phase, Π_{ab} the viscosity term and R_{ab} the term due to tensile instability and S_a the couple term due to solid particles. The symmetrical form of the pressure gradient term is taken in order to reduce error arising from particle inconsistence. The stress

terms $\Pi_{\rm ab}$ representing the viscous diffusion derived by [19] is incorporated into the momentum equation,

$$\Pi_{ab} = \sum_{b} \frac{m_{b}(\mu_{a} + \mu_{b})\mathbf{r}_{ab}\cdot\nabla_{a}W_{ab}(|\mathbf{r}_{ab}|, h)}{\rho_{a}\rho_{b}\left(\mathbf{r}_{ab}^{2} + 0.01h^{2}\right)}\mathbf{v}_{ab}$$
(10)

where μ is the dynamic viscosity. In the weakly compressible SPH, tensile instability is often attributed as the cause of unphysical particle clumping. This phenomenon is especially significant in materials with an equation of state which can give rise to negative pressures, which reduces numerical accuracy due to the uneven particle distribution. To remove the instability, [46] introduced an additional pressure between particles to prevent particle from forming small clumps. This artificial pressure term R_{ab} is added to the momentum equation, given as,

$$R_{ab} = 0.01 \left(\frac{P_a}{\overline{\rho}_a^2} + \frac{P_b}{\overline{\rho}_b^2} \right) \left(\frac{W(|\mathbf{r}_{ab}|, h)}{W(\Delta p, h)} \right)^4$$
(11)

where Δp is the initial particle spacing.

2.1.4. Moving the particles

To keep an orderly flow of particle, the particles are moved using the XSPH variant,

$$\frac{d\mathbf{r}_{a}}{dt} = \mathbf{v}_{a} - \epsilon \sum_{b} \frac{m_{b}}{\overline{\rho}_{ab}} \mathbf{v}_{ab} W_{ab}(|\mathbf{r}_{ab}|, h)$$
(12)

where $\overline{\rho}_{ab} = (\rho_a + \rho_b)/2$ and ϵ is a problem-dependent constant, ranging from 0 to 1. It moves a particle at a velocity close to the average velocity of its neighbors. In this study, ϵ is set to 0.3.

2.1.5. Boundary treatment

When a SPH particle approaches a rigid boundary, the support domain of its kernel will be truncated by the boundary. Ideally, the boundary treatment should compensate for the lack of particles beyond the boundary and provide enough repulsive force to prevent particles from penetration. Different treatments have been proposed to this end, including kernel re-normalization [47], ghost particles [48], layers of fixed fluid particles, repulsive particles [45] and dynamic particles [49–51]. Sun et al. [6] introduced a correction to the SPH approximation where the boundary information are explicit included without using extra wall particles. Together with level-set functions, a unified boundary was enabled for both the solid phase and the fluid phase. In the present study, a generalized wall boundary treatment that is capable of handling arbitrarily shaped geometries was adopted [52]. As shown in Fig. 1, solid wall is discretised into dummy particles whose properties do not evolve with time. These wall particles provide support for the kernel interpolants of the fluid particles. The pressure and velocity at the position of a wall particle is interpolated from surrounding fluid



Fig. 1. Boundary treatment used in SPH calculation: boundary particles acting as both dummy fluid particles and repulsive particles.

particles to ensure a non-slip boundary condition of the solid walls, which are given by,

$$\mathbf{v}_{w} = 2\mathbf{v}_{a} - \frac{\sum_{b} \mathbf{v}_{b} W_{ab}(|\mathbf{r}_{ab}|, h)}{\sum_{b} W_{ab}(|\mathbf{r}_{ab}|, h)}$$
(13)

$$P_{w} = \frac{\sum_{f} P_{f} W_{wf}(|\mathbf{r}_{wf}|, h) + (\mathbf{g} - \mathbf{a}_{w}) \sum_{f} \rho_{f} \mathbf{r}_{wf} W_{wf}(|\mathbf{r}_{wf}|, h)}{\sum_{f} W_{wf}(|\mathbf{r}_{wf}|, h)}$$
(14)

$$\rho_{w} = \rho_{0} \left(\frac{P_{w}}{B} + 1\right)^{\frac{1}{\gamma}} \tag{15}$$

with \mathbf{v}_{a} the prescribed wall velocity, \mathbf{a}_{w} the acceleration of the wall and | \mathbf{r}_{wf} | being the distance between fluid particle and wall particle. In practice, however, wall penetration cannot be fully avoid where violent fluid-wall interactions exist. Therefore, in this study, the repulsive force proposed by Monaghan [45] is combined with the above boundary treatment to fully prevent wall penetration.

