## PHYSICAL REVIEW E 91, 043010 (2015)

# Reduced description of exact coherent states in parallel shear flows

## Cédric Beaume\*

Department of Aeronautics, Imperial College London, London SW7 2AZ, UK

# Gregory P. Chini<sup>†</sup>

Department of Mechanical Engineering and Program in Integrated Applied Mathematics, University of New Hampshire, Durham, New Hampshire 03824, USA

## Keith Julien<sup>‡</sup>

Department of Applied Mathematics, University of Colorado at Boulder, Boulder, Colorado 80309, USA

# Edgar Knobloch§

Department of Physics, University of California, Berkeley, California 94720, USA (Received 20 February 2015; published 15 April 2015)

A reduced description of exact coherent structures in the transition regime of plane parallel shear flows is developed, based on the Reynolds number scaling of streamwise-averaged (mean) and streamwise-varying (fluctuation) velocities observed in numerical simulations. The resulting system is characterized by an effective unit Reynolds number mean equation coupled to linear equations for the fluctuations, regularized by formally higher-order diffusion. Stationary coherent states are computed by solving the resulting equations simultaneously using a robust numerical algorithm developed for this purpose. The algorithm determines self-consistently the amplitude of the fluctuations for which the associated mean flow is just such that the fluctuations neither grow nor decay. The procedure is used to compute exact coherent states of a flow introduced by Drazin and Reid [Hydrodynamic Stability (Cambridge University Press, Cambridge, UK, 1981)] and studied by Waleffe [Phys. Fluids 9, 883 (1997)]: a linearly stable, plane parallel shear flow confined between stationary stress-free walls and driven by a sinusoidal body force. Numerical continuation of the lower-branch states to lower Reynolds numbers reveals the presence of a saddle node; the saddle node allows access to upper-branch states that are, like the lower-branch states, self-consistently described by the reduced equations. Both lower- and upper-branch states are characterized in detail.

DOI: 10.1103/PhysRevE.91.043010 PACS number(s): 47.10.Fg, 47.11.St, 47.27.ed

## I. INTRODUCTION

Exact, fully nonlinear, three-dimensional (3D) solutions of the Navier-Stokes equations play an important role in our understanding of the transition to turbulence in parallel shear flows and of the recurrence properties of the turbulence that results. These solutions, first computed by Nagata [1] and Waleffe [2] and now called exact coherent states (ECS), may take the form of time-independent states (i.e., equilibria) or time-periodic states (e.g., traveling waves). ECS have now been computed by numerous investigators for a number of different flows, including plane Couette flow [3] and pipe flow [4-7]. Typically, these solutions consist of streamwiseoriented streaks and vortices that bear a striking qualitative and even quantitative resemblance to the coherent structures commonly observed in turbulent wall flows, although they are generally unstable. In fact, despite their instability, ECS are frequently observed as transients in both shear flow simulations and experiments. Analysis of a low-order model by Waleffe [2] and a more systematic numerical study by Schmiegel [8] reveal that the ECS in plane Couette flow

(PCF) are born in saddle-node bifurcations as the Reynolds number Re increases, and continue as upper- and lower-branch solutions; this ECS bifurcation scenario seems generic in that it is commonly found in other shear flows. Much of the interest in ECS can be attributed to the possibility that upper-branch solutions comprise the "skeleton" of a high-dimensional "turbulent" attractor in the shear-flow phase space. Both Waleffe [9] and Kawahara and Kida [10] demonstrate that certain low-order statistics of PCF turbulence, particularly the mean and root-mean-square (rms) velocity profiles, can be accurately reproduced using these unstable solutions, quantitatively attesting to their physical relevance.

In two particularly insightful papers, Waleffe [2,9] identifies a fully nonlinear process, involving the interaction of streamwise-oriented streaks and rolls, that sustains lower-branch ECS in plane parallel shear flows (including PCF), following earlier work by Hall and Smith [11] on the closely related phenomenon of vortex-wave interaction. Certain lower-branch solutions have the remarkable property that they have only a *single* unstable eigendirection. These coherent states appear to separate, in phase space, disturbances that decay, causing relaminarization of the flow, from those that follow an excursion toward a turbulent (or at least transiently chaotic) state. For this reason, investigation of these "edge states" and their stability is of great interest, offering tantalizing opportunities for flow control. Importantly, the recent discovery by Schneider *et al.* [12] of spatially localized edge states in PCF

<sup>\*</sup>ced.beaume@gmail.com

<sup>†</sup>greg.chini@unh.edu

<sup>‡</sup>keith.julien@colorado.edu

<sup>§</sup>knobloch@berkeley.edu

has served to further increase interest in ECS by establishing that they are not special solutions found only in small laterally periodic domains but, rather, that they exist in unbounded flows and may thus play a role in transition in open shear flows.

In general, the extraction of both lower- and upper-branch ECS in PCF, as in other shear flows, has required substantial computational effort. In particular, because ECS are typically disconnected from the structureless base shear flow, standard methods based on linear theory cannot be used to identify them, and more sophisticated algorithms (or considerable ingenuity and physical intuition [2,9]) are required. These challenges are exacerbated for flows at large Reynolds numbers and in large domains. To alleviate these challenges, we propose in this paper a reduced description of ECS in the transition regime. Our approach, like the related approach of Hall and Sherwin [13] and Blackburn et al. [14], is based on a decomposition of the flow into streamwise-invariant (i.e., mean) and streamwise-varying (i.e., fluctuation, or "wave") components and subsequent exploitation of the properties of these fields as observed in numerical simulations. Specifically, we use the fact that along the lower ECS branch streamwise-invariant streaks are O(1) as Re  $\rightarrow \infty$  while the streamwise-invariant rolls scale as  $O(Re^{-1})$ , as originally suggested in [15] and confirmed by Wang et al. [16] for PCF. Fundamental streamwise-varying modes also scale roughly as  $O(Re^{-1})$  (Wang et al. [16] cite an exponent of approximately -0.9), but higher harmonics are found to be  $o(Re^{-1})$  (see Table I).

In a parallel development Hall and Sherwin [13] used the critical layer scalings suggested by Hall and Smith [11] to develop an asymptotic analysis valid in the limit  $Re \to \infty$ . The resulting theory leads to a two-dimensional (2D) system for the streamwise-averaged fields at unit rescaled Reynolds number coupled to a quasilinear inviscid eigenvalue problem for neutral disturbances to a mean streaky streamwise (xdirected) flow  $u_0(y,z)$ , where y and z are wall-normal and spanwise coordinates, respectively. This eigenvalue problem is singular, possessing a nonplanar critical layer at  $u_0(y,z) = 0$ . The authors perform a careful matched asymptotic analysis to incorporate a viscously regularized critical layer, ultimately deriving jump conditions across the layer that link the mean fields on either side. In addition to reducing the computational cost of numerically solving for the lower-branch ECS at large Re, their analysis clearly demonstrates the physical mechanism by which the fluctuations sustain the mean fields in this limit: namely, steady streaming is driven within the critical

TABLE I. Summary of the scalings obtained by Wang *et al.* [16] for lower-branch ECS in plane Couette flow. These authors decompose the ECS into streamwise Fourier modes:  $\mathbf{u}(x,y,z,t) = y\hat{\mathbf{x}} + \sum_{n=-N/2}^{N/2} \mathbf{u_n}(y,z)e^{in\theta} + \text{c.c.}$ , where *n* denotes the index of the Fourier mode  $\mathbf{u_n}$ , *N* is the number of Fourier modes retained, and  $\theta = \alpha(x-ct)$ . Here,  $\alpha$  is the fundamental streamwise wave number, *c* is the speed of the wave (in the case of a traveling wave solution), and c.c. denotes the complex conjugate. The last column shows that higher harmonics (n > 3) decay faster than the primary ones.

$$\frac{\text{Mode} \quad u_0 \quad (v_0, w_0) \ (u_1, v_1, w_1) \ (u_2, v_2, w_2) \ (u_3, v_3, w_3) \ (u_n, v_n, w_n)}{\text{Scaling} \ O(1) \ O(\text{Re}^{-1}) \ O(\text{Re}^{-0.9}) \ O(\text{Re}^{-1.6}) \ O(\text{Re}^{-2.2}) \ o(\text{Re}^{-2.2})}$$

layer, which in turn drives the mean flow outside the layer. Despite this, Hall and Sherwin [13] are obliged to employ a sophisticated high-order domain-decomposition numerical scheme to solve their equations, which requires the numerical grid to be adaptively updated since the location of the critical layer is not known *a priori*.

For all its merit, the asymptotic analysis of Hall and Sherwin [13] generates solutions whose dynamical significance remains to be established since at extreme values of the Reynolds number the flow is likely to be fully turbulent. In contrast, our primary interest is in transitional and low Re turbulent flows, for which the dynamical relevance of the ECS is, in fact, clear [3]. At these more moderate values of the Reynolds number (yet still large compared to unity), the critical layer broadens into a critical region. Consequently, we seek a uniform approximation over the entire domain. To obtain a uniformly valid system, we retain formally small dissipation terms in the fluctuation equations, as described by Beaume [17] and subsequently employed by Blackburn et al. [14]. This procedure obviates the need for explicit introduction and subsequent smoothing of jump conditions and for any further regularization of the fluctuation equations. Since jump conditions are not imposed, there is no need for adaptive mesh refinement associated with dynamic tracking of the critical layer. Of course, for very large Re, sufficiently many modes or grid points must still be used to resolve the inevitable sharp gradient regions that arise. This issue is mitigated, however, in the transition regime.

Since we are interested in equilibrium ECS we choose a simple body-forced parallel shear flow first proposed by Drazin and Reid [18] and later studied by Waleffe [2], which we refer to as "Waleffe flow" (hereafter WF, see Sec. II). This flow is a close relative of PCF, being linearly stable for all Re, implying a likely connection between the ECS we compute and those found by Nagata [1], Clever and Busse [19], and later continued via homotopy by Waleffe [20] to other flows. Our reasons for studying this flow instead of plane Couette flow are threefold: (a) the properties of the ECS in this flow have not been studied, (b) trigonometric basis functions may be employed in both the wall-normal and spanwise directions to provide a higher mesh density within the critical region, and (c) it provides an excellent example of a nontrivial configuration on which the new algorithm can be tested. In particular, we employ and, as necessary, refine an equispaced grid in both coordinate directions, and as a result are able to identify structures not only in the direction normal to the broadened critical layer, but also along it. Finally, and somewhat remarkably, we demonstrate in Sec. IV that our asymptotically reduced partial differential equation (PDE) model admits both lower-branch and upper-branch solutions [21]: in spite of the large Reynolds number scaling incorporated into the theory, the approach proves sufficiently robust to capture the saddle-node bifurcation at which the lower- and upper-branch ECS are born. Thus, our reduced PDEs should prove useful for a variety of further studies of parallel shear flows that aim, for example, to investigate streamwise and spanwise localization.

The remainder of this paper is organized as follows. In Sec. II we describe in detail the multiscale averaging approach we use to obtain our reduced description. Section III outlines

the customized numerical algorithm we develop to solve the reduced equations, while Sec. IV summarizes the results obtained. The paper ends with brief conclusions in Sec. V.

## II. MULTISCALE SYSTEM

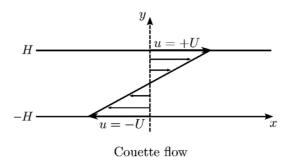
Incompressible channel flow driven by a volume force  $\mathbf{f}(y)$  is governed by the nondimensional Navier-Stokes equation

$$\partial_t \mathbf{v} + (\mathbf{v} \cdot \nabla) \mathbf{v} = -\nabla p + \frac{1}{\text{Re}} \nabla^2 \mathbf{v} + \mathbf{f}(y),$$
 (1)

along with the incompressibility constraint

$$\nabla \cdot \mathbf{v} = 0. \tag{2}$$

Here and throughout, a Cartesian coordinate system is adopted in which x, y, and z are the dimensionless streamwise, wallnormal, and spanwise directions, respectively. The velocity vector v has dimensionless components (u, v, w), and p is the dimensionless fluid pressure. In (1), all lengths have been scaled by H, i.e., half the (dimensional) distance separating the plane parallel walls, and all velocities by a characteristic velocity U. In PCF, U is the dimensional speed of the upper wall, the flow being driven by in-plane but opposing motion of the no-slip boundaries; in this case, the body force vanishes:  $\mathbf{f}(y) = \mathbf{0}$ . This configuration admits a structureless laminar solution, namely Couette flow, as depicted in Fig. 1 (top panel). As is well known, this solution is linearly stable even for asymptotically large values of the Reynolds number Re = UH/v [22,23], although stability is observed experimentally only for Re < Re<sub>u</sub>  $\approx$  310 [24–26]. Careful parameter studies



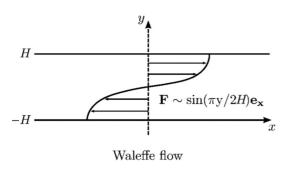


FIG. 1. Sketch of plane Couette and Waleffe flows. Plane Couette flow is driven by wall motion in the x direction, the top and bottom walls moving with opposite velocities  $\pm U$ . Plane Waleffe flow is driven by an x-directed body force. The forcing profile is a halfperiod of a sinusoid in y. Note that the laminar flow is stable despite the presence of an inflection point owing to the proximity of the (stress-free) walls.

have revealed, for larger Reynolds numbers, a variety of structured flow regimes [27]: in  $\text{Re}_u < \text{Re} < \text{Re}_g \approx 325$ , perturbations to Couette flow evolve into evanescent turbulent spots before Couette flow is restored. The lifetime of the transient spots diverges as  $\text{Re}_g$  is approached [28], indicating the onset of sustained turbulence above  $\text{Re}_g$ , where most of the turbulent spots survive and organize themselves into turbulent bands oblique to the streamwise direction [29,30]. As Re is increased further, turbulence progressively invades the domain until  $R_t \approx 415$ , where space-filling turbulence is observed.