2.2. Solid phase: DEM

In DEM, the motion of solid particles is tracked by Newton's second law of motion, written as,

$$m\frac{d\mathbf{v}}{dt} = \mathbf{F}_{\rm f} + \mathbf{F}_{\rm c} + m\mathbf{g} \tag{16}$$

$$I\frac{d\boldsymbol{\omega}}{dt} = \mathbf{T}_{\mathrm{f}} + \mathbf{T}_{\mathrm{c}} \tag{17}$$

where *m*, *l*, **v** and ω are, the mass, inertia, translational and rotational velocities of the element, respectively. The force and torques acting on each particle consists of several contributions, including the hydrodynamic components, **F**_f and **T**_f, arising from fluid-particle interaction, the collision components, **F**_c and **T**_c, due to solid-solid interaction and *m***g** the gravity. The collisions between particles are handled by a soft-sphere model that allows for inter-particle overlap. The collision force includes the normal contact force **F**_n, normal damping force **F**_{d,n}, tangential contact force **F**_t and tangential damping force **F**_{d, t}. The normal contact is based on Mindlin and Deresiewicz theory [53]. The normal and tangential contact forces are given as,

$$\mathbf{F}_{n} = \frac{4}{3} E^{*} R^{*1/2} \delta_{n}^{3/2} \hat{\mathbf{n}}$$
(18)

$$\mathbf{F}_{t} = -\frac{\delta_{t}\mu_{t}|\mathbf{F}_{n}|}{|\boldsymbol{\delta}_{t}|} \left[1 - \left(1 - \frac{\min(|\boldsymbol{\delta}_{t}|, \boldsymbol{\delta}_{t, \text{ max }})}{\boldsymbol{\delta}_{t, \text{ max }}}\right)^{3/2}\right]$$
(19)

in which the R^* , E^* and $\delta_{t, \max}$ are calculated as,

$$\frac{1}{R^*} = \frac{1}{R_i} + \frac{1}{R_j}$$
(20)

$$\frac{1}{E^*} = \frac{1 - \nu_i^2}{E_i} + \frac{1 - \nu_j^2}{E_i}$$
(21)

$$\delta_{t, \max} = \frac{(2-\nu)}{2-2\nu} \mu_t \delta_n \tag{22}$$

with R_i and R_j being the radius of two particles in contact. E and ν are the Young's Modulus and Poisson's ratio of solid particles, respectively. δ_n and δ_t represent the overlap in normal and tangential directions and μ_t is the sliding friction.

The equations used to calculate damping in normal and tangential directions are given by,

$$\mathbf{F}_{d,n} = -c_n \left(8m^* E^* \sqrt{R^* \delta_n}\right)^{1/2} \mathbf{v}_n \tag{20}$$

$$\mathbf{F}_{d,t} = -c_t \left(6\mu_t m E^* |\mathbf{F}_n| \frac{\sqrt{1 - |\boldsymbol{\delta}_t| / \delta_{t, \max}}}{\delta_{t, \max}} \right)^{1/2} \times \mathbf{v}_t$$
(21)

where c_n and c_t are the normal and tangential damping coefficient, respectively. The normal damping coefficient can be directly linked to the restitution coefficient *e* in the normal direction by,

$$c_{\rm n} = - \ln e / \sqrt{\pi^2 + \ln^2 e} \tag{23}$$

The normal restitution coefficient *e* is defined as the ratio of post-collisional contact velocity to pre-collisional contact velocity.

The collision torque T_c is composed of the torque due to the tangential force T_t and the torque T_r due to particle rolling friction resulting from the elastic hysteresis losses or viscous dissipation [54], calculated as,

$$\mathbf{T}_{t} = \left(\mathbf{F}_{t} + \mathbf{F}_{d,t}\right) \times \mathbf{R} \tag{24}$$

$$\mathbf{T}_{\mathrm{r}} = \boldsymbol{\mu}_{\mathrm{r}} R |\mathbf{F}_{\mathrm{n}}| \hat{\boldsymbol{\omega}}_{\mathrm{n}} \tag{25}$$

where μ_r is the rolling friction and $\hat{\omega}_n = \omega_n / |\omega_n|$ with ω_n the angular velocity.

2.3. Phase coupling

The local porosity at the position of fluid particle a is calculated by a summation over neighboring DEM particles within a coupling length h_c , given as,

$$\varepsilon_a = 1 - \sum_j W_{aj}(h_c) V_j \tag{26}$$

with V_j the volume of DEM particle j, $W_{aj}(h_c)$ the SPH kernel and h_c the coupling length for the interaction between two phases. The coupling length should be larger than the diameter of solid particle but small enough to capture local feature of the porosity field. Here, the coupling length is set to be same as the SPH smoothing length.

For the solid particles, forces due to fluid flow are modelled. The total fluid force can be split into a pressure gradient force and a drag force.

$$\mathbf{F}_{i} = -V_{i}(-\nabla P + \nabla \cdot \mathbf{\tau}) + \mathbf{F}_{d}$$
(27)

with V_i the particle volume, ∇P the pressure gradient and \mathbf{F}_d the drag force. The pressure gradient is evaluated at solid particle *i* using a Shepard corrected SPH interpolation [55], given as,

$$(-\nabla P + \nabla \cdot \mathbf{\tau})_{i} = \frac{1}{\sum_{b} \frac{m_{b}}{\rho_{b}} W_{ab}(h_{b})} \sum_{b} \frac{m_{b}}{\rho_{b}} \theta_{b} W_{ib}(h_{b})$$
(28)

$$\theta_{\rm a} = -\sum_{\rm b} m_{\rm b} \left(\frac{P_{\rm a}}{\overline{\rho}_{\rm a}^2} + \frac{P_{\rm b}}{\overline{\rho}_{\rm b}^2} \right) \nabla_{\rm a} W_{\rm ab}(h_{\rm b}) + \Pi_{\rm ab} \tag{29}$$

The fluid-particle drag force F_d depends on the local porosity and relative velocity between fluid and particle. For the dense particle fluid flow, the drag model of Ergun and Wen & Yu [56, 57] is used, which is based on experimental measurements.

$$\mathbf{F}_{d} = \frac{\beta V_{i}}{1 - \varepsilon} (\mathbf{u} - \mathbf{v}) \tag{30}$$

with β the interphase momentum exchange coefficient, which is given by,

$$\beta = \begin{cases} 150 \frac{\mu (1-\varepsilon)^2}{\varepsilon d_i^2} + 1..75 \frac{(1-\varepsilon)\rho_f}{d_i} |\mathbf{u} - \mathbf{v}_i| & (\varepsilon < 0.8) \\ \frac{3}{4} C_D \frac{\varepsilon (1-\varepsilon)}{d_i} \rho_f |\mathbf{u} - \mathbf{v}_i| \varepsilon^{-2.65} & (\varepsilon > 0.8) \end{cases}$$
(31)

where C_D is the drag coefficient. It can be calculated as,

$$C_{D} = \begin{cases} \frac{24(1.0 + 0.15Re_{p}^{0.687})}{Re_{p}} & (Re_{p} \le 1000) \\ 0.44 & (Re_{p} > 1000) \end{cases}$$
(32)

in which the particle Reynolds number Re_p is defined as $Re_p = \rho d_p \varepsilon | \boldsymbol{u} - \boldsymbol{v}_p | / \mu$.

The rate of momentum exchange in the right hand of Eq. (2) for each SPH particle is calculated by a weighted average of fluid-particle coupling force acting on the surrounding DEM particles within the coupling length h_c , so that Newton's third law of motion is satisfied, which is given by,

$$S_{a} = -\frac{m_{a}}{\rho_{a}} \sum_{j} \frac{1}{\sum_{b} \frac{m_{b}}{\rho_{b}} W_{ab}(h_{b})} F_{i} W_{aj}(h_{c})$$

$$(33)$$

2.4. GPU implementation

2.4.1. Neighbor searching

The locality of interaction in both SPH and DEM methods highlights the importance of an efficient neighbor searching algorithm. Neighbor searching is often regarded as the most time demanding aspect of a particle-based method. Although extensive studies have been conducted to optimize the searching process, implementation of a searching algorithm on the GPU platform is still not straightforward for the coupled SPH-DEM method. The reason is attributed to the size difference between the support domain, 2h, of the SPH particles and the diameter, *d*, of the DEM particles. DEM calculations are mainly concerned with particles in direct contact or with short-range interactions, whilst long-ranged interactions are present in the SPH calculations. If the size of the search grid is set as the smoothing length of the SPH particles, 2h, it would inevitably include redundant DEM particles. A dual-grid searching approach was therefore implemented to reduce the size of the search domain for the solid phase and thus accelerate the neighbor searching process. Fig. 2 shows a schematic view of the dual-grid searching approach applied to a coupled system where particles of two phases are of the same size. The hollow circles represent the DEM particles while the solid circles represent the SPH particles. Based on a traditional linked-cell method, the implemented searching method consists of two steps: particle mapping and all-level searching.

Firstly, particles are mapped onto different grids according to their type: solid particles are mapped onto *Level 1* while fluid particles are mapped onto *Level 2*. A radix sort algorithm from the *Thrust* library is performed on the resulting array to sort the particles according to their cell indices [58]. The cell size for the SPH particles is set



Fig. 2. Schematic demonstrating neighbor searching algorithm used in coupled SPH-DEM approach, in which (a) dual-grid system, (b) searching solid neighbors for solid particle in DEM grid, (c) searching fluid neighbors for fluid particle, (d) searching fluid neighbors for solid particle and (e) searching solid neighbors for fluid particle.

as $2h + \Delta_{\text{SPH}}$ while it is set as $d_{\text{max}} + \Delta_{\text{DEM}}$ for the DEM particles, in which d_{max} is the maximum particle diameter. Consequently, only particles in the neighboring cells contribute to the update of a same phase (Fig. 2(b) and 2(c)). The searching area is shaded for illustration purposes in Fig. 2. The purpose of introducing an extra searching gap Δ is to avoid conducting the particle mapping and searching at every time step. A large value of Δ means more time are needed to conduct neighbor searching for each time. It is thus a balance between the frequency of searching and the time cost of each searching routine. Its value depends on factors like the particle velocities and solid concentration, but normally smaller than the particle size. In this study, it is set as 0.3 times of the particle size. Potential neighbors are then identified by looping through all levels of the searching grid. For SPH particles, the fluid neighbors are detected from Level 2 while its solid neighbors are searched on Level 1. For the DEM particles, the contact detection between solid particles is conducted on Level 1 while the interaction with fluid particles is performed on Level 2. For interactions between phases, the search area depends on the size ratio between the size of the SPH particle support domain and the DEM particle size. To find neighboring DEM particles, the SPH particle is mapped into Level 1, as shown in Fig. 2(e). In this illustrative example, $\lceil 2h/d \rceil = 3$ (where the nomenclature $\lceil x \rceil$ returns the smallest integer larger than x). Searching is therefore conducted by looping through the three surrounding layers of the DEM cells. On the other hand, to find neighboring SPH particles for a given DEM particle, searching is only performed within the surrounding SPH cells and the mapped cell itself as $\lceil d/2h \rceil = 1$, as shown in Fig. 3(d).