We focus here on a close relative of PCF, namely Waleffe flow (WF), depicted in Fig. 1 (bottom panel). This flow is driven by an x-directed body force that varies sinusoidally in the wall-normal direction, viz.,  $\mathbf{f}(y) = \frac{\sqrt{2}\pi^2}{4\text{Re}} \sin(\frac{\pi y}{2})\hat{\mathbf{x}}$ , where  $\hat{\mathbf{x}}$  is a unit vector in the *x* direction. Moreover, stress-free rather than no-slip conditions are imposed along stationary boundaries located at  $y = \pm 1$ . This flow was suggested by Waleffe [2] as an alternative to PCF that is more convenient for low-order modeling. Indeed, WF can be naturally expanded in Fourier modes in all three coordinate directions, and the laminar basic state  $\mathbf{v} = [\sqrt{2}\sin(\pi y/2), 0, 0]$  is itself a low-order mode in this basis. Note that in WF the velocity scale U is the root-mean-square velocity of the corresponding dimensional laminar base flow. Although this base flow has an inflection point, it is nevertheless linearly stable for all Re [18]. This is a consequence of the blocking effect of the walls at  $y = \pm 1$ ; in contrast, the related Kolmogorov flow [31] is defined with periodic boundary conditions in y, thereby eliminating the stabilizing effect of the walls and permitting linear instability on large scales [32–34]. However, WF does admit finite amplitude solutions supported by the selfsustaining process identified by Waleffe [2]. These ECS also cannot bifurcate from the base flow, but instead appear through saddle-node bifurcations as Re increases, much as in PCF.

# A. Multiscale analysis

In this section we derive the basic equations used in this paper. The procedure is motivated by earlier work on flows with strong restraints [35], particularly Langmuir circulation [36]. In these flows, the strong restraining force reduces the effective dimensionality of the system, leading to a simplified description.

We begin by decomposing the velocity  $\mathbf{v}$  into a streamwise component u and the perpendicular components  $\mathbf{v}_{\perp} = (v, w)$ , i.e.,  $\mathbf{v} \equiv (u, \mathbf{v}_{\perp})$ . Thus,

$$\partial_t u + u \partial_x u + (\mathbf{v}_\perp \cdot \nabla_\perp) u$$

$$= -\partial_x p + \frac{1}{\text{Re}} (\partial_x^2 + \nabla_\perp^2) u + \frac{\sqrt{2}\pi^2}{4 \text{Re}} \sin\left(\frac{\pi y}{2}\right), \quad (3)$$

$$\partial_t \mathbf{v}_{\perp} + u \partial_x \mathbf{v}_{\perp} + (\mathbf{v}_{\perp} \cdot \nabla_{\perp}) \mathbf{v}_{\perp} = -\nabla_{\perp} p + \frac{1}{\text{Re}} (\partial_x^2 + \nabla_{\perp}^2) \mathbf{v}_{\perp},$$
(4)

$$\partial_x u + \nabla_\perp \cdot \mathbf{v}_\perp = 0. \tag{5}$$

These equations are supplemented with the following boundary conditions along the walls at  $y = \pm 1$ :

$$\partial_{y}u = v = \partial_{y}w = 0. (6)$$

In addition, all fields are taken to be periodic in the streamwise and spanwise directions, with periods  $L_x$  and  $L_z$ , respectively.

To allow for slow modulation in the streamwise direction, we explicitly introduce a slow streamwise coordinate  $X \equiv \epsilon x$  and an associated slow time scale  $T \equiv \epsilon t$ , where the small parameter  $\epsilon \equiv 1/\text{Re}$ . Thus, all field variables are interpreted as functions of (x, X, y, z, t, T). Replacing the original x and t derivatives by  $\partial_x + \epsilon \partial_X$  and  $\partial_t + \epsilon \partial_T$ , respectively, Eqs. (3)–(5) become

$$[\partial_t + \epsilon \partial_T] u + [\partial_x + \epsilon \partial_X] u^2 + \nabla_\perp \cdot (\mathbf{v}_\perp u)$$

$$= -[\partial_x + \epsilon \partial_X] p + \epsilon \frac{\sqrt{2}\pi^2}{4} \sin\left(\frac{\pi y}{2}\right)$$

$$+ \epsilon \left[\partial_x^2 + 2\epsilon \partial_X \partial_x + \epsilon^2 \partial_X^2 + \nabla_\perp^2\right] u, \tag{7}$$

$$[\partial_t + \epsilon \partial_T] \mathbf{v}_{\perp} + [\partial_x + \epsilon \partial_X] (u \mathbf{v}_{\perp}) + \nabla_{\perp} \cdot (\mathbf{v}_{\perp} \mathbf{v}_{\perp})$$
  
=  $-\nabla_{\perp} p + \epsilon [\partial_x^2 + 2\epsilon \partial_X \partial_x + \epsilon^2 \partial_X^2 + \nabla_{\perp}^2] \mathbf{v}_{\perp}, \quad (8)$ 

$$[\partial_x + \epsilon \partial_X] u + \nabla_\perp \cdot \mathbf{v}_\perp = 0. \tag{9}$$

Next, we decompose all variables into a fast (x,t) average plus a fluctuation with zero mean. For example, for the x velocity component, we write

$$u(x, X, y, z, t, T) = \bar{u}(X, y, z, T) + u'(x, X, y, z, t, T),$$

where the overbar denotes the fast (x,t) average. The motivation for averaging only in x rather than over the entire horizontal (x,z) plane is that nontrivial spanwise structure of the streamwise-averaged streamwise velocity, associated with the streamwise streaks, plays a crucial role in the process that sustains the streamwise rolls. With this decomposition, the mean equations can be expressed as

$$\epsilon \partial_{T} \bar{u} + \epsilon \partial_{X} [\bar{u}\bar{u} + \overline{u'u'}] + \nabla_{\perp} \cdot [\bar{\mathbf{v}}_{\perp}\bar{u} + \overline{\mathbf{v}'_{\perp}u'}] 
= -\epsilon \partial_{X} \bar{p} + \epsilon \nabla_{\perp}^{2} \bar{u} + \epsilon^{3} \partial_{X}^{2} \bar{u} 
+ \epsilon \frac{\sqrt{2}\pi^{2}}{4} \sin\left(\frac{\pi y}{2}\right),$$

$$\epsilon \partial_{T} \bar{\mathbf{v}}_{\perp} + \epsilon \partial_{X} [\bar{u}\bar{\mathbf{v}}_{\perp} + \overline{u'}\underline{\mathbf{v}'_{\perp}}] + \nabla_{\perp} \cdot [\bar{\mathbf{v}}_{\perp}\bar{\mathbf{v}}_{\perp} + \overline{\mathbf{v}'_{\perp}}\underline{\mathbf{v}'_{\perp}}] 
= -\nabla_{\perp} \bar{p} + \epsilon \nabla_{\perp}^{2} \bar{\mathbf{v}}_{\perp} + \epsilon^{3} \partial_{X}^{2} \bar{\mathbf{v}}_{\perp},$$
(11)

$$\epsilon \partial_X \bar{u} + \nabla_{\perp} \cdot \bar{\mathbf{v}}_{\perp} = 0. \tag{12}$$

We proceed by positing an appropriate expansion for the various fields. To this end, we are motivated in part by the scaling behavior identified in [16] and recalled in Sec. I for lower-branch ECS although we emphasize that the resulting reduced system is not limited to lower-branch states. For large Re the rolls comprising the streamwise-averaged flow in the perpendicular plane are weak, O(1/Re), relative to the deviation of the streamwise-averaged streamwise flow from the base laminar profile (i.e., the streaks). A *closed*, self-

consistent reduced model may be obtained by further positing that the perpendicular (streamwise-varying) fluctuations are similarly weak relative to the mean streamwise flow, an assumption that is consistent with the scalings reported in Table I. Thus, we expand the velocity components and pressure as follows:

$$u \sim (\bar{u}_0 + u_0') + \epsilon(\bar{u}_1 + u_1') + \dots,$$
 (13)

$$\mathbf{v}_{\perp} \sim \epsilon(\bar{\mathbf{v}}_{1\perp} + \mathbf{v}'_{1\perp}) + \epsilon^2(\bar{\mathbf{v}}_{2\perp} + \mathbf{v}'_{2\perp}) + \dots, \tag{14}$$

$$p \sim (\bar{p}_0 + p'_0) + \epsilon(\bar{p}_1 + p'_1) + \epsilon^2(\bar{p}_2 + p'_2) + \dots$$
 (15)

At O(1), Eqs. (7)–(9) imply

$$\partial_t u_0' + (\bar{u}_0 + u_0') \partial_x u_0' = -\partial_x p_0', \tag{16}$$

$$0 = -\nabla_{\perp}(\bar{p}_0 + p_0'),\tag{17}$$

$$\partial_x u_0' = 0. (18)$$

From this equation set we conclude that  $u_0' \equiv 0$  and  $p_0' \equiv 0$ . Note that  $\bar{p}_0$ , if nonzero, can only depend on X and T; this term is set to zero for PCF and WF, but may be retained for flows driven by externally imposed mean pressure gradients (such as plane Poiseuille flow).

Using these leading-order results, Eq. (7) yields at  $O(\epsilon)$ 

$$\partial_t u_1' + \partial_T \bar{u}_0 + \bar{u}_0 \partial_x u_1' + \bar{u}_0 \partial_x \bar{u}_0 + [(\bar{\mathbf{v}}_{1\perp} + \mathbf{v}_{1\perp}') \cdot \nabla_\perp] \bar{u}_0$$

$$= -\partial_x p_1' - \partial_x \bar{p}_0 + \nabla_\perp^2 \bar{u}_0 + \frac{\sqrt{2}\pi^2}{4} \sin\left(\frac{\pi y}{2}\right). \tag{19}$$

Averaging this equation over the fast x and t variables, we obtain

$$\partial_T \bar{u}_0 + \bar{u}_0 \partial_X \bar{u}_0 + (\bar{\mathbf{v}}_{1\perp} \cdot \mathbf{\nabla}_{\perp}) \bar{u}_0$$

$$= -\partial_X \bar{p}_0 + \nabla_{\perp}^2 \bar{u}_0 + \frac{\sqrt{2}\pi^2}{4} \sin\left(\frac{\pi y}{2}\right). \tag{20}$$

Subtracting Eq. (20) from Eq. (19) yields an equation for the streamwise fluctuating velocity  $u'_1$ :

$$\partial_t u_1' + \bar{u}_0 \partial_x u_1' + (\mathbf{v}_{1\perp}' \cdot \nabla_\perp) \bar{u}_0 = -\partial_x p_1'.$$
 (21)

At  $O(\epsilon)$ , the perpendicular momentum equation (8) takes the form

$$\partial_t \mathbf{v}'_{1\perp} + \bar{u}_0 \partial_x \mathbf{v}'_{1\perp} = -\nabla_{\perp} (\bar{p}_1 + p'_1),$$
 (22)

from which we conclude that

$$\nabla_{\perp} \bar{p}_1 = 0 \tag{23}$$

and

$$\partial_t \mathbf{v}'_{1\perp} + \bar{u}_0 \partial_x \mathbf{v}'_{1\perp} = -\nabla_\perp p'_1. \tag{24}$$

Finally, the  $O(\epsilon)$  continuity equation requires

$$\partial_X \bar{u}_0 + \nabla_\perp \cdot \bar{\mathbf{v}}_{1\perp} = 0 \tag{25}$$

and

$$\partial_x u_1' + \nabla_\perp \cdot \mathbf{v}_{1\perp}' = 0. \tag{26}$$

To obtain a closed reduced system, we average the  $O(\epsilon^2)$  perpendicular momentum equation

$$\partial_{t} \mathbf{v}'_{2\perp} + \partial_{T} (\bar{\mathbf{v}}_{1\perp} + \mathbf{v}'_{1\perp}) + \bar{u}_{0} \partial_{x} \mathbf{v}'_{2\perp} + (\bar{u}_{1} + u'_{1}) \partial_{x} \mathbf{v}'_{1\perp}$$

$$+ \bar{u}_{0} \partial_{X} (\bar{\mathbf{v}}_{1\perp} + \mathbf{v}'_{1\perp})$$

$$+ [(\bar{\mathbf{v}}_{1\perp} + \mathbf{v}'_{1\perp}) \cdot \nabla_{\perp}] (\bar{\mathbf{v}}_{1\perp} + \mathbf{v}'_{1\perp}) = -\nabla_{\perp} (\bar{p}_{2} + p'_{2})$$

$$+ (\partial_{x}^{2} + \nabla_{\perp}^{2}) (\bar{\mathbf{v}}_{1\perp} + \mathbf{v}'_{1\perp}),$$

$$(27)$$

and obtain, on using Eq. (26), the following equation for the evolution of  $\bar{\mathbf{v}}_{1\perp}$ :

$$\partial_T \bar{\mathbf{v}}_{1\perp} + \partial_X [\bar{u}_0 \bar{\mathbf{v}}_{1\perp}] + \nabla_{\perp} \cdot [\bar{\mathbf{v}}_{1\perp} \bar{\mathbf{v}}_{1\perp} + \overline{\mathbf{v}}_{1\perp}' \mathbf{v}_{1\perp}']$$

$$= -\nabla_{\perp} \bar{p}_2 + \nabla_{\perp}^2 \bar{\mathbf{v}}_{1\perp}. \tag{28}$$

The fluctuation equations (21) and (24) are linear and nondissipative and hence the steady version of these equations exhibits a critical-layer singularity. For this reason, these equations require an appropriate regularization before the above equation set can be used to compute ECS. This procedure is discussed next.

## B. Structure and regularization of the reduced model

For ease of reference, we collect here the key results of the analysis. Specifically, the multiscale reduced model consists of Eq. (20) with  $\bar{p}_0 \equiv 0$ , as appropriate for both PCF and WF, and Eqs. (28) and (25),

$$\partial_T \bar{u}_0 + \bar{u}_0 \partial_X \bar{u}_0 + (\bar{\mathbf{v}}_{1\perp} \cdot \nabla_\perp) \bar{u}_0 = \nabla_\perp^2 \bar{u}_0 + \frac{\sqrt{2}\pi^2}{4} \sin\left(\frac{\pi y}{2}\right),\tag{29}$$

$$\partial_T \bar{\mathbf{v}}_{1\perp} + \partial_X [\bar{u}_0 \bar{\mathbf{v}}_{1\perp}] + \nabla_{\perp} \cdot [\bar{\mathbf{v}}_{1\perp} \bar{\mathbf{v}}_{1\perp} + \overline{\mathbf{v}}_{1\perp}' \mathbf{v}_{1\perp}']$$

$$= -\nabla_{\perp} \bar{p}_2 + \nabla_{\perp}^2 \bar{\mathbf{v}}_{1\perp}, \tag{30}$$

$$\partial_X \bar{u}_0 + \nabla_\perp \cdot \bar{\mathbf{v}}_{1\perp} = 0, \tag{31}$$

which govern the mean (i.e., fast x and t averaged) dynamics, and Eqs. (21), (24), and (26), appropriately regularized,

$$\partial_t u_1' + \bar{u}_0 \partial_x u_1' + (\mathbf{v}_{1\perp}' \cdot \nabla_\perp) \bar{u}_0 = -\partial_x p_1' + \epsilon \nabla_\perp^2 u_1', \quad (32)$$

$$\partial_t \mathbf{v}'_{1\perp} + \bar{u}_0 \partial_x \mathbf{v}'_{1\perp} = -\nabla_\perp p'_1 + \epsilon \nabla^2_\perp \mathbf{v}'_{1\perp}, \tag{33}$$

$$\partial_x u_1' + \nabla_\perp \cdot \mathbf{v}_{1\perp}' = 0 \tag{34}$$

for the fluctuating fields. These equations are to be solved subject to the mean and fluctuating boundary conditions obtained by applying the mean and fluctuation decomposition to the conditions (6), viz.,

$$\partial_{y}\bar{u}_{0} = \partial_{y}u'_{1} = \bar{v}_{1} = \partial_{y}\bar{w}_{1} = v'_{1} = \partial_{y}w'_{1} = 0$$
 (35)

on  $y = \pm 1$ .

Physically, the averaged equations constrain the slow temporal and streamwise evolution of the streaks  $(\bar{u}_0)$  and rolls  $(\bar{\mathbf{v}}_{1\perp})$ . The presence of an effective Reynolds number equal to unity [the prefactor in front of the Laplacian in Eqs. (29) and (30) equals one], together with the absence of fast streamwise and temporal variation, suggests that these equations should be more computationally tractable than the full Navier-Stokes equations at large Re. Indeed, if the slow

streamwise (X) variation is suppressed, the averaged equations are spatially 2D and may be expected to exhibit quasilaminar behavior. Thus, deviations from the base laminar flow, if nonzero, are driven solely by the fluctuation-induced Reynolds stress divergence in Eq. (30); this correlation involves only the perpendicular fluctuating velocity field, all other Reynolds stress components being smaller than the retained mean terms.

Presuming fluctuation gradients remain O(1), the fluctuating fields themselves evolve in accord with the equations governing the inviscid stability of streamwise streaks [under the consistent approximation that the O(1/Re) rolls may be neglected]. In particular, spanwise inflections in the profile of  $\bar{u}_0(X, y, z, T)$  may be expected to give rise to an x-varying 3D instability whose primary effect will be to reenergize the streamwise rolls through the Reynolds stress term, in accord with the self-sustaining process of Waleffe [2]. As explicitly demonstrated in [16], the fluctuation (or wave) fields, which are necessarily steady (neutral) for equilibrium ECS, exhibit a critical layer structure along the isosurface  $\bar{u}_0(y,z) = 0$ . In the neighborhood of the critical layer, the fluctuation gradients are large, resulting in a distinct leading-order dominant balance involving diffusion. In Eqs. (32) and (33), this balance is captured, as in [17], by retaining the formally small diffusion terms  $\epsilon \nabla_{\perp}^2 u_1'$  and  $\epsilon \nabla_{\perp}^2 \mathbf{v}_{1\perp}'$ , respectively, where  $\epsilon \equiv 1/\text{Re}$ . From the point of view of asymptotics the retention of these terms may be justified by appeal to the method of composite asymptotic approximations or to the related method of composite asymptotic equations [37], but in our reduced model these terms are retained because they capture correctly the broadening of the critical layer in the transition region that is of interest here. A similar regularization was employed in recent work on high Reynolds number Couette flow by Blackburn et al. [14] in order to avoid the intricacies associated with carrying out a systematic matched asymptotic analysis valid in the limit Re  $\rightarrow \infty$  [13].

It is important to compare our approach with that of Hall and Sherwin [13] and Blackburn et al. [14] in more detail. The analysis of Hall and Sherwin [13] demonstrates that in the limit  $Re \rightarrow \infty$  the amplitude of the fluctuating fields in the Nagata-Busse-Clever-Waleffe lower-branch PCF equilibrium solution scales as  $Re^{-7/6}$ , not  $Re^{-1}$ , away from the critical layer. Within the critical layer, the fluctuation velocity components tangent to the critical layer are amplified, becoming  $O(Re^{-5/6})$ , again rather than  $O(Re^{-1})$  as prescribed here. Thus, in the limit Re  $\rightarrow \infty$ , the fluctuation-induced forcing of the rolls is asymptotically confined to the critical layer, justifying the jump condition formulation derived in [13]. In contrast, our reduced equations are not optimized to capture the properties of ECS in the limit Re  $\rightarrow \infty$ . Instead, the equations have been developed for transition values of the Reynolds number where no singular critical layer is present and one can confirm that all of the terms required to capture the physics of this region are retained. Since our procedure leads ultimately to the retention of the same terms as that of Hall and Sherwin [13] and Blackburn et al. [14], it follows that our fluctuations will grow near the critical layer and decay away from it with increasing Re when Re is very large. This is not an issue since we apply our equations in the transition regime where ECS are known to be dynamically relevant [16]. In this regime, the results of Wang *et al.* [16] indicate that the scale separation between the magnitude of the fluctuation and roll fields and the streak field, on which the reduced model is founded, is already evident. Moreover, at these moderate values of Re [e.g., Re =  $O(10^3)$ ], the quantity Re<sup>-1/3</sup> is not particularly small, and the distinction between the fluctuation amplitude within and outside the critical layer becomes blurred. Of course, there may also be other, perhaps nonequilibrium (e.g., a periodic-orbit), ECS that do *not* exhibit critical layer structure at all but that can nevertheless be captured by our self-consistent reduced model.

It is significant that the fluctuation equations (32)–(34) do not mix x modes, a fact we exploit in our computations of ECS for WF using the reduced system. Specifically and in accord with the scalings given in Table I, we retain only the fundamental streamwise Fourier mode for each fluctuation field, and write

$$u'_1(x, y, z, t) = \hat{u}_1(y, z, t)e^{i\alpha x} + \text{c.c.},$$
 (36)

where  $\alpha \equiv 2\pi/L_x$  is the dimensionless fundamental streamwise wave number and c.c. denotes the complex conjugate; similar expressions are written for  $v_1'$ ,  $w_1'$ , and  $p_1'$ . In the following, we drop the hat over fluctuating variables for brevity of notation. In very long domains, a nearly continuous band of modes with similar streamwise wave numbers will be neutral or very weakly damped, leading to a description of the flow in terms of an evolving linear superposition of these modes exhibiting a slowly varying envelope. This evolution will in turn drive slow streamwise modulations of the mean fields through the Reynolds stress divergence term in Eq. (30). A mechanism of this type may provide an explanation for the streamwise localization of ECS observed in a variety of plane parallel shear flows [12], further attesting to the value of the reduced structure identified here.

In the first instance, however, slow streamwise variations can be suppressed. The resulting averaged equations (29)–(31) can then be further simplified by introducing a streamwise-invariant stream function  $\phi_1$ :  $\bar{v}_1 = -\partial_z \phi_1$ ,  $\bar{w}_1 = \partial_y \phi_1$ , yielding the streamwise-invariant vorticity  $\omega_1 = \nabla_{\perp}^2 \phi_1$ . Consequently, the averaged system can be expressed as

$$\partial_T u_0 + J(\phi_1, u_0) = \nabla_{\perp}^2 u_0 + \frac{\sqrt{2}\pi^2}{4} \sin(\pi y/2),$$
 (37)

$$\partial_T \omega_1 + J(\phi_1, \omega_1) + 2(\partial_{yy}^2 - \partial_{zz}^2)[\text{Re}(v_1 w_1^*)]$$

$$+2\partial_{y}\partial_{z}(w_{1}w_{1}^{*}-v_{1}v_{1}^{*})=\nabla_{\perp}^{2}\omega_{1},$$
(38)

where  $J(\phi_1, f) = \partial_y \phi_1 \partial_z f - \partial_z \phi_1 \partial_y f$  and  $\text{Re}(v_1 w_1^*)$  denotes the real part of  $v_1 w_1^*$ . In writing these equations, we have dropped the overbar on the mean streamwise velocity component  $u_0$ , again for notational brevity. The fluctuation equations can also be simplified, in particular by taking the divergence of Eqs. (32) and (33) and using Eq. (34) to obtain a Helmholtz equation for the pressure  $p_1$ . The resulting fluctuation equations can be written in the form

$$(\alpha^2 - \nabla_{\perp}^2)p_1 = 2i\alpha(v_1\partial_v u_0 + w_1\partial_z u_0), \tag{39}$$

$$\partial_t \mathbf{v}_{1\perp} + i\alpha u_0 \mathbf{v}_{1\perp} = -\nabla_{\perp} p_1 + \epsilon \nabla_{\perp}^2 \mathbf{v}_{1\perp}. \tag{40}$$

The boundary conditions at  $y = \pm 1$  are

$$\partial_{\nu}u_0 = \omega_1 = \phi_1 = v_1 = \partial_{\nu}w_1 = 0,$$
 (41)

together with periodic boundary conditions in z. Observe that  $u_1$  does not appear in these equations although it can be recovered from Eq. (34).

Equations (37)–(40) capture the self-sustaining process explicitly: the rolls  $\omega_1$  ( $\phi_1$ ) deform the structure of the streamwise velocity  $u_0$  to generate streaks as described by Eq. (37). These streaks lead to the formation of a fluctuating structure  $\mathbf{v}_{1\perp}$  through the advection term in Eq. (40). Lastly, these fluctuations feed the rolls through the Reynolds stresses in Eq. (38). The reduced model (37)–(40) thus isolates the self-sustaining process described by Waleffe [2].