2.4.2. Memory management

A unique feature of the DEM calculation is the need to record the contact status between two contacted particles. It is used to determine the friction status, either in static friction or in dynamic friction. If plastic deformation or inter-particle bonding is considered, a large amount of memory is required to keep the contact history information [43]. Additionally, double-precision floating point accuracy is required to minimize numerical errors. Consequently to enable the efficient solution of large-scale problems, it is important to optimize the algorithm to balance memory consumption and computing efficiency. To this end, a neighbor list is only constructed for the DEM particles as each SPH particle can host a large amount of neighbors due to its large size of support domain. Instead, the step of particle mapping for the SPH particles is conducted occasionally while the step of all-level searching is conducted at every time step. The reconstruction of DEM neighbor list and the particle mapping step for SPH particles are triggered when accumulated displacement of any particles exceeds a specified threshold: $\Delta_{\text{DFM}}/2$ for DEM particles and $\Delta_{\text{SPH}}/2$ for SPH particles, respectively.

In this study, both the neighbor list and associated contact history information are saved in the global memory on the GPU. Each time step, neighbor searching and force calculations require frequent access to this data. The memory layout has therefore been optimized to boost the efficiency of data fetching. GPU threads are grouped into warps of 32 threads and each memory operation is issued per warp, meaning that one memory fetch can return a cache line of 128 bytes. Consequently to promote coalesced memory access, and therefore maximize the efficiency of each fetch, the neighbor index and contact history data arrays of each DEM particle are organized in a column-major pattern.

2.4.3. Program flow

The coupled SPH-DEM method was implemented using C++ and Compute Unified Device Architecture (CUDA) developed by NVIDIA. The GPU program was formulated using a single-program multipledata (SPMD) technique, where the same program is executed by multiple threads simultaneously. Due to the discrete nature of the particle methods, GPU threads are assigned to each particle (either DEM or SPH particle). As a result, neighbor searching, force calculation and



Fig. 3. Flow chart of the algorithm of the coupled SPH-DEM method. The steps in the shaded boxes are conducted on the GPU.

time integration of the equation of motion can be carried out independently for each particle using different GPU kernel functions. For efficient use of the GPU memory, parameters that remain the same during simulation, such as material properties, are stored in the constant memory (a type of read-only memory on the GPU with fast data fetching) while other particle-related information, including positions, velocities, forces and contact histories, are stored in the global memory on the GPU.

Fig. 3 shows the flow chart of the algorithm that runs on a single GPU. The whole program can be divided into two major parts: DEM calculation and SPH calculation. For each type of calculation, there are three major components: i) neighbor searching (DEM) or particle mapping (SPH), ii) force computation and iii) time integration of the

equation of motion using an explicit time integration method (forward Euler method). The time step for time integration in SPH is limited by the CFL-condition based on the artificial sound speed and the maximum flow speed and viscous condition [13] while the time step used for DEM calculation is determined based on a Rayleigh wave propagation criteria [59].

The program starts by reading pre-defined initial positions and the associated properties of the SPH particles from files generated from a pre-processing step. The solid particles are subsequently randomly generated and allowed to settle within the required region or the geometry. Once this initialization is complete the DEM and SPH calculations are started. During each step, the DEM calculation are first performed following an order of searching neighbors, updating contact histories, calculating forces and updating the particle velocities and positions. The DEM calculation iterates multiple times until the DEM time is synchronized with the SPH time. The SPH calculation is then started by mapping the fluid particles onto the searching grid, namely assigning the fluid particles into axially-aligned cells, ready for neighbor searching in the following steps. The porosity at the position of each fluid particle is calculated from surrounding solid particles (Eq. (26)). The pressure of fluid particles is updated using the equation of state based on the fluid superficial density (Eq. (7)). Then, the interaction between fluid particles is calculated by solving the continuity (Eq. (6)) and momentum equations (Eq. (9)). The phase coupling is achieved by calculating the fluid force acting on the solid particles (Eq. (27)) and then the reaction force on the fluid particles is calculated using a weighted-average of the forces from the solid particles (Eq. (33)). Finally, the fluid particles' density, velocity and position are updated. These calculations are repeated over the simulation time. It should be noted that the bandwidth between CPU and GPU limits the efficiency of memory transfer. Therefore, we only retrieve simulation data occasionally back to the CPU for data recording. With the exception of the data transfer all the calculations are performed on the GPU by means of issuing a set of CUDA kernel functions.