#### III. NUMERICAL STRATEGY

A common approach for computing edge solutions is to apply a so-called edge-tracking algorithm, which only requires use of a time stepper for the equations [38,39]. In the present case, however, implementation of this technique is complicated by the occurrence of two time scales t and  $T = \epsilon t = t/\text{Re}$  in our reduced equations: the mean variables  $u_0, \omega_1$  evolve on the long time scale T, while the fluctuating variables  $\mathbf{v}_{1\perp}$  and  $p_1$  vary on the fast time scale t. In the small  $\epsilon$  limit, the slow mean variables are quasisteady during the evolution of the fast fluctuating variables. Thus, the fluctuation equations (39) and (40) are effectively quasilinear, and the mean variables only respond to the evolution of the fluctuations on a longer time scale. Systems of this type are best solved by treating the fluctuation system as an eigenvalue problem [17], thereby avoiding edge tracking altogether; a similar strategy was followed by Hall and Sherwin [13] and Blackburn et al. [14].

We now describe the details of our algorithm. We consider a two-dimensional domain  $\mathcal{D}$  initially of size  $L_v \times L_z = 2 \times \pi$ , where  $L_v = 2$  is the (dimensionless) distance between the walls and  $L_z = \pi$  is the (dimensionless) imposed period in the spanwise direction, and set the streamwise wave number  $\alpha = 0.5$ . For PCF this domain yields the least unstable lowerbranch solution [40]. The two-dimensional domain is meshed using equidistributed points and the solutions expressed in a Fourier basis. The equations are treated in spectral space using the fast Fourier transform in the periodic direction z and either the fast cosine transform-I or the fast sine transform-I in the wall-bounded direction y, depending on the boundary conditions:  $u_0$  and  $w_1$  are expanded in a cosine basis while  $\omega_0$ and  $v_1$  are expanded in a sine basis [41]. All spatial derivatives are computed pseudospectrally in physical space. The usual  $\frac{2}{3}$  dealiasing is applied for quadratic nonlinearities to avoid mode contamination by spectral convolution.

By analogy with PCF we seek solutions that are shift-reflect-symmetric, i.e., solutions that are invariant under the operation  $[u,v,w](x,y,z)=[u,v,-w](x+L_x/2,y,-z)$ , where  $L_x$  is the imposed period in the streamwise direction, in our case  $L_x=4\pi$ . Within the reduced model framework, this operation becomes  $[u_0,\omega_1,v_1,w_1](y,z)=[u_0,-\omega_1,-v_1,w_1](y,-z)$ . In addition, the solutions can be translated in  $x:[u,v,w](x,y,z)\to [u,v,w](x+\ell,y,z)$ , where  $\ell$  is an arbitrary real quantity. Within our approach, this symmetry

corresponds to  $[u_0,\omega_1,v_1,w_1](y,z) \rightarrow \{u_0,\omega_1,\operatorname{Re}(v_1)\cos(\ell') - \operatorname{Im}(v_1)\sin(\ell') + i[\operatorname{Re}(v_1)\sin(\ell') + \operatorname{Im}(v_1)\cos(\ell')],\operatorname{Re}(w_1) \cos(\ell') - \operatorname{Im}(w_1)\sin(\ell') + i[\operatorname{Re}(w_1)\sin(\ell') + \operatorname{Im}(w_1)\cos(\ell') + i[\operatorname{Re}(w_1)\sin(\ell') + \operatorname{Im}(w_1)\cos(\ell')]\}(y,z)$ , where  $\operatorname{Re}(\ldots)$  [resp.  $\operatorname{Im}(\ldots)$ ] denotes the real (resp. imaginary) part and  $\ell' \equiv \alpha \ell$ .

The numerical algorithm for solving the above problem consists of two parts: an iterative strategy to obtain a good initial condition (described in Sec. III A) and a Newton method to converge the initial condition to an exact ECS solution of the reduced model (37)–(40) (described in Sec. III B). The latter step requires a preconditioner also described in Sec. III B. This two-step process forms the basis for our numerical continuation of the resulting ECS in Re  $\equiv \epsilon^{-1}$ .

## A. Initial iterate

We start by decoupling the mean variables  $u_0$  and  $\omega_1$ , which evolve slowly, from the fluctuations  $v_1$  and  $w_1$ , which evolve more rapidly. We thus consider in succession the slow equations (37) and (38) in which the Reynolds stresses are fixed, and the fast equations (39) and (40) in which the quantity  $u_0$  is maintained constant. Within this framework, the fluctuation equations are linear and autonomous. We take advantage of this structure by treating the fluctuation system as an eigenvalue problem, i.e., we seek solutions with exponential dependence in time:  $\mathbf{v}_{1\perp}(y,z,t) \equiv \hat{\mathbf{v}}_{1\perp}(y,z)e^{\lambda t}$  and  $p_1(y,z,t) \equiv \hat{p}_1(y,z)e^{\lambda t}$ , where  $\lambda$  is the growth rate of the fluctuations. This approach is critical as it provides more information on the fast dynamics than is available with time steppers, which only determine the dominant modes. The basic idea is straightforward: if one of the fluctuating modes is marginal, it corresponds to a stationary solution of the fluctuation equations and if the associated mean variables are also stationary, then the combined mean and fluctuation fields comprise a stationary solution of the reduced system (37)–(40).

The separate treatment of the mean and fluctuation problems implies that the scalar amplitude of the fluctuations in Eqs. (39) and (40) is not fixed by the eigenvalue solve, but must be self-consistently determined as part of the iterative procedure. We refer to this *a priori* unknown scalar as *A* and define it mathematically in Sec. IV. Finding a solution of the problem (37)–(40) is then equivalent to finding the correct fluctuation amplitude *A* for which, given stationary mean variables  $u_0$  and  $\omega_1$ , there exists a fluctuating mode with vanishing growth rate. To obtain a good first approximation to an ECS for subsequent refinement and continuation via Newton iteration, we use the following multistep iterative algorithm:

- 1. Arbitrarily choose the fluctuation amplitude *A*.
- 2. If the growth rate  $\lambda$  of the fastest nonoscillatory growing (or slowest decaying) mode is nonzero:
  - 2.1. Compute the fastest nonoscillatory growing (or slowest decaying) fluctuating mode and its growth rate  $\lambda$  from Eqs. (39) and (40).
  - 2.2. Time advance  $u_0$  and  $\omega_1$  to steady state using Eqs. (37) and (38).
  - 2.3. Repeat steps 2.1 and 2.2 until a converged growth rate  $\lambda(A)$  is obtained.
- 3. Adjust *A* to drive  $\lambda(A)$  to zero by repeating steps 2.1–2.3.

Although developed for canonical wall-bounded shear flows [17], we mention that a similar procedure has been employed with considerable success in a recent study of the saturation of the von Kármán vortex shedding instability of the flow past a cylinder [42].

To use the above algorithm, an initial condition for  $u_0$  alone is required. Solutions of the eigenvalue problem in step 2.1 are obtained using the exponential power method and the package ARPACK [43]. Note that this computation is equivalent to finding the stability of  $u_0$  with respect to streamwise fluctuating perturbations of wave number  $\alpha$ . As we are interested in the least unstable, and hence the most dynamically influential, solutions, we focus on the fastest growing or slowest decaying mode. Other equilibria or periodic orbits may be found by looking at subsequent eigenvalues, but this is outside the scope of this paper. The time integration of Eqs. (37) and (38) in step 2.2 is carried out using a semi-implicit third-order Runge-Kutta scheme [44]. This step is very fast compared to the eigenvalue computation. To simplify the entire computation, we impose the shift-reflect symmetry during the eigenvalue search and the time integration of the mean equations.

#### **B.** Preconditioned Newton method

Following the computation of good approximates using the iterative algorithm just described, the final step is to converge these solutions to the desired accuracy. Typically, this is done using a Newton method for which an inner iteration is required to invert a certain Jacobian matrix that we define in the following. The Jacobian resulting from the reduced system (37)–(40) is poorly conditioned and complicates the use of a Newton method. A suitable preconditioner is therefore required and we designed one based on that originally proposed by Tuckerman [45,46]. Here, we describe its implementation on the generic system

$$\gamma_t \partial_t U = N(U) + \gamma_D L U, \tag{42}$$

where U is the dependent variable, N is a nonlinear operator, L is a linear (Laplace) operator, with  $\gamma_t$ ,  $\gamma_D$  real constants. We look for stationary solutions and so aim to solve

$$0 = N(U) + \gamma_D L U. \tag{43}$$

The preconditioner introduced by Tuckerman is constructed from an implicit Euler scheme with time step  $\Delta t$  applied to Eq. (42):

$$U(t + \Delta t) = \left(I - \frac{\Delta t \, \gamma_D}{\gamma_t} \, L\right)^{-1} \left\{ U(t) + \frac{\Delta t}{\gamma_t} \, N[U(t)] \right\},\tag{44}$$

where U(t) stands for the value of U at time t and I represents the identity operator. We note that by substracting U(t) from expression (44), we obtain

$$U(t + \Delta t) - U(t) = \frac{\Delta t}{\gamma_t} \left( I - \frac{\Delta t \, \gamma_D}{\gamma_t} \, L \right)^{-1} \times \{ N[U(t)] + \gamma_D L U(t) \}, \quad (45)$$

where the right-hand side of Eq. (43) is recovered and preconditioned by  $P = I - \Delta t \gamma_D/\gamma_t L$ .

A Newton method can be obtained by writing for iterate  $U^{[k]}$  at iteration k

$$N(U^{[k]}) + \gamma_D L U^{[k]} = J(U^{[k]}) \delta U, \tag{46}$$

solving for  $\delta U$  and correcting  $U^{[k+1]} = U^{[k]} - \delta U$ . In writing Eq. (46), we have introduced  $J(U^{[k]}) = \delta N(U^{[k]}) + \gamma_D L$ , the Jacobian of the right-hand side operator in Eq. (42) with  $\delta N(U^{[k]}) \equiv \delta N/\delta U(U^{[k]})$ . On multiplying both sides of Eq. (46) by  $(\Delta t/\gamma_t)P^{-1}$ , one obtains

$$\frac{\Delta t}{\gamma_t} P^{-1} [N(U^{[k]}) + \gamma_D L U^{[k]}] = \frac{\Delta t}{\gamma_t} P^{-1} J(U^{[k]}) \, \delta U. \quad (47)$$

The left-hand side of Eq. (47) can be obtained directly using Eq. (45) by computing one implicit Euler time step of the full equation (42) and substracting the initial condition. Moreover, applying the same method to Eq. (42) linearized around  $U^{[k]}$  we obtain a linearized version of Eq. (45) that can then be used to calculate the right-hand side of Eq. (47). In the small  $\triangle t$  limit,  $P \approx I$  and the equation is not preconditioned, while for sufficiently large  $\Delta t$ , we get the so-called Stokes preconditioner  $P \approx \Delta t (\gamma_D/\gamma_t) L$ . This preconditioning method is easy to implement as it only requires a first order implicit Euler time-integration scheme and its use is natural within matrix-free methods, where the Jacobian is not explicitly constructed. The Stokes preconditioner has been widely used in problems that are dominated by diffusion, like coupled convection [47–49]. In contrast, most shear flow studies are carried out at large Reynolds numbers and involve weakly diffusive flows. The required computations are then often performed without preconditioning [50,51].

In the present case, none of these approaches was efficient and we extended the previous preconditioning method to develop a mixed preconditioner. We note that the Jacobian is influenced by two terms: the linearized nonlinear term and the diffusive operator:  $J(U) = \delta N(U) + \gamma_D L$ . In the case of weakly diffusive flows,  $\gamma_D \ll 1$  and the spectrum of the Jacobian is dominated by modes resulting from  $\delta N(U)$ . This is what happens in large Reynolds number studies. On the other hand, in the aforementioned convection problems  $\gamma_D = O(1)$ . These problems are poorly conditioned due to the prominence of eigenvalues generated by the diffusion operator. This difficulty is natural and can be understood by the following heuristic argument: on a given Fourier grid, the condition number of a periodic Laplace operator scales in proportion to  $k^2$ , where k is the largest wave number allowed. Thus, accuracy is reached at the expense of poorer conditioning. This difficulty is typically handled by the use of Stokes preconditioning. In our reduced model, the mean equations (37) and (38) are diffusion dominated:  $\gamma_t = \epsilon^{-1}$  and  $\gamma_D = 1$ . According to the above considerations, these equations require Stokestype preconditioning and we have found that  $\Delta t = \epsilon^{-1} = \text{Re}$ (hence  $\triangle t \gamma_D / \gamma_t = 1$  and P = I - L) provides good results. The fluctuation equation (40), with Eq. (39) solved as a preliminary step, is weakly diffusive with  $\gamma_t = 1$  and  $\gamma_D = \epsilon$ but was not efficiently solved without preconditioning. To improve the efficiency, we observed that the contribution of the nonlinear and the diffusive terms to the Jacobian depends strongly on the gradients within the critical layer. For the lower-branch solution, the width of this layer scales like  $(\alpha \text{ Re})^{-1/3}$ , yielding a diffusion operator  $(1/\text{Re})\nabla_{\perp}^2 =$ 

 $O(\alpha^{2/3} \mathrm{Re}^{-1/3}) = O(\alpha^{2/3} \epsilon^{1/3})$ . We incorporate this scaling by setting  $\Delta t = \xi^2 \alpha^{2/3} \epsilon^{-2/3}$  so that  $P = I - \xi^2 \alpha^{2/3} \epsilon^{1/3} L$ , where  $\xi = O(1)$  is a tuning constant. Several values of  $\xi$  were tested and we adopted  $\xi = 0.5$  for the computations that follow, unless stated otherwise.

To compute the desired ECS using a Newton search, we embed the biconjugate-gradient-squared routine from NSPCG [52] within the Newton algorithm and impose the shift-reflect symmetry on the solutions of Eqs. (37)–(40). Extra care is necessary to eliminate errors arising from the x invariance of the solutions. Within our Fourier decomposition, translations correspond to the eigenvector  $[u_0, \omega_1, v_1, w_1](y, z) = [0, 0, iv_1, iw_1](y, z)$ . The evaluation of the left side of the system corresponding to Eq. (47) is carried out without any constraint while the right side is projected onto the space orthogonal to the above eigenvector, thereby removing the singularity of the Jacobian arising from translation invariance of the solution in x. The above procedure is implemented at each step of the Newton search and forms part of the continuation algorithm used to continue the converged solutions in parameter space.

The above procedure generates numerically exact stationary solutions of the reduced model (37)–(40) for fixed values of the Reynolds number Re, the streamwise wave number  $\alpha$ , and the cross-stream domain size. These converged ECS are characterized by a self-consistently determined amplitude A of the fluctuations that is required to distort the mean velocity profile in just such a way that the fluctuations neither grow nor decay. Thus A also provides a convenient measure of the nonlinear distortion of the streamwise velocity, and we can use it to track the evolution of the self-consistent ECS with the Reynolds number Re, the streamwise wave number  $\alpha$ , or the cross-stream domain size, just like other measures of the solution such as kinetic energy or maximum vorticity.

## IV. EXACT COHERENT STATES

In this section, we present the ECS we have computed using the reduced model (37)–(40) with stress-free boundary conditions (41) at  $y = \pm 1$  and periodic boundary conditions in z.

### A. Initial search

We set Re = 400 (or equivalently  $\epsilon = \frac{1}{400}$ ) and employ the iterative strategy introduced in Sec. III on a 32 × 32 mesh. The accuracy of the results presented here is confirmed by computations on a 64 × 64 grid. We recall that the use of the iterative algorithm introduced in Sec. III A only requires an initial condition on  $u_0$ . We generate such an initial condition by advecting the structureless Waleffe flow by a steady sinusoidal roll  $\omega_1(y,z) = 20\sin(\pi y/2)\sin(2z)$  [or equivalently  $\phi_1(y,z) = -5(\pi^2/16+1)^{-1}\sin(\pi y/2)\sin(2z)$ ], and integrating Eq. (37) with a fixed  $\phi_1$  until a steady state is reached. The roll structure and amplitude have been chosen such that the resulting initial condition on  $u_0$  resembles the ECS in PCF [13,16,40]. The resulting initial profile for the iterative algorithm is shown in Fig. 2.

We define the amplitude of the fluctuations numerically as the maximum value of any component of the in-plane

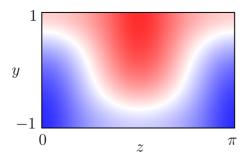


FIG. 2. (Color online) Initial condition for the iterative algorithm obtained by advecting and diffusing  $u_0$  given a steady roll structure with  $\omega_1(y,z) = 20\sin(\pi y/2)\sin(2z)$ . The upper red (resp. lower blue) region has  $u_0 > 0$  (resp.  $u_0 < 0$ ).

fluctuating velocities on the mesh grid:

$$A = \max[|v_1(y_i, z_j)|; |w_1(y_i, z_j)|] \quad \text{for}$$
  

$$i = 1, M, \ j = 1, N,$$
(48)

where  $y_i$  (resp.  $z_i$ ) represents the *i*th (resp. *j*th) mesh point in y (resp. z), M and N are the number of points in y and z, and  $|f| = \sqrt{f_r^2 + f_i^2}$  where the subscript r (resp. i) denotes the real (resp. imaginary) part. We employed the iterative algorithm for different values of A and observed two distinct regimes with different behavior of the leading real eigenvalue. In the first regime, observed for  $A \leq A_H \approx 6.81$ , the leading real eigenvalue converges to  $\lambda = \lambda_c(A)$ . This regime is illustrated in Fig. 3(a). For A = 0 the solution converges to WF, but as A increases the converged flow departs from it. For  $A > A_H$ , a second regime is present in which the algorithm does not converge but instead displays undamped oscillations [Figs. 3(b)-3(d)]. During the oscillations, the streaks  $u_0$ alternately decay towards the trivial solution and then regrow into a more nonlinear structure that exaggerates the traits of the exact coherent state, a process that repeats in a periodic fashion. This process is straightforward for sufficiently small values of A but becomes increasingly complex as A is increased. For example, the oscillations in  $\lambda$  and the accompanying solution have a period of only 6 iterations for A = 6.9 [Fig. 3(b)] but 11 iterations per oscillation for A = 7.5 and 23 iterations per oscillation for A = 8 with  $\lambda$  showing increasingly complex behavior [Figs. 3(c) and 3(d)].

The difference between these two regimes can be traced to the way in which A, the scalar amplitude of the fluctuations, enters Eq. (38), where the Reynolds stress term has amplitude  $A^2$ . For a given fluctuation mode, increasing (decreasing) A leads to a greater (lesser) forcing of the rolls  $\omega_1$ . The induced rolls deform the streaks  $u_0$  generating a new eigenvalue problem for the fluctuations. Hence, A is a forcing parameter that tunes one step in the self-sustaining process. If the forcing is too weak, the iterative algorithm relaxes to the trivial solution perturbed by a latent forcing induced by the nonvanishing fluctuations. If the forcing is too strong, the feedback from the mean variables (rolls  $\omega_1$ , then streaks  $u_0$ ) is also too strong, causing overshooting of a potential "steady" solution of the iterative algorithm. Since A is fixed during the iteration process, successive overshoots occur, generating the observed oscillatory behavior. By analogy, one can think

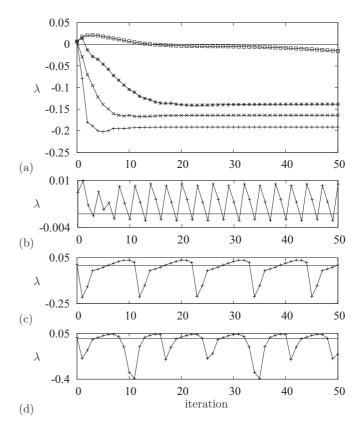


FIG. 3. (a) Steady regime of the iterative algorithm ( $A \le A_H \approx 6.81$ ). The curves represent the value of the largest real eigenvalue  $\lambda$  plotted against the iteration number. Only the first 50 iterations are shown and the curves correspond to A=5,5.5,6, and 6.5 from bottom to top. The A=6.5 eigenvalue converges at a later iteration (not shown). The oscillatory regime ( $A>A_H$ ) is represented in the same way for A=6.9 in (b), A=7.5 in (c), and A=8 in (d). In all cases, the algorithm is initialized using the initial condition shown in Fig. 2.

of a simple dynamical system which admits a stable steady solution at low A before undergoing a supercritical Hopf bifurcation at  $A = A_H$  to produce stable oscillations while the steady solution has become unstable. The variable in which the solution oscillates, the iteration number, is discrete which may account for the small departures from strictly periodic oscillations that can be observed in the data [Figs. 3(b)-3(d)], as the period may vary continuously with A.

We look for ECS that are stationary, and so seek solutions with  $\lambda_c$  close to zero. Figure 4 shows  $\lambda_{50}$ , the largest real eigenvalue after 50 iterations, for values of A spanning the interval  $5 \le A \le 8$ . Although our results have been checked using a refined mesh ( $64 \times 64$  modes) and the leading eigenvalue converged to  $10^{-4}$  in most cases, we emphasize that convergence is not required at this stage as this algorithm is only intended to provide a good initial condition for a Newton iteration. The results reveal two possible candidates, corresponding to values of  $\lambda_{50}$  close to 0:  $A_1 \approx 6.55$  and  $A_2 = A_H \approx 6.81$ . The state corresponding to  $A_1$  is undoubtedly a good initial condition as  $\lambda_{50} < 0$  for  $A < A_1$  and  $\lambda_{50} > 0$  for  $A_2 > A > A_1$ . That corresponding to  $A_2$  is seemingly less secure: for A = 6.81, the converged

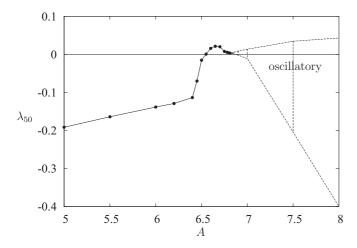


FIG. 4. The leading stationary eigenvalue  $\lambda_{50}$  after 50 iterations of the algorithm in the steady regime ( $A \le A_H \approx 6.81$ ) shown using a solid line with the dots indicating the value actually computed. Most of these eigenvalues decrease by less than  $10^{-4}$  per iteration after 50 iterations. The eigenvalues for  $6.4 \le A \le 6.6$  (corresponding to the region of steepest increase in the figure) are not fully converged after 50 iterations, but do converge to slightly smaller values after a larger number of iterations. In fact, convergence is not required as the aim is to generate a good initial condition for subsequent refinement by the preconditioned Newton method. The amplitude of the eigenvalue oscillations after 50 iterations in the oscillatory regime ( $A > A_H$ ) is indicated by vertical dashed lines at the values of A actually used.

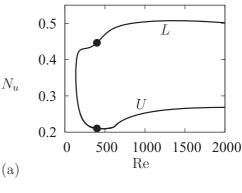
eigenvalue is  $\lambda_c(A=6.81) \approx 0.003\,191$  and approaches 0 from above as A is increased but the oscillatory regime is reached by A=6.82 leading to small amplitude oscillations close to but not crossing zero.

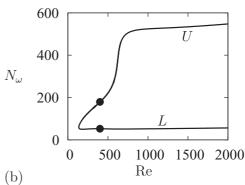
These impressions can be confirmed by inserting the approximate solution with amplitude  $A_1$  into the Newton algorithm described in Sec. III B. On a 32 × 64 mesh grid, the solution readily converges to a lower-branch state at this Reynolds number (Re = 400). In contrast, when the solution with amplitude  $A_2$  is used, the solution converges to the corresponding upper-branch state.

## B. Continuation in Reynolds number

The regularized equations (37)–(40) contain the parameter  $\epsilon \equiv 1/\text{Re}$ . The presence of this parameter allows us to continue the solutions obtained above to both larger and smaller Reynolds numbers. The results for a 32 × 64 mesh grid are displayed in Figs. 5 and 6.

The solution branches are plotted in six different ways. In Fig. 5(a), we present  $N_u \equiv 2E_u$ , where  $E_u \equiv (2D)^{-1} \int_{\mathcal{D}} u_0^2(y,z) \, dy \, dz$  is the streamwise-invariant streamwise kinetic energy per unit volume and  $D \equiv \int_{\mathcal{D}} dy \, dz$ . In Fig. 5(b), we present  $N_\omega \equiv 2\,\mathrm{Re}^2\,E_\omega \equiv D^{-1}\int_{\mathcal{D}}\omega_1^2(y,z)\,dy\,dz$ , a quantity related to the streamwise-invariant in-plane enstrophy per unit volume. In Fig. 5(c), we present the quantity  $N' \equiv 2\,\mathrm{Re}^2\,E'$ , where  $E' \equiv (2D\mathrm{Re}^2)^{-1}\int_{\mathcal{D}}(v_1^2+w_1^2)\,dy\,dz$  measures the streamwise-fluctuating in-plane kinetic energy per unit volume. The lower branch (labeled L) passes a saddle node at  $\mathrm{Re} \approx 136$ , giving rise to an upper branch (labeled U). Figures 6(a) and 6(b) show a projection of our solutions onto





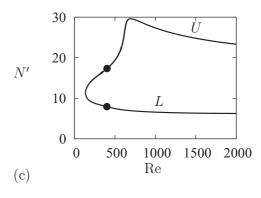
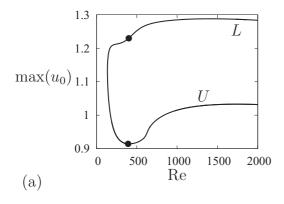
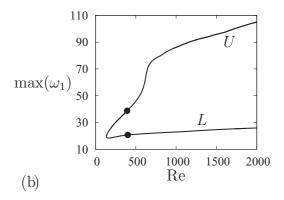


FIG. 5. Bifurcation diagrams showing branches of exact coherent states as a function of the Reynolds number Re obtained by continuation of converged solutions (solid dots) starting from approximates generated by the iterative algorithm with initial amplitude  $A_1$  (converged to a lower-branch state, indicated by L) and  $A_2$  (converged to an upper-branch state, indicated by U). (a)  $N_u$ , (b)  $N_\omega$ , (c) N'.

pointwise maxima of the corresponding quantities, thereby providing a complementary representation of the results.