3. Results and discussion

In this section, validation, application and performance evaluation of the GPU-based models are carried out. The model validation is focused on the SPH model and the coupled SPH-DEM model as the GPU execution of the DEM model has been validated and applied in previous studies of powder flow and compaction [41–43]. Dam break simulations with single and two-phase flow are chosen for this purpose due to their simplicity and wide acceptance as validation tests for free-surface flows. The validated models are then applied to two systems: a novel tubular reactor where the fluid flow is agitated by a perforated tube and a quasi-steady solid-liquid flow in a rotating cylindrical drum. These were selected to examine the capability of the models in handling complex systems encountered in engineering practice.

3.1. Single phase flow

3.1.1. Model validation: single phase dam break

3.1.1.1. Validation and effect of fluid resolution.

The single phase dam break system modelled here has been widely used for SPH method validation [6, 7, 14, 16] by comparison with the experimental results of Koshizuka et al. [60]. The fluid particles are set up initially on a Cartesian lattice, as shown in Fig. 4.

The predicted flow patterns were compared with those from the experiments at a series of time instants, as shown in Fig. 5. Particles are colored by the velocity magnitude. A wedge-shaped water front is generated and moves rapidly towards the right after the sudden removal of the confinement (Fig. 5(a)). The water front starts to deflect and deform once hitting on the vertical wall, a significant amount of water is deflected vertically (Fig. 5(b)) and then falls back due to gravity,



Fig. 4. Particle representation of the dam-break test case. *W* denotes the width of the water column. Blue particles represent the solid wall while red particles represents the fluid. (The side wall and moving gate wall are not shown for clarity.)

generating a plunging surface wave which travels back towards the left side of the tank (Fig. 5(c) and (d)). The flow patterns agree well with those observed experimentally, indicating that the present model is capable of qualitatively capturing the flow behavior in the dam-breaking.

The effect of the fluid particle size on the predicted flow patterns was investigated by comparing the results form 2, 3 and 4 mm fluid particles. The overall flow patterns were very similar however more detailed structure was seen with smaller particles. This is illustrated in Fig. 6, which compares the flow patterns seen at 0.8 s. It can be seen that the size of the void formed under the leading front of the water wave increases with the smaller particle size.

A quantitative estimate of the model accuracy is made by comparing predicted and experimental propagation of the wave front before hitting the right side vertical wall. To this end, two dimensionless numbers are defined: the position of the leading wave front x^* and the characteristic time t^* , which are given as,

$$x^* = x/a \tag{30}$$

$$t^* = t\sqrt{2g/a} \tag{31}$$

x is the position of the wave front in the horizontal direction; *a* is the width of the water column before collapsing; *t* is the physical time and *g* is gravitational acceleration. As shown in Fig. 7, the SPH slightly underpredict the position of the leading front at the initial stage while a better agreement can be seen after $t^* > 1.5$. The fair agreement of the results gives a difference within 5% of the predicted value, demonstrating that the model is able to provide accurate quantitative data for single phase free-surface flows. Fig. 7 also shows the effect of particle resolution on the position of the wave front, with decreasing particle size, the wave front propagates slightly quicker, indicating that a better agreement can be reached when using finer particle resolution.

3.1.1.2. Performance evaluation.

The performance and scalability of the GPU-based SPH method were evaluated by increasing the dam width in the simulation above from 200 mm to 12,800 mm, while other dimensions remain the same. This increased the particle number to 14.142 million in the largest



Fig. 5. Flow pattern during single-phase dam break: experiments [60] (top) and simulation (bottom) at time (a) 0.2 s, (b) 0.4 s, (c) 0.6 s and (d) 0.8 s. Fluid particle size 4 mm.

simulation. The calculation time required for these simulations is shown in Fig 8. A comparison of the same implementation run on two different Nvidia GPU cards, Tesla K80 and P100, is also shown. The mean calculation time per iteration was found to scale linearly with the number of particles ($\mathbb{R}^2 > 0.99$). It shows that more than 10 million particles can be simulated on both cards. The performance of the P100 card is consistently better than that of the K80 card in both the procedures of particle mapping and a calculation cycle. The ratio of the scalability (defined as the slope of the linear fit) between K80 and P100 is 1.83 for the process of particle mapping and 3.19 for a calculation cycle. This is primarily because the P100 card provides 1.6 times more GFLOPS (double precision) and 3 times more memory bandwidth than the K80 card.

3.1.2. Application of SPH: agitated tubular reactor

The validated model was applied to an agitated tubular reactor (ATR), a novel, intensified, reactor used for continuous chemical processing. The ATR system consists of a series of tubes with free-moving internal agitators, which are mounted on a shaking platform. The internal agitators have a cylindrical shape with caps attached to both ends to prevent direct contact with the tube surface, thus reducing milling of the catalyst particles. Compared with conventional mechanically agitated reactors, such as stirred tank reactors, the ATR has no driving shafts and baffles. Instead, the lateral shaking of the system drives the agitator motion. The resulting radial mixing is independent on the axial flow rate. The flexibility to independently control axial flow and

mixing make it well suited for handling slurries, gas/liquid mixture and catalysed reactions [62, 63].