The emergence of the upper branch is rather unexpected since the reduced system (37)–(40) was developed by appealing to lower-branch scalings. Evidently, the asymptotic procedure is sufficiently robust to capture both lower *and* upper solution branches. However, the computation of the upper branch is more delicate. As observed by Beaume *et al.* [21], upper-branch solutions and their critical layer have a different spatial structure which dramatically increases the computational cost. To continue these solutions, we used a  $64 \times 128$  mesh grid and adjusted the preconditioner as necessary. Specifically, we started by testing a few values of  $\Delta t$  (see Sec. III B) and selected the most efficient one





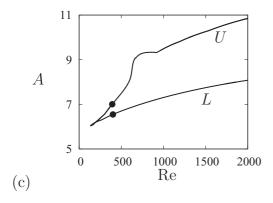


FIG. 6. The same bifurcation diagrams as in Fig. 5 rendered in terms of the maximum values of (a) the streamwise velocity  $u_0$ , (b) the vorticity  $\omega_1$ , and (c) the amplitude A of the self-consistent fluctuation field.

for continuation near the saddle node. We then continued the upper branch until the algorithm failed. Each time this occurred we tested a few values of  $\triangle t$  to determine the most suitable one, repeating this process as many times as necessary to continue the branch up to the desired value of the Reynolds number. Upper-branch solutions can also be computed directly by starting from the approximate solution with amplitude  $A_2$ , obtained using the iterative scheme of Sec. III A, and applying the Newton algorithm with the modified preconditioner described above. In particular, the upper branch is identified with the value of  $A \approx 7.04$  at which the (unstable) fixed point of the iterative process corresponds to zero eigenvalue.

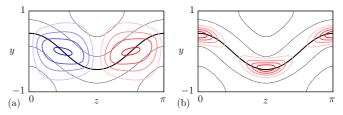


FIG. 7. (Color online) Lower-branch solution computed at Re  $\approx$  1500 represented by (a) contours of the streamwise-constant stream function  $\phi_1$  and (b) contours of  $|(v_1, w_1)|_{L_2}$  representing the amplitude of the in-plane fluctuations. Positive streamwise contours are shown in red (right roll) while negative contours are shown in blue (left roll); contours are equidistributed to give a sense of local gradients. Each contour plot is overlaid on the streak profile shown in black, with the solid line representing the critical layer  $u_0 = 0$ . Three-dimensional visualizations of the fluctuating variables are displayed in Fig. 8.

Some care is required in interpreting the values of the energies in Fig. 5, as  $N_{\omega}$  and N' are proportional to Re<sup>2</sup>. The trivial solution, for which  $N_u = 1$ ,  $N_\omega = N' = 0$ , represents the state of maximal transport and hence has the greatest kinetic energy. All other ECS are found to have a lower kinetic energy, as indicated by lower values of  $N_u$ . The quantities  $N_\omega$  and N', as well as the maximum of  $\omega_1$  and the amplitude A, remain O(1)along the lower branch, reflecting the relevance of the reduced model. Interestingly, a similar observation can be made for the upper-branch solution for which the fluctuations also remain O(1). However, possible departures from the assumed scaling may be observed in the enstrophy-related norm of the upper-branch states. Indeed,  $N_{\omega} = O(100)$ , max $(\omega_1) \sim 80$ –90 at Re = O(1000), suggesting that the vorticity becomes larger and larger in an increasingly narrow region. However, these values, despite being relatively large at low Re, do not appear to increase sufficiently with Re to violate the assumed ordering in the expansion.

Figure 7 depicts the lower-branch solution at Re  $\approx 1500$ using streamwise-averaged quantities, while Fig. 8 provides a three-dimensional rendition of this solution. Figures 9 and 10 provide analogous representations of the upper-branch solution at the same Reynolds number. The lower-branch solution possesses a smoothly undulating critical layer that is maintained by two nearly circular rolls [cf. Fig. 7(a)]. This structure is supported by fluctuations accumulating in the critical layer. Figure 7(b) shows that these fluctuations have a rapid variation in the direction perpendicular to the critical layer [its thickness being proportional to  $(\alpha Re)^{-1/3}$ ], while slow variations are observed along the critical layer. The three-dimensional representations in Fig. 8 confirm these observations and shed some additional light on the streamwise dynamics of the lower-branch solution. The streamwisefluctuating streamwise velocity  $u_1$  is essentially concentrated in the regions of stronger streamwise-invariant stream function  $\phi_1$  [compare Figs. 7(a) with Fig. 8(a)] and therefore away from the crests of the critical layer. As a consequence of the incompressibility of the fluctuations [Eq. (34)], the in-plane fluctuating dynamics accumulate at the extrema of the critical layer, away from the location of the streamwise rolls, as documented in Fig. 7(b). Figure 8 shows that at x = 0 (defined arbitrarily as the front section in the figure), in the region

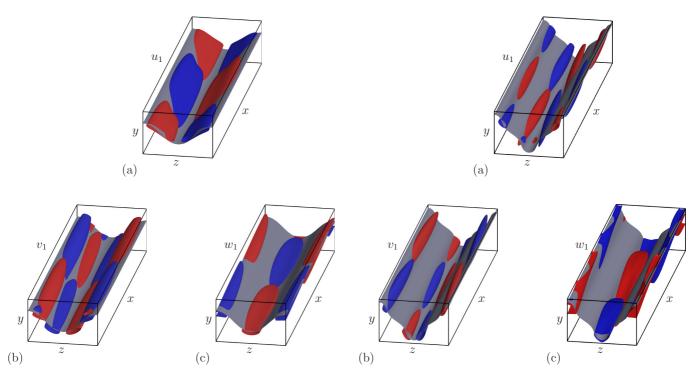


FIG. 8. (Color online) Three-dimensional rendition of the fluctuating flow associated with the lower-branch solution at  $\text{Re} \approx 1500$ . The surfaces represented in color correspond to two equal and opposite values at half the maximum value of (a) the streamwise-fluctuating streamwise velocity  $u_1$ , (b) the streamwise fluctuating wall-normal velocity  $v_1$ , and (c) the streamwise-fluctuating spanwise velocity  $w_1$ . Red (light gray) color corresponds to positive values while blue (dark gray) corresponds to negative values. The gray surface shows the critical layer  $u_0=0$ .

around the lowest point of the critical layer,  $z = \pi/2$ , the fluid flows from left to right along the  $u_0 = 0$  surface. The reverse occurs half a period downstream and at the highest point of the critical layer [located at the boundary of the (periodic) domain when x = 0].

In comparison to the lower-branch solution, the upperbranch solution has stronger variations along the critical layer, the extrema of which approach the top and bottom walls. This change in shape is a signature of stronger rolls. The resulting structure is shown in Fig. 9(a), where by comparison with Fig. 7(a), it is evident that the rolls are stretched diagonally and split, displaying a bimodal structure. This last feature

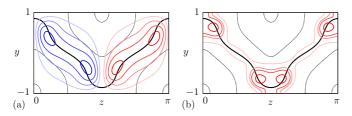
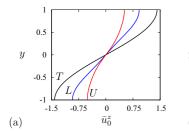


FIG. 9. (Color online) Same representation as in Fig. 7 but for the upper-branch solution at Re  $\approx$  1500. Three-dimensional visualizations of the fluctuating variables for this solution are displayed in Fig. 10.

FIG. 10. (Color online) Same representation as in Fig. 8 but for the upper-branch solution at Re  $\approx$  1500. Intersections of nonzero fluctuations with the upper and lower walls can be observed in (c) and are allowed by the stress-free boundary conditions.

is responsible for the sharper crests of the  $u_0 = 0$  surface relative to that for the lower-branch solution. This change in structure is reminiscent of the differences between lower- and upper-branch states in PCF (see Fig. 7 from [53]) and hints at the usefulness of our reduced model for states beyond the lower-branch states for which it was developed. Associated with the bimodal structure in  $\phi_1$  is a similar bimodal structure of the fluctuations which are now strongly localized on either side of the critical layer turning points. Figure 9 shows that as a result, the location of the streamwise rolls almost coincides with the maxima of the fluctuation field, suggesting that the increased shear in the streamwise rolls suppresses fluctuations, with the location of the self-sustaining process moving towards the critical layer turning points. This evolution in turn implies that for the upper-branch states, the width of the critical layer depends strongly on location along the critical layer: the layer appears broader near its maximum deflection from y = 0 and is substantially thinner in the intervals in-between. In addition, the amplitude of both the rolls and the spanwise fluctuations peaks strongly in the vicinity of these turning points. Although we have not pursued this phenomenology further, the results suggest that in the limit  $Re \to \infty$  the common assumption of uniform critical layer thickness may require reexamination, with the critical layer "breaking up" into something more akin to critical "spots," where most of the critical layer forcing is concentrated [Fig. 9(b)].

These properties of the lower- and upper-branch solutions are reflected in the associated horizontally-averaged streamwise velocity profiles  $\bar{u}_0^z$  shown in Fig. 11. As expected, the ECS in each case reduces the shear across the layer. The



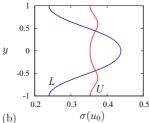


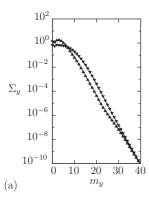
FIG. 11. (Color online) Streamwise velocity profiles at Re  $\approx$  1500. (a) z-averaged streamwise velocity  $\bar{u}_0^z$  as a function of the wall-normal coordinate y. (b) Standard deviation  $\sigma(u_0)$  of the streamwise velocity  $u_0$  from  $\bar{u}_0^z(y)$ . The trivial solution is shown in black (labeled T), the lower branch (labeled L) in blue, and the upper branch (labeled U) in red.

reduction is less for the weaker lower-branch ECS than for the upper-branch ECS [Fig. 11(a)]. Of particular interest is the standard deviation of the velocity from these profiles  $\sigma(u_0,y) = \sqrt{\int_z [u_0(y,z) - \bar{u}_0^z]^2 dz}, \text{ shown in Fig. 11(b)}. \text{ For the lower-branch ECS } \sigma \text{ peaks at mid-height where the streamwise rolls are strongest and falls off quite strongly towards the walls at <math>y = \pm 1$  [Fig. 11(b)]. In contrast, on the upper branch  $\sigma$  is quite uniform across the layer, with a local minimum at mid-height, an effect that can be directly attributed to the stretching of the streamwise rolls along the critical layer and the location of their peak amplitude near regions of maximum deviation of the critical layer from y = 0.

## C. Spectra

To check the accuracy of the solutions, we plot in Fig. 12 the one-dimensional spectra of the fluctuation velocity in the wall-normal and spanwise directions. These are defined in terms of the normalized partial sums

$$\Sigma_{y}(m_{y}) = \frac{1}{2(M-1)N} \left\{ S^{2}(m_{y},0) + \sum_{m_{z}=1}^{N} \left[ S^{2}(m_{y},m_{z}) + S^{2}(m_{y},-m_{z}) \right] \right\}^{1/2}, (49)$$



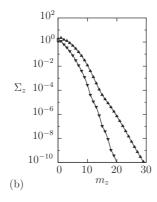


FIG. 12. Spectra of the lower- (downward triangles) and upper-(upward triangles) branch solutions represented through the normalized partial sums  $\sum_y (m_y)$  and  $\sum_z (m_z)$  defined in Eqs. (49) and (50).

$$\Sigma_z(m_z) = \frac{1}{2(M-1)N} \left[ \sum_{m_y=0}^{M} S^2(m_y, m_z) \right]^{1/2}, \quad (50)$$

where  $S^2(m_v, m_z) = |v_1(m_v, m_z)|^2 + |w_1(m_v, m_z)|^2$  and M (resp. N) is the maximum wave number in the y (resp. z) direction. The quantity  $\Sigma_{\nu}(m_{\nu})$  [resp.  $\Sigma_{\tau}(m_{\tau})$ ] has been defined in such a way that it is proportional to the sum of the amplitudes of the fluctuations with wave number  $m_v$  (resp.  $m_z$ ) in the y (resp. z) direction. The plots confirm that the amplitude of the upper-branch fluctuations is larger than that along the lower-branch solutions. In addition, the spectra in the wall-normal direction decay exponentially at the same rate for both lower- and upper-branch states while the spectrum of the upper-branch solution decays more slowly in the spanwise direction than that of the lower-branch solution. These results reflect the fact that the scales of the wall-normal variation remain comparable as one goes from the lower branch to the upper one while the smallest spanwise scale shrinks. These results highlight the fact that the change in structure between the lower and upper branches is primarily associated with differences in the spanwise variation of the fields and inform the numerical requirements to compute these solutions accurately: while the wall-normal mesh can be designed independently of the solution sought with 30 wavelengths sufficient at Re = 1500 (i.e., approximately 30 modes in the cosine-sine basis, or 60 modes in the complex Fourier basis), the number of points in the spanwise direction needs to be increased by a factor of about 1.5 for upper-branch states at Re = 1500. Obviously, increasing the Reynolds number or any other factor that sharpens the critical layer impacts these requirements.