3.1.2.1. Flow system.

In the present study, a periodic section of a single external tube is subjected to a sinusoidal oscillation. The passive internal agitator that drives the flow is a perforated tube with ellipse-shaped surface holes. Fig. 9(a) shows a schematic view of the geometrical and particle representation of the modelled system in SPH.

The collision of the agitator with the external tube is modelled using an approach which is analogous to a soft-sphere collision model used in DEM calculations; the agitator is treated as a single element whose motion is tracked by Newton's second law of motion [64]. Factors determining its motion include gravity and forces due to collision with the external tube. The contact between the agitator and the tube wall only occurs at the end caps, consequently the collision contact diameter is set as the cap size. For simplicity, however, fluid forces are not considered in the present model due to the fact that the motion of the agitator is dominated by the collision between structures. The temporal velocity of the tube is described as a sinusoidal motion, given by,

$$U(t) = 2\pi f A \cos(2\pi f t) \tag{32}$$



with *f* the shaking frequency and *A* the shaking amplitude. As a result, the computational domain is not stationary but constantly changing

Fig. 6. Effect of particle resolution on the flow pattern at time of 0.8 s during single-phase dam break, with different particle size (a) 4 mm, (b) 3 mm and (c) 2 mm.



Fig. 7. Comparison of the propagation of the wave front as a function of characteristic time between SPH results and experiments [61].



Fig. 8. Computational cost of a calculation cycle as a function of the number of particles on different GPU cards. The results are shown in the form of the wall clock time per simulated time step averaged over 1000 timesteps.

with the shaking of the external tube, which would complicate the model implementation and data analysis. In order to enable a stationary domain, simulations are performed in the reference frame of the shaking tube. To this end, an acceleration is imposed to both the agitator and the fluid, given by,

$$a(t) = -4\pi^2 f^2 A \sin(2\pi f t) \tag{33}$$

with a direction opposite to the shaking. Computationally, it is prohibitive to model the whole length of the reactor, a section of ATR system is thus modelled with periodic boundary condition applied in the axial direction. The working fluid is water. Other modelling parameters are summarized in Table 1, which are typical operational parameters. A total physical time of 5s is simulated.

3.1.2.2. Motion of the agitator.

The agitator presents a well-behaved periodic motion due to the sinusoidal oscillation of the reactor tube. The motion of the agitator quickly reaches a stable state after the first two periods. The behaviour of the agitator in a typical period are shown in Fig. 10. From phase 0 to 0.5π , the reactor tube moves toward the right hand side with a decreasing velocity, while the agitator accelerates towards the bottom of the tube followed by a deceleration when it moves upwards. The

Table 1Parameters used in simulation.

External tube	
Amplitude, A (mm)	7.1
Frequency, f (Hz)	4.06
Diameter, D (mm)	25.4
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Internal agitator	
Inner diameter, D _i (mm)	13.8
Outer diameter, D _o (mm)	14.4
Density, ρ_s (kg/m ³)	7800
Cap size, S _{cap} (mm)	1.5
Moment of Inertia, I (kg·m ²)	2.516×10^{-7}
Young's modulus, E (Pa)	$1.0 imes 10^8$
Poisson ratio, ν	0.3
Sliding friction coefficient, μ_{t}	0.3
Restitution coefficient, e	0.6
ent et al.	
Fluid particles	
Fluid density, $\rho_{\rm f}(\rm kg/m^{3})$	1000
Viscosity, μ (kg/m·s)	0.001



Fig. 9. (a) Schematic of the cross-section of the reactor tube in the Coflore ATR1 reactor (courtesy AM Technology), (b) Geometrical representation of the agitator and (c) particle representation of the simulated ATR system, in which the size of the fluid particle is 0.2 mm.



Fig. 10. Evolution of (a) position, (b) translational velocity, (c) velocity magnitude, (d) angular velocity of the agitator in the reference frame of the shaking tube and (e) an illustration of the motion of the agitator in the global reference frame, in which phase $A = 0.581\pi$, $B = 0.906\pi$, $C = 1.150\pi$, $D = 1.393\pi$ and $E = 1.637\pi$.

agitator reaches the highest point around phase **A** shortly after the tube reverses its moving direction at 0.5π . At this stage, the tube continues to move to the left hand side at an increasing speed while the agitator slides down towards the bottom at an increasing speed. The maximum velocity of the agitator is around 0.076 m/s which corresponds to 42% of the maximum shaking velocity (0.181 m/s). After the tube pass the middle point of its moving region (>1.0\pi), the agitator reaches the bottom of the tube at phase **C**. After that, the agitator start to move upward at a decreasing speed.