## D. ECS dependence on the domain size

We next investigate how the solutions computed in the previous section depend on the spanwise domain size  $L_z$ and the imposed streamwise wave number  $\alpha = 2\pi/L_x$ . We begin by fixing  $\alpha = 0.5$  and studying the effect of varying  $L_z$ . The resulting bifurcation diagrams are shown in Fig. 13. Continuation of the lower-branch states to domains with smaller spanwise extent reveals that they pass a saddle node at  $L_z \approx 2.1$  before terminating on a branch of solutions with two wavelengths per period when  $L_z \approx 2.5$ . These solutions are depicted in the top two panels in Fig. 14 together with the initial lower-branch solution at  $L_z = \pi$  in the third panel. The rolls present at  $L_z = \pi$  gradually tilt as the domain period is reduced (Fig. 14, second panel). Beyond the saddle node at  $L_z \approx 2.1$ , these tilted vortices continue to stretch diagonally, in a direction transverse to the critical layer. As this happens, the center of each roll gradually splits forming two corotating rolls, one on either side of the critical layer, a process that greatly reduces the deflection of the critical layer from its laminar location at y = 0. At the same time, small counter-rotating rolls appear in the corners above and below each tilted structure and these grow in strength as  $L_z$  increases, ultimately forming a period two state at  $L_z \approx 2.5$  with an unperturbed y = 0 critical layer (Fig. 14, top panel). The resulting period-doubled ECS bears a number of similarities with the solutions EQ7 and EQ8

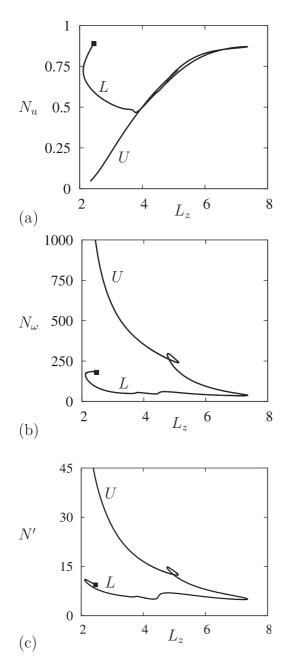


FIG. 13. The ECS at Re  $\approx$  1500 as a function of the spanwise period  $L_z$ . The diagrams show (a)  $N_u$ , (b)  $N_\omega$ , and (c) N'. The full square indicates the termination of the branch on a branch of solutions with two wavelengths in the domain (Fig. 14, upper panel). Solutions along the lower branch (denoted by L) are shown in Fig. 14, while those on the upper branch (denoted by U) are shown in Fig. 15.

first observed by Gibson *et al.* [3] and reported in Fig. 16 of Gibson and Brand [54].

Continuing the lower-branch state at  $L_z = \pi$  in the opposite direction, towards larger  $L_z$ , reveals a new type of behavior. The increasing domain size stretches the rolls, which evolve into a bimodal structure reminiscent of the upper-branch solution with  $L_z = \pi$  [compare Fig. 14, fourth panel, with Fig. 9(a)]. Increasing  $L_z$  further leads to the progressive breakup of each of the original rolls into a pair of corotating rolls (Fig. 14, fourth panel). Once formed, these rolls are pulled

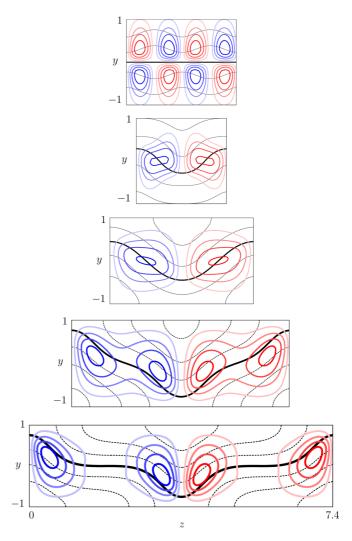


FIG. 14. (Color online) Structure of the lower-branch solutions with different spanwise periods  $L_z$  represented in the same way as in Fig. 7. From top to bottom: end point of the branch at  $L_z\approx 2.5$ , left saddle node at  $L_z\approx 2.1$ , solution at  $L_z=\pi$  (taken from Fig. 7), solution at  $L_z\approx 5$ , and solution at the right saddle node at  $L_z\approx 7.4$ . The contour values of the streamwise velocity are the same throughout, but different values of the stream function are used from panel to panel for better representation of the flow. The contours are in all cases equidistributed and the scale of the domain is kept the same for all solutions.

farther apart as  $L_z$  increases, resulting in a periodic array of pairs of counter-rotating rolls supporting a highly deformed critical layer interspersed with connecting zones where the trivial laminar flow is only weakly perturbed (Fig. 14, bottom panel). The resulting state cannot be continued to larger domain sizes and passes a saddle node at  $L_z \approx 7.4$  where it connects with states originating along the upper branch (see below and Fig. 15). This type of behavior is similar to that observed for PCF by Deguchi *et al.* [55], but is not related to spatial localization in the spanwise direction as conventionally understood since true localized states must become independent of the domain size.

From the saddle-node solution (last panels in Figs. 14 and 15) one can continue the branch back to lower values

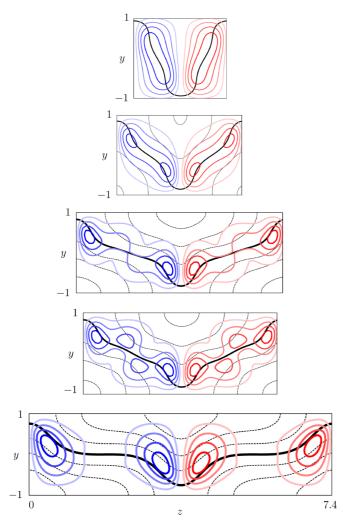


FIG. 15. (Color online) Structure of the upper-branch solutions with different spanwise periods  $L_z$ . From top to bottom: diverging solution at  $L_z \approx 2.3$ , solution at  $L_z = \pi$  (taken from Fig. 9), solution at the right saddle node of the loop at  $L_z \approx 5.1$ , solution at the left saddle node of the loop at  $L_z \approx 4.8$ , and solution at the right saddle node at  $L_z \approx 7.4$  (taken from Fig. 14). The same figure style is used as for Fig. 14.

of  $L_z$  but in the direction of increasing fluctuation intensity, i.e., along the upper branch [see Fig. 13(c)]. The resulting upper-branch states are shown in Fig. 15. As the period  $L_z$  is reduced from  $L_z \approx 7.4$  along the upper branch, the stretching gradually disappears, but the corotating rolls do not merge, in contrast to the behavior along the lower branch. Instead, two additional rolls are nucleated between the original pair of corotating rolls, and these also corotate (Fig. 15, fourth panel). The net result is an array of four corotating rolls whose combined action deforms the critical layer further from the laminar case, and these are paired with a similar set of four corotating rolls in the other half of the domain, but rotating in the opposite sense. This four-roll structure is destroyed as the branch passes through a loop between  $L_z \approx 4.8$  and  $L_z \approx 5.1$ : the two weaker middle rolls that have appeared along the upper branch below  $L_z \approx 7.4$  gradually fade, thereby restoring the bimodal structure (Fig. 15, second panel) and generating the state in Fig. 9. When  $L_z$  is decreased further,

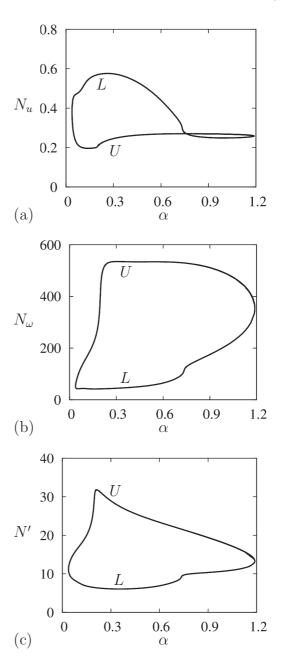


FIG. 16. Bifurcation diagrams showing the ECS at Re = 1500 and  $L_z = \pi$  as a function of the streamwise wave number  $\alpha$ . (a)  $N_u$ . (b)  $N_{\omega}$ . (c) N'. Solutions at the left and right saddle nodes are shown in Fig. 17. The letter L (resp. U) denotes the lower (resp. upper) branch.

the bimodal structure gradually disappears as the rolls are squeezed together (Fig. 15, top panel). At the same time, both the mean stream function and the fluctuation fields grow without bound, while the streamwise velocity  $u_0$  becomes increasingly homogenized and ultimately approaches zero. These developments are reflected in the dramatic decrease in the streamwise velocity norm  $N_u$ , together with increases in the enstrophy norm  $N_\omega$  and the fluctuation norm N' shown in Fig. 13, and indicate that the postulated form of the solutions is starting to break down. Thus, solutions in this regime are unlikely to be physically relevant. We mention that in related

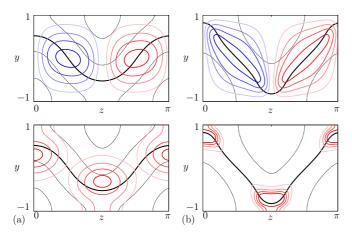


FIG. 17. (Color online) Structure of the ECS at (a) the left saddle node at  $\alpha \approx 0.0380$  ( $L_x \approx 165$ ) and (b) the right saddle node at  $\alpha \approx 1.1890$  ( $L_x \approx 5.3$ ) of the isola shown in Fig. 16. The solutions are represented through their mean streamwise velocity with contours of the stream function (top panel) and the fluctuation norm (bottom panel). The contours are in all cases equidistributed.

calculations for PCF, Melnikov *et al.* [56] find that the lowerand upper-branch states form an isola in  $L_z$  with no additional bifurcations (except for saddle nodes) as  $L_z$  varies.

We have also studied how the morphology of the ECS in a domain with  $L_z = \pi$  varies at fixed Re = 1500 when the streamwise wave number  $\alpha$  is changed. The bifurcation diagrams obtained are shown in Fig. 16. The figure reveals that the ECS lie on an isola and therefore do not connect to any other solution. Thus, the isola defines an interval of existence for the ECS at  $L_7 = \pi$  and Re = 1500: 0.0380 <  $\alpha$  < 1.1890. While the lower bound for  $\alpha$  hints at the persistence of these structures for very long domains ( $L_x \approx 165$ ), the upper bound indicates that the required streamwise periodicity of the domain be at least  $L_x \approx 5.3$  for these structures to be self-sustaining, a value close to that observed in PCF [13]. The ECS at the left and right saddle nodes along the isola are represented in Fig. 17. For small  $\alpha$  (long streamwise domain), the critical layer is distorted approximately sinusoidally by nearly circular rolls and the fluctuations do not exhibit sharp gradients despite being located close to the critical layer  $u_0 = 0$ . In contrast, for large  $\alpha$  and therefore short streamwise domains, the fluctuations become very localized and the associated critical layer is deformed into a sawtooth profile. The associated rolls are highly elongated and align with the approximately constant slope sections of the critical layer. These developments are a consequence of the  $(\alpha \text{ Re})^{-1/3}$  critical layer scaling [57]. As Re is kept fixed, decreasing  $\alpha$  increases the width of the critical layer thereby weakening the strength of the fluctuations. This is a direct consequence of the fact that  $\alpha$  affects the amplitude of the  $u_0$ -induced advection of fluctuations, as described by Eq. (40). As a result, decreasing the value of  $\alpha$  decreases the coupling between the mean and fluctuation fields. Thus, the fluctuations no longer track the critical layer efficiently, and so remain weak and do not deform the rolls. The associated critical layer is sinusoidal. The converse is true when  $\alpha$  is increased: the coupling becomes stronger, leading to fluctuations that are strongly focused on the critical layer

and that substantially deform the rolls and hence generate a strongly distorted critical layer.

## V. DISCUSSION

In this paper, we have presented a simple rational procedure that leads to a reduced description of plane parallel shear flows. The method assumes that the flow is dominated by the mean streamwise flow component, with spanwise components of the velocity field (the rolls) that are much weaker. Despite this, the Reynolds stress generated by the fluctuating fields modifies the mean spanwise velocity and hence the mean streamwise flow, as described by Eqs. (29)–(31), and this in turn modifies the fluctuations as described by Eqs. (32)-(34). Of these, the former are simplified in having an O(1) effective Reynolds number, while the latter constitute a viscously regularized (cf. [14,17]) but quasilinear system that admits solutions of arbitrary amplitude. In our approach, this amplitude is determined by a self-consistency requirement: in steady state the Reynolds stress generated by the fluctuations must be such as to produce a streamwise flow for which the fluctuations neither grow nor decay, and we have described an iterative process whereby the amplitude of the fluctuations can be adjusted to realize this requirement. Our approach therefore captures the essence of the self-sustaining mechanism identified by Waleffe [2] and leads to reduced equations that capture the universality in the behavior of plane parallel shear flows. We believe that these equations are suitable not only for studying steady ECS with critical layers, but also exact traveling waves and indeed other nonequilibrium structures with no critical layer at all. In this respect, the equations possess advantage over detailed studies of particular flows using flow-specific scalings. However, to justify these claims, our results for WF and other flows will have to be compared quantitatively with solutions of the corresponding fully three-dimensional problems. In addition, such comparisons will determine, on a case by case basis, the range of Reynolds numbers for which our results provide a reliable guide to the solutions of the full

We have applied the numerical algorithm developed here to compute a variety of exact coherent states in a body-force driven flow we refer to as Waleffe flow. The fundamental assumptions we make turn out to capture not only the expected lower-branch states, but also the corresponding upper-branch states, reached via numerical continuation in the parameter Re  $\equiv 1/\epsilon$ . The results we obtain are similar to the corresponding PCF results obtained by Blackburn et al. [14] for lower-branch states and by Deguchi et al. [58] for upper-branch states via continuation in  $L_z$ . This fact is significant since there is no guarantee that continuation in  $L_7$  will identify the same states as continuation in Re. In both systems, the lower-branch critical layer deforms into a sinusoidal surface through the action of the rolls, while the deformation corresponding to the upper-branch states is both stronger and bimodal. Particularly intriguing is our discovery that along the upper branch the width of the critical layer is no longer uniform and that the bimodal structure of the rolls and streaks concentrates the critical layer forcing in regions of maximum departure from the unperturbed critical layer. Current asymptotic approaches do not take this possibility into account. However, the intrinsic self-consistency of our reduced equations implies that this new critical layer structure is likely a property of upper-branch states in the full system at large Re.

We have also used numerical continuation to continue our solutions in the spanwise domain length  $L_z$  and in the streamwise wave number  $\alpha$ . The former determines the existence region for the solutions we have found and shows that the lower-branch solutions at small  $L_z$  bifurcate from a period two spatially periodic state but undergo a saddle-node bifurcation at larger  $L_z$  that connects the lower- and upperbranch states. The solutions near this fold are stretched in the spanwise direction relative to O(1) domains but are not spatially localized in the conventional sense, in contrast to the suggestion made by Deguchi et al. [55], since they cannot be continued to larger  $L_z$  and hence to larger separations. The continuation in the streamwise wave number  $\alpha$  leads to simpler results; this time the lower- and upper-branch states are connected by folds at either end and the solutions lie on an isola. As a result, they do not extend to either very small or very large values of  $\alpha$ . We believe that these states, as well as those obtained via continuation in Re, will prove of great value in further explorations of the full set of reduced equations, including studies of spatial modulation and possible localization in the streamwise direction. We hope to report on these explorations in a future publication.

The approach described here has been used with considerable success to derive reduced descriptions of other strongly

anisotropic flows, including convection in a strong magnetic field [35] and rapidly rotating convection [59–61]. In all these cases, the mean field adjusts on a slower time scale than the time scale on which the fluctuations evolve. As a result, the mean flow can be taken to be quasistatic and the fluctuations can be computed at fixed values of this field. The mean can then be updated based on the fluctuations computed on this basis, and the process repeated. Alternatively, the reduced system can be treated as a dynamically coupled system for the fluctuations and the mean, and the combined system time stepped together. The latter procedure has been used successfully in the context of rapidly rotating convection [59–61] and we anticipate that time stepping Eqs. (29)–(34) for transition values of the Reynolds number could lead to similar insights. This, too, is a topic for future study.

## ACKNOWLEDGMENTS

C.B. acknowledges the Geophysical Fluid Dynamics Program at the Woods Hole Oceanographic Institution, where the first results were obtained, for support through a fellowship. The work was supported by the National Science Foundation under Grants No. DMS-1211953 and No. DMS-1317596 (C.B. and E.K.), No. OCE-0934827 (G.P.C.), and No. OCE-0934737 and No. DMS-1317666 (K.J.). E.K. wishes to acknowledge additional support from the Chaire d'Excellence Pierre de Fermat de la région Midi-Pyrénées (France).

- [1] M. Nagata, J. Fluid Mech. 217, 519 (1990).
- [2] F. Waleffe, Phys. Fluids 9, 883 (1997).
- [3] J. F. Gibson, J. Halcrow, and P. Cvitanović, J. Fluid Mech. 638, 243 (2009).
- [4] H. Faisst and B. Eckhardt, Phys. Rev. Lett. 91, 224502 (2003).
- [5] H. Wedin and R. R. Kerswell, J. Fluid Mech. 508, 333 (2004).
- [6] Y. Duguet, A. P. Willis, and R. R. Kerswell, J. Fluid Mech. 613, 255 (2008).
- [7] Y. Duguet, A. P. Willis, and R. R. Kerswell, J. Fluid Mech. 663, 180 (2010).
- [8] A. Schmiegel, Transition to Turbulence in Linearly Stable Shear Flows, Ph.D. thesis, Marburg University, 1999.
- [9] F. Waleffe, J. Fluid Mech. 435, 93 (2001).
- [10] G. Kawahara and S. Kida, J. Fluid Mech. 449, 291 (2001).
- [11] P. Hall and F. T. Smith, J. Fluid Mech. 227, 641 (1991).
- [12] T. M. Schneider, D. Marinc, and B. Eckhardt, J. Fluid Mech. 646, 441 (2010).
- [13] P. Hall and S. Sherwin, J. Fluid Mech. 661, 178 (2010).
- [14] H. M. Blackburn, P. Hall, and S. J. Sherwin, J. Fluid Mech. 726, R2 (2013).
- [15] G. I. Taylor, Philos. Trans. R. Soc. London A 223, 289 (1923).
- [16] J. Wang, J. Gibson, and F. Waleffe, Phys. Rev. Lett. 98, 204501 (2007).
- [17] C. Beaume, in *Proceedings of the Geophysical Fluid Dynamics Program* (Woods Hole Oceanographic Institution, Woods Hole, MA, 2012), pp. 389–412.
- [18] P. G. Drazin and W. H. Reid, *Hydrodynamic Stability* (Cambridge University Press, Cambridge, UK, 1981).
- [19] R. M. Clever and F. H. Busse, J. Fluid Mech. 344, 137 (1997).

- [20] F. Waleffe, Phys. Fluids 15, 1517 (2003).
- [21] C. Beaume, E. Knobloch, G. P. Chini, and K. Julien, Fluid Dyn. Res. 47, 015504 (2015).
- [22] V. A. Romanov, Funct. Anal. Appl. 7, 137 (1973).
- [23] P. J. Schmid and D. S. Henningson, *Stability and Transition in Shear Flows*, Applied Mathematical Sciences, Vol. 142 (Springer, Berlin, 2001).
- [24] O. Dauchot and F. Daviaud, Phys. Fluids 7, 335 (1995).
- [25] O. Dauchot and F. Daviaud, Phys. Fluids 7, 901 (1995).
- [26] N. Tillmark, Europhys. Lett. 32, 481 (1995).
- [27] P. Manneville, Theor. Comput. Fluid Dyn. 18, 169 (2004).
- [28] S. Bottin, F. Daviaud, P. Manneville, and O. Dauchot, Europhys. Lett. **43**, 171 (1998).
- [29] A. Prigent, G. Grégoire, H. Chaté, O. Dauchot, and W. van Saarloos, Phys. Rev. Lett. 89, 014501 (2002).
- [30] D. Barkley and L. S. Tuckerman, Phys. Rev. Lett. 94, 014502 (2005).
- [31] V. I. Arnold and L. D. Meshalkin, Seminar led by A. N. Kolmogorov on selected problems of analysis, Usp. Mat. Nauk. 15, 20 (1960).
- [32] L. D. Meshalkin and Y. G. Sinai, J. Appl. Math. Mech. 25, 1700 (1961).
- [33] P. Love, Bifurcations in Kolmogorov and Taylor-vortex flows, Ph.D. thesis, California Institute of Technology, 1999.
- [34] D. Lucas and R. R. Kerswell, J. Fluid Mech. 750, 518 (2014).
- [35] K. Julien and E. Knobloch, J. Math. Phys. 48, 065405 (2007).
- [36] G. P. Chini, K. Julien, and E. Knobloch, Geophys. Astrophys. Fluid Dyn. 103, 179 (2009).
- [37] F. Giannetti and P. Luchini, J. Fluid Mech. 547, 21 (2006).

- [38] J. D. Skufca, J. A. Yorke, and B. Eckhardt, Phys. Rev. Lett. 96, 174101 (2006).
- [39] T. M. Schneider and B. Eckhardt, Chaos 16, 041103 (2006).
- [40] T. M. Schneider, J. F. Gibson, M. Lagha, F. De Lillo, and B. Eckhardt, Phys. Rev. E 78, 037301 (2008).
- [41] M. Frigo and S. G. Johnson, Proc. IEEE **93**, 216 (2005), special issue on "Program Generation, Optimization, and Platform Adaptation."
- [42] V. Mantič-Lugo, C. Arratia, and F. Gallaire, Phys. Rev. Lett. 113, 084501 (2014).
- [43] R. B. Lehoucq, D. C. Sorensen, and C. Yang, ARPACK Users Guide: Solution of Large Scale Eigenvalue Problems by Implicitly Restarted Arnoldi Methods, 1997.
- [44] P. R. Spalart, R. D. Moser, and M. M. Rogers, J. Comput. Phys. 96, 297 (1991).
- [45] L. S. Tuckerman, in 11th International Conference on Numerical Methods in Fluid Dynamics, Lecture Notes in Physics, Vol. 323, edited by D. Dwoyer, M. Hussaini, and R. Voigt (Springer, Berlin, 1989), pp. 573–577.
- [46] C. K. Mamun and L. S. Tuckerman, Phys. Fluids 7, 80 (1995).
- [47] A. Bergeon and E. Knobloch, Phys. Fluids 14, 3233 (2002).
- [48] I. Mercader, O. Batiste, and A. Alonso, Int. J. Numer. Methods Fluids 52, 707 (2006).

- [49] C. Beaume, A. Bergeon, H.-C. Kao, and E. Knobloch, J. Fluid Mech. 717, 417 (2013).
- [50] J. F. Gibson, Channelflow: A spectral Navier-Stokes simulator in C++, Technical Report, University of New Hampshire, 2012, Channelflow.org.
- [51] Y. Duguet (private communication).
- [52] T. Oppe, W. Joubert, and D. Kincaid, NSPCG User's Guide: A Package for Solving Large Sparse Linear Systems by Various Iterative Methods, 1988.
- [53] J. Jimenez, G. Kawahara, M. P. Simens, M. Nagata, and M. Shiba, Phys. Fluids 17, 015105 (2005).
- [54] J. F. Gibson and E. Brand, J. Fluid Mech. 745, 25 (2014).
- [55] K. Deguchi, P. Hall, and A. Walton, J. Fluid Mech. 721, 58 (2013).
- [56] K. Melnikov, T. Kreilos, and B. Eckhardt, Phys. Rev. E 89, 043008 (2014).
- [57] S. A. Maslowe, Annu. Rev. Fluid Mech. 18, 405 (1986).
- [58] K. Deguchi and P. Hall, J. Fluid Mech. 750, 99 (2014).
- [59] M. Sprague, K. Julien, E. Knobloch, and J. Werne, J. Fluid Mech. 551, 141 (2006).
- [60] K. Julien, E. Knobloch, A. M. Rubio, and G. M. Vasil, Phys. Rev. Lett. 109, 254503 (2012).
- [61] A. M. Rubio, K. Julien, E. Knobloch, and J. B. Weiss, Phys. Rev. Lett. 112, 144501 (2014).