3.1.2.3. Agitated flow field.

Fig. 11(b) shows the contours of fluid velocity magnitude at the five selected phases marked in Fig. 10(a). Combining the information from Fig. 10(a)-(d), an illustration of the agitator's motion and the position of the shaking tube in the global reference frame are given in Fig. 11 (a). In general, the motion of the fluid is mainly driven by the shaking of the reactor tube as suggested by the rotation of the fluid as a whole. However, local variations can still be seen due to the presence of the agitator. For example, at phase A, local maximum of the fluid particles are found located at the interstice between the agitator and the reactor, similar to that of the phase *E*. This is primarily due to the small relative velocity between the agitator and the reactor tube at these phases. The fluid particles at the contact region are being squeezed out by the agitator. Due to the presence of the surface holes, a repeated pattern of the fluid velocity along the axial direction can be observed at the free surface. The complex dynamics captured in the ATR system shows the strong potential of applying the developed GPU-based SPH in understanding and further optimizing the design and the operational conditions of chemical reactors where the free-surface flows present.

3.2. Particle-fluid flow

3.2.1. Model validation: two-phase dam break

To validate the coupling between SPH and DEM, a two-phase dam break is simulated and is compared with the experimental results reported by Sun et al. [6]. This test case has also been adopted to validate coupled SPH-DEM approaches in other studies [7, 15]. The water tank has overall dimensions: 200 mm \times 150 mm \times 150 mm, and is split into two volumes by a movable gate 50 mm from one end. Water with a depth of 100 mm along with a packed particle bed are blocked by the gate. The dam break is initiated by moving the gate upward at a constant speed of 0.68 m/s. The initial particle configuration of the SPH-DEM simulation is shown in Fig. 12.

For the solid phase, a total mass of 200 g spherical particles are first randomly generated behind the moving gate. Then, they are allowed to settle under gravity until the total kinetic energy essentially vanishes. For the fluid phase, SPH particles with material properties of water are orderly distributed behind the gate. Other modelling parameter can be found in Table 2.

Fig. 13 compares the simulation with the experiments at a time interval of 0.5 s. The SPH particles are colored by volume fraction while the DEM particles are colored by the velocity magnitude. The dam break is initiated by moving the gate upward at a constant velocity of 0.68 m/s. With restriction of the vertical gate, particles are driven by



Fig. 11. Velocity profile in a half-filled ATR system at different phases, in which phase $A = 0.581\pi$, $B = 0.906\pi$, $C = 1.150\pi$, $D = 1.393\pi$ and $E = 1.637\pi$. Fluid particles are colored by velocity magnitude in the reference frame of the tube.

the fluid drag to move along with the flow direction. In comparison, both the fluid and the solid phase observed in experiments are well reproduced, indicating qualitatively comparable simulation results. The minimum fluid volume fraction produced by the simulation is around 0.33 which is close to the packing fraction of random loose packing (0.64). In the simulation, however, the wave front of the water lags the experiment very slightly, this can be seen in Fig. 13d; in the experiment the water has reached the left hand wall, where as in the simulation the water wave has not quite made impact.

Quantitative comparisons of the extent of propagation of the leading front of both the fluid and the solid phases are shown in Fig. 14. The same dimensionless numbers are used as the previous single phase case. It can be seen that the simulation matches well with the experiments. Discrepancies, however, are seen at initial stage of the dam break for the fluid phase (0.015 s to 0.035 s) and at the later stage for



Fig. 12. Particle configuration of the two-phase dam break test case for SPH-DEM, solid particles are not shown. Fluid particles are colored by volume of fraction.

the solid phase (>0.14 s). Several factors may contribute to the discrepancy, including the use of uncalibrated DEM parameters, such as the friction coefficient and the restitution coefficient, as also noted by Markauskas et al. [7], and the lack of the lubrication mechanism in the present simulation which would also lead to an underestimation of the position of the solid front.

3.2.2. Application of coupled SPH-DEM: rotating drum

In mineral and chemical processing, rotating drums are often used for mixing or grinding. Water is added to suppress dust or to modify the operational conditions, leading to a typical slurry flow. In this section, the GPU-based coupling method is applied to model the particle-fluid flow in a rotating cylindrical drum. The setup is the same as reported in the work of Sun et al. [6]. The predicted results are first compared with experiments in terms of the bed shape and dimensions.

Fabl	e 2		

Modelling parameters used in the simulation.

Solid phase	
Number of particles	7762
Density (kg/m ³)	2500
Young's modulus (Pa)	$1.0 imes 10^{8}$
Friction coefficient	0.2
Rolling friction coefficient	0.01
Restitution coefficient	0.9
Time step (s)	$2.5 imes 10^{-6}$
Fluid phase	
Density (kg/m ³)	1000
Viscosity (Pa·s)	8.9×10 ⁻⁴
Fluid resolution (mm)	3.0
Boundary particle separation (mm)	2.1
Smoothing length, h (mm)	3.9
Time step (s)	5×10 ⁻⁶



Fig. 13. Comparison of flow patterns in two-phase dam break at different time: experiments [6] (left), fluid phase (middle) and solid phase (right).



Fig. 14. Normalized leading front as a function of scaled time: comparison between simulation and experiments [6].

Then, the performance of the GPU implementation is evaluated by scaling up the device.

3.2.2.1. Model validation.

In this study, the diameter and the length of the drum are both 100 mm. The drum rotates at a constant speed of 104 rpm, with half of it filled with water. A quasi-steady state of the solid particle bed is reached at this speed. In the simulation, a total of 7755 solid particles with a diameter of 2.7 mm are used, replicating the simulation reported by Sun et al. [6]. The rest of the parameters are the same as summarized in Table 2. Fig. 15(a) shows the initial configuration of the SPH particles. A single layer of wall particles are used. The randomly generated solid particles are allowed to settle down in the drum without the influence of fluid force to form a packed bed. Then the drum starts to rotate. After reaching a steady state, a particle bed with bilinear slope is observed. As shown in Fig. 15(b) and (c), the present simulation is capable of qualitatively reproducing the bilinear slope close to that observed in the experiment.

Quantitative comparisons are made of both the average height and the width of the bed. The dimensions are obtained by averaging over 50 equally spaced sample times over a period of 10 s. The comparisons with experimental results are given in Table 3, together with a comparison to the simulation results of Sun et al. [6]. The present simulation



Comparison of the bed dimensions between simulation and results of Sun et al. [6].

Bed dimensions (standard deviation)	Experiment [6]	Simulation of [6]	Current study
Width, mm	73.41 (1.25)	71.45 (0.79)	73.78
Height, mm	62.18 (1.71)	59.80 (0.61)	(0.708) 56.93 (0.86)

accurately captured the width of the bed, with a difference smaller than 1%. However, the height of the bed is slightly under-predicted with a difference around 8% compared to the experiment. Compared to the simulation of Sun et al. [6], the difference is about 4.8%. This discrepancy seems to arise from differences in contact models and no proper calibration of the parameters. For example, we used the Hertz model for the normal contact force and the Mindlin and Deresiewicz theory for the tangential elastic frictional contact while [6] used a spring-dashpot model.

3.2.2.2. Performance evaluation.

The rotating drum is scaled to evaluate the performance of the coupled SPH-DEM on the Tesla P100 card. Four different diameters of the drum, 100 mm, 200 mm, 300 mm and 400 mm, are tested. The length of the drum is the same as its diameter in all cases. The water and article volume fractions were kept constant across all simulations. The computational time cost for the DEM calculations and the SPH calculations were recorded and are shown in Fig. 16. The elapsed time per step is calculated by averaging over 10,000 steps. It can be seen that the time cost of the SPH part is consistently more than an order of magnitude higher than that of the DEM part. Moreover, the computational cost for both parts increase linearly with increasing number of particles.

4. Conclusion

A GPU-based program has been developed to accelerate the simulations of single phase fluid flow and particle-fluid flow involving freesurfaces, with the goal of an efficient technique for application to multi-phase chemical processes. The free-surface fluid flow is resolved using SPH, while DEM is adopted to track the motion of solid particles if present, thus a unified particle-based modelling framework was established. The coupling between the two phases is achieved using local averaging techniques. A dual-grid neighbor searching method was proposed to handle coupling between phases, due to the difference



Fig. 15. (a) Particle representation of the simulated rotating drum, in which the size of the fluid particle is 3 mm and the typical snapshot of (b) experiment [6] and (b) the present simulation, in which the solid particles are represented by yellow spheres while the fluid particles are represented by blue cubes. The bilinear slope is indicated by the red lines in the figure.



Fig. 16. Comparison of computational time for four drum sizes, presented as a function of (a) the number of DEM particles and (b) the number of SPH particles on the Tesla P100 GPU card.

in the size of influencing area between solid particles and fluid particles. The algorithm and the memory management was specifically designed to suit the GPU platform.

For the single phase flow, a single-phase dam break flow was simulated to validate the SPH model. The model accuracy was evaluated in terms of the flow pattern and evolution of the position of the wave front, all showed good agreement with published experimental results. A sensitivity test indicates more detailed flow structure can be obtained using finer fluid particle resolution. The performance of the SPH algorithm implemented on GPU was evaluated by changing the width of the dam. A linear scalability was obtained between the averaged calculation time per step and the number of particles. The overall performance achieved on the Tesla P100 card was about 2 times faster than that on the Tesla K80 card. The model's ability to predict free-surface flows in complex engineering problems was demonstrated by a novel tubular reactor with a free-moving perforated agitator.

For particle-fluid flow, validation was conducted on a similar example case, this time with particles present. A good agreement with experimental results was obtained for the temporal evolution of both the solid particle bed and the fluid wave front. A typical application to chemical process was demonstrated by a simulation of a quasi-steady particle-fluid flow in a rotating drum. The obtained results have shown satisfactory predictions in terms of bed shape and bed dimensions. The performance of the coupled SPH-DEM model was evaluated by scaling up the size of the rotating drum. The SPH related calculations consumed more time than those of the DEM. The computational time for both the DEM and the SPH particle scales linearly with the particle number.

With the reliable results and acceptable scalability obtained from the GPU platform, the coupled SPH-DEM model has demonstrated its potential to contribute to an improved understanding of the practical problems encountered in chemical processing. Future work on extending the implementation to multiple GPUs by means of Message Passing Interface is on course to further enhance the computational capabilities.

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